

DYNAMIC SIMULATION OF

LARGE STIFF SYSTEMS

DYNAMIC SIMULATION  
OF LARGE STIFF SYSTEMS  
IN A MODULAR SIMULATION FRAMEWORK

by

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## ABSTRACT

"DYNSYS" is a digital simulation package for modelling the dynamic behaviour and automatic control of complex industrial systems. An acronym for Dynamic Systems Simulator, it was developed in the late 1960s in the chemical engineering department at McMaster University. The underlying principle of "DYNSYS" is that of modularity; i.e., the user assembles mathematical models of process units and control devices to build the process whose transient behaviour is to be studied. A fundamental aspect of dynamic simulation is the numerical solution of ordinary differential equations (o.d.e.s). The original version of "DYNSYS" used a third order Adams-Moulton-Shell routine; however, this is not sufficient to handle stiff systems, i.e., systems where the time constants differ greatly in magnitude. In chemical engineering, stiff o.d.e.s occur widely in reaction kinetics and to some extent in multistage systems.

Conventional numerical techniques are restricted by stability to using a very small step size resulting in large computer times. There have been many new numerical techniques published in the recent literature directed at

the efficient numerical solution of systems of stiff d.d.e.s. A literature survey of these has been made.

Numerical testing of several methods indicated Gear's method to be superior. It is a variable order, variable step, linear, multistep method.

Most stiff techniques are implicit and require a technique such as Newton-Raphson iteration to converge. Each iteration involves the solution of a system of linear algebraic equations (usually sparse) equal in size to the number of o.d.e.s. For a large stiff system, this requires considerable computer time. Various sparse linear equation solvers have been evaluated and that of Bending and Hutchison appears to be the most efficient. Their routine stores and operates on only the nonzero elements of the equations. When the equations are solved for the first time, a string of integers called the "operator list" is created which stores the particular solution process by Gaussian elimination. If the system is re-solved using the operator list, the amount of computer time required is greatly diminished. If the zero elements remain zero and the nonzero elements change, the same "operator list" can be used to solve the new system. This is essentially what occurs during numerical integration. The operator list could be set up on the first integration step and used on later steps to solve each new linear system.

Gear's integration algorithm in conjunction with the Bending-Hutchison linear equation solve has been implemented into DYNYSYS version 2.0. An option for stiff systems with tridiagonal Jacobian matrix is also included. The procedure for writing modules is outlined.

Four small examples are presented to illustrate the new executive:

- (1) The level control of a stirred tank system (nonstiff) with time delay.
- (2) A network of 15 stirred tank reactors, stiff and nonstiff, 2 o.d.e.s per reactor.
- (3) A tubular reactor with 222 stiff o.d.e.s resulting from the discretization of the partial differential equations.
- (4) A tubular reactor with 49 stiff o.d.e.s with tridiagonal Jacobian matrix.

A simulation of a fictitious chemical plant proposed by Williams and Otto is also described.

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## 1. INTRODUCTION TO DYNAMIC SIMULATION

Dynamic simulation in this thesis will refer to the study, via digital computer simulation, of the dynamic behaviour of a process which is changing with time. In our context, the process will be all or part of a chemical plant.

There are many analysis and design problems that may require a flexible dynamic model for their solution, for example:

- (1) Difficulties in start up or shut down of a process.
- (2) Control strategies including finding suitable controller settings.
- (3) The influence of frequent disturbances or fluctuations in the process.
- (4) The response of the plant to equipment failure and testing of corrective action.
- (5) Operating strategies for multiproduct plants.

In addition there may be some steady state process problems whose solution can be obtained by driving a



dynamic model of the process to the steady state (thus avoiding numerical instabilities in a steady state iterative process, for example).

Various executive computer programs have been developed to handle the information transfer and calculations for performing a dynamic simulation.

### 1.1 Equation-Oriented Approach

The equation-oriented approach was developed first. Here, the user would write, possibly in some coded manner, all the algebraic and differential equations describing his process and the executive would then solve all the equations simultaneously over a period of time. There have been over thirty packages developed since 1955 (Franks, 1967). Table 1.1 lists some of the more common, recently developed programs: MIMIC (Northcott, 1967), CSMP (CSMP, 1967) and IMP (Brandon, 1972).

Table 1.1: Equation-Oriented Executive Programs For Dynamic Simulation

Acronym	Date	Full Name	Institution Where Developed
MIMIC	1965		Wright-Patterson Air Force Base, Ohio
CSMP	1967	Continuous System Modeling Program	IBM
IMP	1972	Implicit Solution Software System	University of Connecticut

## 1.2 Modular Approach

In the past few years, several executive programs have been developed, mostly by chemical engineers, which are more problem-oriented. A summary of these appears in Table 1.2: SWAPSO (Utsumi, 1969), KARDAZ (Brambilla *et al.*, 1971; Kardasz, 1969; Kardasz and Molnar, 1971, 1974), DYNYSYS (Bobrow, Johnson and Ponton, 1970, 1971), FLEX (Shern and Petty, 1970), PRODYC (Ingels and Motard, 1970), REMUS (Ham, 1969), EARLYBIRD (Weaver, 1974), ACME (Loibl, Camp and Wilkins, 1973), DYFLO (Franks, 1972a) and OSUSIM (Koenig, 1972). More information on these packages appears in Section 3.4.

These problem-oriented programs use variations of what is known as the modular approach. A real plant is made up of processing units such as reactors, heat exchangers, compressors, etc. with connecting lines of material flow. The flows, temperatures, concentrations and other variables in the process may be measured and, based on these measurements, controllers can activate control devices in an attempt to achieve desired conditions in the plant. Figure 1.1 illustrates a very simple mixing operation which is made up of a processing unit with controllers and control valves in a process flow diagram. Each of the interconnected units can be described by mathematical relationships relating inputs and outputs.

Table 1.2: Problem-Oriented Executive Programs For  
Dynamic Simulation

Acronym	Date	Full Name	Institution Where Developed
SWAPSO	1969	Stone and Webster All Purpose Simulator and Optimizer	Stone and Webster
KARDAZ	1969	Kardasz Program	University of Pisa
DYNSYS	1970	Dynamic Systems Simulator	McMaster University
FLEX	1970		Procter and Gamble
PRODYC	1970	A System for Simulating Chemical Process Dynamics and Control	University of Houston
BEMUS	1970	Routine For Executive Multi-Unit Simulation	University of Pennsylvania
EARLYBIRD	1971		Tulane University
ACME	1972	Analyzer For Computer Modeling of Chemical Engineering Processes	University of Detroit
DYFLO	1972		DuPont
OSUSIM	1972	Ohio State University Simulator	Ohio State University

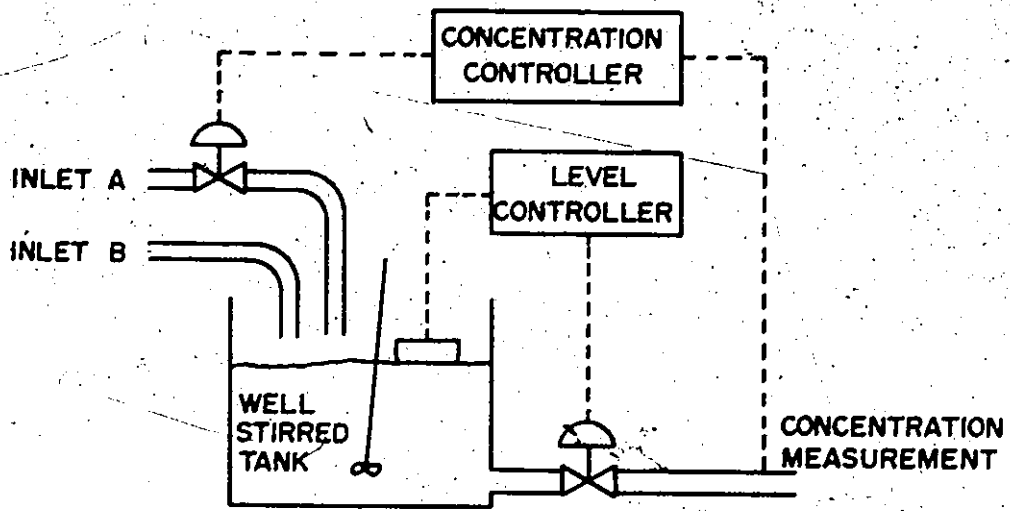


FIGURE I.1 : PROCESS FLOW DIAGRAM

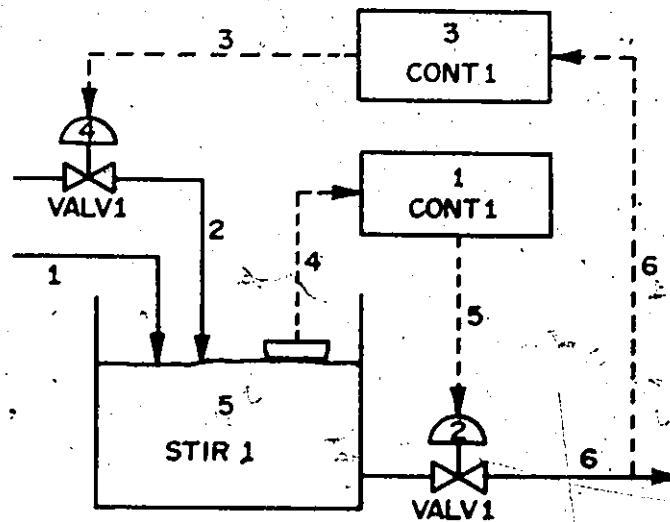


FIGURE I.2 : DYNAMIC INFORMATION FLOW DIAGRAM

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The assembly of the set of mathematical equations for simulating one of these process units, including controllers and control devices, in a computer program subroutine is called a unit computation or computation module.

Just as a chemical plant is made up of assemblies of the physical units, a mathematical model of a plant can be assembled as a network of modules, among which information will flow in a manner analogous to the material flow or control signals in the real plant. Figure 1.2 is a dynamic information flow diagram that corresponds to Figure 1.1. It can be seen that there is a very close similarity between the process flow diagram and the dynamic information flow diagram.

This building-block approach is very convenient for simulating systems which are very modular, i.e., which have many different pieces of physical equipment and where it is desired to study different combinations or configurations of these equipments and their effect on the dynamic behaviour. It is not as efficient computationally as the equation-oriented approach since there is more information transfer; however, it does allow the user to visualize his process more easily since each piece of equipment is usually represented by a corresponding module.

Several modules of different levels of sophistication can be developed for the same equipment. As a library of equipment and control modules becomes available, they may be readily used for the simulation of new plants, provided that the modules have a reasonable level of generality. This reduces the programming effort of new plant studies. A new simulation would require perhaps one third of the modules to be created, while the remaining two thirds could be taken from the library.

The modular approach deals with the real variables of the process, rather than transformed variables, and furthermore, the modules may be quite nonlinear in behaviour. This should encourage control studies by design and process engineers, even if they are unfamiliar with modern control systems theory and terminology.

### 1.3 Thesis Objectives

The objectives of this thesis are:

- (1) to critically evaluate numerical techniques for the solution of systems of stiff ordinary differential equations such as those encountered in the analysis and design of complex chemical processes and to develop improved techniques for these situations.
- (2) to critically analyze the DYNSSYS package which is under development and to improve its approach for analyzing and designing complex systems.
- (3) to demonstrate the application of an improved DYNSSYS to several examples including a dynamic simulation of a realistic chemical process and to make recommendations for a library of unit computations.





## 2. NUMERICAL SOLUTION OF ORDINARY DIFFERENTIAL EQUATIONS

A fundamental part of dynamic simulation is the numerical solution of ordinary differential equations (o.d.e.s). Partial differential equations can be broken down into o.d.e.s by discretizing one or more of the independent variables. Higher order o.d.e.s can usually be broken down into a set of first order equations. Thus we are primarily concerned with the numerical solution of a set of first order ordinary differential equations. These are usually initial value problems where each of the independent variables is specified at some initial time value (normally  $t = 0$ ), rather than boundary value problems where the independent variables are not all specified at the same time value. There are a large number of numerical integration techniques for solving these equations. This chapter discusses some aspects of these techniques.

## 2.1 Introduction

We are considering the system:

$$\dot{y} = \frac{dy}{dt} = f(t, y) \quad (2.1.1)$$

with initial condition:

$$y(t_0) = y_0 \quad (2.1.2)$$

where  $y$  may be scalar or vector.

$f(t, y)$  represents any function of the dependent variable  $y$  and the independent variable  $t$ .

$t_0$  represents the initial value of  $t$  and  $y_0$  is the initial value of  $y(t)$  at  $t = t_0$ .

By a solution to the above system, we mean a curve in the  $y(t)$  versus  $t$  domain which passes through the point  $(t_0, y_0)$  and which satisfies (2.1.1). By a numerical solution, we mean a discrete set of values of  $y(t)$  called  $\{y_n\}$  corresponding to a discrete set of  $t$  values called  $\{t_n\}$  which approximate the equivalent continuous  $y(t)$  versus  $t$  curve.

To obtain these discrete sequences, we consider a finite set of points  $\{t_n\}$  which form a grid along the  $t$  coordinate. Each point in the sequence will be related to the previous point by the relationship:

$$t_{n+1} = t_n + h_n \quad n = 0, 1, \dots, N \quad (2.1.3)$$

where  $h_n$  is the grid spacing.

Numerous techniques have been developed to solve o.d.e.s, the two principal types being single and multiple-step methods.

## 2.2. Existence and Uniqueness Theorem

Before we attempt to find a solution to (2.1.1) and (2.1.2), we must be assured that a unique solution does exist. The following two conditions are necessary and sufficient for a unique solution to exist in the time interval  $[a, b]$  (Henrici, 1962)

1.  $f(t, y)$  is defined and continuous in the interval  $a \leq t \leq b$ ,  $-\infty < y < \infty$  where  $a$  and  $b$  are finite.
2. There exists a Lipschitz constant  $L$  such that for any  $t \in [a, b]$  and any two numbers  $u$  and  $v$ :

$$|f(t, u) - f(t, v)| \leq L |u - v| \quad (2.2.1)$$

In the nonscalar case, vector norms replace absolute values.

The author is not aware of any equations resulting from chemical engineering applications where these conditions are not satisfied.

### 2.3 Single-Step Methods

Single-step methods do not require any information prior to  $(t_n, y_n)$  to calculate  $y_{n+1}$ . The most common single-step methods are the Euler and Runge-Kutta types.

The simple Euler method is:

$$y_{n+1} = y_n + h f(t_n, y_n) \quad (2.3.1)$$

Runge-Kutta methods use evaluations of  $f(t, y)$  within the interval  $(t_n, y_n)$  and  $(t_{n+1}, y_{n+1})$ .

The general single-step equation is:

$$y_{n+1} = y_n + \sum_{i=1}^v w_i k_i \quad (2.3.2)$$

where  $w_i$  are weighting coefficients

$v$  is the number of  $f(t, y)$  substitutions  
and  $k_i = h f(t_n + c_i h, y_n + \sum_{j=1}^{i-1} a_{ij} k_j)$

$$c_1 = 0, \quad i=1, 2, \dots, v \quad (2.3.3)$$

A typical fourth order formula is given by:

$$y_{n+1} = y_n + \frac{1}{6} [k_1 + 2k_2 + 2k_3 + k_4]$$

where  $k_1 = h f(t_n, y_n)$

$$k_2 = h f\left(t_n + \frac{1}{2}h, y_n + \frac{1}{2}k_1\right) \quad (2.3.4)$$

$$k_3 = h f\left(t_n + \frac{1}{2}h, y_n + \frac{1}{2}k_2\right)$$

$$k_4 = h f\left(t_n + h, y_n + k_3\right)$$

## 2.4 Multiple-Step Methods

A multiple-step or multistep method is one which uses information prior to  $y_n$  to calculate  $y_{n+1}$ . The general linear,  $k$ -step differential-difference equation with constant coefficients is:

$$y_{n+1} = \sum_{i=1}^k \alpha_i y_{n+1-i} + h \sum_{i=0}^k \beta_i \dot{y}_{n+1-i} \quad (2.4.1)$$

If  $\beta_0 = 0$ , the resulting equation is called an explicit or predictor formula. However, if  $\beta_0 \neq 0$ , the resulting equation is referred to as an implicit or corrector equation.

The normal mode of usage is to use an explicit form of (2.4.1) to get a first approximation to  $y_{n+1}$  (the predicted value) and then use an implicit form of (2.4.1) to improve on this value with  $\dot{y}_{n+1}$  evaluated using the predicted value of  $y_{n+1}$ . The process may be repeated using the new value of  $\dot{y}_{n+1}$ , calculated from the new

value of  $y_{n+1}$ . This may be done several times until  $y_{n+1}$  converges to a fixed value. The term PECE means a predictor step followed by a derivative evaluation, then a corrector step and another derivative evaluation. For PE(CE)<sup>s</sup> the latter step is repeated s times. These are also known as predictor-corrector methods although a single step method can also be a predictor-corrector.

A commonly used combination is the Adams-Bashforth predictor with the Adams-Moulton corrector equation. The third order equations are:

$$\text{PREDICTOR: } y_{n+1} = y_n + \frac{h}{12} [23y_n - 16y_{n-1} + 5y_{n-2}] \quad (2.4.2)$$

$$\text{CORRECTOR: } y_{n+1} = y_n + \frac{h}{12} [5y_{n+1} + 8y_n - y_{n-1}]$$

## 2.5 Stability

In the numerical solution of an o.d.e., a sequence of approximations  $y_n$  to the true solution  $y(t_n)$  is generated. The stability of a numerical method refers to the behaviour of the difference or accumulated error  $y(t_n) - y_n$  as  $n$  becomes large.

Stability analysis is usually performed on the scalar equation:

$$\frac{dy}{dt} = \lambda y \quad (2.5.1)$$

$$y(0) = 1 \quad (2.5.2)$$

with analytical solution:

$$y(t) = \exp(\lambda t) \quad (2.5.3)$$

For the numerical solution, if  $h$  is constant

$$t = nh, \quad y_n = e^{nh\lambda} \quad n = 0, 1, 2, \dots \quad (2.5.4)$$

or

$$y_{n+1} = e^{h\lambda} y_n \quad (2.5.5)$$



### 2.5.1 Stability Of Multistep Methods

Applying the general linear multistep method (2.4.1), we get the characteristic equation:

$$\mu^k - \sum_{i=1}^k \alpha_i \mu^{k-i} - h\lambda \sum_{i=0}^k \beta_i \mu^{k-i} = 0 \quad (2.5.6)$$

which has  $k$  characteristic roots  $\mu_i$ ,  $i = 1, 2, \dots, k$ .

The numerical solution is thus:

$$y_n = \sum_{i=1}^k d_i \mu_i^n \quad (2.5.7)$$

where  $d_1, \dots, d_k$  are constants determined by the initial conditions.

One of the characteristic roots approximates the Taylor Series expansion of the true solution  $y = \exp(\lambda t)$  with a truncation error corresponding to the order  $p$  of the method. If we let this root be  $\mu_1$ , then as  $h \rightarrow 0$

$$\mu_1 = \exp(h\lambda) + o(h^{p+1}) \quad (2.5.8)$$

This root, called the principal root, is the root which we wish to be represented in the numerical solution, since  $\mu_1^n$  approximates  $\exp(nh\lambda)$ . The other  $k-1$  roots are extraneous and are as a result of the use of a difference equation of degree  $k$  to represent a first order

differential equation. The extraneous roots have no relation to the exact solution, but nevertheless are unavoidable.

The characteristic roots of (2.5.7) are the same as those of the difference equation for the error:

$$\epsilon_n = y_n - y(t_n) \quad (2.5.9)$$

i.e. 
$$\epsilon_n = \sum_{i=1}^k c_i \mu_i^n \quad (2.5.10)$$

For a valid numerical solution we require that  $\epsilon_n$  not grow with  $n$ . A linear multistep method is called absolutely stable if  $|\mu_i| \leq 1$ ,  $i = 2, \dots, k$ .

The critical problems of numerical stability in o.d.e.s are associated with inherently stable o.d.e.s ( $Re(\lambda) < 0$ ) in which absolute stability is the important factor.

The value of  $h\lambda$  for which  $|\mu_i| = 1$  and for which a small increase in  $h\lambda$  makes  $|\mu_i| > 1$  is called the general stability boundary. Any method with a finite stability boundary is called conditionally stable, whereas any method with an infinite general stability boundary is called unconditionally stable or A-stable. The recent literature contains many new stability definitions (Lambert, 1973; Widlund, 1967; Cryer, 1973).

Most conventional methods have stability boundaries roughly like those in Figure 2.1. A-stability would require absolute stability for the entire left hand plane (Dahlquist, 1963).

Dahlquist (1963) has proven several important theorems for multistep methods:

- (1) An explicit linear multistep method cannot be A-stable.
- (2) The order of an A-stable implicit linear multistep method cannot exceed two.
- (3) The second order A-stable implicit linear multistep method with the smallest error constant is the trapezoidal rule:

$$y_{n+1} = y_n + \frac{h}{2} (\dot{y}_n + \dot{y}_{n+1}) \quad (2.5.11)$$

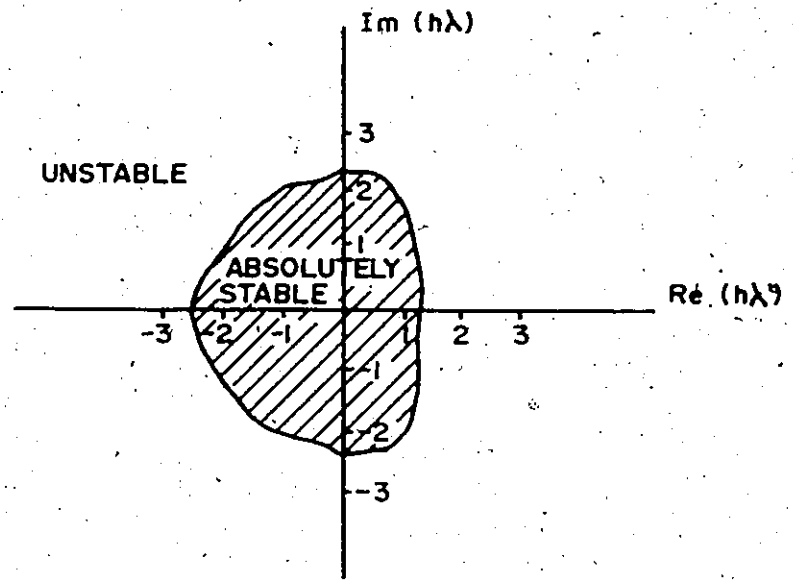


FIGURE 2.1 : TYPICAL STABILITY BOUNDARY FOR CONVENTIONAL INTEGRATION TECHNIQUE

### 2.5.2 Stability Of Runge-Kutta Methods

The difference equation resulting from an explicit Runge-Kutta method is:

$$y_{n+1} = \left[ \sum_{i=0}^p \frac{(h\lambda)^i}{i!} + \sum_{i=p+1}^v a_i \frac{(h\lambda)^i}{i!} \right] y_n \quad (2.5.12)$$

or

$$y_{n+1} = \mu_1 y_n \quad (2.5.13)$$

where  $p$  is the order of the method

$v$  the number of derivative substitutions

$a_i$  are constants depending on the specific formula

and  $\mu_1$  is the characteristic root of the equation

analogous to the principal root of a multistep method.

The numerical solution is thus:

$$y_n = c_1 \mu_1^n \quad (2.5.14)$$

and as for the linear multistep equation, we require for absolute stability  $|\mu_1| \leq 1$ ; this usually requires

$|h\lambda| \leq k$  where  $k$  is of the order 1 to 10.

Note the difference in the basic causes of instability for the two types of solution. The difference equation for the multistep method yields extraneous roots

which can cause instability if any one of them is greater than 1. However, for the Runge-Kutta technique, there are no extraneous roots and instability is caused only by taking a step size which is too large for the series to represent the characteristic root accurately.

### 2.5.3 Nonlinear Systems

The preceding stability analysis has been applied to the linear system  $\dot{y} = \lambda y$ . There is no general stability theory for the nonlinear o.d.e.:  $\dot{y} = f(t, y)$ , but at any point in the integration, the equation can be linearized by a Taylor Series about that point,  $(t_n, y_n)$ .

$$\dot{y} = f(t_n, y_n) + \left. \frac{\partial f}{\partial y} \right|_n (y - y_n) + O(h^2) \quad (2.5.15)$$

Hildebrand (1956) has shown that the stability characteristics of the linearized o.d.e. are very similar to those of the original equation for small  $h$ . Thus the stability analysis for the linear case represents the nonlinear problem at least locally.

The eigenvalues of (2.5.15) corresponding to  $\lambda$  of  $\dot{y} = \lambda y$  are the eigenvalues of the Jacobian matrix  $\frac{\partial f}{\partial y}$ .

For the system  $\dot{y} = f(y)$  the Jacobian matrix is:

$$\begin{bmatrix}
 \frac{\partial f_1}{\partial y_1} & \frac{\partial f_1}{\partial y_2} & \dots & \frac{\partial f_1}{\partial y_n} \\
 \frac{\partial f_2}{\partial y_1} & \frac{\partial f_2}{\partial y_2} & \dots & \frac{\partial f_2}{\partial y_n} \\
 \cdot & \cdot & \cdot & \cdot \\
 \cdot & \cdot & \cdot & \cdot \\
 \frac{\partial f_n}{\partial y_1} & \frac{\partial f_n}{\partial y_2} & \dots & \frac{\partial f_n}{\partial y_n}
 \end{bmatrix}$$

(2.5.16)

## 2.6 Stiffness

A stiff o.d.e. is one in which one component of the solution decays much faster than others. This may occur in a single o.d.e. or in a system of o.d.e.s. The term "stiff" probably arose from the study of mechanical or structural problems which had both soft and stiff spring constants (Steeper, 1970).

An example of a single stiff equation is:

$$\frac{dy}{dt} = -200 [y - z(t)] + \dot{z}(t) \quad (2.6.1)$$

$$y(0) = 10 \quad (2.6.2)$$

$$z(t) = 10 - (10 + t)e^{-t} \quad (2.6.3)$$

The analytical solution is:

$$y(t) = z(t) + 10e^{-200t} \quad (2.6.4)$$

It is more common, however, for this condition to occur in systems of equations. As an example:

$$\dot{y} = \begin{bmatrix} -500.5 & 499.5 \\ 499.5 & -500.5 \end{bmatrix} y \quad (2.6.5)$$



$$y(0) = \begin{bmatrix} 0 \\ 2 \end{bmatrix} \quad (2.6.6)$$

with analytical solution:

$$y_1(t) = e^{-t} - e^{-1000t} \quad (2.6.7)$$

$$y_2(t) = e^{-t} + e^{-1000t} \quad (2.6.8)$$

The eigenvalues of the matrix of coefficients are  $\lambda_1 = -1000$  and  $\lambda_2 = -1$ . In terms of time constants, they are  $\tau_1 = 0.001$  and  $\tau_2 = 1$ ; the eigenvalues and time constants are negative reciprocals.

Both  $y_1$  and  $y_2$  have a rapidly decaying component, corresponding to  $\lambda_1$ , which very quickly becomes insignificant. After the brief initial phase of the solution in which the  $\lambda_1$  component is not negligible, we would like to use a step size  $h$  which is determined only by the component of the solution corresponding to  $\lambda_2$ . However, stability considerations demand that both  $|h\lambda_1|$  and  $|h\lambda_2|$  be bounded for the entire integration, usually by a number from 1 to 10. As an example, Euler's method requires  $|h\lambda_2| = 2$ .

Although the component of the solution corresponding to  $\lambda_1$  is of no practical interest, the criterion of absolute stability forces us to use an extremely small

value of  $h$  over the entire range of integration. As a result, the computation time necessary to integrate a highly stiff system can become excessive. It is desirable then, to use a method which is not so restrictive in step size, for example, a method that is A-stable.

The ratio of the largest to the smallest eigenvalue in absolute value is called the stiffness ratio. In (2.6.5), the stiffness ratio is 1000, a moderate value. In practice, ratios of the order  $10^6$  have been encountered, for example, the hydrogen-bromine reaction (Creighton, 1971).

Mathematically, for a stiff system, the real part of the largest eigenvalue is much less than zero and the absolute value of the ratio of the real part of the largest eigenvalue to the real part of the smallest eigenvalue is much greater than one.

$$\text{i.e. } \operatorname{Re}(\lambda_{\max}) \ll 0 \quad (2.6.9)$$

$$\left| \frac{\operatorname{Re}(\lambda_{\max})}{\operatorname{Re}(\lambda_{\min})} \right| \gg 1 \quad (2.6.10)$$

One way of determining stiffness is to calculate the eigenvalues of the Jacobian matrix, but for all but the most simple cases, this is very tedious. In practice, an easy way to identify a stiff system is to use a

conventional method to integrate it. If an extremely small value is needed for the step size (say  $h < 0.01$ ) to avoid instability, the system is probably stiff and a special method should be used.

Stiff o.d.e.s occur commonly in reaction kinetics, control theory, circuit theory and to some extent in multistage systems. They are obtained in theoretical studies of reaction kinetic models of systems where the rate constants for the reactions involved are widely separated. Such systems occur in many fields of chemistry, particularly when radicals or quasistationary species are involved. The stiffness in control theory could, for instance, stem from the difference in time constants between the fast electrical control circuits and some slow mechanical devices. Stiffness in circuit theory often occurs in situations where transistors or other nonlinear semiconductors with very small time constants are connected to networks, often linear, with much larger time constants. Stiffness can arise in multistage systems, for example, in a distillation column because the time constants for the various components and plates may be very different and also since the time constant for the reboiler is large compared with the time constant of a plate.

Many chemical engineering systems give rise to stiff systems. Such situations occur in chemical reactors

(Amundson, 1965; Amundson and Luss, 1968; Creighton, 1971; DeGroat and Abbett, 1965; Emanuel, 1963; Eschenroeder, Boyer and Hall, 1962; Gelinas, 1972; Moretti, 1965; Robertson, 1967 and Williams and Otto, 1960) and in multistage systems (Davison, 1968; Distefano, 1968a,b,c; Haines, 1969; Mah, Michaelson and Sargent, 1962 and Rosenbrock, 1957).

## 2.7 Accuracy

In numerical integration there is usually an error associated with the method.

One source of error is the round-off error which is introduced by the machine due to the finite number of significant figures carried. Double precision arithmetic can be used, if it is felt round-off error is critical; however, it is very difficult to estimate and is usually neglected (Henrici, 1962).

The global truncation error of a method is the difference between the actual value and the calculated value:

$$G.T.E. = y_n^{ACT} - y_n^{CALC} \quad (2.7.1)$$

It is usually very difficult or impossible to calculate.

The local truncation error is the error committed in one integration step from  $y_n$  to  $y_{n+1}$ . It is often of the form:

$$L.T.E. = Ch^{p+1} y^{(p+1)}(\xi) \quad (t_n < \xi < t_{n+1}) \quad (2.7.2)$$

If as  $h \rightarrow 0$ , the L.T.E. becomes  $o(h^{p+1})$ ,  $p$  is called the order of the method.

The global truncation error is actually the accumulation of the local truncation errors. Because the global error is so difficult to calculate, the local truncation error is usually controlled. By keeping the local error below some specified value at each step, it is hoped that the global error is also kept under control. The step size can be chosen automatically to do this, if there is an estimate for the local error.

### 2.7.1 Single-step Methods

Among single-step methods, Euler's method has no error estimate, nor do most of the Runge-Kutta methods. Merson (1957) devised the first Runge-Kutta method with an error estimate:

$$k_1 = hf(t_n, y_n)$$

$$k_2 = hf(t_n + \frac{1}{3}h, y_n + \frac{1}{3}k_1)$$

$$k_3 = hf(t_n + \frac{2}{3}h, y_n + \frac{1}{6}k_1 + \frac{1}{6}k_2)$$

$$k_4 = hf(t_n + h, y_n + \frac{1}{8}k_1 + \frac{3}{8}k_2)$$

$$k_5 = hf(t_n + \frac{5}{6}h, y_n + \frac{1}{2}k_1 - \frac{3}{2}k_3 + 2k_4)$$

$$y_{n+1} = y_n + \frac{1}{6}(k_1 + 4k_4 + k_5) \quad (2.7.3)$$

$$L.T.E. = \frac{1}{30}(2k_1 - 9k_3 + 8k_4 - k_5) \quad (2.7.4)$$

Other Runge-Kutta error estimators have also been developed. Lapidus and Seinfeld (1971) list a number of these.

## 2.7.2 Predictor-Corrector Methods

Predictor-corrector methods have very convenient local truncation error estimates, if the predictor and corrector are of the same order. For example, consider the third order Adams-Bashforth predictor, Adams-Moulton corrector (2.4.2).

$$\text{predictor error}_p = \frac{9}{24} h^4 y^{(4)}(\xi_p)$$

$$t_{n-1} < \xi_p < t_{n+1} \quad (2.7.5)$$

$$\text{corrector error}_c = -\frac{1}{24} h^4 y^{(4)}(\xi_c)$$

$$t_n < \xi_c < t_{n+1} \quad (2.7.6)$$

The exact value of  $y$  at  $t_{n+1}$  is given by:

$$y(t_{n+1}) = y_{n+1}^{\text{PRED}} + \frac{9}{24} h^4 y^{(4)}(\xi_p) \quad (2.7.7)$$

and 
$$y(t_{n+1}) = y_{n+1}^{\text{CORR}} - \frac{1}{24} h^4 y^{(4)}(\xi_c) \quad (2.7.8)$$

$$\text{Thus } y_{n+1}^{\text{PRED}} + \frac{9}{24} h^4 y^{(4)}(\xi_p) = y_{n+1}^{\text{CORR}} - \frac{1}{24} h^4 y^{(4)}(\xi_c) \quad (2.7.9)$$

Assuming the fourth derivative remains fairly constant over the interval

$$y_{n+1}^{\text{CORR}} - y_{n+1}^{\text{PRED}} = \frac{5}{12} h^4 y^{(4)}(\xi) \quad t_{n-1} < \xi < t_{n+1} \quad (2.7.10)$$

$$\text{thus predictor error} = \frac{9}{10} (y_{n+1}^{\text{CORR}} - y_{n+1}^{\text{PRED}}) \quad (2.7.11)$$

$$\text{corrector error} = -\frac{1}{10} (y_{n+1}^{\text{CORR}} - y_{n+1}^{\text{PRED}}) \quad (2.7.12)$$

A possible PECE algorithm is:

- (1) calculate  $y_{n+1}^{PRED}$
- (2) add predictor error (from previous step, assuming little change)
- (3) evaluate derivative
- (4) calculate  $y_{n+1}^{CORR}$
- (5) add corrector error
- (6) evaluate derivative

### 2.7.3 Changing Step Size

Usually the local error is kept below some specified tolerance. If the error is too large (greater than tolerance) or too small (less than say  $\frac{1}{10}$  of the tolerance), the step size is adjusted. One common algorithm is to double the step when the error is too small and halve the step when the error is too large.

Another scheme if the error varies as say,  $h^4$

$$E = Ch^4 \quad (2.7.13)$$

Then if,

$$E_{ACTUAL} = C_1 h_{OLD}^4 > TOLERANCE \quad (2.7.14)$$



If the desired error is say, half the tolerance,

$$E_{DESIRED} = 0.5(TOLERANCE) = C_2 h_{NEW}^4 \quad (2.7.15)$$

Assuming  $C_1 \approx C_2$

$$\frac{E_{ACTUAL}}{E_{DESIRED}} = \left( \frac{h_{OLD}}{h_{NEW}} \right)^4 \quad (2.7.16)$$

Thus the new step size is chosen as:

$$h_{NEW} = h_{OLD} \left[ \frac{E_{DESIRED}}{E_{ACTUAL}} \right]^{1/4} \quad (2.7.17)$$

$$= h_{OLD} \left[ \frac{0.5(TOLERANCE)}{E_{ACTUAL}} \right]^{1/4} \quad (2.7.18)$$

#### 2.7.4 General Techniques

Usually relative error is controlled but if the independent variable becomes small (less than 1), it is better to control absolute error.

For a system of equations, the maximum error may be controlled. Gear (1971a) controls the Euclidean norm of the relative local truncation errors.

There are many other numerical integration techniques which may or may not have error estimators. An efficient and convenient local error estimate is a

desirable feature of any integration technique, assuming a great deal of computation time is not required for estimation and step-changing.

If the method does not have an estimate, one strategy is to integrate the method using a step  $h$  and then repeat the integration for two steps each of  $\frac{1}{2}h$ . By Richardson extrapolation, an expression for the local truncation error is (Carnahan, Luther and Wilkes, 1969):

$$L.T.E. = \frac{2^p (y_{n+1,2} - y_{n+1,1})}{2^p - 1} \quad (2.7.19)$$

where  $p$  is the order of the method

$y_{n+1,1}$  is the value calculated by step  $h$

$y_{n+1,2}$  is the value calculated by 2 steps of  $\frac{h}{2}$

## 2.8 Accuracy Versus Stability

For a stable numerical solution, conventional methods for solving o.d.e.s require that  $|h\lambda|$  for each eigenvalue be less than a number of the order 1 to 10 and hence the step size is restricted to be not much greater than the smallest time constant present. Ordinarily this restriction on  $h$  does not control the step size since the accuracy requirement, or the prescribed bound on the local truncation error, usually forces the step size to satisfy an even more stringent restriction. However, for stiff

equations, the bound on the step size necessary to maintain stability can be three to five orders of magnitude smaller than that necessary to maintain accuracy (Walters, 1972), making it uneconomical to solve stiff systems with standard methods.

Also in stiff systems, some components may be so small that it is not necessary to simulate them accurately to get an accurate solution for the main components. What is required is that the solution be stable.

As a result many numerical methods not subject to the above stability condition have been developed for solving stiff equations. A literature survey of these appears in the following chapter.

### 3. LITERATURE SURVEY OF NUMERICAL METHODS TO SOLVE STIFF SYSTEMS

The first mention in the literature of the stiffness problem was probably Curtiss and Hirschfelder (1952). Since then, the subject has received a great deal of attention, notably in the last five or ten years. This chapter surveys the methods which have been proposed to handle stiffness.

#### 3.1 Conventional Methods

Conventional methods are restricted to a very small step size of the order  $|h\lambda|$  less than 1 to 10. If a conventional method is to be chosen, it is reasonable to use one which requires the least work per time step, since they will use roughly the same step size and they will all be extremely accurate with such a small step. This method is the Euler method which has been recommended by Franks (1971, 1972b). However Euler's method will still use a great deal of computer time in solving stiff problems as shown in Chapter 6.

### 3.2 Pseudo Steady State Approach

Franks (1972a) suggests that the proper approach to stiff problems may be to simply eliminate those differential equations having small time constants and solve them as algebraic equations instead. This is generally known as the pseudo steady state approach.

Snow (1966) developed a computer program to calculate the product distribution in any homogeneous reaction mechanism. The program includes a numerical method to apply the steady state assumption when a mechanism involves intermediates present in low concentrations.

Brayton, Gustavson and Liniger (1966) analyze the transient behaviour of a transistor circuit. They reduce the original system of differential equations by setting some small resistors equal to zero, thereby eliminating the fast components. They point out that the new steady state differs from the original one. The error is roughly of the order of the ratio of the resistors neglected to those retained. For some problems this accuracy might be sufficient.

The pseudo steady state approach does introduce some large errors initially in some components. It is possible for these errors to be propagated throughout the solution so it is risky to use this procedure (Emanuel, 1967; Kolbrack, 1967; Snow, 1967; Gelinas, 1972). Also

with a complex nonlinear system, it may be very difficult or even impossible to eliminate the small time constants.

Perhaps future research will find the conditions under which the pseudo steady state approach is applicable and the amount of error incurred.

### 3.3 Stiff Techniques

In the past decade, many new sophisticated methods have been developed to overcome the instability problem.

The first major attempt to survey stiff techniques was Bjurel *et al.* (1970). There have been many review papers on a smaller scale (Lapidus and Seinfeld, 1971; Walters, 1971; Sigurdsson, 1970; Gear, 1969b; Steeper, 1970; Calahan, 1969; Hull, 1969).

This survey is not as detailed as that of Bjurel *et al.* but it extends the literature to early 1974, but since this is an area of current research, it should soon be obsolete.

Categorizing the methods is difficult since some methods would occur in more than one category; here we use several rough categories. Methods which are actually tested in Chapter 6 are explained in greater detail.

### 3.3.1 Extension Of Conventional Methods

Many authors have slightly extended the region of stability of conventional methods. Some of these are mentioned here.

Lawson (1966, 1967b) has developed fifth and sixth order Runge-Kutta formulae with extended ranges of stability.

Schoen (1971) develops fifth and sixth order predictors for use with Adams-Moulton correctors of the same order. The resulting PECE algorithms have larger regions of absolute stability than the Adams algorithms of corresponding order.

Emanuel (1964) examines the interaction between a stiff equation and several common integration procedures. He derives a convergence condition for the Runge-Kutta method which can be used to control the integration step size. He also introduces a technique for maximizing the step size and applies it to the generalized Adams predictor-corrector procedure.

Robertson (1966) combined linearity with a free parameter in the fourth order Simpsons Rule and the third order Adams-Moulton equation. For certain values of the parameter, he obtained stability regions superior

to those of either method (Sigurdsson, 1970).

### 3.3.2 Explicit Methods

Treanor (1966) has proposed a modified Runge-Kutta method. He observed that many stiff systems are of the form  $\dot{y} = -P(y-s)$ , where  $P$  is a large number and  $s$  is a slowly varying function of time.  $s$  can be approximated by a power series in time containing unknown parameters which are determined in the course of the integration. In particular, the method assumes that the derivative can be approximated in any interval by:

$$\dot{y}_i = -(P_i)_n y_i + (A_i)_n + (B_i)_n t + (C_i)_n t^2 \quad (3.3.1)$$

The four constants  $A_i$ ,  $B_i$ ,  $C_i$  and  $P_i$  are evaluated by determining the value of the derivative at four points in the interval  $[t_n, t_n + h]$  and solving for the constants from evaluation of the above equation at the four points. The four points are  $t_n$ ,  $t_n + \frac{1}{2}h$ ,  $t_n + \frac{1}{3}h$  and  $t_n + h$ .

The following algorithm is obtained:

$$\begin{aligned} y_{n+\frac{1}{2}}^{(1)} &= y_n + \left(\frac{h}{2}\right) f_n \\ y_{n+\frac{1}{2}}^{(2)} &= y_n + \left(\frac{h}{2}\right) f_{n+\frac{1}{2}}^{(1)} \\ y_{n+1}^{(3)} &= y_n + h \left[ 2f_{n+\frac{1}{2}}^{(2)} F_2 + f_{n+\frac{1}{2}}^{(1)} PhF_2 + f_n (F_1 - 2F_2) \right] \end{aligned} \quad (3.3.2)$$



$$y_{n+1} = y_n + h f_n F_1 + h v_3 (P y_n + f_n) + h v_2 (P y_{n+1} + f_{n+1}^{(1)}) \\ + h v_2 (P y_{n+1} + f_{n+1}^{(2)}) + h v_1 (P y_{n+1} + f_{n+1}^{(3)})$$

where

$$F_1 = \frac{e^{-Ph} - 1}{-Ph}, \quad F_2 = \frac{e^{-Ph} - 1 + Ph}{(Ph)^2}$$

$$F_3 = \frac{e^{-Ph} - 1 + Ph - \frac{1}{2}(Ph)^2}{(Ph)^3}$$

(3.3.3)

$$v_1 = -F_2 + 4F_3$$

$$v_2 = 2(F_2 - 2F_3)$$

$$v_3 = 4F_3 - 3F_2$$

$$P_i = \frac{f_{i,n+1}^{(2)} - f_{i,n+1}^{(1)}}{y_{i,n+1}^{(2)} - y_{i,n+1}^{(1)}}$$

The value of  $P$  is taken to be the largest value of  $P_i$ ; if this value is negative,  $P$  is set to zero.

For small  $h$ , the method is identical to the fourth order Runge-Kutta method (Eqn. 2.3.4).

The step size control is to bound  $\left| \frac{y_{n+1} - y_n}{y_{n+1}} \right|$  at each step between an upper and lower tolerance. Double-halving is used.

Jung (1967) has written a modified version of Treanor's method.

Lomax and Bailey (1967) have analyzed the method of Treanor and shown that it is efficient if there is only one large real negative eigenvalue or if all the large negative eigenvalues are close to each other and real. They also suggest some improvements of the method.

Richards, Lanning and Torrey (1965) devised a scheme for large stiff systems based on Euler's method. The method is based on two qualitative observations:

- (1) It is possible to take a large integration step even with Euler's method, if the value of  $y$  is sufficiently close to the desired trajectory.
- (2) The onset of instability is indicated by sudden reversals of the derivative.

The algorithm uses Euler's method with step size  $h = \beta \frac{\| \dot{y}_n \|}{\| y_n \|}$ , where the double bars indicate Euclidean norms.  $\beta$  is a parameter that can be adjusted for stability. Essentially it limits the fractional change in the large components of  $y$ .  $\beta = 0.01$  is a reasonable value for most problems.

If the derivatives change greatly; i.e., if

$$\cos\theta = \frac{\dot{y}_n \cdot \dot{y}_{n+1}}{\|\dot{y}_n\| \cdot \|\dot{y}_{n+1}\|} < -\frac{1}{S} \quad (3.3.4)$$

then the initial value of  $y_{n+1}$  is abandoned and replaced by an interpolated value:

$$\begin{aligned} t_{n+1} &= t_n + S(\Delta t) \\ y_{n+1}^{NEW} &= y_n + S(y_{n+1} - y_n) \\ S &= \frac{\dot{y}_n \cdot (\dot{y}_n - \dot{y}_{n+1})}{\|\dot{y}_{n+1} - \dot{y}_n\|^2} \end{aligned} \quad (3.3.5)$$

Fowler and Warten (1967) derived an explicit second order method for large stiff systems, where the smallest time constant is real.

Consider the differential equation  $\dot{y} = f(t, y)$  with solution  $y_T$ . At each step, let  $y_C$  be the computed solution.  $y_C$  is assumed to be the sum of two functions,  $y_A$ , an asymptotic part and  $y_{PS}$  a perturbation from the asymptote.  $y_A$  is to be determined from present and past values of  $y_T$ . In practice of course,  $y_T$  is not available and the approximations,  $y_C$ , obtained at the previous steps are used in the calculations instead of  $y_T$ .

Assume  $y_A$  and  $y_{PE}$  are of the form:

$$y_A(t + \xi) = y_A(t) + \xi \dot{y}_A(t)$$

$$y_{PE}(t + \xi) = y_{PE}(t) + \frac{e^{\lambda_P \xi} - 1}{\lambda_P} \dot{y}_{PE}(t) \quad (3.3.6)$$

$0 \leq \xi \leq h$ , thus:

$$\begin{aligned} y_C(t + \xi) &= y_A(t + \xi) + y_{PE}(t + \xi) \\ &= y_A(t) + \xi \dot{y}_A(t) + y_{PE}(t) + \frac{e^{\lambda_P \xi} - 1}{\lambda_P} \dot{y}_{PE}(t) \end{aligned}$$

Five conditions are imposed to determine the five constants  $y_A(t)$ ,  $y_{PE}(t)$ ,  $\dot{y}_A(t)$ ,  $\dot{y}_{PE}(t)$ ,  $\lambda_P$ . The first three of these make the method second order exact and the last two determine the constants uniquely:

$$(1) \quad y_A(t) + y_{PE}(t) = y_T(t)$$

$$(2) \quad \dot{y}_A(t) + \dot{y}_{PE}(t) = \dot{y}_T(t)$$

$$(3) \quad \lambda_P \dot{y}_{PE}(t) = \ddot{y}_T(t) \quad (3.3.7)$$

$$(4) \quad y_A(t) = y_T(t)$$

$$(5) \quad \dot{y}_A(t) = [y_T(t) - y_T(t - h_0)]/h_0$$

There are several possible modifications under certain conditions and a complex step size control.

Nigro (1969) uses multistep formulae of the form:

$$y_{n+1} = \sum_{i=1}^3 \alpha_i y_{n+1-i} + h \sum_{i=1}^3 \beta_i \dot{y}_{n+1-i} \quad (3.3.8)$$

He constructs arbitrarily large stability intervals,  $h\lambda_{max}$ , by choosing appropriate values of the  $\alpha$  and  $\beta$ . By perturbing the  $\alpha$  and  $\beta$ , larger relative stability intervals can be created.

The method is not self-starting; it needs three calculated points before it can take over. The step size is fixed, as there is not a good error or step changing analysis.

Lomax (1968) proposed some second order explicit Runge-Kutta methods which extended the region of stability considerably.

Pope (1963) proposes the explicit difference equation:

$$y_{n+1} = y_n + h f_n + \frac{h^2}{2} f_y^2 [\exp(hf_y) - 1 - hf_y] \quad (3.3.9)$$

Kubicek (1975) has developed a complex one-step integration method, based on boundary value technique. The algorithm is nonlinear, explicit and second order. Kubicek also claims it is A-stable.

Chu and Berman (1974) describe an explicit complex single-step method. It is second order.

### 3.3.3 Implicit Multistep Methods

Klopfenstein and Davis (1971) present a study of a class of PECE algorithms consisting of an application of a predictor followed by application of one iteration of a pseudo Newton-Raphson method to a corrector.

They use algorithms of the form:

$$p_{n+1} = y_n + h [\alpha f_n + \beta f_{n-1}]$$

$$c_{n+1} = y_n + h [v f(t_{n+1}, p_{n+1}) + u f_n] \quad (3.3.10)$$

$$y_{n+1} = p_{n+1} + [\alpha I - v h J]^{-1} [c_{n+1} - p_{n+1}]$$

The  $\alpha$ ,  $\beta$ ,  $u$ ,  $v$  and  $\alpha$  are real constants,  $I$  is the identity matrix and  $J$  approximates the Jacobian.

The algorithm with  $\alpha = 1-\beta$ ,  $v = 1-u$ ,  $u < \frac{1}{2}$  is first order with a first order error estimate:

$$e_{n+1} = p_{n+1} - y_{n+1} + \frac{\beta + \frac{1}{2}}{\beta - u + 1} (c_{n+1} - p_{n+1}) \quad (3.3.11)$$

$\alpha = 1-u$  will shift the stability region so that the centre of an inscribed circle is at the origin.

The algorithm with  $\alpha = \frac{3}{2}$ ,  $\beta = -\frac{1}{2}$ ,  $v = u = \frac{1}{2}$  is second order with first order error estimate:

$$e_{n+1} = p_{n+1} - y_{n+1} + \frac{5}{6} (c_{n+1} - p_{n+1}) \quad (3.3.12)$$

$\alpha = 0.71$  will shift the stability region to the origin.

There are variants of the  $\theta$  method, known for many years:

$$y_{n+1} = y_n + h[(1 - \theta)\dot{y}_{n+1} + \theta \dot{y}_n] \quad (3.3.13)$$

Liniger and Willoughby (1970) and Brandon (1972) discuss methods of this type. See Appendix B for an analysis of Brandon's method. Zein and Hakimi (1971) use Liniger and Willoughby's method with Newton-Raphson iteration to solve circuit problems.

Sandberg and Shichman (1968), Shichman (1969) recommend taking one Newton-Raphson iteration of the implicit Euler method with  $y_n$  used as the estimate of  $y_{n+1}$ :

$$y_{n+1} = y_n + h \dot{y}_{n+1}$$

$$y_{n+1} = y_n + \left[1 - h \frac{\partial f}{\partial y} \Big|_{y=y_n}\right]^{-1} [h \dot{y}_n] \quad (3.3.14)$$

A rough error estimate and step control scheme is given. Sloate (1970) uses an extension of this method.

Hodgkins (1969) presents an algorithm based on the trapezoidal rule in conjunction with global extrapolation.

Lindberg (1971a,b) gives a simple smoothing procedure which damps out the oscillation of the trapezoidal rule applied to stiff systems.

The FACE program (Price, 1970) uses a simple, first order, one-step algorithm, known as a linear extrapolation method. There is provision for using a special integrator which permits fast portions of the problem to be run with a reduced step size. This is an approximation which assumes the inputs from the main portion of the problem are constant during a normal time step.

Curtiss and Hirschfelder (1952), Krogh (1971, 1973) and Gear (Appendix A) have recommended the numerical differentiation formulae for stiff systems. Cryer (1972) discusses a stability theorem for the formulae. Calahan (1971) also discusses Gear's method. Dill and Gear (1971) used an interactive computer graphics program to find stiffly stable methods of orders seven and eight.

Branin et al. (1971) have developed a version of the numerical differentiation algorithm different from Gear's which they claim is more stable than his version under certain conditions.

Klopfenstein (1971) generalizes the numerical differentiation formulae of orders one through six to increase the angular width of the wedge of stability at only modest cost in increased local truncation error.



Bickart and Picel (1973) derive some high order stiffly stable composite multistep methods.

Cooke (1972, 1973) analyzes stiff stability.

Williams and de Hoog (1974) derive a class of A-stable advanced multistep methods with order up to ten. It compares favourably with Gear's method.

Brunner (1972) derives linear k-step methods ( $k \geq 2$ ) with constant coefficients by choosing as the second characteristic polynomial of the method, a Schur polynomial whose coefficients depend on a certain set of parameters. For the case  $k = 2$ , the corresponding two-step methods are always A-stable.

Bjorel (1969, 1972) derives implicit and explicit Adams-like multistep formulae for equations of the type  $P\dot{y} = f(t, y)$  where  $P$  is a polynomial with constant coefficients and  $|\partial f/\partial y|$  is small compared with the roots of  $P$ .

Ehle (1968) shows how one can obtain one-step methods of arbitrarily high order which are A-stable. These are n-stage implicit Runge-Kutta processes of order  $2n$  and a generalized class of linear one-step methods of the form:

$$y_{n+1} = y_n + \sum_{i=1}^j \alpha_{ij} h^i [(-1)^{i+1} y_{n+1}^{(i)} + y_n^{(i)}]$$

( $j=1, 2, 3, \dots$ )

(3.3.15)

Lindberg (1972) describes the package IMPEX which consists of two procedures for the treatment of the transient phase, and two algorithms, one of order two and one of order four; for computation of the smooth solution after the transient.

The Lipschitz constant at the starting point is estimated. From this estimate, a starting step size for the calculation of the transient phase is computed. Throughout the transient phase, a fourth order Runge-Kutta method is used. When the transients are negligible, the variation of the solution is examined in order to get a reasonable starting step size for the main program. The smooth solution is computed by the implicit midpoint rule with second order smoothing and fourth order extrapolation (Bulirsch and Stoer, 1966). The step size is controlled using an estimate of the global truncation error.

IMPEX is the only program found containing different algorithms for both the transient and smooth phases. However, in a practical simulation, there may be transients constantly arising from system disturbances. IMPEX assumes the transients arise only at the beginning of the simulation.

Zavorin and Khesina (1974) have developed three A-stable third order single-step methods.

Sloate (1971), Bickart, Burgess and Sloate (1971) and Sloate and Bickart (1973) use two simultaneous linear multistep difference equations to integrate stiff systems. In this way Dahlquist's limitation that no A-stable linear multistep formula can exceed order two is bypassed. They present a fourth order A-stable method.

Davison (1973) has devised a single-step implicit algorithm for integrating very large stiff differential equations of the type:

$$\dot{y} = Ay + Bu + f(t, y) \quad (3.3.16)$$

where  $f(t, y)$  has a small Lipschitz constant.

Roe (1967) describes a four-step, predictor-corrector algorithm which is derived by fitting the formula to an exponential plus a quadratic. The method works best if the output versus time is exponential-like and degenerates to the Adams-Moulton method, if the response is not similar to an exponential.

Guderley and Hsu (1972) studied a predictor-corrector method for systems of the form:

$$\dot{y} + Dy = -Ay + Bt \quad (3.3.17)$$

where  $D$  is a diagonal matrix which may have some large

elements and the right hand side is considered as non-stiff. The operator on the left is inverted and the right hand side is approximated by Lagrangian interpolation polynomials at grid points. The integration of the exponential functions is done analytically.

Norsett (1969) gives new finite difference formulae which are exact for the problem  $\dot{y} = Py + Q(t)$  where  $P$  is a constant and  $Q(t)$  is a polynomial of degree  $q$ . When  $P = 0$ , the method is identical with the Adams-Bashforth formulae.

Stineman (1965) used an implicit one-step method for the integration of first and second order equations of the type:

$$a(t) \ddot{y} + \dot{y} = b(t)$$

(3.3.18)

$$c(t) \ddot{y} + d(t) \dot{y} + y = e(t)$$

where the functions  $a$ ,  $b$ ,  $c$ ,  $d$  and  $e$  are locally approximated by linear functions.

Sarkany and Ball (1969) treat the solution of  $\dot{y} = -Dy + f(t, y)$  where  $D$  is a diagonal matrix. A predictor-corrector method is used.

Spicer (1968, 1969) defines exact stability as the extraneous roots being constrained to be zero. He derives

predictor-corrector formulae having this property.

Certaine (1960) has developed an iterative multi-step method for solving systems of the form  $\dot{y} + Dy = f(t, y)$  where  $D$  is a constant diagonal matrix with large positive elements and  $f(t, y)$  is a slowly varying function of time.

Snider and Fleming (1974) use aliasing to reduce the computation of finding an accurate trigonometric interpolation for a function with dominant high frequencies. The technique is applied to stiff differential equations, extending the applicability of the method of Certaine to systems with oscillatory forcing functions.

### 3.3.4 Runge-Kutta Methods

The basic Runge-Kutta equation is explicit:

$$y_{n+1} = y_n + \sum_{i=1}^v w_i k_i \quad (3.3.19)$$

$$k_i = hf(x_n + c_i h, y_n + \sum_{j=1}^{i-1} a_{ij} k_j) \quad c_1=0, i=1, 2, \dots, v$$

Butcher (1964) introduced implicit Runge-Kutta methods. Ehle (1968) and Gear (1970) discuss these

further:

$$k_i = hf(x_n + c_i h, y_n + \sum_{j=1}^v a_{ij} k_j) \quad c_1=0, i=1, 2, \dots, v \quad (3.3.20)$$

Rosenbrock (1963), Calahan (1968b), Allen and Pottle (1968), Allen (1969), Haines (1969) and Caillaud and Padmanabhan (1971) introduce semi-implicit methods of the form:

$$k_i = hf(x_n + c_i h, y_n + \sum_{j=1}^i a_{ij} k_j) \quad c_1=0, i=1, 2, \dots, v \quad (3.3.21)$$

The implicit and semi-implicit forms are A-stable.

Chipman (1971) defines the concept of strong A-stability. He then introduces a class of strongly A-stable Runge-Kutta processes.

Axelsson (1972) gives formulae for a class of A-stable quadrature methods or equivalently for a certain implicit Runge-Kutta scheme. The methods are strongly A-stable. †

Lopes and Phares (1965) and Lawson (1967a) have developed an A-stable Runge-Kutta method. The system must first be transformed by  $z(t) = \exp(-\lambda t)y(t)$ .

Stanton and Talukdar (1970) use a fourth order Runge-Kutta and a complex transformation of variables to increase the stability interval and permit a large step size.

### 3.3.5 Miscellaneous Methods

Jain (1972) gives some A-stable methods of order  $2n$ , with variable coefficients based on Hermite interpolation polynomials, making use of  $n$  starting values.

Axelsson (1969) gives A-stable methods of a quadrature type:

$$y_{n,r} = y_{n,r-1} + h \sum_{k=1}^n a_{ik} f(y_{k,r}) \quad i=1,2,\dots,n \quad (3.3.22)$$

$r=1,2,\dots,y_{n,0}$

where  $a_{ik}$  are quadrature coefficients over the zeros of  $P_n - P_{n-1}$  ( $v=1$ ) or  $P_n - P_{n-2}$  ( $v=2$ ), where  $P_n$  is the Legendre polynomial orthogonal on  $[0,1]$  and normalized such that  $P_n(1) = 1$ .

Dahlquist (1969) devised a method based on local polynomial approximations for two stiff o.d.e.s of particular type. Oden (1971) gives further details of the method.

Watts and Shampine (1972) develop  $r$ -block implicit one-step methods which compute a block of  $r$  new values simultaneously with each set of application. A subclass of formulae is derived which is related to Newton-Cotes quadrature and it is shown that for block sizes  $r=1,2,\dots,8$ , these methods are A-stable, while those for  $r=9,10$  are not. They construct A-stable and strongly A-stable

formulae having arbitrarily high orders of accuracy.

Giloi and Grebe (1968) discuss multistep algorithms and the computation of optimum coefficients. They point out that each problem has its own optimum formula, and z-transform techniques are used to study this. This approach may not be practical for general purpose use, since the optimum formula would be difficult or impossible to determine, and the method is limited to linear equations.

Andrus (1967) uses a quasi-analytical approach which takes the inverse Laplace transform of the system equations, using a summation of the partial fractions. Some approximate techniques for handling nonlinearities are discussed, but this appears to be a limitation.

Liniger and Willoughby (1970) introduced the concept of exponential fitting. They proposed single-step methods of the form:

$$y_{n+1} = \alpha_1 y_n + h[\beta_0 \dot{y}_{n+1} + \beta_1 \dot{y}_n] + h^2[\gamma_0 \ddot{y}_{n+1} + \gamma_1 \ddot{y}_n] \quad (3.3.23)$$

If the characteristic root is chosen by adjusting the free parameters to be  $\exp(q_0)$ , then we say the method is exponentially fitted at  $q_0$ .



Liniger (1968) gives two easy-to-check conditions which together are sufficient and "almost necessary" for A-stability of linear multistep integration formulae.

Liniger (1969) discusses global accuracy and A-stability for one and two-step integration formulae.

Liniger (1971) gives a stopping criterion for the Newton-Raphson method in implicit multistep integration algorithms.

Odeh and Liniger (1972) discuss unconditional fixed step size stability of linear multistep formulae.

Prothero and Robinson (1974) discuss the stability and accuracy of implicit one-step methods to stiff equations of the form:

$$\dot{y} = g(t) + \lambda(y - g(t))$$

They describe some new stability properties and recommend a family of methods based on a compromise between accuracy and stability considerations.

Brunner (1974) uses recursive collocation. He approximates the solution on each subinterval by a linear combination of exponential functions which involve only the significant eigenvalues of the Jacobian. The unknown vectors are computed recursively by requiring they satisfy the given system at certain suitable points (collocation),

with the additional condition that the collection of these functions represent a continuous function satisfying the given initial conditions.

Liniger and Odeh (1972) discuss an algorithm wherein several low-order solutions are computed by repeated integration using a multistep method with parameters. By forming suitable linear combinations of such solutions, higher order solutions are obtained. If the parameters are properly chosen, the underlying solutions, and thus the higher order one, can be made A-stable and strongly damping with respect to the stiff components of the system.

Blue and Gummel (1970) discuss rational function approximations for the matrix exponential  $\exp(h\lambda)$ .

The use of Padé approximations for  $\exp(h\lambda)$  is discussed by Calahan (1967), Jung (1968) and Osborne (1969) for linear systems.

Cooper (1969) gives a brief review of stiffness, and an iteration procedure likely to give convergence, both in multistep methods and in the steady state approach.

Miller (1964) gives a second order stiff equation and suggests a scheme for solving it. Cash (1972) extends this method and develops an entirely new algorithm for differential equations of order three and higher, to enable these equations to be integrated with a reasonable step length.

Miranker (1971) gives a description and analysis of a class of matricial difference schemes. This class of schemes is based in part on a generalization of the feature of classical numerical methods of being characterized by approximations at a single point in the complex plane.

The following four methods are restricted to linear systems or systems which can be linearized easily.

Mah, Michaelson and Sargent (1962) developed a stable linearization technique for solving the dynamic behaviour of multistage systems. The coefficients of the differential system are taken as constant over a short time interval. The exact solution of the linearized system is used to predict the values at the end of the interval and hence the coefficients for the next interval.

Buffam and Kropholler (1969) extend the concept of network combing to give the concentration history at all the nodes in a flow network whose nodes have exponential dynamic mixing characteristics with arbitrary time constants. They show that the transition matrix for the flow network is a special form of stochastic matrix whose power series has absolute convergence. The generalization results in a very stable method for integrating linear systems of o.d.e.s.

Davison (1968) has considered the problem of the numerical integration of large systems of constant

coefficient linear o.d.e.s. The poles and zeros of the solution are obtained and the solution constructed in terms of a sum of exponentials.

Moretti's (1965) technique requires nonlinear o.d.e.s to be reduced to linear form. The numerical solution then requires the evaluation of the complex eigenvalues of an n-th order matrix, thus limiting the analysis to fairly small systems. DeGroat and Abbett (1965) and Maheshwari (1968) have extended this approach.

The following four methods use state transition matrices.

Liou (1966) has devised a fast and accurate method. However, it is for linear systems only; one output is available and the problem must be formulated as a ratio of two polynomials in  $s$  (the Laplace operator). Nichol (1970) has a similar method, but simple nonlinearities can be handled.

Giесе (1967) also uses state variable techniques and discusses their use in quasi-linear systems. He also discusses a combination approach for stiff equations, where this method is used for the fast portion of the problem and a conventional explicit numerical procedure is used for the slow portion.

Oswald and Smith (1970) also use state transition matrices in their method.

Lee (1967) uses matrix filters to separate the differential equations into slow and fast equations. These are then computed separately and the results are added together. It is limited to linear equations.

Allen and Fath (1969) propose techniques for systems which can be separated into a stiff linear part and a nonstiff linear part. Padé approximations are used.

Jain (1971) transforms the system by  $z(t) = \exp(At)y(t)$ , where  $A$  is a real square matrix and obtains modified Adams and linear multistep methods which are  $A$ -stable. Jain gives a working algorithm to obtain  $A$  and to calculate  $\exp(hA)$ .

Shannon (1971) developed a technique using a changeable independent variable of integration. It can be used with any integration method. Appendix C contains an analysis of the technique.

Singular perturbation theory is applicable when the system can be written as:

$$\epsilon \dot{y}_1 = f_1(t, y_1, y_2)$$

$$\dot{y}_2 = f_2(t, y_1, y_2)$$

(3.3.24)

where  $\epsilon$  is a small real parameter and the  $y_2$  components are stable in the sense that if  $y_2$  is fixed at

any time, then the  $y_i$  components rapidly reach an equilibrium solution.

MacMillan (1968) proposes a method based upon this theory.

Miranker (1973) describes an equivalence between a subclass of stiff systems and differential equations subjected to singular perturbations. He uses the characterization of the solution of this class of equations in terms of boundary layers as a means of generating numerical procedures for solving the stiff equations. The numerical procedures have the desirable feature of improving with increasing stiffness.

Diperna (1971) describes a digital simulation method for interconnected continuous systems.

#### 3.4 Methods Used In Dynamic Simulation Executive Packages

As mentioned in Chapter 1, there have been several new problem-oriented executive packages developed in the past five years. Almost all of these programs contain an algorithm for the numerical solution of ordinary differential equations; i.e., the system model contains ordinary differential equations. Table 3.1 summarizes some of the numerical methods used.

Table 3.1: Numerical Methods Used in Dynamic Simulation Packages

Package	Method Used To Integrate Differential Equations	Suited For Stiff Eqns.
MIMIC	4th order Runge-Kutta	No
CSMP	various conventional methods, latest version contains Gear's method	Yes
IMP	Brandon's method	Yes
KARDAZ	explicit finite difference method used to solve a set of quasilinear hyperbolic partial differential equations	No
DYNSYS	Adams-Moulton-Shell	No
PRODYC	any method in CSMP	Yes
REMUS	16 conventional options: Simpson's Rule, various Runge-Kutta, Milne and Adams-Moulton formulae	No
ACME	2nd and 4th order Runge-Kutta, Euler's method	No
DYFLO	2nd and 4th order Runge-Kutta, Euler's method	No
OSUSIM	Euler predictor, trapezoidal rule as corrector	Yes

MIMIC (Northcott, 1967) uses the fourth-order Runge-Kutta method.

CSMP (CSMP, 1967) offers a variety of conventional methods, but the latest version contains Gear's method.

IMP (Appendix B) offers Brandon's method, a new A-stable method very suitable for stiff equations.

Franks (1971) has made a detailed review of the packages DYNYSYS, PRODYC, REMUS, DYFLO and OSUSIM.

DYNYSYS (Bobrow, Johnson and Ponton, 1970, 1971) uses the Adams-Moulton-Shell method (Shell, 1958; Fairchild, Wengrow and May, 1965).

PRODYC (Ingels and Motard, 1970) has a preprocessor which converts the system model into CSMP statements after which any of the methods available in CSMP (CSMP, 1967) can be used to solve the equations.

REMUS (Ham, 1969) offers sixteen conventional options: Simpson's Rule, and various Runge-Kutta, Milne and Adams-Moulton formulae.

DYFLO (Franks, 1972) offers a choice of simple Euler or second or fourth order Runge-Kutta.

OSUSIM (Koenig, 1972) uses an Euler predictor and a trapezoidal rule as corrector. OSUSIM emphasizes the formation and solution of implicit and explicit algebraic equations.

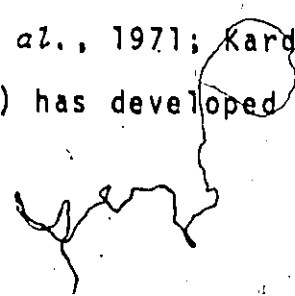


ACME (Loibl, Camp and Wilkins, 1973) is a new modular package. It simulates both the steady state and dynamic behaviour of chemical engineering systems. It uses several options for solving o.d.e.s: second and fourth order Runge-Kutta and Euler's method.

The packages DYNYSYS, REMUS, PRODYC, OSUSIM and ACME basically use the modular approach. All of these executives are highly structured and have their own input language which the user must master. The author has not noticed any particular advantage of any of these, but Franks claims DYFLO is the most convenient program from the user's point of view. DYFLO is not exactly modular; it is somewhere in between the modular approach and the equation-oriented approach. The user of DYFLO writes his own computer program using essentially a series of CALL statements to utilize a rather large number of service subroutines that Franks has provided. Culver (1972) used DYFLO to study distillation column dynamics and he attests to its convenience.

The IMP package requires a good deal of sophistication on the part of the user to set up the system equations for simulation and to select the best options for a particular study.

Kardasz (Brambilla *et al.*, 1971; Kardasz, 1969; Kardasz and Molnar, 1971, 1974) has developed a package



for the simulation of the dynamic behaviour of chemical plants. It is quite different than the other packages just discussed. The language used is not FORTRAN but is based on SIMULA-67. It is structure-oriented according to the flowsheet of the plant to be simulated. The method of simulation is based on the concept of a cell model. Under the assumption of a one-dimensional model, a chemical plant composed of a set of devices can be represented mathematically by a set of quasilinear hyperbolic partial differential equations of first order. After appropriate discretization, each device is subdivided into a finite number of cells. An explicit finite difference method is used to solve the equations.

Other attempts have been made to form dynamic simulation packages. Utsumi (1969) describes the SWAPSO package developed at Stone & Webster, but it has not been pursued (Crean, 1974). A package called EARLYBIRD has been developed at Tulane University but the dynamic simulation portion of it has been discontinued (Weaver, 1974). Procter & Gamble (Shern and Petty, 1970) has a modular package called FLEX. It uses Euler's method for numerical integration.

As seen in Table 3.1, only several of the packages contain a numerical method suitable for stiff o.d.e.s. The latest version of CSMP contains Gear's method. The

IMP package contains Brandon's method. OSUSIM uses the trapezoidal rule which is A-stable. An equation-oriented package, FORSIM (Carver, 1972, 1973) developed at AECL uses Gear's and the Fowler-Warten method. Franks (1971, 1972b) recommends Euler's method for stiff systems.

However, except for IMP, none of the packages are oriented to solving large systems of stiff o.d.e.s. In Chapter 6, we shall see a comparison of some of these techniques.

#### 4. CONVERGENCE OF IMPLICIT METHODS

Of the methods developed to integrate stiff systems, implicit methods are usually preferred, but particular problems arise in solving the corrector or implicit equation, especially for large stiff systems.

Consider the general linear multistep formula:

$$y_{n+1} = \alpha_1 y_n + \alpha_2 y_{n-1} + \dots + \alpha_k y_{n+1-k} + h[B_0 \dot{y}_{n+1} + B_1 \dot{y}_n + \dots + B_k \dot{y}_{n+1-k}] \quad (4.1)$$

or

$$y_{n+1} = \sum_{i=1}^k \alpha_i y_{n+1-i} + h \sum_{i=0}^k B_i \dot{y}_{n+1-i} \quad (4.2)$$

$$y_{n+1} = h B_0 \dot{y}_{n+1} + \sum_{i=1}^k (\alpha_i y_{n+1-i} + B_i \dot{y}_{n+1-i}) \quad (4.3)$$

Since the last term is known, we can replace it by  $w_n$ :

$$y_{n+1} = h B_0 f(t_{n+1}, y_{n+1}) + w_n \quad (4.4)$$

$$y_{n+1} - h B_0 f(t_{n+1}, y_{n+1}) - w_n = 0 \quad (4.5)$$

From the above equation we wish to determine  $y_{n+1}$ .  
 For a system of  $m$  o.d.e.s, this is equivalent to solving  $m$  nonlinear equations. Several common methods of doing this are outlined in this chapter, but this is an ongoing area of research and perhaps better methods will be found.

#### 4.1 Jacobi Iteration

The obvious way of solving this is by repeated substitution:

$$y_{n+1}^{(s+1)} - hB_0 f(t_{n+1}, y_{n+1}^{(s)}) - w_n = 0 \quad (4.1.1)$$

until  $y_{n+1}^{(s+1)}$  is equal to  $y_{n+1}^{(s)}$  to the degree of accuracy desired.

This approach, called Jacobi iteration, is commonly used for nonstiff o.d.e.s, but ~~for~~ stiff problems, it does not converge unless the step size is very small.

#### 4.2 Accelerated Iteration

A modification of Jacobi iteration is:

$$(1+\alpha)y_{n+1}^{(s+1)} - hB_0 f(t_{n+1}, y_{n+1}^{(s)}) - w_n - \alpha y_{n+1}^{(s)} = 0 \quad (4.2.1)$$

where  $\alpha$  is an acceleration parameter.

This will increase the rate of convergence, but the step size condition for convergence is just as stringent as Jacobi iteration.

#### 4.3 Backward Iteration

A backward iteration with much better convergence properties is:

$$y_{n+1}^{(s)} - hB_0 f(t_{n+1}, y_{n+1}^{(s+1)}) - w_n = 0 \quad (4.3.1)$$

however, this still requires the solution of nonlinear equations.

#### 4.4 Newton-Raphson Iteration

The most commonly used method is Newton-Raphson iteration.

$$\text{For } g(y_{n+1}) = y_{n+1} - hB_0 f(t_{n+1}, y_{n+1}) - w_n = 0 \quad (4.4.1)$$

The Newton-Raphson formula is:

$$y_{n+1}^{(s+1)} = y_{n+1}^{(s)} - \left[ \left( \frac{\partial g}{\partial y} \right)_{n+1}^{(s)} \right]^{-1} g_{n+1}^{(s)} \quad (4.4.2)$$

$$y_{n+1}^{(s+1)} = y_{n+1}^{(s)} - \left[ 1 - hB_0 \left( \frac{\partial f}{\partial y} \right)_{n+1}^{(s)} \right]^{-1} [y_{n+1}^{(s)} - hB_0 f_{n+1}^{(s)} - w_n] \quad (4.4.3)$$

The theoretical convergence condition is quite conservative, but in practice, the method converges for fairly large  $h$  and does not slow down the integration process.

$y_{n+1}^{(0)}$  can be a predicted value or if no predictor equation is used, it can be taken as the previous value,

$y_n$ .

Note that if the term to be inverted is singular, the step size  $h$  can always be reduced to make it non-singular.

#### 4.5 Quasilinearization

The IMP package (Appendix B) uses a method called quasilinearization.

For a single equation:

$$\dot{y} = f(t, y) \quad (4.5.1)$$

Expand the right hand side in a Taylor Series about the present point,

$$\text{i.e. } \dot{y}_n = f(t_n, y_n) + \left. \frac{\partial f}{\partial y} \right|_n (y - y_n) + \dots \quad (4.5.2)$$

$$\dot{y}_n = \left. \frac{\partial f}{\partial y} \right|_n y + f(t_n, y_n) - \left. \frac{\partial f}{\partial y} \right|_n y_n \quad (4.5.3)$$

$$\dot{y}_n = J y + B \quad (4.5.4)$$

For a system of equations,  $J$  is the Jacobian matrix, while  $B$  is called the augmented constant vector.

When the system is linearized in this way, we no longer have to solve a set of nonlinear equations but a set of linear equations at each time step.



#### 4.6 Large Stiff Systems

Two useful methods for solving stiff corrector equations are Newton-Raphson and quasilinearization. Whichever of the two is used, a set of linear algebraic equations equal in size to the number of o.d.e.s must be solved.

Lapidus and Seinfeld (1971) recommend Jacobi or accelerated iteration if the stiffness ratio is of the order of 10. If it is greater than the order of 10, they recommend Newton-Raphson or backward iteration with the step size selected on the basis of the number of iterations desired per step.

In quasilinearization, the linear equations arise directly from the Taylor Series linearization of the system.

In Newton-Raphson iteration, we must evaluate the term:

$$\left[ I - hB_0 \left( \frac{\partial f}{\partial y} \right)_{n+1}^{(e)} \right]^{-1} \left[ y_{n+1}^{(e)} - hB_0 f_{n+1}^{(e)} - w_n \right] \quad (4.6.1)$$

In general this means solving a set of linear equations :

$$A X = b \quad (4.6.2)$$

where

$$A = [I - hB_0 \left( \frac{\partial f}{\partial y} \right)_{n+1}^{(s)}] \quad (4.6.3)$$

$$b = [y_{n+1}^{(s)} - hB_0 f_{n+1}^{(s)} - w_n] \quad (4.6.4)$$

and  $x = A^{-1}b \quad (4.6.5)$

This set of linear equations must be solved for each corrector equation.

For large systems of o.d.e.s, the solution of this set of linear equations can require a large amount of computer time.

The matrix of coefficients is usually quite sparse; i.e., there are many zero elements. A method which takes advantage of the sparsity by storing and operating on only the nonzero elements will reduce the amount of computer time appreciably. Therefore we will examine ways of solving sparse sets of linear equations in the following chapter.

## 5. NUMERICAL SOLUTION OF SPARSE LINEAR ALGEBRAIC EQUATIONS

We saw in Chapter 4 that an implicit numerical integration technique for stiff o.d.e.s requires the solution of a set of linear algebraic equations equal in size to the number of o.d.e.s. For large systems of o.d.e.s, the solution of this set of linear equations can use a large amount of computer time.

The coefficient matrix of the linear equations is a function of the Jacobian matrix for the system. In general, the Jacobian matrix has many zeros so that using a method which stores and operates on only the nonzero elements will reduce the computer time greatly. Such methods are called sparse matrix techniques.

There are two major ways of solving linear equations: direct elimination techniques such as Gaussian elimination, and iterative methods such as Jacobi and Gauss-Seidel. Traditionally, iterative techniques have been used to handle sparse systems (Varga, 1962; Young, 1971); however, in recent years, elimination methods have been more commonly used (Reid, 1971; Rose and Willoughby, 1972; Westlake, 1968; Willoughby, 1969). Decomposition techniques (Himmelblau, 1973) where the system is broken into two or more subsystems have also been developed, but these require

extensive a priori analysis.

Several conventional methods and some sparse techniques were tested on a set of banded test examples. We will assume that our "sparse" equations contain at most 15 nonzeros per equation.

## 5.1 Methods Tested For Solving Linear Algebraic Equations

### 5.1.1 Conventional Methods

Five conventional methods were studied:

- |                  |  |
|------------------|--|
| (1) MINV         | An SSP matrix inversion algorithm<br>(SSP, 1970)                         |
| (2) SOLVE        | Gaussian elimination with no pivoting                                    |
| (3) DECOMP-SOLVE | Gaussian elimination with partial<br>pivoting (Forsythe and Moler, 1967) |
| (4) JACOBI       | Jacobi iterative method (initial<br>estimate of zero used)               |
| (5) GAUSS-SEIDEL | Gauss-Seidel iterative method (initial<br>estimate of zero used).        |

### 5.1.2 Key (1973)

Key (1973) has written a program called SIMULT based on Gauss-Jordan elimination with seven pivoting options:

- (1) Simple Gauss-Jordan elimination. Diagonal elements are used in order as pivots.
- (2) Gauss-Jordan partial pivoting. The largest coefficient in absolute value in each column in order is selected as pivot, provided this coefficient is not in a previously selected pivotal row.
- (3) Gauss-Jordan full pivoting. The largest coefficient in the entire coefficient matrix that is not in a previously selected pivotal row or column is used.
- (4) Minimum row-Minimum column. The coefficient matrix is searched to find the row with the least number of nonzero coefficients. Then for the nonzero elements in this row, the column with the most zeros is chosen as the pivotal column. If more than one row or column is chosen, the row or column with the smallest index is used.

- (5) Minimum column-Minimum row. This is similar to the above except that the pivotal column is determined first.
- (6) Maximum column-Minimum row. The column with the most nonzero elements is chosen, followed by the row with the least nonzero elements.
- (7) Minimum of row entries times column entries. The pivot is chosen from the nonzero element with the smallest product of row and column entries.

The storage scheme is illustrated by the following example:

The matrix

$$A = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 \\ 0 & 2 & 0 & 0 & 1 \\ 0 & 2 & 4 & 0 & 0 \\ 1 & 0 & 0 & 2 & 0 \\ 0 & 0 & 2 & 0 & 3 \end{bmatrix} \quad (5.1.1)$$

is stored as

$$A = \begin{bmatrix} 3 & 0 \\ 2 & 1 \\ 2 & 4 \\ 1 & 2 \\ 2 & 3 \end{bmatrix} \quad \text{and } ICOL = \begin{bmatrix} 1 & 0 \\ 2 & 5 \\ 2 & 3 \\ 1 & 4 \\ 3 & 5 \end{bmatrix} \quad (5.1.2)$$

A contains the nonzero elements and *ICOL* the column index of the corresponding elements. The disadvantage of this is that one full row will make the A matrix as large as the original matrix.

### 5.1.3 IMP (1972)

IMP (Brandon, 1972) is a software system developed by D.M. Brandon at the University of Connecticut. It is now offered as a standard CDC software product through the Application Services Division of CDC. An object copy of IMP, Version 1.1, FTN version 4.0 compiler and optimization level 2 was used.

Five options with the IMP package were tested:

- (1) Gauss-Seidel
- (2) Gradient
- (3) Crout elimination, fixed order, written for general sparse matrices
- (4) Crout elimination, variable order, written for general sparse matrices
- (5) Crout elimination, variable order, written for variable or constant banded matrices.

The Crout method is a variation of Gaussian elimination which uses less storage.

In addition, a program supplied by D.M. Brandon (Brandon, 1974c), which is a simplified version of the method (4) above, was used. It should run faster because there is none of the overhead associated with the IMP routines.

The storage scheme known as cumulative indexing is as follows:

The matrix

$$P = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 \\ 0 & 2 & 0 & 0 & 1 \\ 0 & 2 & 4 & 0 & 0 \\ 1 & 0 & 0 & 2 & 0 \\ 0 & 0 & 2 & 0 & 3 \end{bmatrix} \quad (5.1.3)$$

is stored as

$$A = \begin{bmatrix} 3 \\ 2 \\ 1 \\ 2 \\ 4 \\ 1 \\ 2 \\ 2 \\ 3 \end{bmatrix} \quad JHA = \begin{bmatrix} 1 \\ 2 \\ 5 \\ 2 \\ 3 \\ 1 \\ 4 \\ 3 \\ 5 \end{bmatrix} \quad ICUMA = \begin{bmatrix} 1 \\ 3 \\ 5 \\ 7 \\ 9 \end{bmatrix} \quad (5.1.4)$$

where  $A$  is the vector of nonzero entries of  $P$  by rows



$JHA$  is the vector of column numbers of nonzero elements for each row

$ICUMA$  is a cumulative vector which separates  $JHA$  by rows.

#### 5.1.4 Schappelle (1967)

Schappelle (1967) developed a program called LINEQ4 which is available from VIM, the CDC 6000 Series User Association.

The method first performs a systematic rearrangement, called pre-triangularization, of the rows and columns of the coefficient matrix. The equations then appear as:

$$\begin{bmatrix} A & B \\ C & D \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \end{bmatrix} = \begin{bmatrix} C_1 \\ C_2 \end{bmatrix} \quad (5.1.5)$$

where  $A$  is upper triangular,  $B$  and  $D$  are rectangular, and  $C$  is quasi lower triangular meaning no lead element of any row of the matrix is further to the right than the lead element of its preceding row.

$$\begin{aligned} A X_1 + B X_2 &= C_1 \\ C X_1 + D X_2 &= C_2 \end{aligned} \quad (5.1.8)$$

solving for  $X_2$

$$(D - C A^{-1} B) X_2 = (C_2 - C A^{-1} C_1) \quad (5.1.7)$$

$X_2$  is solved by Gaussian elimination with full pivoting

$$A X_1 = C_1 - B X_2 \quad (5.1.8)$$

$X_1$  is solved by back-substitution, since  $A$  is upper triangular.

The solution is corrected by iterating on the residuals.

The storage scheme is as illustrated below:

The matrix

$$P = \begin{bmatrix} 3 & 0 & 0 & 0 & 0 \\ 0 & 2 & 0 & 0 & 1 \\ 0 & 2 & 4 & 0 & 0 \\ 1 & 0 & 0 & 2 & 0 \\ 0 & 0 & 2 & 0 & 3 \end{bmatrix} \quad (5.1.9)$$

is stored as

$$\begin{bmatrix} 1 & 1 & 3 \\ 2 & 2 & 2 \\ 2 & 5 & 1 \\ 3 & 2 & 2 \\ 3 & 3 & 4 \\ 4 & 1 & 1 \\ 4 & 4 & 2 \\ 5 & 3 & 2 \\ 5 & 5 & 3 \end{bmatrix}$$

i.e., row number, column number, and value for each nonzero element in any order.

#### 5.1.5 Bending-Hutchison (1973)

Bending and Hutchison (1973) developed the concept of an "operator list" to store the solution process by Gaussian elimination. The process consists of two stages, triangularization and back-substitution.

##### 5.1.5.1 *Triangularization*

For the system

$$\begin{bmatrix} a_1 & 0 & a_9 & a_{11} \\ 0 & a_6 & 0 & a_2 \\ a_4 & 0 & a_8 & 0 \\ a_5 & 0 & 0 & a_3 \end{bmatrix} \begin{bmatrix} X_1 \\ X_2 \\ X_3 \\ X_4 \end{bmatrix} = \begin{bmatrix} a_7 \\ a_{10} \\ 0 \\ 0 \end{bmatrix} \quad (5.1.10)$$

(1) eliminate  $a_2$ , row 2 = row 2 -  $\frac{a_2}{a_3}$  row 4

$$\begin{array}{cccccc}
 a_1 & 0 & a_9 & a_{11} & a_7 & \\
 a_{12} & a_8 & 0 & 0 & a_{10} & \\
 a_4 & 0 & a_8 & 0 & 0 & \\
 a_5 & 0 & 0 & a_3 & 0 & 
 \end{array}
 \quad a_{12} = 0 - \frac{a_2}{a_3} a_5 = -a_5 \frac{a_2}{a_3}$$

(5.1.11)

(2) eliminate  $a_{11}$ , row 1 = row 1 -  $\frac{a_{11}}{a_3}$  row 4

$$\begin{array}{cccccc}
 a_1^* & 0 & a_9 & 0 & a_7 & \\
 a_{12} & a_8 & 0 & 0 & a_{10} & \\
 a_4 & 0 & a_8 & 0 & 0 & \\
 a_5 & 0 & 0 & a_3 & 0 & 
 \end{array}
 \quad a_1^* = a_1 - a_5 \frac{a_{11}}{a_3}$$

(5.1.12)

(3) eliminate  $a_9$ , row 1 = row 1 -  $\frac{a_9}{a_8}$  row 3

$$\begin{array}{cccccc}
 a_1^{**} & 0 & 0 & 0 & a_7 & \\
 a_{12} & a_8 & 0 & 0 & a_{10} & \\
 a_4 & 0 & a_8 & 0 & 0 & \\
 a_5 & 0 & 0 & a_3 & 0 & 
 \end{array}
 \quad a_1^{**} = a_1^* - a_4 \frac{a_9}{a_8}$$

(5.1.13)

The above process can be represented by a string of integers:

$$\text{New element } a_i = -a_j \frac{a_k}{a_l} \rightarrow -k \ l \ i \ j \quad (5.1.14)$$

$$\text{Element changed } a_i = a_i - a_j \frac{a_k}{a_l} \rightarrow -k \ l \ i \ j \quad (5.1.15)$$

For this example:

-2 3 12 5 -11 3 1 5 -9 8 1 4 0 0 terminates string

#### 5.1.5.2 Back-Substitution

$$X_0 = -1 \quad (\text{dummy variable})$$

$$X_1 = -a_7 X_0 / a_1^{**}$$

$$X_2 = (-a_{10} X_0 - a_{12} X_1) / a_8 \quad (5.1.16)$$

$$X_3 = -a_4 X_1 / a_8$$

$$X_4 = -a_5 X_1 / a_3$$

This process can also be characterized by a string of integers:

$$a_i = -a_j \frac{X_k}{a_l} \rightarrow j \ k \ -i \ l \quad (5.1.17)$$

or

$$a_i = (-a_j X_k - a_m X_n) / a_l \rightarrow j \ k \ m \ n \ -i \ l \quad (5.1.18)$$

For this example:

7 0 -1 1 10 0 12 1 -2 6 4 1 -3 8 5 1 -4 3 0 0

These two strings of integers form what is called the operator list, i.e., the particular solution process by Gaussian elimination. Subroutine TRGB solves the equations for the first time and creates the operator list which can be stored in core or on tape if the system is large. Subroutine TRGB2 can re-solve the system using only the operator list. If the zero elements remain zero and the nonzero elements change, the same operator list can be used to solve the new system. This is usually what occurs during numerical integration. The operator list could be set up on the first integration step by TRGB and used on later steps by TRGB2 to solve each new linear system. The operator list can change during the integration; however, in all examples tested, the original list could be used for the entire integration:

The pivot is chosen by the row and column each with the most zeros. The row is selected first. In case of ties, the largest element is chosen. The storage scheme is row number, column number, value:

## 5.2 Numerical Testing

Banded systems of 50, 100 and 200 linear equations with bandwidths 3, 5, 7, 9, 11 and 15 were used as test systems. The nonzero elements were generated randomly by rows from -10 to +10 using the FTN library random number generator RANF (Control Data, 1971). An initial seed of one was used. The right hand side was adjusted to yield a vector of ones as the solution.

The banded systems were used mainly for convenience; none of the algorithms tested takes advantage of this structure. However, with banded matrices, no new elements will be generated during the solution; thus the testing does not reflect how well the methods handle this problem. The Bending-Hutchison method, however, does handle this easily.

Execution times (See Appendix E) for the various methods are shown in Tables 5.2, 5.3 and 5.4. Table 5.1 provides a key to the methods.

The conventional methods are obviously unsuitable for these problems. Matrix inversion takes much time and Gaussian elimination is quite slow. The iterative methods, Jacobi iteration and Gauss-Seidel iteration both diverged from initial estimates of zero. None of the conventional techniques were able to solve 200 equations as this required more than the 49K of core available.

Of the sparse routines, many gave inaccurate answers. This may be attributed to the fact that the non-zero elements were generated over a wide range, producing badly conditioned systems. If instead the elements are generated from 1-2, these programs work quite well. The execution times were independent of the spread of the non-zero elements.

Of the seven pivoting options given by Key, the best was minimum-row, minimum-column which Key also recommends.

The Bending-Hutchison routine, TRGB2, gave the best performance of the sparse methods. This is almost to be expected since all of the work of determining the solution process has already been done by TRGB. The method is ideal for repeated solution of linear equations.

There is another program developed at IBM by Gustavson, Liniger and Willoughby (1970) in which the first run generates the FORTRAN code to solve the specific linear system. Further solutions of the same system structure can use the FORTRAN code already produced (Calahan, 1968a). This process has been implemented along with a modified version of Gear's method (Brayton, Gustavson and Hachtel (1972); Hachtel, Brayton and Gustavson, 1971) into the ECAP II Package (Branin *et al.*, 1971). "



The version of TRGB-TRGB2 used in the testing stored the operator list completely in core. For 200 equations, the method could not store the list for bandwidth of 11 or more. The list can be put into auxiliary storage however, but this will require more computer time to transfer the list or parts of it to and from core. If this is done, there would be no limit to the number of equations which can be solved. Table 5.5 gives the length of the operator list for the test systems.

As the bandwidth increases, one of the IMP options, method 17 of Table 5.1 becomes more competitive. Possibly as the bandwidth increases still further, the IMP method will become even faster than TRGB2. This opinion is held by Brandon (Brandon, 1974e).

Other algorithms not included in the numerical testing were a conjugate gradient program of Reid (1970), an elimination algorithm of Curtis and Reid (1971) and a program based on arc-graph structure (Rheinboldt, 1973).

Table 5.1: Key To Linear Equation Solvers

NUMBER	METHOD
1 2 3 4 5	<b>CONVENTIONAL METHODS</b> (1) Matrix Inversion (MINV) (2) Gaussian Elimination, No Pivoting (SOLVE) (3) Gaussian Elimination, Partial Pivoting (DECOMP-SOLVE) (4) Jacobi Iteration (5) Gauss Seidel Iteration
6 7 8 9 10 11 12	<b>KEY</b> (1) Simple Gauss-Jordan Elimination (2) Gauss-Jordan Partial Pivoting (3) Gauss-Jordan Full Pivoting (4) Minimum Row-Minimum Column (5) Minimum Column-Minimum Row (6) Maximum Column-Minimum Row (7) Minimum Of Row Entries Times Column Entries
13 14 15 16 17	<b>IMP</b> (1) Gauss-Seidel (2) Gradient (3) Crout, Fixed Order, General Sparse (4) Crout, Variable Order, General Sparse (5) Crout, Variable Order, Variable Or Constant Band
18 19 20 21	<b>MISCELLANEOUS</b> (1) Brandon, Crout, Variable Order, General Sparse (2) LINEQ4 (3) TRGB (4) TRGB2

Table 5.2) Execution Times For 50 Linear Equations

NO.	BANDWIDTH						
	3	5	7	9	11	13	15
1	5.17	5.15	5.13	5.15	5.18	5.20	5.16
2	0.838	0.850	0.816	0.833	0.819	0.818	0.818
3	0.790	0.791	0.780	0.779	0.755	0.764	0.756
4	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
5	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
6	0.647	0.798	0.935	1.09	1.27	1.44	1.62
7	0.748	0.986	1.23	1.38	1.64	1.91	2.14
8	0.523	1.79	2.30	3.27	5.05	5.39	5.15
9	0.215	0.327	0.495	0.691 <sup>I</sup>	0.897	1.12 <sup>I</sup>	1.36 <sup>SI</sup>
10	0.293	0.437	0.612	0.814 <sup>I</sup>	1.06	1.31 <sup>SI</sup>	1.60
11	0.768	0.970	1.22	1.49	1.78	2.06	2.39
12	0.320	0.538	0.736	0.994 <sup>I</sup>	1.28	1.58 <sup>SI</sup>	1.88
13	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
14	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
15	0.112	0.161	0.243	0.373	0.537	0.771	1.02
16	0.101	0.158	0.223	0.298	0.391	0.509	0.645
17	0.047	0.060	0.080	0.092	0.111	0.131	0.138
18	0.073	0.097	0.161	0.234	0.320	0.424	0.564
19	0.960 <sup>I</sup>	1.35 <sup>I</sup>	1.68 <sup>I</sup>	2.04 <sup>I</sup>	2.46 <sup>I</sup>	2.97 <sup>I</sup>	3.52 <sup>I</sup>
20	0.185	0.268	0.346	0.453	0.550	0.674	0.797
21	0.012	0.023	0.033	0.049	0.065	0.086	0.106

I - inaccurate    SI - slightly inaccurate (1-3 figure accuracy)  
 DIV - diverged

Table 5.3: Execution Times For 100 Linear Equations

NO.	BANDWIDTH						
	3	5	7	9	11	13	15
1	40.1	39.8	40.0	39.8	39.7	39.8	40.6
2	6.45	6.43	6.37	6.28	6.32	6.26	6.26
3	5.81	5.78	5.72	5.67	5.62	5.59	5.54
4	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
5	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
6	2.60	3.15	3.71	4.32	4.94	5.57	6.20
7	2.07	3.88	4.79	5.69	6.40	7.47	8.33
8	1.91	8.68	11.6	15.1	24.7	24.9	34.9
9	0.722 <sup>I</sup>	1.05 <sup>I</sup>	1.49 <sup>I</sup>	1.95 <sup>I</sup>	2.48 <sup>I</sup>	3.10 <sup>I</sup>	3.76 <sup>I</sup>
10	1.02 <sup>I</sup>	1.38 <sup>I</sup>	1.84 <sup>I</sup>	2.35 <sup>I</sup>	2.95 <sup>I</sup>	3.65 <sup>I</sup>	4.37 <sup>I</sup>
11	3.05	3.81	4.64	5.51	6.43	7.42	8.47
12	1.22	1.81	2.51	3.25	4.08	4.97	5.86
13	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
14	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
15	0.179	0.286	0.463	0.741	1.10	1.58	2.20
16	0.277	0.379	0.518	0.674	0.879	1.13	1.43
17	0.118	0.144	0.178	0.216	0.249	0.295	0.327
18	0.152	0.247	0.350	0.503	0.690	0.923	1.20
19	2.36 <sup>I</sup>	3.87 <sup>I</sup>	5.03 <sup>I</sup>	6.40 <sup>I</sup>	7.95 <sup>I</sup>	9.74 <sup>I</sup>	11.7 <sup>I</sup>
20	0.651	0.933	1.24	1.54	1.88	2.25	2.67
21	0.023	0.044	0.068	0.101	0.138	0.181	0.229

I - inaccurate SI - slightly inaccurate (1-3 figure accuracy)

DIV - diverged

Table 5.4: Execution Times For 200 Linear Equations

NO.	BANDWIDTH						
	3	5	7	9	11	13	15
1	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.
2	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.
3	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.
4	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.
5	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.	T.L.
6	10.4	12.5	14.8	17.1	19.4	21.8	24.2
7	6.17	15.5	20.0	23.3	26.4	29.9	33.8
8	6.64	33.5	60.3	95.2	110.0	147.7	148.2
9	2.60 <sup>I</sup>	3.57 <sup>I</sup>	4.70 <sup>I</sup>	5.98 <sup>I</sup>	7.43 <sup>I</sup>	9.00 <sup>I</sup>	10.8 <sup>I</sup>
10	3.85 <sup>I</sup>	4.85 <sup>I</sup>	6.05 <sup>I</sup>	7.51 <sup>I</sup>	8.97 <sup>I</sup>	10.7 <sup>I</sup>	12.7 <sup>I</sup>
11	12.2	15.0	18.0 <sup>I</sup>	21.1 <sup>I</sup>	24.4 <sup>I</sup>	27.7 <sup>I</sup>	31.1 <sup>I</sup>
12	4.56 <sup>I</sup>	6.59 <sup>I</sup>	8.81 <sup>I</sup>	11.2 <sup>I</sup>	13.7 <sup>I</sup>	16.5 <sup>I</sup>	19.3 <sup>I</sup>
13	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
14	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.	DIV.
15	0.293	0.530	0.897	1.46	2.22	3.24	4.52
16	0.816	1.02	1.28	1.62	2.04	2.56	3.17
17	0.315	0.372	0.440	0.506	0.577	0.650	0.743
18	0.417	0.587	0.824	1.15	1.53	2.00	2.59
19	7.26 <sup>I</sup>	13.2 <sup>I</sup>	17.5 <sup>I</sup>	22.7 <sup>I</sup>	28.5 <sup>I</sup>	35.1 <sup>I</sup>	42.3 <sup>I</sup>
20	2.45	3.47	4.51	5.59	T.L.	T.L.	T.L.
21	0.046	0.086	0.137	0.201	T.L.	T.L.	T.L.

I - inaccurate      SI - slightly inaccurate (1-3 figure accuracy)  
 T.L. - too large (requires more than 49K)      DIV - diverged

Table 5.5: Length of TRGB Operator List

BANDWIDTH	NUMBER OF EQUATIONS		
	50	100	200
3	600	1200	2400
5	1176	2376	4776
7	1928	3928	7928
9	2848	5848	11848
11	3928	8128	-
13	5160	10760	-
15	6536	13736	-

## 6. NUMERICAL TESTING OF STIFF TECHNIQUES

### 6.1 Testing By Other Workers

Several numerical comparisons of methods for solving stiff o.d.e.s have been made.

Steeper (1970) compares Sloate's method (Sloate, 1970), Liou's method (Liou, 1966), Pope's method (Pope, 1963), second and fourth order Runge-Kutta, Jung's variation of Treanor's method (Jung, 1967), a linear extrapolation method from the FACE program (Price, 1970) and MINTA5, a modified version of Adams-Moulton-Shell (Shell, 1958). Based on two test examples, he recommends MINTA5.

Lapidus and Seinfeld (1971), also Seinfeld, Lapidus and Hwang (1970), have compared fourth order Runge-Kutta, fourth order Adams, Treanor's method (Treanor, 1966), the modified midpoint rule, the trapezoidal rule with and without extrapolation, Calahan's method (Calahan, 1968b) and two methods of Liniger and Willoughby (1970). They found the last four implicit methods to be superior and roughly comparable in terms of accuracy and computing time. For highly stiff systems, they recommend one of Liniger and Willoughby's techniques. Four test examples were used.

Brandon (1972, 1974f) compares his own method with the stiff techniques compared by Lapidus and Seinfeld and finds his method to be superior. He uses two test examples.

Hronsky and Martens (1973) compare Runge-Kutta-Newton (Kreyszig, 1967) which is the pseudo steady state approach used with the fourth order Runge-Kutta method, Treanor's method (Treanor, 1966), a method of Liniger and Willoughby (1970), and the backward differentiation formula (Brayton *et al.*, 1972) which is a modification of Gear's method (Appendix A). Based on five test examples, they recommend the methods in the order above, except if the number of equations is less than six, Liniger and Willoughby's method is recommended over that of Treanor.

Hull and Enright have been doing extensive comparisons of stiff techniques, but their final results are not yet available. Their preliminary testing indicated the three best methods to be Gear's method, trapezoidal rule with extrapolation and the second derivative multi-step method (Enright, 1974).

Several other comparisons appear in the literature but, unfortunately, most authors of new techniques compare their algorithm only with an inefficient conventional method such as fourth order Runge-Kutta or with a stiff technique very similar to theirs, to demonstrate a



superior modification.

## 6.2 Test Methods and Examples

The following methods were selected for numerical testing:

four conventional methods none of which are oriented to solving stiff systems:

- (1) Euler
- (2) Adams-Moulton-Shell (Shell, 1958)
- (3) Runge-Kutta-Merson (Merson, 1957)
- (4) Gear's nonstiff option (Appendix A)

four explicit stiff techniques:

- (1) Richards, Lanning and Torrey (1965)
- (2) Nigro (1969)
- (3) Treanor (1966)
- (4) Fowler and Warten (1967)

four implicit stiff techniques:

- (1) Klopfenstein and Davis (1971)
- (2) Sandberg and Shichman (1968)
- (3) Brandon (1972, 1974f)
- (4) Gear's stiff option (Appendix A).

The IMP package (Appendix B) containing Brandon's method was also tested. Obviously it is impractical to test all the methods surveyed in Chapter 3, but these provide a cross-section of promising explicit and implicit techniques. The explicit methods have the advantage of avoiding the solution of linear algebraic equations.

The four implicit methods used matrix inversion (MINV of Chapter 5) in solving the corrector. Gear's method was also used with two other linear equation solvers: DECOMP-SOLVE and TRGB-TRGB2 described in Chapter 5. TRGB was used only on the first time step to set up the operator list and TRGB2 used thereafter.

Eleven test systems were used (Appendix D). They can be divided into three groups:

- (1) small stiff systems: I, II, III, IV, V
- (2) large stiff systems: VI, VII
- (3) nonstiff systems : VIII, IX, X, XI

Each of the fifteen test programs were tested on each example. Ideally the same method could handle both nonstiff and stiff methods efficiently; otherwise, two methods would have to be used.

The two complex eigenvalue systems III and VIII had small imaginary parts to avoid introducing the problem

of simulating rapidly oscillating components.

Most of the methods used had a local truncation error estimate: Gear, IMP, Brandon, Fowler-Warten, Klopfenstein-Davis, Sandberg-Shichman, Adams-Moulton-Shell and Merson. Gear's error control is described in Gear (1971a,b) and briefly in Appendix A. IMP uses a complex algorithm (Brandon, 1972, 1974f) to calculate the step size. In all of these methods, however, the maximum relative local error was kept below an upper tolerance and usually above a lower tolerance (except for Gear and IMP). The upper tolerance used was  $10^{-3}$  for systems I-X and  $10^{-6}$  for system XI. The lower tolerance was taken as one-tenth of the upper tolerance. Relative errors were used with the absolute error estimate divided by the maximum value of the corresponding independent variable. This maximum value had a minimum of 1.0, so that in effect, relative errors were used for large  $y$  values and absolute errors for small  $y$  values. Fowler-Warten had a complex error control scheme, but it was not used here. The last six methods mentioned above used the double-halving approach described in Section 2.7.3.

Three methods had no error estimate: Euler, Nigro and Treanor. Lapidus and Seinfeld (1971) recommend the following error control scheme for Treanor's method. If the relative change in  $y$ ,  $|y_{n+1} - y_n| / |y_{n+1}|$ , is greater than

some upper tolerance, the step size is halved and the step repeated. If it is greater than some lower tolerance, the step is doubled. Tolerances of 0.05 and 0.005 were used. This strategy gave unstable results for systems V, VI and VII. It was altered for them to keeping

$$0.001 < \left| \frac{y_{n+1} - y_n}{y_{n+1} + R} \right| < 0.01$$

$$\begin{aligned} \text{where } R &= 0.1 \quad \text{if } y_{n+1} < 0.5 \\ &= 0.0 \quad \text{otherwise} \end{aligned}$$

Euler and Nigro used a fixed step size. The critical step size was used for the stiff examples, i.e., the step size just before the onset of instability. For the nonstiff examples, the step size was adjusted to keep the final error below 1%.

For Nigro's method, the parameters corresponding to  $h\lambda = 120$  were used. Euler's method was used to start the integration. A step size of 0.01 of the Nigro step size was used to calculate the starting points. This was not included in the execution time.

The first order Klopfenstein-Davis algorithm tested better than the second order. Tests with various coefficients of  $\alpha$ ,  $\beta$ ,  $u$ ,  $v$  and  $\alpha$  yielded very similar results so the values  $\alpha = 1.0$ ,  $\beta = 0.0$ ,  $u = 0.25$ ,  $v = 0.75$  and  $\alpha = 0.6667$  were used. See Equation (3.3.10).

### 6.3 Test Results

Global errors are given at the end of the simulation with the exception of systems II, V, VI and VII. There, the error is given after 10% of the simulation, since it was expected several methods could require near infinite computer time to reach the end of the integration.

Analytical solutions are available on systems I, II, III, IV, VIII and IX to calculate the error. For the remaining examples, V, VI, VII, X and XI, the Euler method with  $h = 0.00005$  was used to get an accurate solution. This was arbitrarily taken as the "correct" solution. In each case with  $h = 0.0001$ , the results differed at worst in the fourth significant figure.

None of the examples came close to steady state, so an error taken at the end of the simulation will be reasonably indicative of the accuracy of the trajectory.

Table 6.1 shows the execution times and errors for the explicit methods and the Euler step size. Tables 6.2 and 6.3 show results for the explicit and implicit stiff methods respectively. Nigro's step size is also shown. Table 6.4 shows IMP and three different linear equation solvers in conjunction with Gear's method. Gear-MINV is repeated from Table 6.3. See Appendix E for a discussion on computer timings.

In general the conventional methods did not perform well on the stiff examples. As expected, Euler's method was the least inefficient, but obviously a method designed specifically for stiff systems is to be preferred. Some of the runs especially on systems II, V, VI and VII were a waste of computer time, as the methods were so inefficient. The conventional methods are not of great concern here except to note how well the stiff methods were able to solve the nonstiff examples by comparison. They do reasonably well with respect to execution time and error. Since the execution times are so small anyway, it would suffice to use a stiff method for both nonstiff and stiff examples.

Hull *et al.* (1972a) consider a method of Krogh (1969) very good for nonstiff systems, but they rate Gear's (nonstiff option) closely behind it. In our limited testing of nonstiff examples here, Runge-Kutta-Merson did very well. AMOS could not handle system XI, probably because of the extremely small tolerance demanded.

In general, the explicit stiff methods did not perform as well as the implicit methods. Of the stiff methods, Gear's was easily the best (Table 6.3). Brandon (1974d) has improved his error estimation procedure recently and it does much better, but it was not used in our testing. In particular, the latest version of the IMP package

integrates system VI and VII in 22.2 and 6.68 seconds respectively on the CDC 6600. These times would roughly be multiplied by three for comparable CYBER 73 times.

The Fowler-Warten method handled the complex eigenvalue systems III and VIII reasonably well in spite of the fact that it is intended for systems with the largest eigenvalue real.

Neither of the step-changing strategies used with Treanor's method were very efficient. Possibly a fixed step size would be a better strategy.

Some of the methods simulated only the extremely small components inaccurately, especially in Systems V and X. In these cases, the second largest error is given in brackets.

Different methods of handling the corrector are illustrated in Table 6.4 with Gear's method. For less than about five equations, matrix inversion is fastest, then Gaussian elimination is preferable up to perhaps ten equations, but for large systems a sparse method such as the Bending-Hutchison algorithm is necessary, as shown by the results for Systems VI and VII. The difference in execution time is slight for the small examples.

System XI ran slightly faster with the tridiagonal option (0.753 seconds) rather than with TRGB-TRGB2 (0.980 seconds). The time savings will increase with the size of the system as shown in Section 8.4.

If we had to choose one method to handle all systems stiff and nonstiff, large and small, it would be Gear's method with stiff option and the Bending-Hutchison method to solve the corrector. It handles small stiff systems and nonstiff systems efficiently enough since little computer time is required anyway. However, the nonstiff option is also available in the same program and only a switch is required to change from one to the other, so that both stiff and nonstiff options are conveniently available. Different coefficients are used, but most of the code is similar for both options.



Table 6.1: Execution Times And Errors For Conventional Methods

TEST SYSTEM	METHOD	EULER	STEP SIZE	ERROR	AMOS	ERROR	RKM	ERROR	GEAR NONSTIFF	ERROR
I		0.066	$1.9 \times 10^{-3}$	$9.50 \times 10^{-4}$	0.872	$1.88 \times 10^{-3}$	0.301	$3.72 \times 10^{-4}$	2.30	$1.13 \times 10^{-3}$
II		71.7	$1.9 \times 10^{-6}$	$9.51 \times 10^{-8}$	866.0	$1.01 \times 10^{-3}$	300.3	$1.36 \times 10^{-4}$	2188.0	$3.81 \times 10^{-4}$
III		0.325	$1.9 \times 10^{-3}$	$4.28 \times 10^{-3}$	2.06	$4.29 \times 10^{-3}$	1.18	$3.55 \times 10^{-3}$	4.90	$4.05 \times 10^{-3}$
IV		1.47	$1.9 \times 10^{-3}$	$9.62 \times 10^{-6}$	9.09	$5.65 \times 10^{-4}$	6.02	$7.27 \times 10^{-6}$	36.4	$2.89 \times 10^{-5}$
V		17.1	$5.0 \times 10^{-4}$	$3.91 \times 10^{-5}$	240.0	114.6 ( $4.70 \times 10^{-3}$ )	79.0	154.6 ( $5.21 \times 10^{-4}$ )	431.6	0.368 ( $3.95 \times 10^{-4}$ )
VI		$U^1$ 23.0 (0-10)	$2.5 \times 10^{-4}$ $1.0 \times 10^{-3}$	$4.21 \times 10^{-4}$	>1500 101.0 (0-10)	$9.95 \times 10^{-3}$	>1000 48.0 (0-10)	$1.99 \times 10^{-4}$	1925.0	$1.86 \times 10^{-4}$
VII		37.6	$6.0 \times 10^{-3}$	$4.55 \times 10^{-3}$	353.0	0.855 <sup>2</sup>	167.8	$7.94 \times 10^{-2}$	656.0	$8.80 \times 10^{-3}$
VIII		0.530	$2.0 \times 10^{-3}$	$9.58 \times 10^{-3}$	0.059	$6.69 \times 10^{-3}$	0.021	$1.89 \times 10^{-3}$	0.101	$8.15 \times 10^{-4}$
IX		1.04	$5.0 \times 10^{-3}$	$9.48 \times 10^{-3}$	0.107	$9.62 \times 10^{-3}$	0.041	$1.56 \times 10^{-2}$	0.213	$6.67 \times 10^{-3}$
X		0.636	$2.0 \times 10^{-3}$	$9.75 \times 10^{-3}$	0.059	3.48 ( $1.81 \times 10^{-4}$ )	0.022	0.394 ( $6.05 \times 10^{-4}$ )	0.164	$4.39 \times 10^{-2}$
XI		1.39	$2.5 \times 10^{-3}$	$9.45 \times 10^{-3}$	>320		0.263	$4.87 \times 10^{-4}$	1.30	$2.25 \times 10^{-4}$

<sup>1</sup>unstable

<sup>2</sup>only extremely small components inaccurate

Table 6.2: Execution Times And Errors For Explicit Stiff Techniques

TEST SYSTEM	METHOD	RLT	ERROR	NIGRO	STEP SIZE	ERROR	TREAHOR	ERROR	FOWLER WARTEN	ERROR
I		0.092	$4.83 \times 10^{-3}$	0.144	$1.8 \times 10^{-3}$	$5.35 \times 10^{-3}$	0.384	$2.24 \times 10^{-4}$	0.495	$2.24 \times 10^{-3}$
II		0.068	$5.03 \times 10^{-4}$	26.5	$1.9 \times 10^{-5}$	$2.15 \times 10^{-2}$	0.393	$8.90 \times 10^{-7}$	49.1	$3.98 \times 10^{-2}$
III		3.32	$3.81 \times 10^{-3}$	0.498	$1.8 \times 10^{-3}$	$6.13 \times 10^{-3}$	1.38	$5.09 \times 10^{-4}$	1.48	$7.49 \times 10^{-5}$
IV		0.356	$5.03 \times 10^{-3}$	0.817	$5.0 \times 10^{-3}$	$6.88 \times 10^{-3}$	6.90	$1.85 \times 10^{-3}$	0.317	$1.79 \times 10^{-2}$
V		0.078	343.0 ( $6.58 \times 10^{-4}$ )	18.5	$1.0 \times 10^{-3}$	$1.06 \times 10^{-2}$	0.505	$3.56 \times 10^{-3}$	0.624	$8.42 \times 10^{-3}$
VI		36.5	8.86 <sup>2</sup>	464.0	$1.0 \times 10^{-3}$	$3.35 \times 10^{-2}$	U <sup>1</sup> 44.5 (0-10)	$4.41 \times 10^{-2}$	117.0	$2.48 \times 10^{-2}$
VII		3.58	2.03 <sup>2</sup>	46.5	$1.0 \times 10^{-2}$	$1.73 \times 10^{-2}$	219.0	$8.24 \times 10^{-2}$	75.2	$1.41 \times 10^{-2}$
VIII		0.159	$8.61 \times 10^{-2}$	1.24	$2.0 \times 10^{-3}$	$9.72 \times 10^{-3}$	0.927	$1.97 \times 10^{-5}$	0.080	$2.64 \times 10^{-2}$
IX		0.317	$5.45 \times 10^{-2}$	2.02	$4.0 \times 10^{-3}$	$1.05 \times 10^{-2}$ <sup>1</sup>	2.69	$3.13 \times 10^{-9}$	0.777	$7.07 \times 10^{-3}$
X		0.118	0.630 ( $2.81 \times 10^{-2}$ )	1.36	$2.5 \times 10^{-3}$	$2.64 \times 10^{-2}$	2.36	$2.46 \times 10^{-4}$	0.060	$1.83 \times 10^{-3}$
XI		0.744	$3.00 \times 10^{-2}$	5.65	$1.0 \times 10^{-2}$	$1.63 \times 10^{-2}$	5.07	$4.06 \times 10^{-5}$	0.666	$1.86 \times 10^{-4}$

<sup>1</sup>unstable

<sup>2</sup>large components reasonably accurate

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Table 6.3: Execution Times And Errors For Implicit Stiff Techniques

TEST SYSTEM	METHOD	KLOP DAVIS	ERROR	SAND SCH.	ERROR	BRADSHAW	ERROR	GEAR	ERROR
I		2.27	$5.85 \times 10^{-4}$	0.274	$1.21 \times 10^{-2}$	0.124	$1.22 \times 10^{-3}$	0.139	$1.07 \times 10^{-3}$
II		>1000 385.0 (0-10)	$5.00 \times 10^{-8}$	0.276	$7.10 \times 10^{-4}$	0.268	$3.98 \times 10^{-4}$	0.196	$1.01 \times 10^{-4}$
III		7.11	$2.67 \times 10^{-3}$	0.788	$4.64 \times 10^{-2}$	4.24	$3.97 \times 10^{-3}$	0.318	$3.77 \times 10^{-3}$
IV		39.2	$5.90 \times 10^{-6}$	1.75	$9.41 \times 10^{-4}$	1.11	$5.66 \times 10^{-4}$	0.505	$1.16 \times 10^{-5}$
V		845.0	0.267 ( $7.56 \times 10^{-3}$ )	0.117	$1.27 \times 10^{-4}$	0.459	$3.10 \times 10^{-5}$	0.168	$4.57 \times 10^{-5}$
VI		>1000 (0-10)		>1000 626.0 (0-10)	$6.83 \times 10^{-3}$	>1500 761.0 (0-10)	$7.72 \times 10^{-4}$	81.7	$8.47 \times 10^{-3}$
VII		>1000 (0-10)		442.0	0.133	299.0	$1.86 \times 10^{-2}$	42.0	$1.71 \times 10^{-2}$
VIII		0.760	0.206	0.721	0.204	0.294	0.308 (0.023)	0.078	$3.23 \times 10^{-3}$
IX		0.788	0.335	0.830	0.316	0.546	$9.68 \times 10^{-3}$	0.192	$8.07 \times 10^{-2}$
X		0.473	0.936 ( $3.0 \times 10^{-2}$ )	0.455	0.870 ( $2.94 \times 10^{-2}$ )	0.313	$8.13 \times 10^{-3}$	0.119	1.38 ( $2.74 \times 10^{-3}$ )
XI		29.1	$3.78 \times 10^{-2}$	29.1	$3.75 \times 10^{-2}$	2.44	$1.77 \times 10^{-4}$	1.03	$4.85 \times 10^{-5}$

Table 6.4: Execution Times And Errors For Gear's Method And IMP

TEST SYSTEM	METHOD	GEAR MINV	GEAR DECOMP-SOLVE	GEAR TRGB	ERROR	IMP	ERROR
I		0.139	0.148	0.162	$1.07 \times 10^{-3}$	0.139 <sup>1</sup>	$1.85 \times 10^{-3}$
II		0.196	0.216	0.235	$1.01 \times 10^{-4}$	0.373 <sup>1</sup>	$2.84 \times 10^{-4}$
III		0.318	0.294	0.315	$3.77 \times 10^{-3}$	3.72 <sup>1</sup>	$1.06 \times 10^{-5}$
IV		0.505	0.489	0.656	$1.16 \times 10^{-5}$	0.967 <sup>1</sup>	$2.69 \times 10^{-2}$
V		0.168	0.165	0.182	$4.57 \times 10^{-5}$	27.3 <sup>2</sup>	$3.91 \times 10^{-5}$
VI		81.7	20.0	7.58	$8.47 \times 10^{-3}$	>1000	
VII		42.0	12.3	6.05	$1.71 \times 10^{-2}$	30.2 <sup>2</sup>	0.169
VIII		0.078	0.079	0.080	$3.23 \times 10^{-3}$	0.258 <sup>1</sup>	0.344
IX		0.192	0.197	0.292	$8.07 \times 10^{-2}$	0.584 <sup>2</sup>	$2.37 \times 10^{-3}$
X		0.119	0.128	0.130	1.38 ( $2.74 \times 10^{-3}$ )	1.12 <sup>1</sup>	$5.27 \times 10^{-3}$
XI		1.03	0.897	0.920 0.753(TR1)	$4.85 \times 10^{-5}$	11.5 <sup>1</sup>	0.967

<sup>1</sup>Crout elimination, variable order, variable or constant banded matrices

<sup>2</sup>Gauss-Seidel

## 7. THE DYNYSYS 2.0 EXECUTIVE PROGRAM

We have tested several integration techniques for handling stiff ordinary differential equations. Of these, Gear's method appears to be the most effective. We have also tested various techniques for solving large systems of linear algebraic equations which arise in solving the implicit part of a stiff numerical integration method. The Bending-Hutchison approach was found to be particularly appropriate.

We would now like to implement these features into a dynamic simulation executive program. Either the equation-oriented or problem-oriented approaches can be used, but we believe the modular approach is especially suited to the engineer, since many engineering processes are modular in structure; i.e., they consist of many different pieces of equipment. It was decided to implement the new techniques into the DYNYSYS package. The resulting program is called DYNYSYS 2.0.

This chapter describes the new features of DYNYSYS 2.0 and how they affect the user. A listing of DYNYSYS 2.0 appears in Appendix I.

7.1 Numerical Integration of O.D.E.s

Both the stiff and nonstiff options of Gear's method are available; however, the same option must be used for the entire simulation.

A simulation will often contain both stiff and nonstiff modules. In this case the stiff option must be used and all modules will use the stiff coefficients of Gear's method. We saw in Chapter 6 that the stiff option handles nonstiff problems reasonably well. It is not possible to use the nonstiff and stiff coefficients in the same simulation as they have different error constants and the error and step-changing analysis could not be done.

The stiff option is assumed by the executive. If the user wishes to have the nonstiff system, i.e., the system is completely nonstiff, he must specify this through the data set. Section 2.6 describes how to identify a stiff system.

The algorithm used differs from Gear's original DIFSUB (Appendix A) because of the modular approach. A predictor step is taken through all of the modules followed by a corrector step. Up to three corrector iterations are made. The derivatives are re-evaluated by returning to the module. If convergence does not occur for any module, the integration step size is reduced to 1/4 of its present size and the step repeated for all modules. The factor 1/4

is given by Gear (1971c) and is empirical.

Another way of using the predictor-corrector would be to predict and correct for each module in turn. However, the manner actually chosen helps to converge any recycle in the system. As yet DYNYSYS has no other way of handling recycle convergence. This is a possible item for future work.

After the corrector pass, the error analysis is done (Gear, 1971c; Appendix A) considering the equations from every module as one system.

## 7.2 Corrector Convergence of Stiff Equations

Gear's algorithm contains an option for both stiff and nonstiff equations. If the nonstiff option is used, the corrector equations will be solved by repeated iteration only. For the stiff option, Newton-Raphson iteration is generally used, but a special option for tridiagonal matrices (Carnahan, Luther and Wilkes, 1969) was included. This was done because tridiagonal systems arise often in chemical engineering in the discretization of partial differential equations and in countercurrent stagewise processes. A special algorithm for the tridiagonal system is much faster than using the Bending-Hutchison solver. See example #4 of Chapter 8.

In a system of nonstiff and stiff modules, all the modules will use the stiff coefficients of Gear's method to facilitate the error analysis, but the nonstiff modules can use Jacobi iteration of the corrector.

Each set of stiff p.d.e.s in each module will generate an operator list. All the lists are created on the first time step and are stored sequentially in core with 10% extra storage in case the list is re-created.

The routine contains a minimum pivot presently set at  $10^{-6}$  which is reasonable for 14 figure single precision accuracy as on CDC machines. This figure may be altered through the data set for other computers (for example, perhaps  $10^{-3}$  for single precision IBM computers). If, during the integration, one of the pivots falls below  $10^{-6}$ , the operator list will be re-calculated using different elements as pivots. It is assumed that the new operator list will not be more than 10% longer than the original one. Several tests were made where the operator list was re-calculated every time step. The greatest change in length was 5% longer than the original length; however, in every example tested, the original operator list was sufficient for the entire simulation. Different operator lists can be generated as the integration proceeds, in addition to the possibility of a pivot falling below  $10^{-6}$ , because at  $t = 0$ , many nonzero elements of the



Jacobian may actually be zero because of the initial conditions, their values may become large enough that they are available as pivots. The present executive does not, however, re-evaluate the operator list for this reason as it is unnecessary and would require too much computer time.

DIFSUB can use a numerical or analytical (user supplied) Jacobian matrix. It was decided to use an analytical Jacobian for DYNSSYS as it is more efficient and it would be extremely difficult to compute a sparse numerical Jacobian and use it with the Bending-Hutchison routine.

Gear re-evaluated the Jacobian only if the corrector did not converge in three iterations or less or if a change in order was made. The present method in DYNSSYS evaluates the Jacobian every time step to avoid storing all the Jacobians together in core at the same time since storage space is critical.

The DYNSSYS version of DIFSUB has about the same execution speed as Gear's published version. The Jacobian is evaluated more often, but this is balanced by fewer corrector iterations being required.

### 7.3 Using the DYNYSYS 2.0 Executive Program

#### 7.3.1 New Parameters in the DYNYSYS 2.0 Data Set

Several new data set parameters relating to the new integration method have been introduced. They are the smallest and largest permissible integration step size (HMIN, HMAX), the option, stiff or nonstiff to be used (NONSTIFF), the maximum permissible order of the integration (ORDER), the error tolerance (EPS), and the minimum permissible pivot element (MINPIVOT). These are summarized in Table 7.1. The other data set parameters are described in the DYNYSYS Manual.

#### 7.3.2 Writing Modules For DYNYSYS 2.0

The user will usually write a module corresponding to each of his pieces of equipment. A general skeleton of such a module is shown in Figure 7.1. The user should study this and the examples in Chapter 8 before he writes his modules.

As shown in the skeleton, a module can be divided into five basic sections:

- (1) calculate module parameters, stream input, initial conditions, ITER, etc.

Table 7.1: New Parameters In the DYN SYS 2.0 Data Set

DATA INPUT	DEFAULT VALUE	DESCRIPTION
HMIN	HMIN=10 <sup>-6</sup>	HMIN:smallest permissible step size to be used by DIFSUB
HMAX	HMAX=0,05	HMAX:largest permissible step size to be used by DIFSUB
NONSTIFF	ISTIFF=1	ISTIFF=0:nonstiff option, i.e., nonstiff coefficients are used in Gear's method ISTIFF=1:stiff option
ORDER	IORDER=6	IORDER:maximum permissible order of integration method to be used IORDER cannot exceed 6 for stiff option or 7 for nonstiff option
TOLERANCE	EPS=0.001	EPS:Euclidean norm of relative local truncation errors
MINPIVOT	XMIN=10 <sup>-6</sup>	XMIN:minimum permissible value of pivot element in TRGB, TRGB2

FIGURE 7.1: SKELETON OF MODULE FOR DYNYSYS 2.0

```

SUBROUTINE TYPE2.0
C
C COMMENTS DESCRIBING MODULE
C
C THE FOLLOWING COMMON BLOCKS ARE OR MAY BE REQUIRED:
C MAT,CON,PTAB,UNIT,GEAR2,MODULE,ROW,COLUMN,JACOB,SUBDI,DIAG,SUPERD
C
COMMON/MAT/MP( , ),EP( , ),S( , , ),EX( )
COMMON/CON/IG,NCOMP,NCS,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON/PTAB/IGFLAG,PP( , )
COMMON/UNIT/IM,NMP
COMMON/GEAR2/EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF
COMMON/MODULE/IDERY,ITER,ITRI,NZERO
C
C NZERO IS NUMBER OF NONZERO ELEMENTS IN JACOBIAN
C
COMMON/ROW/IROW(NZERO) (NORMAL OPTION)
COMMON/COLUMN/JCOL(NZERO) (NORMAL OPTION)
COMMON/JACOB/XJACOB(NZERO) (NORMAL OPTION)
C
C N IS NUMBER OF ODES
C
COMMON/SUBDI/A(N) (TRIDIAGONAL OPTION)
COMMON/DIAG/B(N) (TRIDIAGONAL OPTION)
COMMON/SUPERD/C(N) (TRIDIAGONAL OPTION)
C
C *****
C SECTION #1 : PARAMETER CALCULATIONS
C *****
C
C CALCULATE MODULE PARAMETERS : STREAM INPUT,INITIAL CONDITIONS,
C VALUES OF ITER ETC.
C
C INPUT STREAM INFORMATION IS OBTAINED FROM S(IG, , )
C VALUE OF INDEPENDENT VARIABLE (Y) NEED ONLY BE SPECIFIED ON FIRST
C INTEGRATION STEP (I.E. INITIAL CONDITIONS)
C JSTART=0 ON FIRST INTEGRATION STEP
C =CURRENT ORDER OF INTEGRATION TECHNIQUE ON LATER STEPS
C
C IF ISTIFF=0, NONSTIFF COEFFICIENTS WILL BE USED IN INTEGRATION ALGORITHM
C (DIFSUB) FOR ALL MODULES,
C JACOBIAN MATRIX IS NOT REQUIRED
C
C IF ISTIFF=1, STIFF COEFFICIENTS WILL BE USED FOR ALL MODULES
C THEN, IF ITER=0 DIRECT ITERATION OF CORRECTOR WILL BE USED
C (NONSTIFF MODULE)
C JACOBIAN IS NOT REQUIRED
C IF ITER=1 NEWTON-RAPHSON ITERATION OF CORRECTOR WILL
C BE USED (STIFF MODULE)

```

JACOBIAN MATRIX MUST BE SUPPLIED

ITER MUST BE SPECIFIED 0 OR 1  
IT IS NOT USED UNLESS ISTIFF=1

IF (IG.EQ.2) GO TO 2

.....  
SECTION #2 : JACOBIAN EVALUATION  
.....

SECTION #2 IS OMITTED FOR NONSTIFF MODULE

JACOBIAN MATRIX IS REQUIRED ONLY IF MODULE EQUATIONS ARE STIFF  
CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS ONLY  
JACOBIAN NEED NOT BE VERY ACCURATE, AS IT IS USED ONLY FOR CONVERGENCE  
OF CORRECTOR  
ONLY NONZERO ELEMENTS ARE CALCULATED, BUT ALL DIAGONAL ELEMENTS MUST  
BE STORED WHETHER OR NOT THEY ARE ZERO (NORMAL OPTION)

NORMAL OPTION

IF JACOBIAN MATRIX IS NOT TRIDIAGONAL  
ROW NUMBER, COLUMN NUMBER AND VALUE OF EACH NONZERO ELEMENT ARE STORED IN  
ARRAYS IROW, JCOL AND XJACOB FROM COMMON BLOCKS ROW, COLUMN AND JACOB  
RESPECTIVELY  
ELEMENTS MAY BE STORED IN ANY ORDER

- NZERO - NUMBER OF NONZERO ELEMENTS IN JACOBIAN  
INCLUDING ANY ZERO DIAGONAL ELEMENTS
- IROW(I) - ROW NUMBER OF NONZERO ELEMENT I
- JCOL(I) - COLUMN NUMBER OF NONZERO ELEMENT I
- XJACOB(I) - VALUE OF NONZERO ELEMENT I

NZERO=  
IROW(1)=  
JCOL(1)=  
XJACOB(1)=

.....  
.....

IROW(NZERO)=  
JCOL(NZERO)=  
XJACOB(NZERO)=

TRIDIAGONAL OPTION

FOR TRIDIAGONAL JACOBIAN MATRIX, THE SUBDIAGONAL, DIAGONAL AND SUPERDIAGONAL  
ELEMENT VALUES ARE STORED IN ARRAYS A, B AND C FROM COMMON BLOCKS SUBDI,  
DIAG AND SUPERD RESPECTIVELY

N - NUMBER OF ODES

C A - VALUES OF SUBDIAGONAL ELEMENTS ARE STORED IN A(2)...A(N)  
 C A(1) IS NOT USED  
 C B - VALUES OF DIAGONAL ELEMENTS  
 C C - VALUES OF SUPERDIAGONAL ELEMENTS ARE STORED IN C(1)...C(N-1)  
 C C(N) IS NOT USED  
 C

B(1)=

C(1)=

A(2)=

B(2)=

C(2)=

.

.

A(N-1)=

B(N-1)=

C(N-1)=

A(N)=

B(N)=

1 CONTINUE

C IF TRIDIAGONAL OPTION IS BEING USED,ITRI MUST BE SET TO 1 HERE,  
 C IF NORMAL OPTION IS USED ITRI MAY BE IGNORED  
 C

C ITRI=1  
 C

C 2 CONTINUE  
 C

C \*\*\*\*\*  
 C SECTION #3 : DERIVATIVE CALCULATION  
 C \*\*\*\*\*  
 C

C CALCULATE DERIVATIVES  
 C

C DERY(1)=  
 C .  
 C .  
 C .

C DERY(N)=  
 C

C \*\*\*\*\*  
 C SECTION #4 : CALL DIFSUB  
 C \*\*\*\*\*  
 C

C CALL DIFSUB TO SOLVE ODES FOR MODULE  
 C DIFSUB MAY BE CALLED MORE THAN ONCE FROM A MODULE FOR EXAMPLE IF THE  
 C MODULE CONTAINS A SET OF STIFF O.D.E.S AND ANOTHER SET OF NONSTIFF  
 C O.D.E.S  
 C

C ARGUMENTS : N - NUMBER OF ODES  
 C Y - INDEPENDENT VARIABLE  
 C DERY - DERIVATIVES  
 C

```
C
C CALL DIFSUB(N,Y,DERY)
C
C IF IDERY IS NOT ZERO,THE DERIVATIVES WILL BE RE-EVALUATED AND
C RETURNED TO DIFSUB
C ITRI MUST ALSO BE RESET IF IT IS 1
C
C IF(IDERY.NE.0) GO TO 1
C
C *****
C SECTION #5 : STREAM OUTPUT CALCULATION
C *****
C
C CALCULATE STREAM OUTPUT (STORED IN S(1, , ))
C
C S(1, , ) =
C .
C .
C .
C RETURN
C END
```

- (2) calculate Jacobian matrix (if o.d.e.s are stiff) on corrector pass only
- (3) evaluate derivatives
- (4) CALL DIFSUB to solve o.d.e.s for module
- (5) calculate stream output.

There are several variables used in the skeleton module which relate to the integration. These are summarized in Table 7.2.

It is not necessary to supply a Jacobian matrix for the nonstiff modules. Setting the variable ITER to zero inside the module will cause direct iteration of the corrector. ITER should be set to one for stiff modules so that Newton-Raphson iteration will be used and the Jacobian of course must be supplied. If the module is known to be always stiff or nonstiff, ITER can be specified inside the module; otherwise, an equipment parameter can be used to specify the option.

It is possible to have more than one set of o.d.e.s in a module. This will present no problem as long as each set is defined before the call to DIFSUB.

It may be necessary to use a low value of HMAX, the maximum permissible integration step size, to keep the system under control. If too large a step is used, say



Table 7.2: Summary Of Module Variables Relating To DIFSUB

VARIABLE	DESCRIPTION
A	values of subdiagonal elements of a tridiagonal Jacobian matrix are stored in A(2),...,A(N)
B	values of diagonal elements of a tridiagonal Jacobian matrix are stored in B(1),...,B(N)
C	values of superdiagonal elements of a tridiagonal Jacobian matrix are stored in C(1),...,C(N-1)
DERY	derivative values
IDERY	IDERY is tested immediately after call to DIFSUB IDERY=0 : module execution is continued IDERY=1 : derivatives are re-evaluated and returned to DIFSUB, ITR1 is reset to 1 if necessary
IROW	row numbers of nonzero Jacobian elements and zero diagonal elements in normal option for storing Jacobian
ITER	ITER is used only if ISTIFF=1 (stiff option) ITER=0 : direct iteration of corrector (nonstiff module) ITER=1 : Newton-Raphson iteration of corrector (stiff module)
ITR1	ITR1 specifies option for storing Jacobian ITR1=0 : normal option ITR1=1 : tridiagonal option if ITR1=1, it must be reset to 1 each time DIFSUB is called
JCOL	column numbers of nonzero Jacobian elements and zero diagonal elements in normal option for storing Jacobian
N	number of ordinary differential equations in module
NZERO	number of nonzero elements in Jacobian for normal option (zero diagonal elements are included)
XJACOB	values of nonzero Jacobian elements in normal option
Y	independent variable

near steady state, the truncation error will be acceptable, but the corrector may not converge and thus the step size will be reduced by a factor of four. This may happen repeatedly if HMAX is too large, for both stiff and nonstiff systems.

Problems may also occur if the variables being simulated have widely varying values. If an important variable has a value of the order  $10^{-6}$ , then EPS, the error tolerance, must be set to about  $10^{-9}$ . This can use a great deal of computer time for the other variables. It may be necessary to scale the mathematical model so that there are no important variables with extremely small value.

DYNSYS 2.0 was generally tested with optimization level one of the CYBER 73 FTN compiler. Using the RUN compiler (a nonoptimizing compiler), erroneous results were obtained on some examples probably because of excessive round-off from the unoptimized code. Thus the user should ensure that an optimizing compiler should be used. Both DIFSUB and TRGB-TRGB2 should have at least twelve figure accuracy in the computer word length. Excessive round-off may affect the results, if a smaller word size is used.

### 7.3.3 Jacobian Evaluation

If the module o.d.e.s are stiff (Section 2.6), the Jacobian matrix must be supplied in the module. There are two ways of accomplishing this: the normal option for a general Jacobian matrix and a special option for a tri-diagonal Jacobian. The Jacobian does not have to be extremely accurate as it is needed only for the convergence of the corrector; for example, if the Jacobian is diagonally dominant, it may suffice to use only the diagonal coefficients.

#### 7.3.3.1 Normal Option

The row number, column number and value of each nonzero element are stored in arrays IROW, JCOL and XJACOB respectively. The elements may be stored in any order. For example the matrix:

$$J = \begin{bmatrix} 1.0 & 2.0 \\ 3.0 & 4.0 \end{bmatrix} \quad (7.3.1)$$

would be stored as:

```
NZERO = 4
IROW(1) = 1
JCOL(1) = 1
XJACOB(1) = 1.0
IROW(2) = 1
JCOL(2) = 2
XJACOB(2) = 2.0
IROW(3) = 2
JCOL(3) = 1
XJACOB(3) = 3.0
IROW(4) = 2
JCOL(4) = 2
XJACOB(4) = 4.0
```

NZERO is the number of nonzero elements in the matrix. Note also that a Jacobian element must be created for all diagonal elements whether or not they are nonzero.

Figure 8.7 depicts a simple example of a module using the normal option.

### 7.3.3.2. *Tri-diagonal Option*

The subdiagonal, diagonal and superdiagonal element values are stored in arrays *A*, *B* and *C* respectively.

*A*(1) and *C*(*N*) are not used. For example:



## (1) TRUNCATION ERROR IS TOO LARGE

This message may occur in critical regions of the simulation where some variables may be changing rapidly. The step size is too large and will be adjusted automatically by the executive.

(2) THE CORRECTOR FAILED TO CONVERGE IN 3 ITERATIONS  
DIFFERENTIAL EQUATIONS \_\_\_ TO \_\_\_

This message will be printed when the nonstiff option is used for a stiff system, or if the stiff option is used with direct iteration of the corrector for a stiff system. The stiff option should be used with Newton-Raphson iteration in either case. It may also occur if the problem is formulated incorrectly; i.e., there is an error in the differential equations or in the Jacobian, or the Jacobian is not accurate enough. The differential equations causing the problem are given. They are numbered in order of their occurrence in the simulation.

(3) THE CORRECTOR CANNOT BE SOLVED BECAUSE PW(\_\_\_\_) IS A  
PIVOT ELEMENT WHOSE VALUE IS BELOW XMIN DIFFERENTIAL  
EQUATIONS \_\_\_ TO \_\_\_ OPERATOR LIST WILL BE RE-CREATED

This message can appear in using the stiff option with Newton-Raphson iteration, i.e., the Bending-Hutchison linear equation solver. If it occurs more than once or

or twice, it may help to decrease the variable XMIN through the data set. The formulation of the differential equations and Jacobian should be checked.

(4) THE MAXIMUM ORDER SPECIFIED IS OUT OF RANGE

For the nonstiff option, the maximum order must be in the range 1-7. The range is 1-6 for the stiff option. The default value is 6 for both options, but the user may specify a value through the data set.

(5) CORRECTOR CONVERGENCE COULD NOT BE ACHIEVED FOR  
H.GT.HMIN DIFFERENTIAL EQUATIONS \_\_\_ TO \_\_\_

Message (2) will precede this one and the cause is the same, but this time the program will stop.

(6) THE STEP WAS TAKEN WITH H=HMIN BUT THE REQUESTED  
ERROR WAS NOT ACHIEVED

The user should try a larger value for the error tolerance or a smaller value of HMIN.

(7) THE REQUESTED ERROR IS SMALLER THAN CAN BE HANDLED FOR  
THIS PROBLEM

EPS has been specified equal to zero. As in (6), the user should try a larger tolerance.

## (8) ERROR IN TRGB

This means that the Jacobian matrix has not been properly defined; i.e., there is a redundant or inconsistent equation. It may also occur if the arrays are dimensioned too small.



## 8. SIMULATION EXAMPLES

Four relatively simple examples were simulated with the new DYN SYS 2.0 executive:

- (1) the level control of a simple stirred tank (nonstiff) with time delay
- (2) a network of 15 stirred tank reactors, 2 o.d.e.s per reactor, some stiff and some nonstiff
- (3) a tubular reactor with 222 stiff equations resulting from the discretization of partial differential equations
- (4) a tubular reactor with 49 stiff o.d.e.s with tri-diagonal Jacobian matrix.

The examples are not intended to be rigorously realistic, but they do demonstrate the types of problems which may be handled. A more complex simulation is described in Chapter 9. No system of units is employed since the examples are fictitious. The reader may assume any set of consistent units. Each example contains a listing of the modules and data set. A description of a data set is in Appendix F.

### 8.1 Level Control Example

This example is based on one in the DYNYSYS Manual (Bobrow *et al.*, 1970). Figure 8.1 is the process flow diagram for a simple mixing tank equipped with a level controller.

Liquid flows into a stirred tank at a fluctuating rate. The tank outlet stream is equipped with a valve which is manipulated by a level controller connected to the tank.

The response of the system to a simultaneous step change in inlet flow and temperature is to be simulated. The controller will attempt to keep the tank level at a certain set point by manipulating the flow rate through the outlet valve. Available in the DYNYSYS library of modules are mathematical models for:

- (1) a perfectly mixed tank (STIR1)
- (2) a valve with a parabolic characteristic (VALV1)
- (3) a proportional-integral controller (CONT1)
- (4) a time delay (DELAY).

Figure 8.2 is the dynamic information flow diagram for the system. There is a time delay between the tank and the controller and another between the controller and

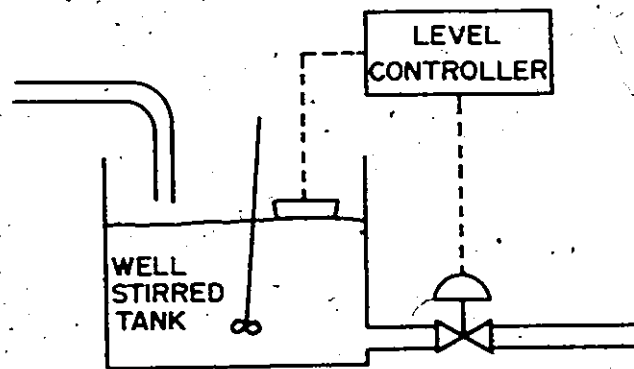


FIGURE 8.1 PROCESS FLOW DIAGRAM - LEVEL CONTROL EXAMPLE

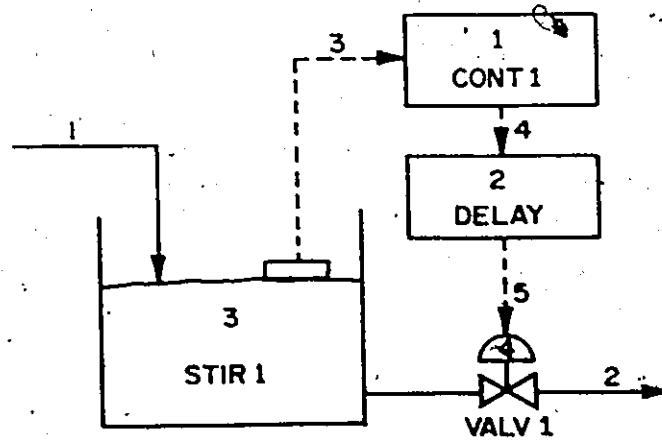


FIGURE 8.2 : DYNAMIC INFORMATION FLOW DIAGRAM - LEVEL CONTROL EXAMPLE

the valve but these can be represented together by one module.

STIR1 is the only differential module, i.e., the only one containing differential equations.

The unsteady state mass balance for each component  $i$  can be written:

$$\frac{dM_i}{dt} = F_{i,in} - F_{i,out} \quad (8.1.1)$$

where  $M_i \equiv$  mass of component  $i$  in the tank

$F_{i,in} \equiv$  inlet mass flow of component  $i$

$F_{i,out} \equiv$  outlet mass flow of component  $i$

in terms of the variables of DYNSSYS

$$\frac{dM_i}{dt} = F_{in} * x_{i,in} - F_{out} * x_{i,out} \quad (8.1.2)$$

where  $x_i \equiv$  component  $i$  mass fraction

$F \equiv$  total mass flow.

This equation can be integrated to give the mass holdup of the component,  $M_i$ .

Summation over all components  $i$  of  $M_i$

$$M = \sum_{i=1}^{NCOMP} M_i \quad (NCOMP - \text{number of components}) \quad (8.1.3)$$

gives the total mass holdup in the tank. The mass fractions of each component in the vessel, and hence in the outlet stream, may be obtained by division:

$$x_i = \frac{M_i}{M} \quad (8.1.4)$$

similarly the heat balance for the vessel:

$$\frac{dH}{dt} = H_{in} - H_{out} \quad (8.1.5)$$

becomes:

$$\frac{dH}{dt} = F_{in} * T_{in} * CP_{in} - F_{out} * T_{out} * CP_{out} \quad (8.1.6)$$

and  $H$  is obtained by integration.

Assuming constant heat capacity, the new vessel temperature is:

$$T = \frac{H}{M * CP} \quad (8.1.7)$$

The modules CONT1 and VALV1 are described in the DYN SYS Manual (Bobrow *et al.*, 1970).

The time delay module, DELAY, uses the "bucket brigade" approach to simulate a delayed variable. The past times and stream information are stored in vectors

and the stream output from the delay is interpolated from these values. The module is described more fully in Appendix G.

A listing of the modules (except DELAY) and the data set is shown in Figure 8.4. A listing of DELAY appears in Appendix G. A graph of the tank level versus time is depicted in Figure 8.3. After a little more than two time units, the tank level returns to its set point of 1000.

The old version of DYNYSYS required 20.8 seconds of execution time to simulate this example, while DYNYSYS 2.0 took 5.9 seconds. In both runs, HMAX, the maximum permissible step size was limited to 0.01. However, with DYNYSYS 2.0, HMAX had to be limited to 0.01 to get accurate results, while the AMOS version could use a maximum step of 0.05 and still be accurate. Possibly the Gear algorithm does not handle forcing functions as well as the Adams-Moulton-Shell routine. The Gear algorithm might function better if the order was limited to one or two when forcing functions are present.

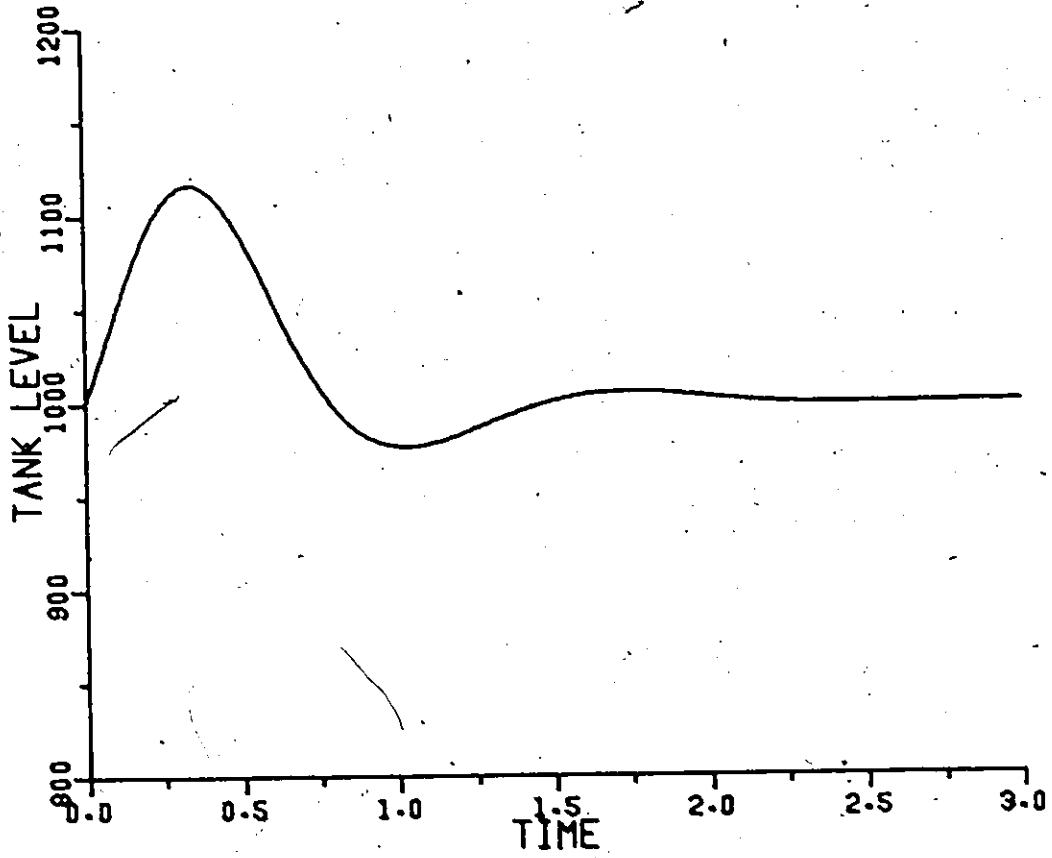


FIGURE 8.3: GRAPH OF TANK LEVEL VERSUS TIME FOR EXAMPLE#1

FIGURE 8.4: LISTING OF MODULES AND DATA SET FOR EXAMPLE #1  
 LEVEL CONTROL SYSTEM WITH TIME DELAY  
 LISTING OF TIME DELAY MODULE IS IN APPENDIX G

	SUBROUTINE TYPE1	T01	1
C		T01	2
C	SUBROUTINE VALV1	T01	3
C		T01	4
C	V-PORT (PARABOLIC) CONTROL VALVE	T01	5
C		T01	6
C	EQUIPMENT PARAMETERS	T01	7
C		T01	8
C	1 - NOT USED	T01	9
C	2 - VALVE CONSTANT	T01	10
C	3 - ACTION (+=DIRECT, -=REVERSE)	T01	11
C		T01	12
	COMMON /UNIT/ IM,NMP	T01	13
	COMMON /MAT/ MP(4,5),EP(4,5),S(2,5,7),EX(1)	T01	14
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NBGRAPH	T01	15
C		T01	16
	MI=MP(IM,3)	T01	17
	MO=IABS(MP(IM,4))	T01	18
C	CHECK THAT THE VALVE SIGNAL IS IN THE RANGE 0-100	T01	19
	V=S(1,MI,3)	T01	20
	IF (V.LT.0.) V=0.	T01	21
	IF (V.GT.100.) V=100.	T01	22
C	ACTION	T01	23
	A=EP(IM,3)	T01	24
	IF (A.LT.0.) S(1,MO,3)=EP(IM,2)*((100.-V)**2)	T01	25
	IF (A.GE.0.) S(1,MO,3)=EP(IM,2)*V**2	T01	26
	RETURN	T01	27
	END	T01	28



SUBROUTINE TYPE4

SUBROUTINE CONT1

PI OR ON/OFF CONTROLLER  
 SECOND OPTIONS ARE FOR ON/OFF CONTROLLER  
 IF EQUIPMENT PARAMETERS 4 AND 5 ARE LT.0, CONTROLLER IS ON/OFF

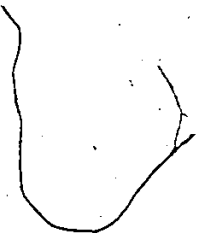
EQUIPMENT PARAMETERS

1 - CONTROLLED VARIABLE  
 2 - RANGE OR UPPER LIMIT  
 3 - SET POINT OR LOWER LIMIT  
 4 - PROPORTIONAL GAIN OR LT. 0  
 5 - INTEGRAL TIME OR LT. 0 , IF .LT. -10.0, OFF WHEN .GT. SETP

COMMON /UNIT/ IM,NMP  
 COMMON /MAT/ MP(4,5),EP(4,5),S(2,5,7),EX(1)  
 COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH

P1=EP(IM,4)  
 P2=EP(IM,5)  
 IN=MP(IM,3)  
 IOUT=IABS(MP(IM,4))  
 K=EP(IM,1)  
 IF (P1.LT.0.0.AND.P2.LE.0.0) GO TO 1  
 MEASURED VARIABLE AT CURRENT TIME  
 SIG1=S(1,IN,K)  
 MEASURED VARIABLE AT LAST TIME  
 SIG2=S(2,IN,K)  
 SCALE THE ERROR  
 SCALE=50.0/EP(IM,2)  
 PRESENT ERROR  
 ERR=(SIG1-EP(IM,3))\*SCALE  
 LAST ERROR  
 OLDER=(SIG2-EP(IM,3))\*SCALE  
 OUTPUT SIGNAL  
 S(1,IOUT,3)=P1\*(ERR-OLDER+P2\*(ERR+OLDER)\*0.5\*H)+S(2,IOUT,3)  
 IF (S(1,IOUT,3).GT.100.0) S(1,IOUT,3)=100.0  
 IF (S(1,IOUT,3).LT.0.0) S(1,IOUT,3)=0.0  
 RETURN  
 CONTINUE  
 ON/OFF CONTROLLER  
 IF (IG.EQ.1) RETURN  
 T=S(1,IN,K)  
 IF (P2.LT.-10.0) GO TO 2  
 IF (T.GE.EP(IM,2)) T=50.0  
 IF (T.LT.EP(IM,3)) T=0.0  
 GO TO 3  
 CONTINUE  
 IF (T.LE.EP(IM,2)) T=50.0  
 IF (T.GT.EP(IM,2)) T=0.0

T04 1  
 T04 2  
 T04 3  
 T04 4  
 T04 5  
 T04 6  
 T04 7  
 T04 8  
 T04 9  
 T04 10  
 T04 11  
 T04 12  
 T04 13  
 T04 14  
 T04 15  
 T04 16  
 T04 17  
 T04 18  
 T04 19  
 T04 20  
 T04 21  
 T04 22  
 T04 23  
 T04 24  
 T04 25  
 T04 26  
 T04 27  
 T04 28  
 T04 29  
 T04 30  
 T04 31  
 T04 32  
 T04 33  
 T04 34  
 T04 35  
 T04 36  
 T04 37  
 T04 38  
 T04 39  
 T04 40  
 T04 41  
 T04 42  
 T04 43  
 T04 44  
 T04 45  
 T04 46  
 T04 47  
 T04 48  
 T04 49  
 T04 50  
 T04 51  
 T04 52



3 CONTINUE  
S(1,IOUT,3)=T  
RETURN  
END

T04 53  
T04 54  
T04 55  
T04 56



```

SURROUTINE TYPE3
C
C STIR01 - TYPE 3
C .....
C GENERAL WELL MIXED, MULTIPLE INPUT/OUTPUT VESSEL
C PRESSURE EFFECTS ARE CONSIDERED( ASSUMING IDEAL GAS
C BEHAVIOUR ) IF A VOLUME IS SPECIFIED
C PARAMETERS.....
C -1- INITIAL HOLDUP
C -2- VOLUME
C -5- 1ST. OUTPUT STREAM, USED INTERNALLY
C 3 AND 4 ARE NOT SPECIFIED BY USER.
C .....
C COMMON /UNIT/ IM,NMP
C COMMON /MAT/ MP(4,5),EP(4,5),S(2,5,7),EX(1)
C COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
C COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF
C COMMON /PTAB/ IGFLAG,PP(10,10)
C COMMON /MODULE/ IDERY,ITER,ITRI,NZERO
C DIMENSION Y(10), CMPT(10)
C REAL MW
C
C USE DIRECT ITERATION IF STIFF OPTION IS USED (ITER=0)
C
C ITER=0
C NC1=NCOMP+1
C VOL=EP(IM,2)
C IF (JSTART.NE.0.OR.IG.EQ.1) GO TO 4
C FIND THE FIRST OUTPUT STREAM
C DO 1 J=3,NMP
C IS=IABS(MP(IM,J))
C IF (ABS(S(1,IS,2)).GT.10.) GO TO 1
C IF (MP(IM,J).LT.0) GO TO 2
1 CONTINUE
2 EP(IM,5)=J
  NOUT=EP(IM,5)
  IOUT=-MP(IM,NOUT)
C CALCULATE INITIAL HOLDUP IF NOT SPECIFIED.
  IF (VOL.GT.0.0.AND.EP(IM,1).LE.0.0) EP(IM,1)=VOL*SG(1,IS)
  IF (ABS(S(1,IS,2)).GT.10.) WRITE (6,15) MP(IM,1)
C CALCULATE INITIAL CONDITIONS FOR VARIABLES TO BE INTEGRATED,
C I.E.,HEAT AND MASSES.
C HEAT
  Y(1)=EP(IM,1)*S(IG,IOUT,4)*CP(IG,IOUT)
C MASSES
  DO 3 K=6,NC5
  KK=K-4
  Y(KK)=EP(IM,1)*S(IG,IOUT,K)
3 CONTINUE
4 CONTINUE
  NOUT=EP(IM,5)
  IOUT=-MP(IM,NOUT)

```

```

T03 1
T03 2
T03 3
T03 4
T03 5
T03 6
T03 7
T03 8
T03 9
T03 10
T03 11
T03 12
T03 13
T03 14
T03 15
T03 16
T03 17
T03 18
T03 19
T03 20
T03 21
T03 22
T03 23
T03 24
T03 25
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T03 30
T03 31
T03 32
T03 33
T03 34
T03 35
T03 36
T03 37
T03 38
T03 39
T03 40
T03 41
T03 42
T03 43
T03 44
T03 45
T03 46
T03 47
T03 48
T03 49
T03 50
T03 51
T03 52

```

C	CALCULATE DERIVATIVES	T03	53
5	CONTINUE	T03	54
	DO 6 J=1,NC1	T03	55
	CMPT(J)=0	T03	56
6	CONTINUE	T03	57
	DO 9 J=3,NMP	T03	58
	IS=IABS(MP(IM,J))	T03	59
	IF (IS.EQ.0) GO TO 9	T03	60
C	DO NOT INCLUDE CONTROLLER SIGNALS IN MASS AND HEAT BALANCES	T03	61
	IF (ABS(S(1,IS,2)).GT.10.) GO TO 9	T03	62
	SIGN=IS/MP(IM,J)	T03	63
C	ALLOW HEAT INPUT STREAM WITH FLAG = 9	T03	64
C	S( ,3) CONTAINS HEAT RATE IN BTU./MIN.	T03	65
	IF (ABS(S(1,IS,2)).LE.8.999) GO TO 7	T03	66
	CMPT(1)=CMPT(1)+S(1,IS,3)	T03	67
	GO TO 9	T03	68
7	CONTINUE	T03	69
C	HEAT	T03	70
	CMPT(1)=CMPT(1)+SIGN*S(IG,IS,4)*CP(IG,IS)*S(IG,IS,3)	T03	71
C	MASS BALANCES	T03	72
	DO 8 K=2,NC1	T03	73
	KK=K+4	T03	74
	CMPT(K)=CMPT(K)+SIGN*S(IG,IS,KK)*S(IG,IS,3)	T03	75
8	CONTINUE	T03	76
9	CONTINUE	T03	77
C	PREDICT/CORRECT NEW VALUES	T03	78
	CALL DIFSUB (NC1,Y,CMPT)	T03	79
C	NORMALIZE MASS FRACTIONS	T03	80
	SUM=0.0	T03	81
	DO 10 I=2,NC1	T03	82
	SUM=SUM+Y(I)	T03	83
10	CONTINUE	T03	84
	DO 11 K=6,NC5	T03	85
	S(1,IOUT,K)=Y(K-4)/SUM	T03	86
11	CONTINUE	T03	87
C	THE NEW TEMPERATURE	T03	88
	S(1,IOUT,4)=Y(1)/(SUM*CP(1,IOUT))	T03	89
	IF (IDERY.NE.0) GO TO 5	T03	90
C	INSERT THE NEW VALUES IN THE OUTPUT STREAMS	T03	91
	DO 13 J=3,NMP	T03	92
	IS=-MP(IM,J)	T03	93
	IF (IS.LE.0) GO TO 13	T03	94
	DO 12 K=4,NC5	T03	95
	S(1,IS,K)=S(1,IOUT,K)	T03	96
12	CONTINUE	T03	97
C	FOR CONTROLLER OUTPUT,HOLD-UP STORED IN PLACE OF FLOW	T03	98
	IF (ABS(S(1,IS,2)).GT.10.) S(1,IS,3)=SUM	T03	99
13	CONTINUE	T03	100
	IF (VOL.LE.0.0) RETURN	T03	101
C	PRESSURE	T03	102
	PRES=SUM*10.71*S(1,IOUT,4)/(VOL*MW(1,IOUT))	T03	103
	DO 14 I=3,NMP	T03	104

```
J=IABS(MP(IM,I))  
IF (J.EQ.0) GO TO.14  
S(1,J,5)=PRES  
14 CONTINUE  
RETURN  
C  
15 FORMAT (1H0,14H ERROR EQ. NO.,I3,22H ONLY OUTPUT IS SIGNAL)  
END
```

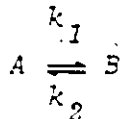
```
T03 105  
T03 106  
T03 107  
T03 108  
T03 109  
T03 110  
T03 111  
T03 112
```

.....  
 TEST OF SIMPLE LEVEL CONTROL SYSTEM, WITH STEP  
 CHANGE IN INLET FLOW RATE AND TEMPERATURE,  
 TIME DELAY INCLUDED.  
 .....

BEGIN					
IN/OUT	3.0				
TOLERANCE	0.001				
HMAX	0.01				
TIME	3.0				
LIBRARY	1.0				
DELAY	9.0				
PROCESS					
CONT1	1.0				
	3.0	-4.0			
	3.	1500.	1000.	2.	3.
DELAY	2.0				
	4.0	-5.0			
	0.1	0.0	100.0	3.0	0.0
STIR1	3.0				
	1.0	-2.0	-3.0		
	1000.				
VALV1	4.0				
	5.0	-2.0			
		1.0	1.0		
END					
STREAMS	5.				
EXPLICIT					
	1.	1.	1000.	100.	14.7
	1.				
	2.	1.	500.	50.	14.7
	1.				
	3.	11.	1000.		
	4.	11.	22.36068		
	5.	11.	22.36068		
END					
PROPERTIES	-1.				
END					
END					

## 8.2 Stirred Tank Reactor Network

Figures 8.5 and 8.6 represent the process flow diagram and the dynamic information flow diagram respectively for a network of CSTRs. The first order reversible reaction:



occurs in each reactor.

The differential equations for the module are the mass balances for components A and B:

$$\frac{dy_1}{dt} = -(k_1 + \frac{1}{\tau})y_1 + k_2y_2 + y_{1,0}/\tau \quad (8.2.1)$$

$$\frac{dy_2}{dt} = k_1y_1 - (k_2 + \frac{1}{\tau})y_2 + y_{2,0}/\tau$$

where  $y_1, y_2$   $\equiv$  mass fraction components A and B respectively

$\tau$   $\equiv$  reactor time constant (volume/volumetric flow rate)

$k_1, k_2$   $\equiv$  reaction rate constants

$y_{1,0}, y_{2,0}$   $\equiv$  mass fraction of components A and B respectively in inlet stream

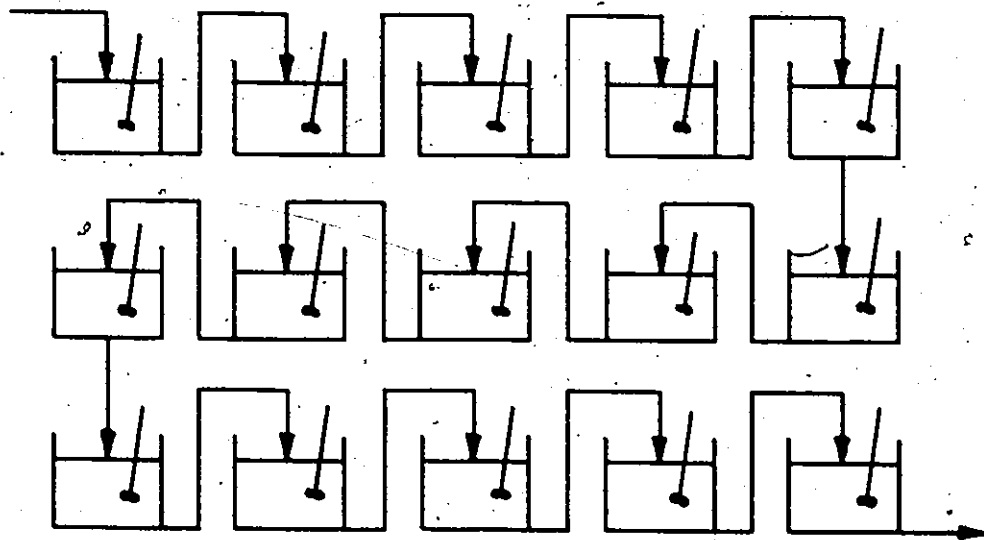


FIGURE 8.5 : PROCESS FLOW DIAGRAM - REACTOR NETWORK EXAMPLE

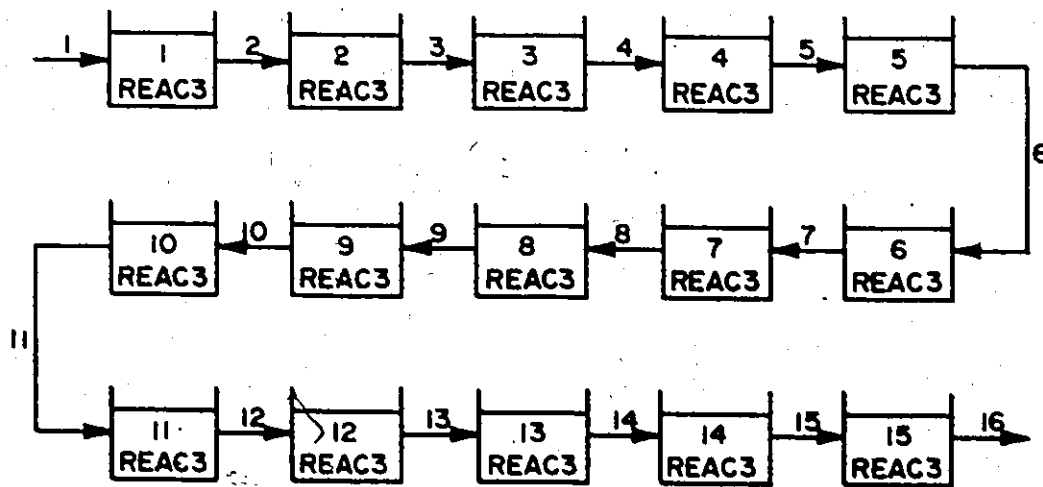


FIGURE 8.6 : DYNAMIC INFORMATION FLOW DIAGRAM - REACTOR NETWORK EXAMPLE



The Jacobian matrix of the system is :

$$J = \begin{bmatrix} -(k_1 + \frac{1}{\tau}) & + k_2 \\ k_1 & -(k_2 + \frac{1}{\tau}) \end{bmatrix} \quad (8.2.2)$$

The eigenvalues are  $-\frac{1}{\tau}$  and  $-(k_1 - k_2 - \frac{1}{\tau})$ .

By choosing values of  $k_1$ ,  $k_2$  and  $\tau$  we may create different stiffness ratios in each of the reactors. In practice the  $k$ 's may vary because of different temperatures in each reactor. Table 8.1 shows the values which were used for the simulation. This creates a mixture of non-stiff and stiff modules. It was found experimentally that if the stiffness ratio is greater than approximately 3.5:1, Newton-Raphson iteration of the corrector should be used, but that below 3.5:1 the execution time is faster for direct iteration.

DYNSYS 2.0 required 3.92 seconds of execution time. A listing of the module and its data set are given in Figure 8.7. A graph of the outlet concentration of A versus time for each reactor is shown in Figure 8.8. The curves are ordered from top to bottom in order of increasing module number. As the module number increases, so does the stiffness ratio as seen in Table 8.1. It can be seen in Figure 8.8 that the higher the stiffness ratio, the faster the reactor reaches steady state.

Table 8.1: Simulation Parameters For Reactor Network  
Example

Module	Tau	K1/K2	K1	K2	Eig1	Eig2	Stiffness Ratio
1	1.0	1.1	.26190	.23810	-1.0	-1.5	1.5
2	1.0	1.2	.54545	.45455	-1.0	-2.0	2.0
3	1.0	1.3	.84783	.65217	-1.0	-2.5	2.5
4	1.0	1.4	1.16667	.83333	-1.0	-3.0	3.0
5	1.0	1.5	5.40000	3.60000	-1.0	-10.0	10.0
6	1.0	1.6	30.15385	18.84615	-1.0	-50.0	50.0
7	1.0	1.7	62.33333	36.66667	-1.0	-100.0	100.0
8	1.0	1.8	127.92857	71.07143	-1.0	-200.0	200.0
9	1.0	1.9	261.41379	137.58621	-1.0	-400.0	400.0
10	1.0	2.0	399.33333	199.66667	-1.0	-600.0	600.0
11	1.0	2.1	541.25806	257.74194	-1.0	-800.0	800.0
12	1.0	2.2	686.81250	312.18750	-1.0	-1000.0	1000.0
13	1.0	2.3	835.66667	363.33333	-1.0	-1200.0	1200.0
14	1.0	2.4	1058.11765	440.88235	-1.0	-1500.0	1500.0
15	1.0	2.5	1427.85714	571.14286	-1.0	-2000.0	2000.0

FIGURE 8.7: LISTING OF MODULE AND DATA SET FOR EXAMPLE #2  
CSTR WITH FIRST ORDER REVERSIBLE REACTION (15 IN SERIES)

```

SUBROUTINE TYPE10
SUBROUTINE REAC1

THIS MODULE REPRESENTS A CSTR WITH A FIRST ORDER REVERSIBLE
REACTION, 1 INPUT-STREAM, 1 OUTPUT STREAM, CONSTANT TEMPERATURE.

EQUIPMENT PARAMETERS

1 - VOL - REACTOR VOLUME - FT**3
2 - K1 - FORWARD REACTION RATE CONSTANT - MIN**-1
3 - K2 - BACKWARD REACTION RATE CONSTANT - MIN**-1
4 - ITER - ITER=0 - USE DIRECT ITERATION WITH STIFF OPTION
           ITER=1 - USE NEWTON-RAPHSON ITERATION WITH STIFF OPTION
           FOR NON-STIFF OPTION ITER IS NOT USED

COMMON /MAT/ MP(15,5),EP(15,5),S(2,16,7),EX(1)
COMMON /CON/ IG, NCOMP, NC5, H, NE, NS, NPR, NPOL, TMAX, IORDER, NGRAPH
COMMON /PTAB/ IGFLAG, PP(10,10)
COMMON /UNIT/ IM
COMMON /ROW/ IROW(4)
COMMON /COLUMN/ JCOL(4)
COMMON /JACOB/ XJACOB(4)
COMMON /MODULE/ IDERY, ITER, ITRI, NZERO
DIMENSION Y(2), DERY(2)
REAL K1, K2

CALCULATE MODULE PARAMETERS.

GET VOLUME OF REACTOR FROM EP - VOL
VOL=EP(IM,1)
GET REACTION RATE CONSTANTS FROM EP - K1, K2
K1=EP(IM,2)
K2=EP(IM,3)
GET ITERATION OPTION FROM EP
ITER=EP(IM,4)
GET STREAM NUMBER OF INPUT STREAM FROM MP - IN
IN=MP(IM,3)
GET STREAM NUMBER OF OUTPUT STREAM FROM MP - IOUT
IOUT=IABS(MP(IM,4))
GET DENSITY OF INPUT STREAM FROM PP - DENS
DENS=PP(1,2)
CALCULATE REACTOR TIME CONSTANT - TAU
TAU=VOL/(S(IG,IN,3)/DENS)
GET INITIAL REACTOR CONCENTRATIONS FROM OUTPUT STREAM
Y(1)=S(IG,IOUT,6)
Y(2)=S(IG,IOUT,7)

```

T10 1  
T10 2  
T10 3  
T10 4  
T10 5  
T10 6  
T10 7  
T10 8  
T10 9  
T10 10  
T10 11  
T10 12  
T10 13  
T10 14  
T10 15  
T10 16  
T10 17  
T10 18  
T10 19  
T10 20  
T10 21  
T10 22  
T10 23  
T10 24  
T10 25  
T10 26  
T10 27  
T10 28  
T10 29  
T10 30  
T10 31  
T10 32  
T10 33  
T10 34  
T10 35  
T10 36  
T10 37  
T10 38  
T10 39  
T10 40  
T10 41  
T10 42  
T10 43  
T10 44  
T10 45  
T10 46  
T10 47  
T10 48

C	CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS	T10	49
C		T10	50
	IF (IG, EQ, 2) GO TO 1	T10	51
C	NZERO IS NUMBER OF NON-ZERO ELEMENTS IN JACOBIAN	T10	52
	NZERO=4	T10	53
	IROW(1)=1	T10	54
	JCOL(1)=1	T10	55
	XJACOB(1)=- (K1+1.0/TAU)	T10	56
	IROW(2)=2	T10	57
	JCOL(2)=1	T10	58
	XJACOB(2)=K1	T10	59
	IROW(3)=1	T10	60
	JCOL(3)=2	T10	61
	XJACOB(3)=K2	T10	62
	IROW(4)=2	T10	63
	JCOL(4)=2	T10	64
	XJACOB(4)=- (K2+1.0/TAU)	T10	65
	CONTINUE	T10	66
C		T10	67
C	CALCULATE DERIVATIVES	T10	68
C		T10	69
C	MASS BALANCE FOR COMPONENT A (UNITS=MIN**-1)	T10	70
C	RATE OF CHANGE OF MASS FRACTION OF COMPONENT A	T10	71
C	= RATE OF INPUT OF A	T10	72
C	-RATE OF OUTPUT OF A	T10	73
C	+RATE AT WHICH B REACTS INTO A	T10	74
C	-RATE AT WHICH A REACTS INTO B	T10	75
C	DERY(1)=- (K1+1.0/TAU)*Y(1)+K2*Y(2)+S(IG, IN, 6)/TAU	T10	76
C	MASS BALANCE FOR COMPONENT B (UNITS=MIN**-1)	T10	77
C	RATE OF CHANGE OF MASS FRACTION OF COMPONENT B	T10	78
C	= RATE OF INPUT OF B	T10	79
C	-RATE OF OUTPUT OF B	T10	80
C	+RATE AT WHICH A REACTS INTO B	T10	81
C	-RATE AT WHICH B REACTS INTO A	T10	82
C	DERY(2)=K1*Y(1)- (K2+1.0/TAU)*Y(2)+S(IG, IN, 7)/TAU	T10	83
C		T10	84
C	CALL DIFSUB TO SOLVE ODES FOR MODULE	T10	85
C		T10	86
	CALL DIFSUB (2, Y, DERY)	T10	87
	IF (IDERY.NE.0) GO TO 1	T10	88
C		T10	89
C	CALCULATE STREAM OUTPUT	T10	90
C		T10	91
C	NORMALIZE CONCENTRATIONS OF A AND B	T10	92
	SUM=Y(1)+Y(2)	T10	93
	Y(1)=Y(1)/SUM	T10	94
	Y(2)=Y(2)/SUM	T10	95
C	PUT MASS FRACTIONS INTO OUTPUT STREAMS	T10	96
	S(1, IOUT, 6)=Y(1)	T10	97
	S(1, IOUT, 7)=Y(2)	T10	98
	RETURN	T10	99
	END	T10	100

\*\*\*\*\*  
 SIMULATION OF 15 CSTRS IN SERIES  
 \*\*\*\*\*

BEGIN				
HMAX	1.0			
TIME	1.0			
COMPS	2.0			
LIBRARY	1.0			
REAC1	10.0			
PROCESS				
REAC1	1.0			
	1.0	-2.0		
REAC1	10.0	0.26190	0.23810	0.0
	2.0			
	2.0	-3.0		
REAC1	10.0	0.54545	0.45455	0.0
	3.0			
	3.0	-4.0		
REAC1	10.0	0.84783	0.65217	0.0
	4.0			
	4.0	-5.0		
REAC1	10.0	1.16667	0.83333	0.0
	5.0			
	5.0	-6.0		
REAC1	10.0	5.4	3.6	0.0
	6.0			
	6.0	-7.0		
REAC1	10.0	30.154	18.846	1.0
	7.0			
	7.0	-8.0		
REAC1	10.0	62.333	36.667	1.0
	8.0			
	8.0	-9.0		
REAC1	10.0	127.929	71.071	1.0
	9.0			
	9.0	-10.0		
REAC1	10.0	261.41	137.59	1.0
	10.0			
	10.0	-11.0		
REAC1	10.0	399.33	199.67	1.0
	11.0			
	11.0	-12.0		
REAC1	10.0	541.26	257.74	1.0
	12.0			
	12.0	-13.0		
REAC1	10.0	686.81	312.19	1.0
	13.0			
	13.0	-14.0		
REAC1	10.0	835.67	363.33	1.0
	14.0			
	14.0	-15.0		

REAC1	10.0	1058.12	440.88	1.0	
	15.0				
	15.0	-16.0			
	10.0	1427.86	571.14	1.0	
END STREAMS EXPLICIT	16.0				
	1.0	1.0	624.0	60.0	14.7
	1.0	0.0			
	2.0	1.0	624.0	65.0	14.7
	1.0	0.0			
	3.0	1.0	624.0	70.0	14.7
	1.0	0.0			
	4.0	1.0	624.0	75.0	14.7
	1.0	0.0			
	5.0	1.0	624.0	80.0	14.7
	1.0	0.0			
	6.0	1.0	624.0	85.0	14.7
	1.0	0.0			
	7.0	1.0	624.0	90.0	14.7
	1.0	0.0			
	8.0	1.0	624.0	95.0	14.7
	1.0	0.0			
	9.0	1.0	624.0	100.0	14.7
	1.0	0.0			
	10.0	1.0	624.0	105.0	14.7
	1.0	0.0			
	11.0	1.0	624.0	110.0	14.7
	1.0	0.0			
	12.0	1.0	624.0	115.0	14.7
	1.0	0.0			
	13.0	1.0	624.0	120.0	14.7
	1.0	0.0			
	14.0	1.0	624.0	125.0	14.7
	1.0	0.0			
	15.0	1.0	624.0	130.0	14.7
	1.0	0.0			
	16.0	1.0	624.0	135.0	14.7
	1.0	0.0			
END PROPERTIES END	-1.0				
	END				

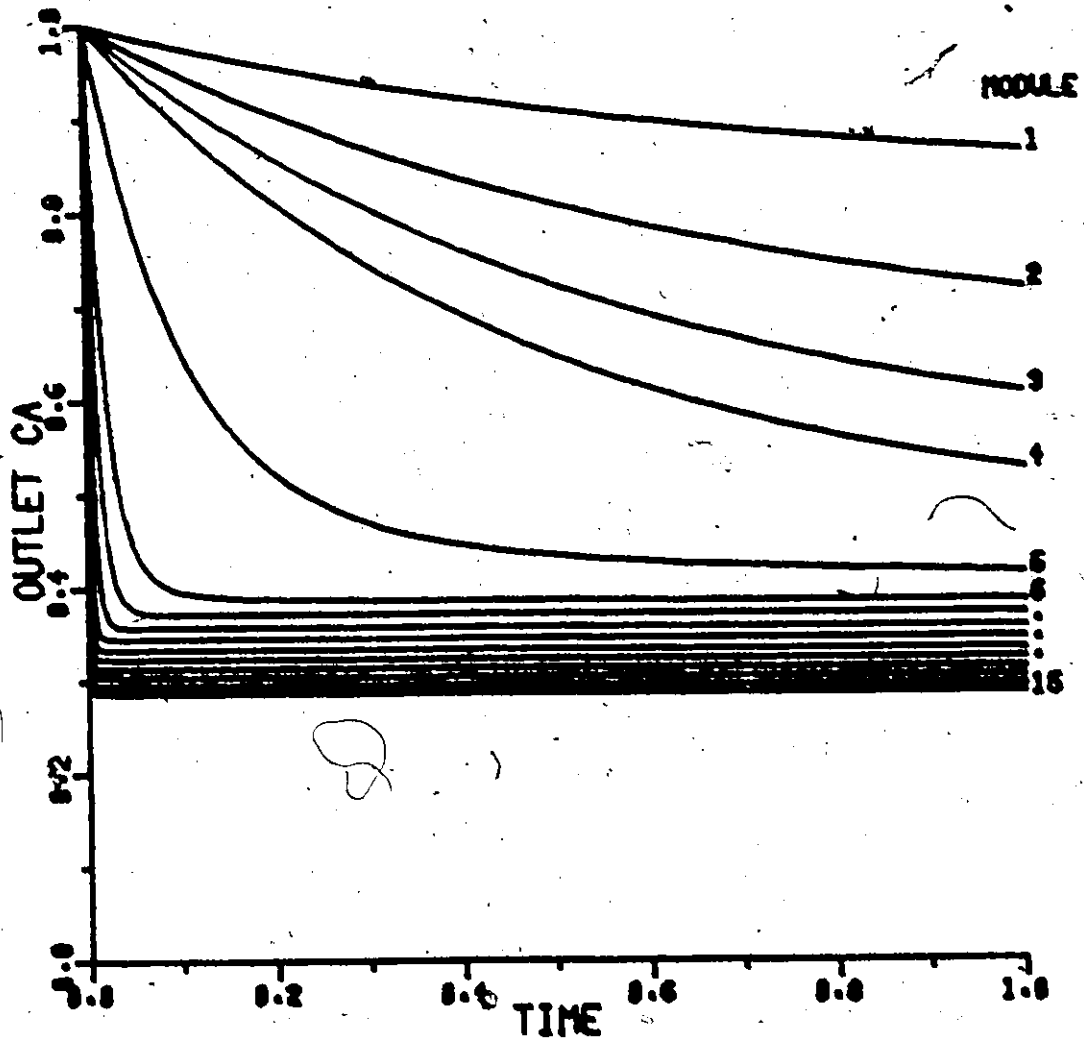


FIGURE 8.8: GRAPH OF OUTLET CONCENTRATION OF A VERSUS TIME FOR EXAMPLE#2

### 8.3 Tubular Reactor Simulation

An exothermal tubular chemical reactor based on Chapter 7 of the IMP Manual (Brandon, 1972) was simulated. A diagram of the reactor is shown in Figure 8.8. The following reaction occurs within the reactor:



The partial differential equations describing the process are:

$$\begin{aligned} \frac{\partial C_A}{\partial t} &= -k_1 e^{a_1 T} C_A^2 + D_1 \frac{\partial^2 C_A}{\partial z^2} - V \frac{\partial C_A}{\partial z} \\ \frac{\partial C_B}{\partial t} &= -k_2 e^{a_2 T} C_B + \frac{k_1}{2} e^{a_1 T} C_A^2 + D_2 \frac{\partial^2 C_B}{\partial z^2} - V \frac{\partial C_B}{\partial z} \\ \frac{\partial T}{\partial t} &= k_1 e^{a_1 T} C_A^2 H_1 + k_2 e^{a_2 T} C_B H_2 + D_3 \frac{\partial^2 T}{\partial z^2} - V \frac{\partial T}{\partial z} \end{aligned} \quad (8.3.2)$$

Reaction, diffusion and convection terms are included in the mathematical model. The following boundary and initial conditions apply:

$$\begin{aligned} C_A &= 0 \text{ at } t = 0 \text{ for all } z \\ C_B &= 0 \text{ at } t = 0 \text{ for all } z \\ T &= 0 \text{ at } t = 0 \text{ for all } z \end{aligned}$$



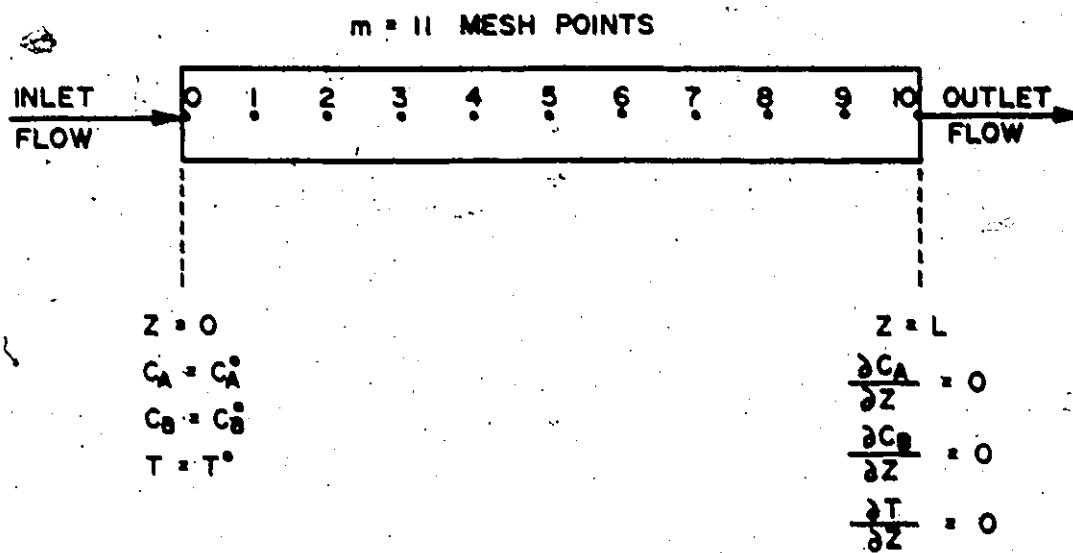


FIGURE 8.9: SCHEMATIC OF TUBULAR REACTOR

$$C_A = C_A^0 \quad \text{at } z = 0 \quad \text{for all } t$$

$$C_B = C_B^0 \quad \text{at } z = 0 \quad \text{for all } t$$

$$T = T^0 \quad \text{at } z = 0 \quad \text{for all } t$$

(8.3.3)

$$\frac{\partial C_A}{\partial z} = 0 \quad \text{at } z = L \quad \text{for all } t$$

$$\frac{\partial C_B}{\partial z} = 0 \quad \text{at } z = L \quad \text{for all } t$$

$$\frac{\partial T}{\partial z} = 0 \quad \text{at } z = L \quad \text{for all } t$$

where

$C_A$   $\equiv$  concentration of component A

$C_B$   $\equiv$  concentration of component B

$T$   $\equiv$  temperature

$t$   $\equiv$  time

$z$   $\equiv$  space coordinate along reactor axis

$C_A^0$   $\equiv$  feed concentration of component A

$C_B^0$   $\equiv$  feed concentration of component B

$T^0$   $\equiv$  inlet temperature

$L$   $\equiv$  reactor length

$v$   $\equiv$  fluid velocity

$D_1$   $\equiv$  diffusivity of component A

$D_2$   $\equiv$  diffusivity of component B

$D_3$   $\equiv$  thermal diffusivity

$k_1$   $\equiv$  reaction one rate constant

$k_2$   $\equiv$  reaction two rate constant

$\alpha_1$   $\equiv$  reaction one activation constant

$a_2$   $\equiv$  reaction two activation constant

$H_1$   $\equiv$  heat of reaction one

$H_2$   $\equiv$  heat of reaction two

The values of the parameters for this simulation are:

$$\begin{array}{lll}
 D_1 = 30.0 & H_1 = 1.0 & a_2 = 0.07 \\
 D_2 = 20.0 & H_2 = 50.0 & C_A^0 = 10.0 \\
 D_3 = 90.0 & k_1 = 1.5 & C_B^0 = 0.0 \\
 V = 100.0 & k_2 = 0.00002 & T^0 = 100.0 \\
 L = 100.0 & a_1 = 0.01 & t_{max} = 5.0
 \end{array} \quad (8.3.4)$$

To use DYNYSYS the set of partial differential equations must be broken down into a set of ordinary differential equations. To do this the length of the reactor is divided into  $M$  mesh points as shown in Figure 8.9. Each mesh point will have 3 o.d.e.s (except for the inlet and outlet):

Replacing  $\frac{\partial X}{\partial Z}$  and  $\frac{\partial^2 X}{\partial Z^2}$  by

$$\frac{\partial X}{\partial Z} = \frac{X|_Z - X|_{Z-\Delta Z}}{\Delta Z} \quad (8.3.5)$$

$$\frac{\partial^2 X}{\partial Z^2} = \frac{X|_{Z+\Delta Z} - 2X|_Z + X|_{Z-\Delta Z}}{(\Delta Z)^2}$$

we obtain:

$$\begin{aligned} \frac{dC_{A,i}}{dt} &= -k_1 e^{a_1 T_i} C_{A,i}^2 + D_1 \left[ \frac{C_{A,i+1} - 2C_{A,i} + C_{A,i-1}}{(\Delta Z)^2} \right] \\ &\quad - V \left[ \frac{C_{A,i+1} - C_{A,i}}{\Delta Z} \right] \\ \frac{dC_{B,i}}{dt} &= -k_2 e^{a_2 T_i} C_{B,i} + \frac{k_1}{2} e^{a_1 T_i} C_{A,i}^2 + D_2 \left[ \frac{C_{B,i+1} - 2C_{B,i} + C_{B,i-1}}{(\Delta Z)^2} \right] \\ &\quad - V \left[ \frac{C_{B,i+1} - C_{B,i}}{\Delta Z} \right] \end{aligned} \quad (8.3.6)$$

$$\begin{aligned} \frac{dT_i}{dt} &= k_1 e^{a_1 T_i} C_{A,i}^{H_1} + k_2 e^{a_2 T_i} C_{B,i}^{H_2} + D_3 \left[ \frac{T_{i+1} - 2T_i + T_{i-1}}{(\Delta Z)^2} \right] \\ &\quad - V \left[ \frac{T_i - T_{i-1}}{\Delta Z} \right] \end{aligned}$$

The equations are very stiff.

The equations at mesh point 1 are:

$$\begin{aligned} \frac{dy_1}{dt} &= -k_1 e^{a_1 y_3} y_1^2 + D_1 \left[ \frac{y_4 - 2y_1 + y_{10}}{(\Delta Z)^2} \right] - V \left[ \frac{y_4 - y_1}{\Delta Z} \right] \\ \frac{dy_2}{dt} &= -k_2 e^{a_2 y_3} y_2 + \frac{k_1}{2} e^{a_1 y_3} y_1^2 + D_2 \left[ \frac{y_5 - 2y_2 + y_{20}}{(\Delta Z)^2} \right] \\ &\quad - V \left[ \frac{y_5 - y_2}{\Delta Z} \right] \quad (8.3.7) \\ \frac{dy_3}{dt} &= k_1 e^{a_1 y_3} y_1^{2H_1} + k_2 e^{a_2 y_3} y_2^{H_2} + D_3 \left[ \frac{y_6 - 2y_3 + y_{30}}{(\Delta Z)^2} \right] - V \left[ \frac{y_5 - y_2}{\Delta Z} \right] \end{aligned}$$

The equations at mesh points 2, 3, ..., M-2 are:

$$\frac{dy_i}{dt} = -k_1 e^{a_1 y_{i+2}} y_i^2 + D_1 \left[ \frac{y_{i+3} - 2y_i + y_{i-3}}{(\Delta Z)^2} \right] - V \left[ \frac{y_{i+3} - y_i}{\Delta Z} \right]$$

$$\begin{aligned} \frac{dy_{i+1}}{dt} = & -k_2 e^{a_2 y_{i+2}} y_{i+1} + \frac{k_1}{2} e^{a_1 y_{i+2}} y_i^2 \\ & + D_2 \left[ \frac{y_{i+4} - 2y_{i+1} + y_{i-2}}{(\Delta Z)^2} \right] - V \left[ \frac{y_{i+4} - y_{i+1}}{\Delta Z} \right] \end{aligned} \quad (8.3.8)$$

$$\begin{aligned} \frac{dy_{i+2}}{dt} = & -k_1 e^{a_1 y_{i+2}} y_i^2 H_1 + k_2 e^{a_2 y_{i+2}} y_{i+1} H_2 \\ & + D_3 \left[ \frac{y_{i+5} - 2y_{i+2} + y_{i-1}}{(\Delta Z)^2} \right] - V \left[ \frac{y_{i+5} - y_{i+2}}{\Delta Z} \right] \end{aligned}$$

The equations at the last mesh point are (i=3M-5):

$$\frac{dy_i}{dt} = -k_1 e^{a_1 y_{i+2}} y_i^2 + D_1 \left[ \frac{y_{i-3} - y_i}{(\Delta Z)^2} \right]$$

$$\frac{dy_{i+1}}{dt} = -k_2 e^{a_2 y_{i+2}} y_{i+1} + \frac{k_1}{2} e^{a_1 y_{i+2}} y_i^2 + D_2 \left[ \frac{y_{i-2} - y_{i+1}}{(\Delta Z)^2} \right] \quad (8.3.9)$$

$$\frac{dy_{i+2}}{dt} = k_1 e^{a_1 y_{i+2}} y_i^2 H_1 + k_2 e^{a_2 y_{i+2}} y_{i+1} H_2 + D_3 \left[ \frac{y_{i-1} - y_{i+2}}{(\Delta Z)^2} \right]$$

where  $y_1, y_4, y_7, \dots, y_{3M-5}$  refer to  $C_A$

$y_2, y_5, y_8, \dots, y_{3M-4}$  refer to  $C_B$

$y_3, y_6, y_9, \dots, y_{3M-3}$  refer to  $T$

76 mesh points were taken giving a total of 222 o.d.e.s.

The execution time for the simulation was 102.5 seconds while an equation-oriented version of Gear's method took 114.2 seconds. The IMP package required 577.7 seconds.

A listing of the module and its data set are given in Figure 8.10.

Graphs of the outlet values of  $C_A$ ,  $C_B$  and  $T$  versus time are depicted in Figures 8.11, 8.12 and 8.13 respectively. No change in the outlet values is seen until about 0.6 time units. Steady state is reached after about 1.5.  $C_A$  rises to a fairly sharp peak at about 1.0 and then falls to a steady state of about 0.226. The graphs of  $C_B$  and  $T$  are sigmoid in appearance.

FIGURE 8.10: LISTING OF MODULE AND DATA SET FOR EXAMPLE #3  
TUBULAR REACTOR (222 STIFF ODES)

C	SUBROUTINE TYPES	T05	1
C		T05	2
C	SUBROUTINE REAC3	T05	3
C		T05	4
C	THIS MODULE REPRESENTS A TUBULAR REACTOR WITH	T05	5
C	REACTION 2C1-C2-C3	T05	6
C	FEED CONCENTRATIONS MUST BE FAIRLY CONSTANT OTHERWISE ODES	T05	7
C	MUST BE SOLVED FOR C3	T05	8
C		T05	9
C	EQUIPMENT PARAMETERS	T05	10
C		T05	11
C	1 - D1 - DIFFUSIVITY OF COMPONENT 1	T05	12
C	2 - D2 - DIFFUSIVITY OF COMPONENT 2	T05	13
C	3 - D3 - THERMAL DIFFUSIVITY	T05	14
C	4 - V - FLUID VELOCITY	T05	15
C	5 - L - LENGTH OF REACTOR	T05	16
C	6 - H1 - HEAT OF REACTION 1	T05	17
C	7 - H2 - HEAT OF REACTION 2	T05	18
C	8 - K1 - REACTION 1 RATE CONSTANT	T05	19
C	9 - K2 - REACTION 2 RATE CONSTANT	T05	20
C	10 - A1 - REACTION 1 ACTIVATION CONSTANT	T05	21
C	11 - A2 - REACTION 2 ACTIVATION CONSTANT	T05	22
C	12 - M - NUMBER OF SECTIONS IN REACTOR	T05	23
C		T05	24
C	COMMON /MAT/ MP(1,5),EP(1,5),S(2,2,8),EX(7)	T05	25
C	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	T05	26
C	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	T05	27
C	COMMON /UNIT/ IM,NMP	T05	28
C	COMMON /ROW/ IROW(1030)	T05	29
C	COMMON /COLUMN/ JCOL(1030)	T05	30
C	COMMON /JACOB/ DFDY(1030)	T05	31
C	COMMON /MODULE/ IDERY,ITER,ITRI,NZERO	T05	32
C	REAL K1,K2,K1K,K2K,L	T05	33
C	DIMENSION Y(222), DERY(222)	T05	34
C		T05	35
C	CALCULATE MODULE PARAMETERS	T05	36
C		T05	37
C		T05	38
C	USE NEWTON-RAPHSON ITERATION WITH STIFF OPTION (ITER=1)	T05	39
C		T05	40
C	ITER=1	T05	41
C	IF (JSTART.NE.0.OR.IG.EQ.1) GO TO 2	T05	42
C	D1=EP(IM,1)	T05	43
C	D2=EP(IM,2)	T05	44
C	D3=EP(IM,3)	T05	45
C	V=EP(IM,4)	T05	46
C	L=EP(IM,5)	T05	47
C	NEX=MP(IM,NMP+1)	T05	48

```

H1=EX(NEX)
H2=EX(NEX+1)
K1=EX(NEX+2)
K2=EX(NEX+3)
A1=EX(NEX+4)
A2=EX(NEX+5)
M=EX(NEX+6)
DZ=L/FLOAT(M)
DZ2=DZ*DZ
D1DZ2=D1/DZ2
D2DZ2=D2/DZ2
D3DZ2=D3/DZ2
VDZ=V/DZ

```

```

C
C
C
CALCULATE INITIAL CONDITIONS

```

```

N=3*M-3
IN=MP(IM,3)
IOUT=IABS(MP(IM,4))
IF=N-2
DO 1 I=1,IF,3
Y(I)=S(2,IOUT,3)*S(2,IOUT,6)
Y(I+1)=S(2,IOUT,3)*S(2,IOUT,7)
Y(I+2)=S(2,IOUT,4)

```

```

1
2
CONTINUE

```

```

CONTINUE

```

```

C
NZERO IS NUMBER OF NONZERO ELEMENTS IN JACOBIAN
NZERO=14*M-20

```

```

C
C
C
CALCULATE FEED CONDITIONS

```

```

C10=S(IG,IN,3)*S(IG,IN,6)
C20=S(IG,IN,3)*S(IG,IN,7)
T0=S(IG,IN,4)

```

```

C
C
C
CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS

```

```

IF (IG.EQ.2) GO TO 4

```

```

C
C
C
MESH POINT 1

```

```

K1K=K1*EXP(A1*Y(3))
K2K=K2*EXP(A2*Y(3))
IROW(1)=1
JCOL(1)=1
DFDY(1)=-2.0*K1K*Y(1)-2.0*D1DZ2-VDZ
IROW(2)=1
JCOL(2)=3
DFDY(2)=-A1*K1K*Y(1)*Y(1)
IROW(3)=1
JCOL(3)=4
DFDY(3)=D1DZ2

```

```

T05 49
T05 50
T05 51
T05 52
T05 53
T05 54
T05 55
T05 56
T05 57
T05 58
T05 59
T05 60
T05 61
T05 62
T05 63
T05 64
T05 65
T05 66
T05 67
T05 68
T05 69
T05 70
T05 71
T05 72
T05 73
T05 74
T05 75
T05 76
T05 77
T05 78
T05 79
T05 80
T05 81
T05 82
T05 83
T05 84
T05 85
T05 86
T05 87
T05 88
T05 89
T05 90
T05 91
T05 92
T05 93
T05 94
T05 95
T05 96
T05 97
T05 98
T05 99
T05 100

```



IROW(4)=2	T05 101
JCOL(4)=1	T05 102
DFDY(4)=K1K*Y(1)	T05 103
IROW(5)=2	T05 104
JCOL(5)=2	T05 105
DFDY(5)=-K2K-2.0*D2DZ2-VDZ	T05 106
IROW(6)=2	T05 107
JCOL(6)=3	T05 108
DFDY(6)=-A2*K2K*Y(2)+0.5*A1*K1K*Y(1)*Y(1)	T05 109
IROW(7)=2	T05 110
JCOL(7)=5	T05 111
DFDY(7)=D2DZ2	T05 112
IROW(8)=3	T05 113
JCOL(8)=1	T05 114
DFDY(8)=2.0*K1K*H1*Y(1)	T05 115
IROW(9)=3	T05 116
JCOL(9)=2	T05 117
DFDY(9)=K2K*H2	T05 118
IROW(10)=3	T05 119
JCOL(10)=3	T05 120
DFDY(10)=A1*K1K*H1*Y(1)*Y(1)+A2*K2K*H2*Y(2)-2.0*D3DZ2-VDZ	T05 121
IROW(11)=3	T05 122
JCOL(11)=6	T05 123
DFDY(11)=D3DZ2	T05 124
MESH POINTS 2,3,....,M-2	T05 125
J=1	T05 126
IF=14*M-44	T05 127
DO 3, I=12, IF, 14	T05 128
J=J+3	T05 129
K1K=K1*EXP(A1*Y(J+2))	T05 130
K2K=K2*EXP(A2*Y(J+2))	T05 131
IROW(I)=J	T05 132
JCOL(I)=J-3	T05 133
DFDY(I)=D1DZ2+VDZ	T05 134
IROW(I+1)=J	T05 135
JCOL(I+1)=J	T05 136
DFDY(I+1)=-2.0*K1K*Y(J)-2.0*D1DZ2-VDZ	T05 137
IROW(I+2)=J	T05 138
JCOL(I+2)=J+2	T05 139
DFDY(I+2)=-A1*K1K*Y(J)*Y(J)	T05 140
IROW(I+3)=J	T05 141
JCOL(I+3)=J+3	T05 142
DFDY(I+3)=D1DZ2	T05 143
IROW(I+4)=J+1	T05 144
JCOL(I+4)=J-2	T05 145
DFDY(I+4)=D2DZ2+VDZ	T05 146
IROW(I+5)=J+1	T05 147
JCOL(I+5)=J	T05 148
DFDY(I+5)=K1K*Y(J)	T05 149
IROW(I+6)=J+1	T05 150
	T05 151
	T05 152

C  
C  
C

JCOL (I+6)=J+1	T05 153
DFDY (I+6)=-K2K-2.0*D2DZ2=VDZ	T05 154
IROW (I+7)=J+1	T05 155
JCOL (I+7)=J+2	T05 156
DFDY (I+7)=-A2*K2K*Y(J+1)+0.5*A1*K1K*Y(J)*Y(J)	T05 157
IROW (I+8)=J+1	T05 158
JCOL (I+8)=J+4	T05 159
DFDY (I+8)=D2DZ2	T05 160
IROW (I+9)=J+2	T05 161
JCOL (I+9)=J-1	T05 162
DFDY (I+9)=D3DZ2+VDZ	T05 163
IROW (I+10)=J+2	T05 164
JCOL (I+10)=J	T05 165
DFDY (I+10)=2.0*K1K*H1*Y(J)	T05 166
IROW (I+11)=J+2	T05 167
JCOL (I+11)=J+1	T05 168
DFDY (I+11)=K2K*H2	T05 169
IROW (I+12)=J+2	T05 170
JCOL (I+12)=J+2	T05 171
DFDY (I+12)=A1*K1K*H1*Y(J)*Y(J)+A2*K2K*H2*Y(J+1)-2.0*D3DZ2=VDZ	T05 172
IROW (I+13)=J+2	T05 173
JCOL (I+13)=J+5	T05 174
DFDY (I+13)=D3DZ2	T05 175
CONTINUE	T05 176
	T05 177
MESH POINT M-1	T05 178
	T05 179
I=14*M-30	T05 180
J=3*M-5	T05 181
K1K=K1*EXP(A1*Y(J+2))	T05 182
K2K=K2*EXP(A2*Y(J+2))	T05 183
IROW (I)=J	T05 184
JCOL (I)=J-3	T05 185
DFDY (I)=D1DZ2+VDZ	T05 186
IROW (I+1)=J	T05 187
JCOL (I+1)=J	T05 188
DFDY (I+1)=-2.0*K1K*Y(J)-D1DZ2=VDZ	T05 189
IROW (I+2)=J	T05 190
JCOL (I+2)=J+2	T05 191
DFDY (I+2)=-A1*K1K*Y(J)*Y(J)	T05 192
IROW (I+3)=J+1	T05 193
JCOL (I+3)=J-2	T05 194
DFDY (I+3)=D2DZ2+VDZ	T05 195
IROW (I+4)=J+1	T05 196
JCOL (I+4)=J	T05 197
DFDY (I+4)=K1K*Y(J)	T05 198
IROW (I+5)=J+1	T05 199
JCOL (I+5)=J+1	T05 200
DFDY (I+5)=-K2K-D2DZ2=VDZ	T05 201
IROW (I+6)=J+1	T05 202
JCOL (I+6)=J+2	T05 203
DFDY (I+6)=-A2*K2K*Y(J+1)+0.5*A1*K1K*Y(J)*Y(J)	T05 204

```

IROW(I+7)=J+2
JCOL(I+7)=J-1
DFDY(I+7)=D3DZ2+VDZ
IROW(I+8)=J+2
JCOL(I+8)=J
DFDY(I+8)=2.0*H1*K1K*Y(J)
IROW(I+9)=J+2
JCOL(I+9)=J+1
DFDY(I+9)=K2K*H2
IROW(I+10)=J+2
JCOL(I+10)=J+2
DFDY(I+10)=A1*K1K*H1*Y(J)*Y(J)+A2*K2K*H2*Y(J+1)-D3DZ2-VDZ
CONTINUE

```

```

CALCULATE DERIVATIVES

```

```

MESH POINT 1

```

```

K1K=K1*EXP(A1*Y(3))
K2K=K2*EXP(A2*Y(3))
DERY(1)=-K1K*Y(1)*Y(1)+D1DZ2*(Y(4)-2.0*Y(1)+C10)-VDZ*(Y(1)-C10)
DERY(2)=-K2K*Y(2)+0.5*K1K*Y(1)*Y(1)+D2DZ2*(Y(5)-2.0*Y(2)+C20)-VDZ*
1(Y(2)-C20)
DERY(3)=K1K*H1*Y(1)*Y(1)+K2K*H2*Y(2)+D3DZ2*(Y(6)-2.0*Y(3)+T0)-VDZ*
1(Y(3)-T0)

```

```

MESH POINTS 2,3,...,M-2

```

```

IF=3*M-8
DO 5 I=4,IF,3
K1K=K1*EXP(A1*Y(I+2))
K2K=K2*EXP(A2*Y(I+2))
DERY(I)=-K1K*Y(I)*Y(I)+D1DZ2*(Y(I+3)-2.0*Y(I)+Y(I-3))-VDZ*(Y(I)-Y(I-3))
DERY(I+1)=-K2K*Y(I+1)+0.5*K1K*Y(I)*Y(I)+D2DZ2*(Y(I+4)-2.0*Y(I+1)+Y(I-2))-VDZ*(Y(I+1)-Y(I-2))
DERY(I+2)=K1K*H1*Y(I)*Y(I)+K2K*H2*Y(I+1)+D3DZ2*(Y(I+5)-2.0*Y(I+2)+Y(I-1))-VDZ*(Y(I+2)-Y(I-1))
CONTINUE

```

```

MESH POINT M-1

```

```

I=3*M-8
K1K=K1*EXP(A1*Y(I+2))
K2K=K2*EXP(A2*Y(I+2))
DERY(I)=-K1K*Y(I)*Y(I)+D1DZ2*(Y(I+3)-Y(I))-VDZ*(Y(I)-Y(I-3))
DERY(I+1)=-K2K*Y(I+1)+0.5*K1K*Y(I)*Y(I)+D2DZ2*(Y(I+2)-Y(I+1))-VDZ*(Y(I+1)-Y(I-2))
DERY(I+2)=K1K*H1*Y(I)*Y(I)+K2K*H2*Y(I+1)+D3DZ2*(Y(I+1)-Y(I+2))-VDZ*(Y(I+2)-Y(I-1))
CALL DIFSUB (N,Y,DERY)

```

```

T05 205
T05 206
T05 207
T05 208
T05 209
T05 210
T05 211
T05 212
T05 213
T05 214
T05 215
T05 216
T05 217
T05 218
T05 219
T05 220
T05 221
T05 222
T05 223
T05 224
T05 225
T05 226
T05 227
T05 228
T05 229
T05 230
T05 231
T05 232
T05 233
T05 234
T05 235
T05 236
T05 237
T05 238
T05 239
T05 240
T05 241
T05 242
T05 243
T05 244
T05 245
T05 246
T05 247
T05 248
T05 249
T05 250
T05 251
T05 252
T05 253
T05 254
T05 255
T05 256

```

IF (IDERY.NE.0) GO TO 4

C  
C  
C  
CALCULATE STREAM OUTPUT

$C30 = S(1, IN, 3) * S(1, IN, 8)$

$C1 = Y(N-2)$

$C2 = Y(N-1)$

$C3 = C30 + 0.5 * (C10 - C1) + C20 - C2$

$SUM = C1 + C2 + C3$

$S(1, IOUT, 4) = Y(N)$

$S(1, IOUT, 6) = C1 / SUM$

$S(1, IOUT, 7) = C2 / SUM$

$S(1, IOUT, 8) = C3 / SUM$

RETURN

END

T05 257

T05 258

T05 259

T05 260

T05 261

T05 262

T05 263

T05 264

T05 265

T05 266

T05 267

T05 268

T05 269

T05 270

T05 271

.....  
 TUBULAR REACTOR SIMULATION  
 .....

```

BEGIN
TIME      5.0
HMAX      1.0
IN/OUT    2.0
COMPS     3.0
LIBRARY   1.0
REAC3     5.0
PROCESS
REAC3     1.0
          1.0      -2.0
          30.0     20.0      90.0      100.0      100.0
EXTRA     7.0
          1.0      50.0      1.5      0.00002     0.01
          0.07     75.0
END
STREAMS
EXPLICIT  2.0
          1.0      1.0      10.0      100.0      14.7
          1.0      0.0      0.0
          1.0      1.0      10.0      0.0        14.7
          0.0      0.0      1.0
END
PROPERTIES -1.0
END
      END
  
```

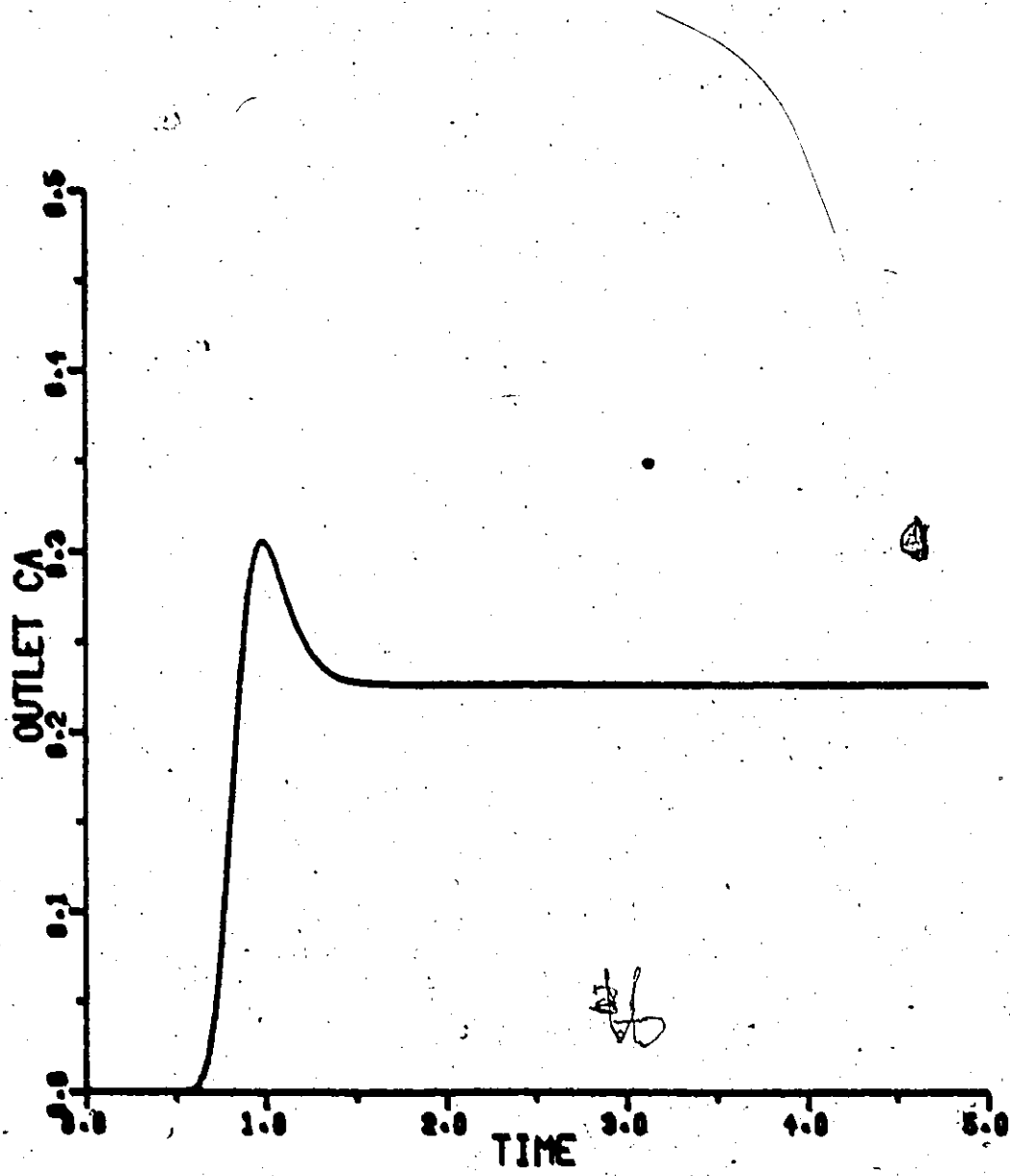


FIGURE 8.11: GRAPH OF OUTLET CONCENTRATION OF A VERSUS TIME FOR EXAMPLE#9

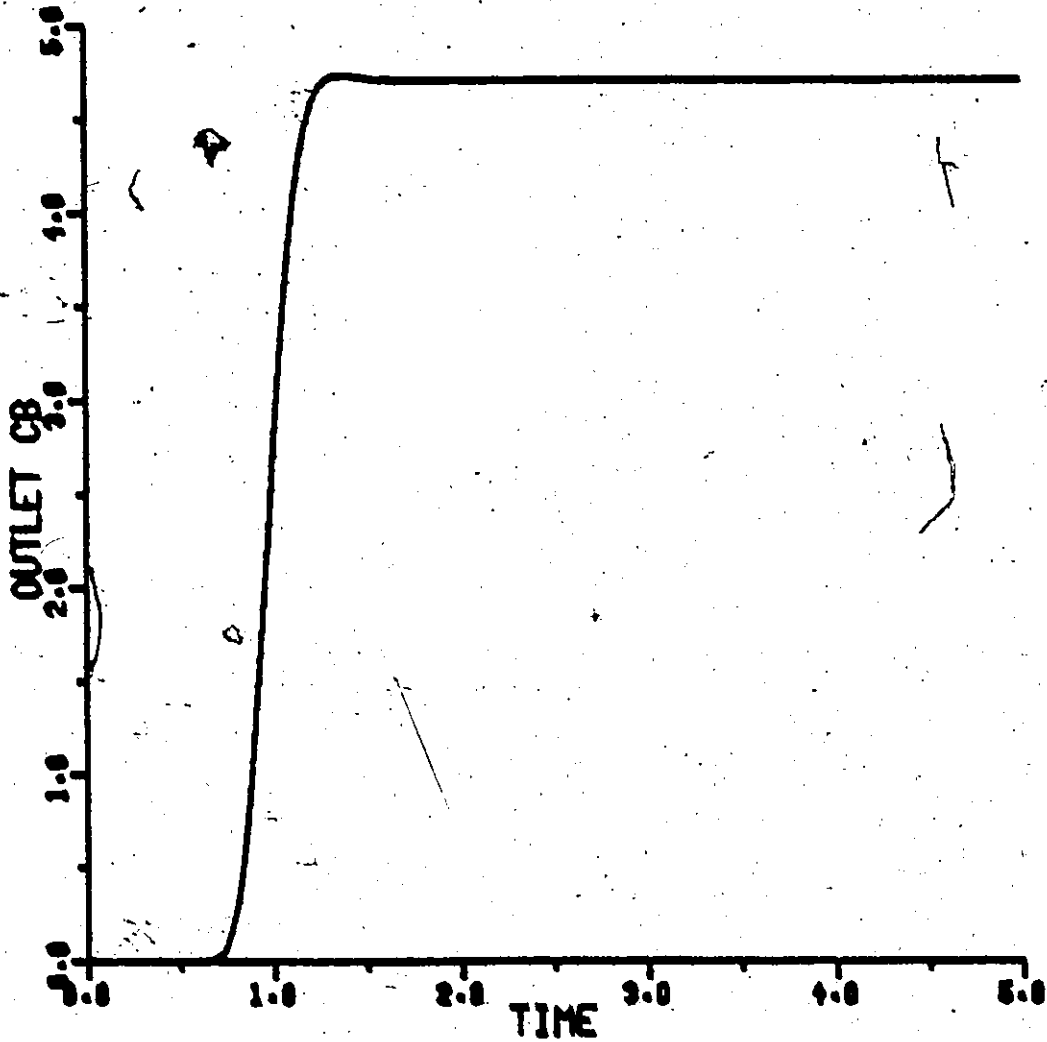


FIGURE 8.12: GRAPH OF OUTLET CONCENTRATION OF B VERSUS TIME FOR EXAMPLE#3

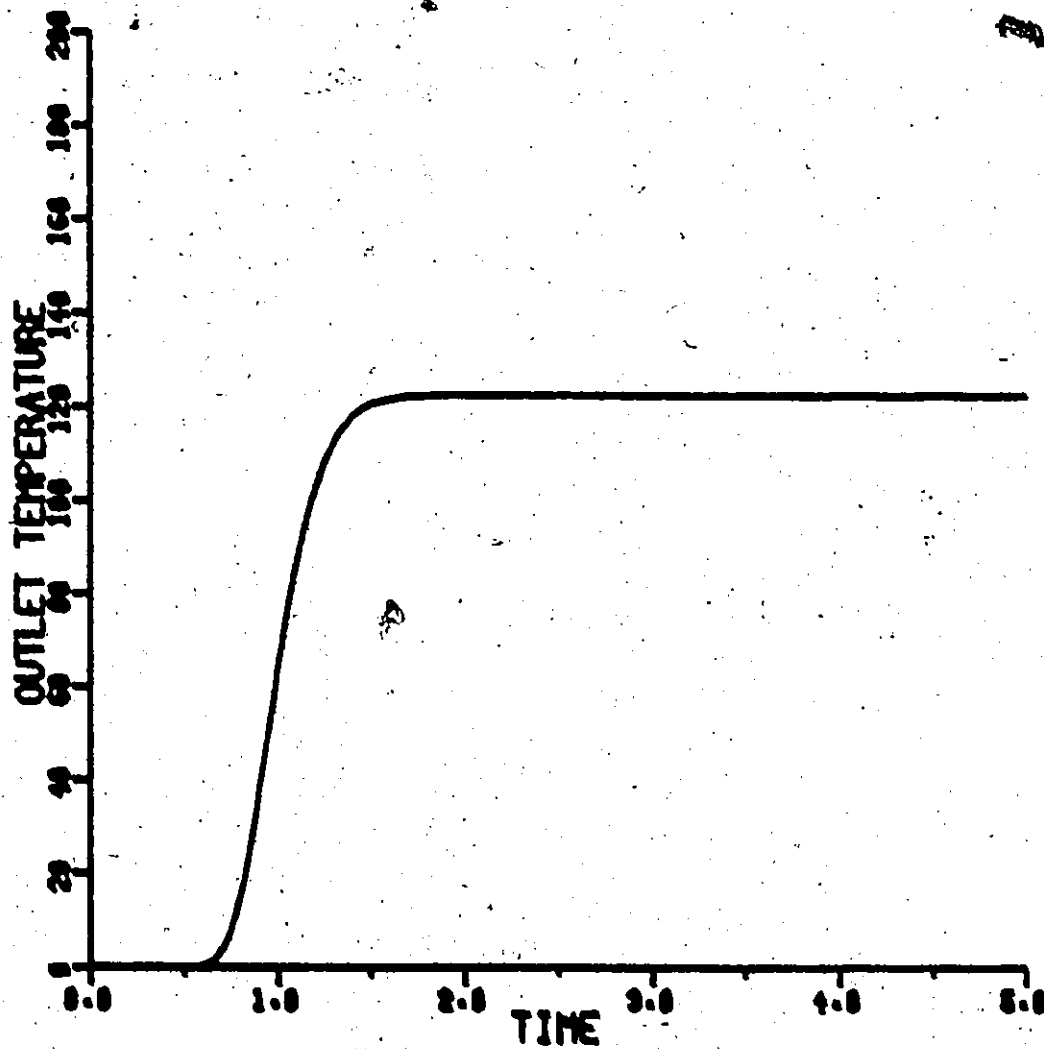


FIGURE 8.13: GRAPH OF OUTLET TEMPERATURE VERSUS TIME FOR EXAMPLE#3



#### 8.4 Tubular Reactor Simulation With Tridiagonal Jacobian Matrix

An isothermal tubular reactor was simulated yielding a tridiagonal Jacobian matrix. The reaction  $A \xrightarrow{k} B$  occurs.

The partial differential equation describing the process is:

$$\frac{\partial C_A}{\partial t} = -k C_A^2 + D_{AB} \frac{\partial^2 C_A}{\partial z^2} - V \frac{\partial C_A}{\partial z} \quad (8.4.1)$$

The following boundary and initial conditions apply:

$$\begin{aligned} C_A &= 0 \text{ at } t=0 \text{ for all } z \\ C_A &= C_A^0 \text{ at } z=0 \text{ for all } t \\ \frac{\partial C_A}{\partial z} &= 0 \text{ at } z=L \text{ for all } t \end{aligned} \quad (8.4.2)$$

where  $C_A$   $\equiv$  concentration of component A

$t$   $\equiv$  time

$z$   $\equiv$  space coordinate along reactor axis

$C_A^0$   $\equiv$  feed concentration of component A

$L$   $\equiv$  reactor length

$V$   $\equiv$  fluid velocity

$D_{AB}$   $\equiv$  diffusivity of component A

$k$   $\equiv$  reaction rate constant

The parameter values used are:

$$\begin{aligned}
 D &= 30.0 \\
 V &= 100.0 \\
 L &= 100.0 \\
 k &= 10.0 \\
 C_A^0 &= 10.0 \\
 C_B^0 &= 0.0 \\
 t_{max} &= 5.0
 \end{aligned}
 \tag{8.4.3}$$

Dividing the reactor into M mesh points as in the previous example, each mesh point will have 1 o.d.e (except at the inlet and outlet)

$$\begin{aligned}
 \frac{dC_{A,i}}{dt} &= -kC_{A,i}^2 + D_{AB} \left[ \frac{C_{A,i+1} - 2C_{A,i} + C_{A,i-1}}{(\Delta z)^2} \right] \\
 &\quad - V \left[ \frac{C_{A,i+1} - C_{A,i}}{\Delta z} \right]
 \end{aligned}
 \tag{8.4.4}$$

The equations are moderately stiff.

The equation at mesh point 1 is:

$$\frac{dy_1}{dt} = -ky_1^2 + D_{AB} \left[ \frac{y_2 - 2y_1 + y_0}{(\Delta z)^2} \right] - V \left[ \frac{y_2 - y_1}{\Delta z} \right]
 \tag{8.4.5}$$

The equations at mesh points 2,3,...M-2 are:

$$\frac{dy_i}{dt} = -ky_i^2 + D_{AB} \left[ \frac{y_{i+1} - 2y_i + y_{i-1}}{(\Delta Z)^2} \right] - V \left[ \frac{y_{i+1} - y_i}{\Delta Z} \right] \quad (8.4.6)$$

The equation at the last mesh point is:

$$\frac{dy_n}{dt} = -ky_n^2 + D_{AB} \left[ \frac{y_{n-1} - y_n}{(\Delta Z)^2} \right] \quad (8.4.7)$$

where  $y$  refers to  $C_A$ .

51 mesh points were taken giving a total of 49 o.d.e.s.

The execution time for the simulation was 4.04 seconds while an equation-oriented version of Gear's method took 3.92 seconds. An equation-oriented version using TRGB-TRGB2 required 6.78 seconds. Thus the tri-diagonal option does permit time savings. The IMP package required 46.1 seconds.

A list of the module and its data set are given in Figure 8.14. A graph of the outlet value of  $C_B$  versus time is shown in Figure 8.15. No change in the outlet concentration is seen until about 0.5 time units. Steady state is reached at about 1.2. The graph is sigmoid in appearance.

FIGURE 8.14: LISTING OF MODULE AND DATA SET FOR EXAMPLE #4  
TUBULAR REACTOR WITH TRIDIAGONAL JACOBIAN MATRIX (49 ODES)

C	SUBROUTINE TYPE6	T06	1
C		T06	2
C	SUBROUTINE REAC4	T06	3
C		T06	4
C	THIS MODULE REPRESENTS A TUBULAR REACTOR WITH REACTION C1-C2	T06	5
C	FEED CONCENTRATIONS MUST BE FAIRLY CONSTANT OTHERWISE	T06	6
C	ODES MUST BE SOLVED FOR C2	T06	7
C	THE JACOBIAN MATRIX IS TRIDIAGONAL	T06	8
C		T06	9
C	EQUIPMENT PARAMETERS	T06	10
C		T06	11
C	1 - D - DIFFUSIVITY OF COMPONENT 1	T06	12
C	2 - V - FLUID VELOCITY	T06	13
C	3 - L - LENGTH OF REACTOR	T06	14
C	4 - K - REACTION RATE CONSTANT	T06	15
C	5 - M - NUMBER OF SECTIONS IN REACTOR	T06	16
C		T06	17
C	COMMON /MAT/ MP(1,5),EP(1,5),S(2,2,8),EX(7)	T06	18
C	COMMON /CON/ IG,NCOMP,NCB,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	T06	19
C	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	T06	20
C	COMMON /UNIT/ IM,NMP	T06	21
C	COMMON /SUBDI/ A(49)	T06	22
C	COMMON /DIAG/ B(49)	T06	23
C	COMMON /SUPERD/ C(49)	T06	24
C	COMMON /MODULE/ IDERY,ITER,ITRI,NZERO	T06	25
C	DIMENSION Y(49), DERY(49)	T06	26
C	REAL K,L	T06	27
C		T06	28
C	USE NEWTON-RAPHSON ITERATION FOR THIS MODULE	T06	29
C		T06	30
C	ITER=1	T06	31
C		T06	32
C	CALCULATE MODULE PARAMETERS	T06	33
C		T06	34
C	IF (JSTART.NE.0.OR.IG.EQ.1) GO TO 2	T06	35
C	D=EP(IM,1)	T06	36
C	V=EP(IM,2)	T06	37
C	L=EP(IM,3)	T06	38
C	K=EP(IM,4)	T06	39
C	M=EP(IM,5)	T06	40
C	DZ=L/FLOAT(M)	T06	41
C	DZ2=DZ*DZ	T06	42
C	DDZ2=D/DZ2	T06	43
C	VDZ=V/DZ	T06	44
C	N=M-1	T06	45
C		T06	46
C	CALCULATE INITIAL CONDITIONS	T06	47
C		T06	48

	N=M-1	T06 49
	IN=MP(IM,3)	T06 50
	IOUT=IABS(MP(IM,4))	T06 51
	DO 1 I=1,N	T06 52
	Y(I)=S(2,IOUT,3)*S(2,IOUT,6)	T06 53
1	CONTINUE	T06 54
2	CONTINUE	T06 55
C		T06 56
C	CALCULATE FEED CONDITIONS	T06 57
C		T06 58
	C10=S(IG,IN,3)*S(IG,IN,6)	T06 59
C		T06 60
C	CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS	T06 61
C		T06 62
	IF (IG,EQ.2) GO TO 4	T06 63
C		T06 64
C	MESH POINT 1	T06 65
C		T06 66
	B(1)=-2.0*K*Y(1)-2.0*DDZ2-VDZ	T06 67
	C(1)=DDZ2	T06 68
C		T06 69
C	MESH POINTS 2,...,M-2	T06 70
C		T06 71
	M2=M-2	T06 72
	DO 3 I=2,M2	T06 73
	A(I)=DDZ2+VDZ	T06 74
	B(I)=-2.0*K*Y(I)-2.0*DDZ2-VDZ	T06 75
	C(I)=DDZ2	T06 76
	CONTINUE	T06 77
3		T06 78
C		T06 79
C	MESH POINT M-1 = N	T06 80
C		T06 81
	A(N)=DDZ2+VDZ	T06 82
	B(N)=-2.0*K*Y(N)-DDZ2-VDZ	T06 83
	CONTINUE	T06 84
4		T06 85
C		T06 86
C	USE TRIDIAGONAL OPTION FOR THIS MODULE (ITRI=1)	T06 87
C		T06 88
	ITRI=1	T06 89
C		T06 90
C	CALCULATE DERIVATIVES	T06 91
C		T06 92
	MESH POINT 1	T06 93
	DERY(1)=-K*Y(1)*Y(1)+DDZ2*(Y(2)-2.0*Y(1)+C10)-VDZ*(Y(1)-C10)	T06 94
		T06 95
C		T06 96
C	MESH POINTS 2,...,M-2	T06 97
		T06 98
	M2=M-2	T06 99
	DO 5 I=2,M2	T06 99
	DERY(I)=-K*Y(I)*Y(I)+DDZ2*(Y(I+1)-2.0*Y(I)+Y(I-1))-VDZ*(Y(I)-Y(I-1))	T06 100
	)))	

```
S  
C  
C  
C  
CONTINUE  
MESH POINT M=1 = N  
DERY(N)=-K*Y(N)*Y(N)+DDZ2*(Y(N-1)-Y(N))-VDZ*(Y(N)-Y(N-1))  
CALL DIFSUB (N,Y,DERY)  
IF (IDERY.NE.0) GO TO 4  
C  
C  
C  
CALCULATE STREAM OUTPUT  
C20=S(1,IN,3)*S(1,IN,7)  
C1=Y(N)  
C2=C20+C10-C1  
SUM=C1+C2  
S(1,IOUT,6)=C1/SUM  
S(1,IOUT,7)=C2/SUM  
RETURN  
END
```

```
T06 101  
T06 102  
T06 103  
T06 104  
T06 105  
T06 106  
T06 107  
T06 108  
T06 109  
T06 110  
T06 111  
T06 112  
T06 113  
T06 114  
T06 115  
T06 116  
T06 117  
T06 118
```

.....  
 TUBULAR REACTOR SIMULATION - TRIDIAGONAL JACOBIAN MATRIX  
 .....

```

BEGIN
TIME      5.0
DELTAT    0.0001
IN/OUT    2.0
HMAX      1.0
COMPS     2.0
LIBRARY   1.0
REAC4     6.0
PROCESS
REAC4     1.0
          1.0
          30.0
          -2.0
          100.0
          100.0
          10.0
          90.0

END
STREAMS
EXPLICIT  2.0
          1.0
          1.0
          2.0
          0.0
          1.0
          0.0
          -1.0

END
PROPERTIES
END
END
END
  
```

REAC4	1.0	1.0	30.0	-2.0	100.0	100.0	10.0	90.0
EXPLICIT	1.0	1.0	2.0	1.0	10.0	100.0	14.7	14.7
EXPLICIT	1.0	0.0	1.0	1.0	10.0	100.0	14.7	14.7
EXPLICIT	2.0	1.0	1.0	1.0	10.0	100.0	14.7	14.7
EXPLICIT	0.0	1.0	1.0	1.0	10.0	100.0	14.7	14.7

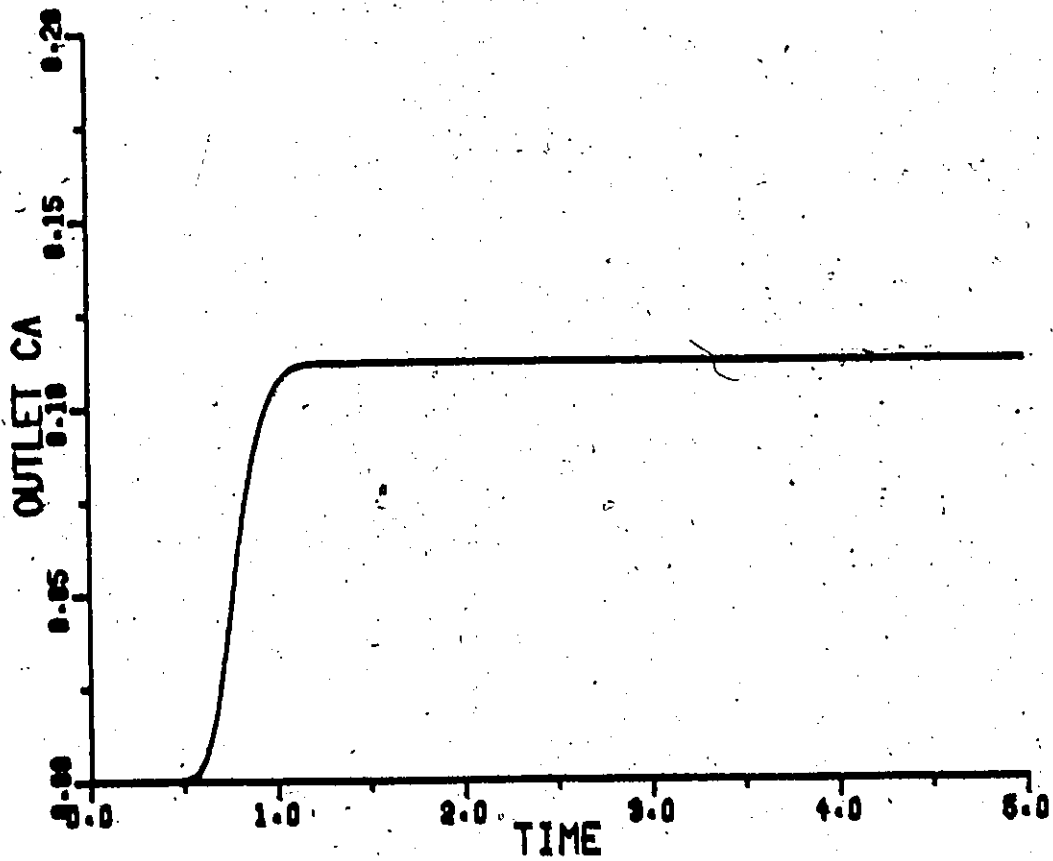


FIGURE 8.15: GRAPH OF OUTLET CONCENTRATION OF A VERSUS TIME FOR EXAMPLE#4



## 9. THE WILLIAMS-OTTO PLANT SIMULATION

The examples of the previous chapter were quite simple since they were only intended to illustrate different capabilities of the DYNYSYS 2.0 package. In this chapter, we would like to show a simulation of a realistic chemical plant. The Williams-Otto plant (Williams and Otto, 1960; Williams, 1961) was chosen.

Around 1960, many computer control schemes had recently been proposed for use by the chemical and petroleum processing industries. There was no direct method of comparing the relative applicability of these schemes. For this reason, the Monsanto Chemical Company proposed this plant as a chemical processing model to those interested in computer control with the hope that it may serve as a basis for a direct comparison of the available computers and their relative capabilities. Every effort was made to have the model include as many as possible of the effects present in typical chemical manufacturing plants.

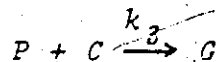
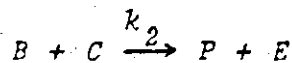
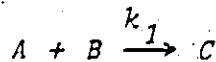
Williams assumed that a plant has already been designed and built for producing a chemical product  $P$ . However, considerable difficulty has been experienced with the stability of operation of the plant, and it is desired to investigate the feasibility of applying computer control

in an effort to achieve stable operation. At the same time, it is desired that the computer maintain an optimum balance of plant operating conditions to assure that the maximum return on investment is being obtained.

The plant consists of four major pieces of equipment, a stirred tank reactor, a heat exchanger, a decanter and a distillation column. Figure 9.1 depicts a simplified process flow diagram for the plant.

### 9.1 The Reactor

The reactor is a continuous stirred tank reactor in which the following reactions occur:



The desired product of the plant is  $P$ , while  $G$  is a heavy oily waste material. Reactants  $A$  and  $B$  are fed to the reactor in pure form and there is a recycle stream containing  $A$ ,  $B$ ,  $C$ ,  $E$  and  $P$ .

Steam is available to heat up the reactor and cooling water is available to keep the temperature under control, once the reactor has been heated up to about  $180^\circ\text{F}$ .

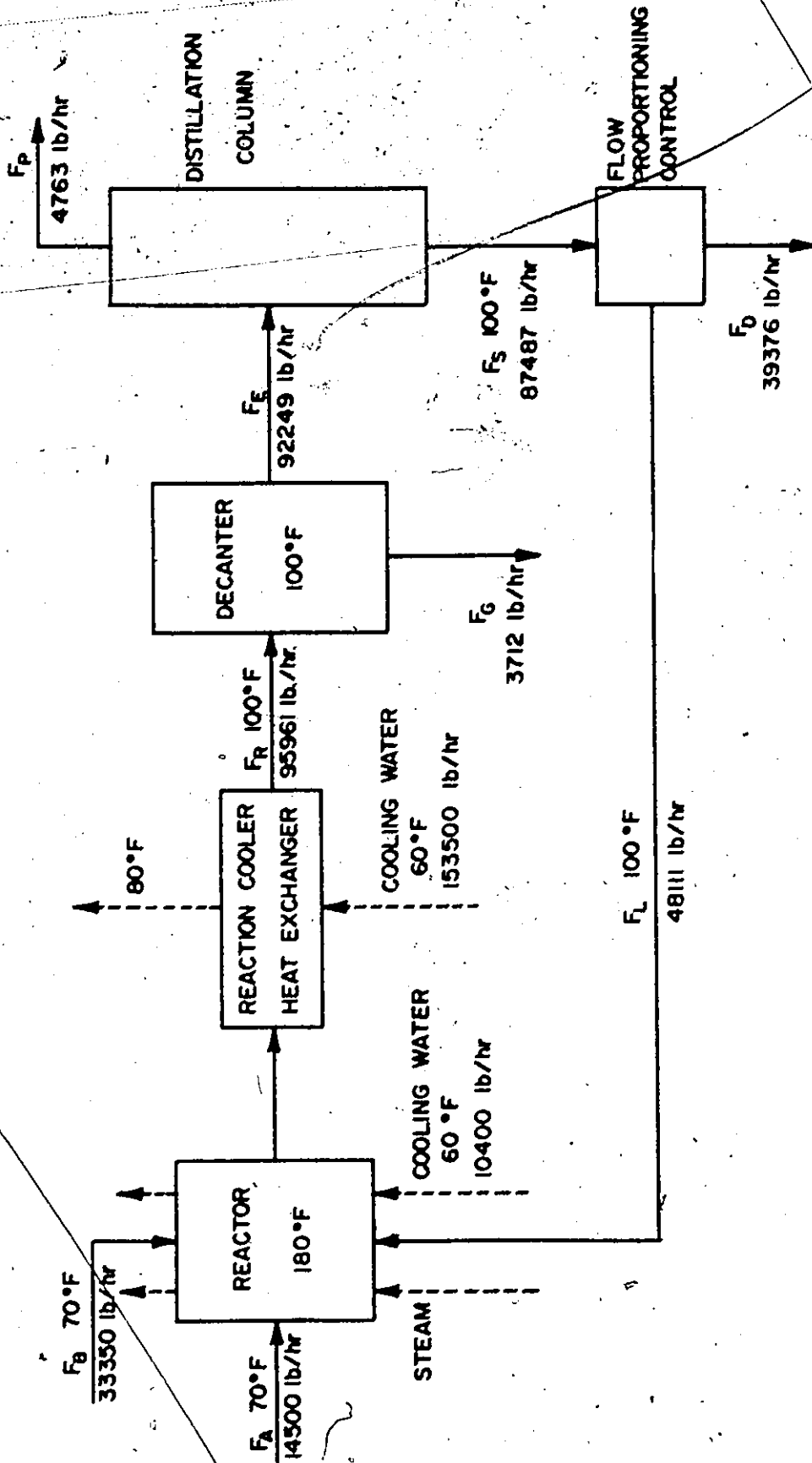


FIGURE 9.1: SIMPLIFIED PROCESS FLOW DIAGRAM - WILLIAMS - OTTO PLANT

The following differential equations were used in the model]:

mass balance on A:

$$\frac{dy_1}{dt} = \frac{1}{V_R} (F_A + F_L A_L - F_R y_1) - k_1 y_1 y_2 \quad (9.1.1)$$

mass balance on B:

$$\frac{dy_2}{dt} = \frac{1}{V_R} (F_B + F_L B_L - F_R y_2) - k_1 y_1 y_2 - k_2 y_2 y_3 \quad (9.1.2)$$

mass balance on C:

$$\begin{aligned} \frac{dy_3}{dt} = \frac{1}{V_R} (F_L C_L - F_R y_3) + 2.0 k_1 y_1 y_2 - 2.0 k_2 y_2 y_3 \\ - k_3 y_3 y_6 \end{aligned} \quad (9.1.3)$$

mass balance on E:

$$\frac{dy_4}{dt} = \frac{1}{V_R} (F_L E_L - F_R y_4) + 2.0 k_2 y_2 y_3 \quad (9.1.4)$$

mass balance on G:

$$\frac{dy_5}{dt} = \frac{1}{V_R} (F_L G_L - F_R y_5) + 1.5 y_3 y_6 \quad (9.1.5)$$

mass balance on P:

$$\frac{dy_6}{dt} = \frac{1}{V_R} (F_L P_L - F_R y_6) + k_2 y_2 y_3 - 0.5 y_2 y_6 \quad (9.1.6)$$

reactor heat balance:

$$\begin{aligned} \frac{dy_7}{dt} = \frac{1}{V_R C_{PR}} [ & -2.0 k_1 y_1 y_2^H V_R - 3.0 k_2 y_2 y_3^H V_R \\ & - 1.5 k_3 y_3 y_6^H V_R - h_W a_W (y_7 - y_8) \\ & + h_S a_S (T_S - y_7) - F_L C_{PR} (y_7 - T_L) \\ & - F_A C_{PR} (y_7 - T_A) - F_B C_{PR} (y_7 - T_B) ] \quad (9.1.7) \end{aligned}$$

cooling coil heat balance:

$$\frac{dy_8}{dt} = \frac{1}{V_W C_{PW}} [h_W a_W (y_7 - y_8) + F_W C_{PW} (T_{INC} - y_8)] \quad (9.1.8)$$

total reactor mass balance:

$$\frac{dy_9}{dt} = F_A + F_B + F_L - F_R \quad (9.1.9)$$

where

$$k_i = a_i \exp[-b_i / (y_7 + 459.69)] \quad (9.1.10)$$

where

- $y_1 - y_6$   $\equiv$  concentrations of  $A$ ,  $B$ ,  $C$ ,  $E$ ,  $G$  and  $P$  respectively in the reactor (mass fraction)
- $y_7$   $\equiv$  reactor temperature ( $^{\circ}\text{F}$ )
- $y_8$   $\equiv$  cooling coil temperature ( $^{\circ}\text{F}$ )
- $v_3, v_R$   $\equiv$  reactor mass holdup (lb)
- $v_W$   $\equiv$  cooling coil mass holdup (lb)
- $h_W, h_S$   $\equiv$  overall heat transfer coefficient of cooling and steam coils respectively (BTU/hrft $^2$  $^{\circ}\text{F}$ )
- $a_W, a_S$   $\equiv$  heat transfer area of cooling and steam coils respectively (ft $^2$ )
- $A_L, B_L, C_L, E_L, G_L, P_L$   $\equiv$  recycle stream concentrations of  $A$ ,  $B$ ,  $C$ ,  $E$ ,  $G$  and  $P$  respectively (mass fraction)
- $F_A, F_B, F_L, F_R, F_W$   $\equiv$  flow rates of  $A$  feed stream,  $B$  feed stream, recycle stream, reactor exit stream and coolant respectively (lb/hr)
- $C_{PR}, C_{PW}$   $\equiv$  heat capacities of reactor mixture (assumed same for all reactants) and coolant respectively (BTU/lb $^{\circ}\text{F}$ )
- $k_1, k_2, k_3$   $\equiv$  reaction rate coefficients (hr $^{-1}$ )
- $H_1, H_2, H_3$   $\equiv$  heats of reaction (BTU/lb)
- $T_{INC}$   $\equiv$  inlet coolant temperature ( $^{\circ}\text{F}$ )
- $T_S$   $\equiv$  steam temperature ( $^{\circ}\text{F}$ )



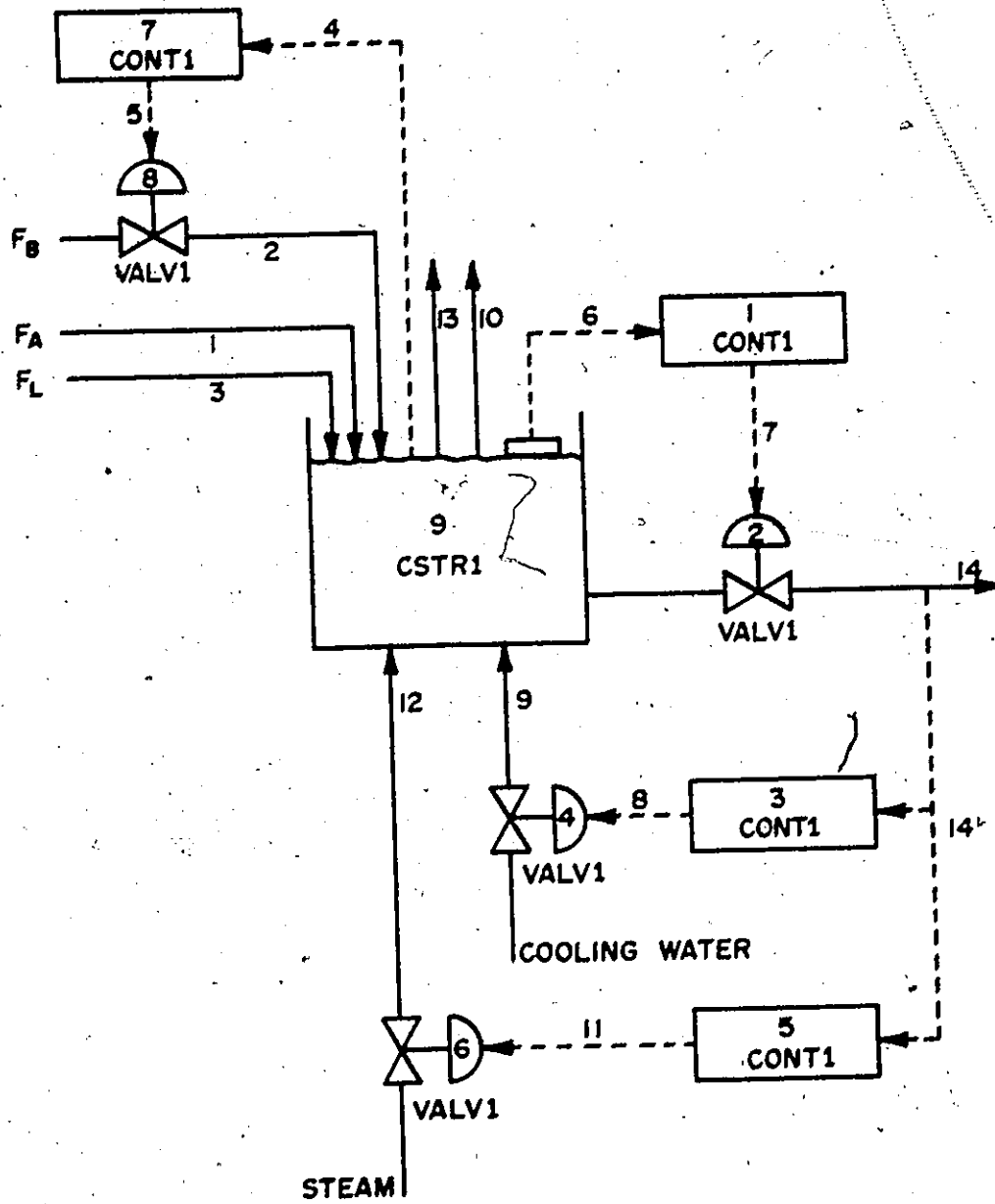


FIGURE 9.2 : REACTOR INFORMATION FLOW DIAGRAM



There are four equipment parameters:

- (1)  $V_R$  - initial holdup in reactor (lb)
- (2)  $V_W$  - holdup in cooling coil (lb)
- (3)  $HWAW$  - product of overall cooling coil heat transfer coefficient and effective heat transfer area (BTU/°Fhr)
- (4)  $HSAS$  - product of overall steam coil heat transfer coefficient and effective heat transfer area (BTU/°Fhr)

## 9.2 The Heat Exchanger

A heat exchanger follows the reactor in order to stop the reaction and prevent an overproduction of the waste material  $G$ . Both the tube and shell side of the exchanger are modelled as perfectly stirred tanks. It is assumed no reaction occurs in the heat exchanger.

The following differential equations were used in the model:

tube side heat balance:

$$\frac{dy_1}{dt} = \frac{1}{V_{HT} C_{PR}} [F_T C_{PR} (T_{INT}^{TR} - y_1) - h_H a_H T_{LM}] \quad (9.2.1)$$

where  $T_{LM}$  is the log mean temperature difference:

$$T_{LM} = \frac{(T_{INT} - y_2) - (y_1 - T_{INS})}{\ln\left(\frac{T_{INT} - y_2}{y_1 - T_{INS}}\right)} \quad (9.2.2)$$

shell side heat balance (coolant):

$$\frac{dy_2}{dt} = \frac{1}{V_{HS} C_{PW}} [F_S C_{PW} (T_{INS} - y_2) + h_{HH} a T_{LM}] \quad (9.2.3)$$

component mass balances:

$$\frac{dy_{i+2}}{dt} = \frac{F_T}{V_{HT}} (C_i - y_{i+1}) \quad i=1,6 \quad (9.2.4 - 9.2.9)$$

where

- $y_1$   $\equiv$  tube side temperature ( $^{\circ}\text{F}$ )
- $y_2$   $\equiv$  shell side temperature ( $^{\circ}\text{F}$ )
- $y_3 - y_8$   $\equiv$  tube side concentrations of A, B, C, E, G and P respectively (mass fractions)
- $T_{INT}, T_{INS}$   $\equiv$  inlet tube and shell side temperatures respectively ( $^{\circ}\text{F}$ )
- $V_{HT}, V_{HS}$   $\equiv$  tube and shell side mass holdups respectively (lb)
- $C_{PT}, C_{PW}$   $\equiv$  tube and shell side heat capacities respectively (BTU/lb $^{\circ}\text{F}$ )
- $F_T, F_S$   $\equiv$  tube and shell side flow rates respectively (lb/hr)

- $C_i$   $\equiv$  inlet tube side concentrations of A, B, C, E, G and P respectively (mass fractions)
- $h_H$   $\equiv$  overall heat transfer coefficient (BTU/°F ft<sup>2</sup>hr)
- $a_H$   $\equiv$  area of heat transfer (ft<sup>2</sup>)

The module is nonstiff. An information flow diagram for the heat exchanger appears in Figure 9.3. The flow of the cooling water is controlled by the outlet temperature of the heat exchanger.

There are five equipment parameters:

- (1)  $V_{HT}$  - tube side mass holdup (lb)
- (2)  $V_{HS}$  - shell side (coolant) mass holdup (lb)
- (3)  $A_H$  - area of heat transfer (ft<sup>2</sup>)
- (4)  $H_H$  - overall heat transfer coefficient (BTU/°F ft<sup>2</sup>hr)
- (5)  $CPW$  - coolant heat capacity (BTU/lb°F)

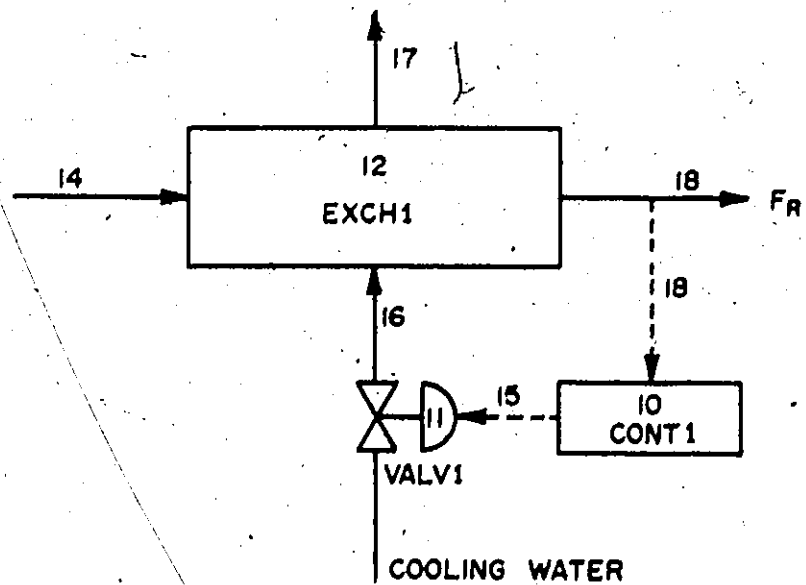


FIGURE 9.3 : HEAT EXCHANGER INFORMATION FLOW DIAGRAM

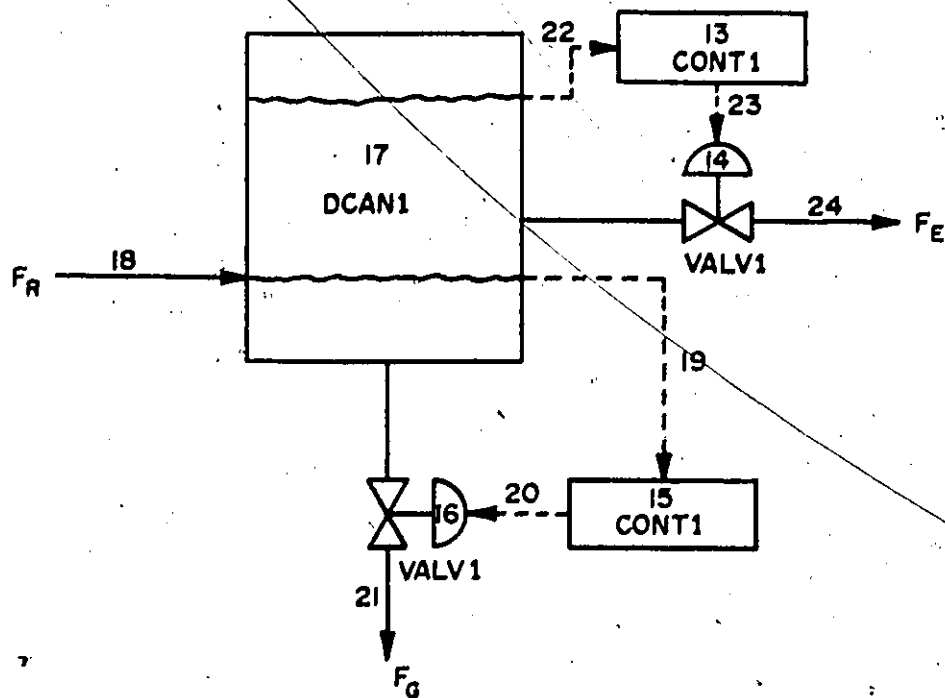


FIGURE 9.4 : DECANTER INFORMATION FLOW DIAGRAM

9.3 The Decanter

The heavy oil waste material, G, is insoluble at temperatures of less than 100°F and since it has a considerably higher specific gravity than the carrier stream in which it is suspended, it may be removed by settling. Thus a decanter follows the heat exchanger.

The top and bottom layers are both modelled as perfectly stirred tanks. The mass fraction of G in the reactant stream as a function of temperature is given. The decanter temperature determines the mass fraction of G entering the top layer.

The following differential equations were used in the model:

total mass balance:

$$\frac{dy_1}{dt} = F_{IN} - F_{OUTT} - F_{OUTB} \tag{9.3.1}$$

total heat balance:

$$\frac{dy_2}{dt} = \frac{1}{y_1} [F_{IN} T_{IN} - (F_{OUTT} + F_{OUTB}) y_2] \tag{9.3.2}$$

component mass balances for the top layer:

$$\frac{dy_{i+2}}{dt} = F_{INT} y_{INT} - F_{OUTT} \frac{y_{i+2}}{\sum_{i=1} y_{i+2}} \tag{9.3.3-9.3.8}$$

mass balance for bottom:

$$\frac{dy_2}{dt} = F_{INB} - F_{OUTB} \quad (9.2.3)$$

where

- $M_2$                    ≡ total mass holdup (lb)
- $T_2$                    ≡ decanter temperature (°F)
- $F_{A1}, F_{B1}, F_{C1}, F_{E1}, F_{G1}, F_{P1}$                    ≡ top layer holdups of A, B, C, E, G and P, respectively (lb/hr)
- $M_3$                    ≡ holdup of G in bottom layer (lb)
- $T_{IN}$                    ≡ inlet temperature (°F)
- $F_{OUTT}, F_{OUTB}$        ≡ outlet flow rates from top and bottom layers respectively (lb/hr)
- $F_{INT}, F_{OUTB}$        ≡ inlet flow rates to top and bottom layers respectively (lb/hr)
- $F_{IN}$                    ≡ total decanter inlet flow rate (lb/hr)

Figure 9.4 shows an information flow diagram for the decanter. The holdups in the top and bottom layers are controlled.

There are three equipment parameters:

- (1)  $H_{UPT}$  - initial holdup in top layer (lb)
- (2)  $H_{UPB}$  - initial holdup in bottom layer (lb)
- (3)  $I_{SEP}$  - number of component to be separated from entering stream

#### 9.4 The Distillation Column

Williams and Otto proposed to model the column as a stirred tank. Rather expectedly, this did not give satisfactory results. Pulido (1975) has created DYN SYS 2.0 modules of a distillation column, condenser, reboiler and reboiler drum. These are discussed in detail in his thesis.

Assumptions used were no heat of mixing, constant holdup on each plate and equilibrium on each plate at all times. The column module contains mass and energy balances about each plate. Bubble point calculations are done. The condenser is either a total or a partial condenser. The reboiler module handles the heat transfer calculations, while the reboiler drum module does the mass and energy balances for the reboiler. The azeotrope in the column was neglected for simplicity, but it can be implemented once a correlation for the liquid activity coefficients is found, such as the Wilson equation.

The column, condenser drum, reboiler and reboiler drum modules are modelled as being stiff and Jacobian matrices are supplied. The condenser module is algebraic.

An information flow diagram for the column appears in Figure 9.5. Since the column uses molar rather than mass quantities, a module CONV1 converts the mass flow and mass fractions to molar flow and mole fractions respectively before entering the column and does the opposite for the

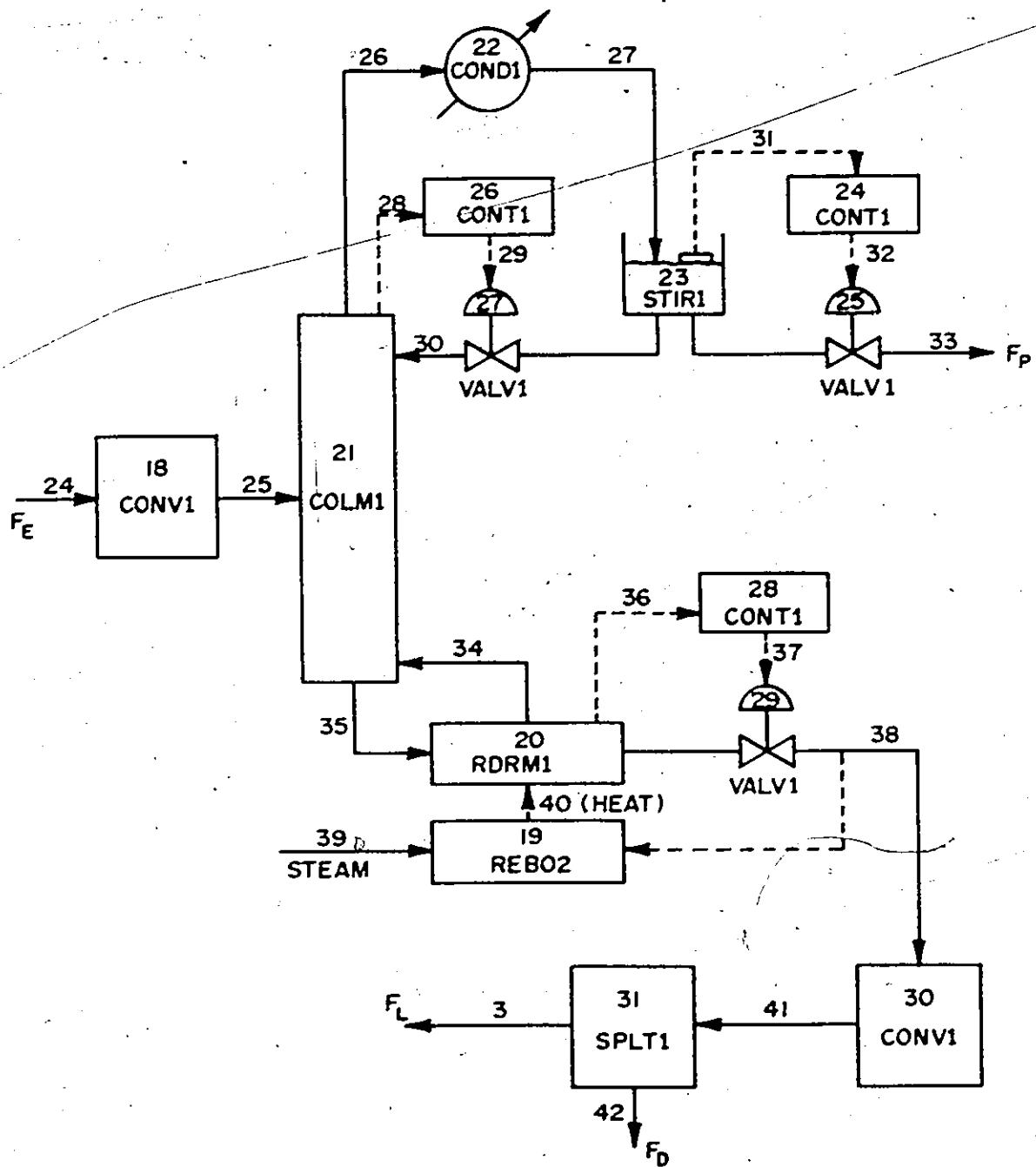


FIGURE 9.5: DISTILLATION COLUMN INFORMATION FLOW DIAGRAM



bottoms stream leaving the column.

A module SPLT1 divides the bottoms flow, according to a given ratio, into two streams, one of which is recycled back to the reactor.

### 9.5 Simulation Results

A listing of all the modules and a possible data set are contained in Appendix H. The modules form the basis of a library of modules for DYNISYS 2.0. In all, 31 modules and 42 streams were required.

The reactor was started up from 70°F assuming a constant recycle stream. Figure 9.6 shows a graph of the reactor concentrations versus time. Figure 9.7 shows a graph of the reactor and exit heat exchanger temperatures versus time. Steady state was reached after about 0.7 hours.

The control scheme for the reactor kept the temperature at 180°F after some slight oscillation. The steam flow is cut off at 175°F and a proportional controller with a high gain controls the cooling water flow so that the temperature does not rise above 180°F. This is similar to the control scheme recommended by Williams. The steady state cooling water flow rate at 180°F with no steam flow was calculated to be 5115 lb/hr. The reactor concentrations at steady state were the same as given by Williams.

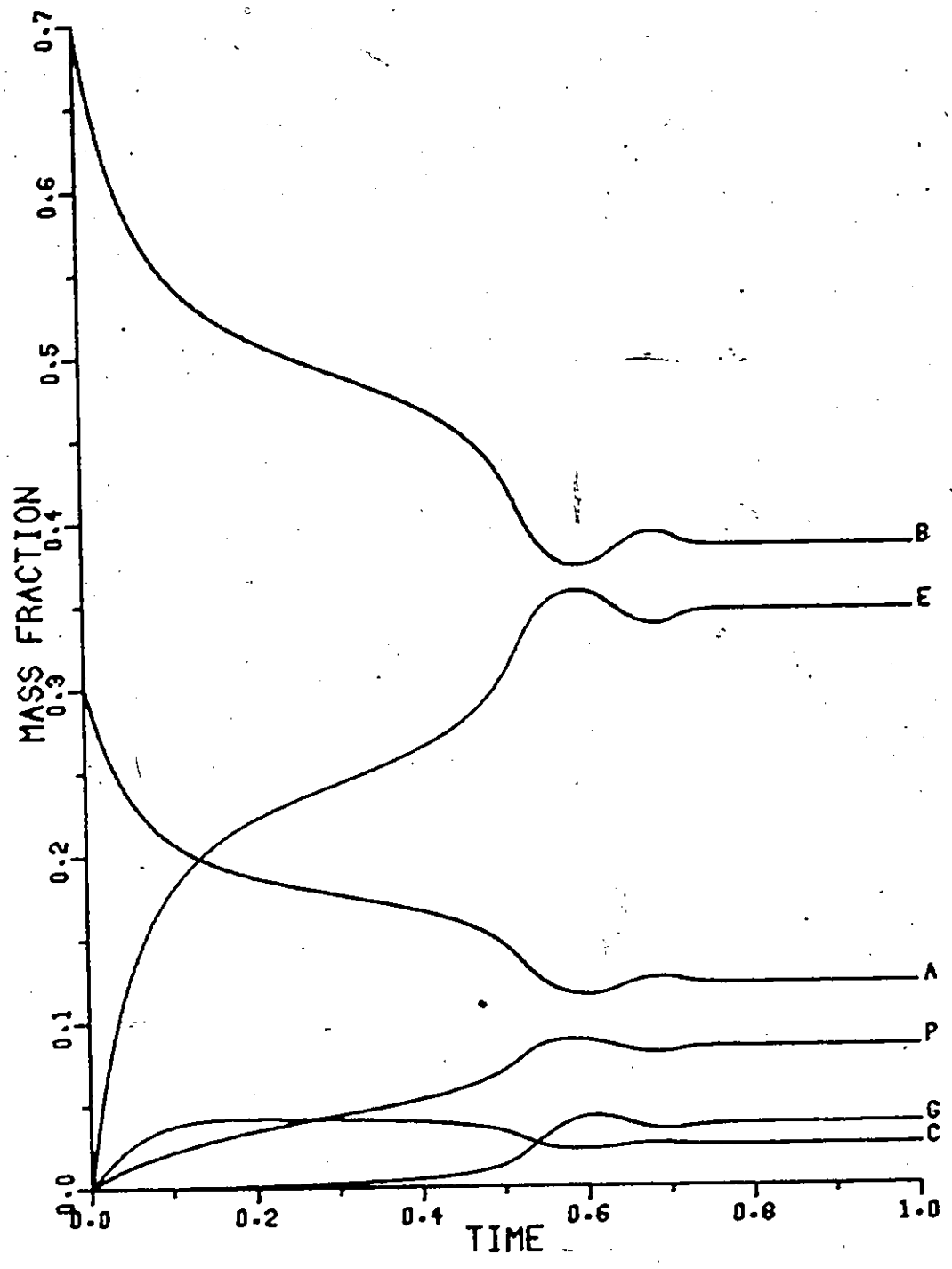


FIGURE 9.6: GRAPH OF REACTOR CONCENTRATIONS OF A, B, C, E, G AND P VERSUS TIME

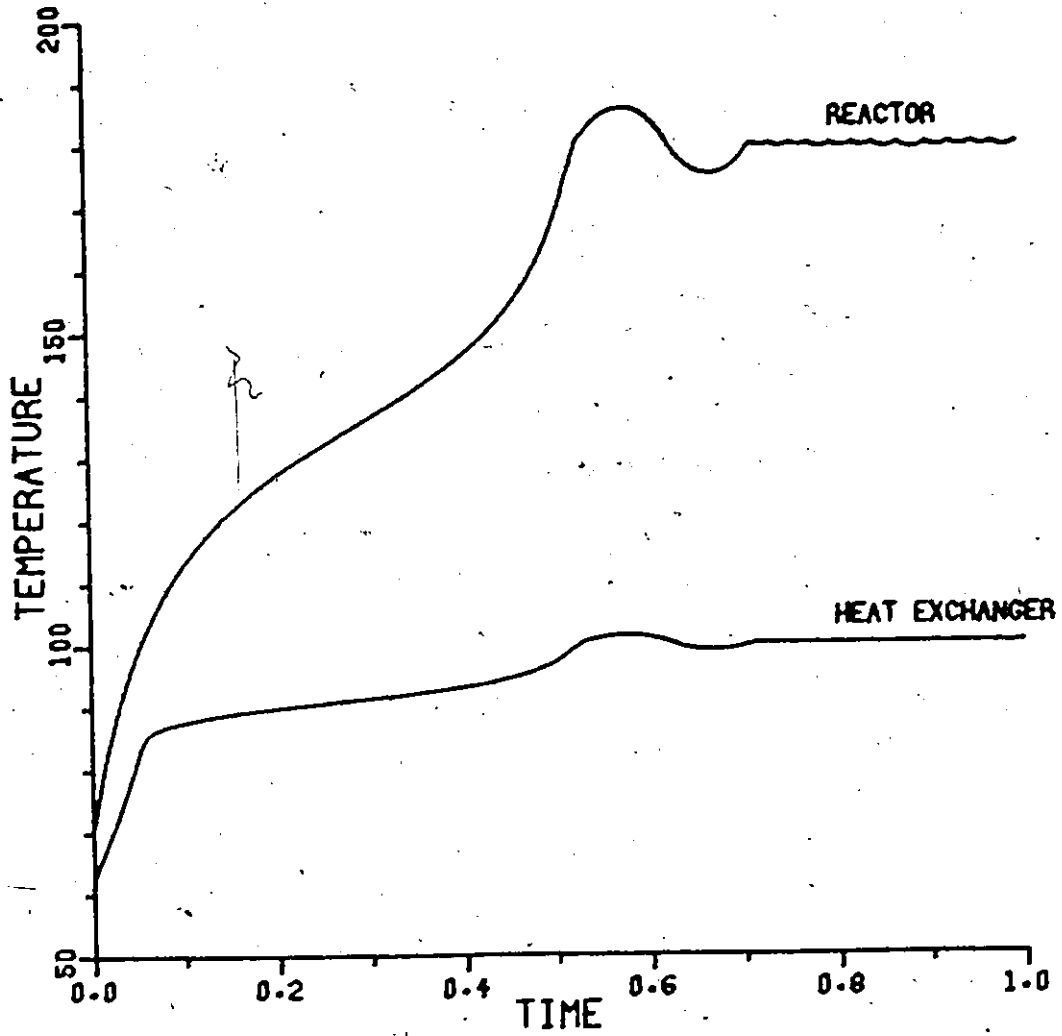


FIGURE 9.7: GRAPH OF REACTOR AND EXIT HEAT EXCHANGER TEMPERATURES VERSUS TIME

A value of 5000.0 was used for  $h_g a_g$ . The value of 10000.0 given by Williams heats the reactor too fast and the temperature becomes out of control.

The heat exchanger outlet temperature was below 102°F at all times. Since the reactor temperature control works well and the heat exchanger cools the reactor outlet stream efficiently, no trouble was encountered in the decanter. The decanter temperature never rose above 100°F and all of the  $G$  was settled out.

The distillation column results were handled by J. Pulido and a discussion of these may appear in his thesis (Pulido, 1975).

DYNSYS 2.0 performed the simulation quite well and is very convenient for simulating a modular plant such as this one. However, during the simulation, corrector convergence problems occurred often for the heat exchanger even with a supplied Jacobian. Possibly the model will have to be refined.

The Williams-Otto plant offers many possibilities for control system studies. More work may be done of this in the future. DYNSYS 2.0 is a very convenient tool for evaluating different control policies. Pulido (1975) has used DYNSYS 2.0 to evaluate various distillation column control schemes.

## 10. CONCLUSION

### 10.1 Summary

The numerical solution of ordinary differential equations is a fundamental aspect of dynamic simulation. The original version of DYNYSYS used a third order Adams-Moulton-Shell routine; however, this is very inefficient for stiff systems where there is a wide variation of time constants. In chemical engineering, stiff o.d.e.s occur widely in reaction kinetics and to some extent in multi-stage systems. Conventional numerical techniques are restricted by stability to using a very small step size resulting in large computer times. There have been many new numerical techniques published in the recent literature directed at the efficient numerical solution of systems of stiff o.d.e.s. A literature survey of these has been made.

Most stiff techniques are implicit and require a technique such as Newton-Raphson iteration to converge. Each Newton-Raphson iteration involves the solution of a system of linear algebraic equations (usually sparse) equal in size to the number of o.d.e.s. For a large stiff system, this requires considerable computer time. Various

recently developed sparse linear equation solvers have been evaluated and that of Bending and Hutchison appears to be the most appropriate.

A comparison of several integration techniques has been done with a number of nonstiff and stiff test examples: Gear's method appears to be the most efficient.

Gear's integration algorithm in conjunction with the Bending-Hutchison linear equation solver has been implemented into DYNYSYS version 2.0. An option for stiff systems with tridiagonal Jacobian matrix is also included. Several simulation examples have been described including the simulation of the Williams-Otto plant. The package is a very powerful one, capable of the simultaneous numerical solution of hundreds of stiff o.d.e.s.

The modular approach has been chosen as the framework of our study, but certainly the results apply equally well to the equation-oriented approach. An equation-oriented program containing Gear's method and the Bending-Hutchison linear equation solver has been created and it is useful for simulating nonmodular systems.

An interactive version of the executive, where the output can be displayed graphically on a CRT screen, has also been developed.

## 10.2 Future Work

A version of the executive which keeps the operator lists and possibly other variables in auxiliary storage can easily be created. Segments of the operator list can be stored or read in, thus allowing much larger systems to be simulated. Thousands of o.d.e.s could then be solved, but the program would not be as fast because of the extra time required to store and access the operator lists. It would also make the program more machine dependent.

A tridiagonal option has been included, but other options might well be included, similar to those in the IMP package, such as options for banded matrices in general rather than just tridiagonal, or iterative options for problems with diagonally dominant Jacobians.

This thesis has concentrated on the numerical solution of stiff systems, particularly large ones. Another area which can be studied is the convergence of recycle in dynamic simulation (Koenig, 1972).

With the present package, it is reasonably difficult for the novice to learn how to write modules. Ways of simplifying the module structure for user convenience should be studied.

A library of modules could be developed to make future programming easier. Pulido (1975) and Millares (1975) are presently using DYN SYS 2.0 to simulate a commercial

process. The modules from their work and from the Williams-Otto plant simulation would form the basis of such a library. Additional modules could be added corresponding to the types of equipments used in chemical plants.

The package should be applied to other industrial processes.

A physical properties package could be added to the system.

A version of DYNSSYS which can use CRT graphical input in the form of an information flow diagram is being developed.

The applicability of the pseudo state approach is a possible area of research. We pointed out in Chapter 3 that it is a dangerous method to apply; however the circumstances under which it can be applied and the amount of error incurred are questions of interest in solving stiff systems.

All of the test examples in Appendix D are of the relaxation type; i.e. the fast components become essentially zero for a very short time. In a control system simulation with disturbances, the fast modes will be constantly re-introduced into the system. Whether Gear's method is the best numerical technique under such conditions should be investigated.



The study of numerical techniques for solving stiff o.d.e.s and for solving sparse linear algebraic equations is a currently active area of research and more efficient techniques than those used in DYNYSYS 2.0 will undoubtedly be developed. Gear himself has a new, more efficient version of DIFSUB which will be available around the end of 1974 (Gear 1974). DYNYSYS 2.0 will be modified to take this into account and the evolution of the package will include the incorporation of more efficient techniques.

## APPENDIX A: GEAR'S METHOD

C.W. Gear (Gear, 1971a,b,c; 1972) has created a subroutine called DIFSUB for solving ordinary differential equations. It uses a variable-order, variable-step, linear, predictor-corrector algorithm.

There are options for nonstiff and stiff equations.

### A.1 Nonstiff Option

The nonstiff option uses an Adams-Bashforth predictor and an Adams-Moulton corrector:

$$\begin{array}{l} \text{PREDICTOR EQN: } y_{n+1} = y_n + h \sum_{i=1}^k \beta_i y_{n+1-i} \\ \text{(Adams-Bashforth)} \end{array} \quad (A.1)$$

$$\begin{array}{l} \text{CORRECTOR EQN: } y_{n+1} = y_n + h \sum_{i=0}^k \beta_i^* y_{n+1-i} \\ \text{(Adams-Moulton)} \end{array} \quad (A.2)$$

The coefficients  $\beta_i$  and  $\beta_i^*$  may be found in Henrici (1962). The order may vary from 1-7.

## A.2 Stiff Option

The stiff option uses a predictor and corrector based on the backward or numerical differentiation methods (Henrici, 1962):

$$\text{PREDICTOR EQN.: } y_{n+1} = h \eta_1 y_n + \sum_{i=1}^k \alpha_i y_{n+1-i} \quad (\text{A.3})$$

$$\text{CORRECTOR EQN.: } y_{n+1} = h \eta_0^* y_{n+1} + \sum_{i=1}^k \alpha_i^* y_{n+1-i} \quad (\text{A.4})$$

The corrector coefficient values are given in Gear (1971c). From 1st to 6th order is available.

The corrector equation has the property of stiff stability (Gear, 1969a; Figure A.1) for up to 6th order; i.e., it is accurate and stable near the origin and stable to the far left of the origin in the  $h\lambda$  plane.

Accuracy is not important far to the left of the origin, since the components here are very small, but absolute stability is required. Around the origin, accuracy is essential for which we need relative or absolute stability. Gear's method is relatively stable in the rectangle shown. Elsewhere there are no requirements, since to the right of  $\text{Re}(h\lambda) = \alpha$  and above and below  $\text{Im}(h\lambda) = \pm \theta$ , the function is varying too rapidly to be represented by points spaced a distance  $h$ , but is not decaying so fast that we can ignore such terms. For order 1-6,  $D > -6.1$ ,  $\theta < 0.5$  and  $\alpha = 0$  (Gear 1969a).

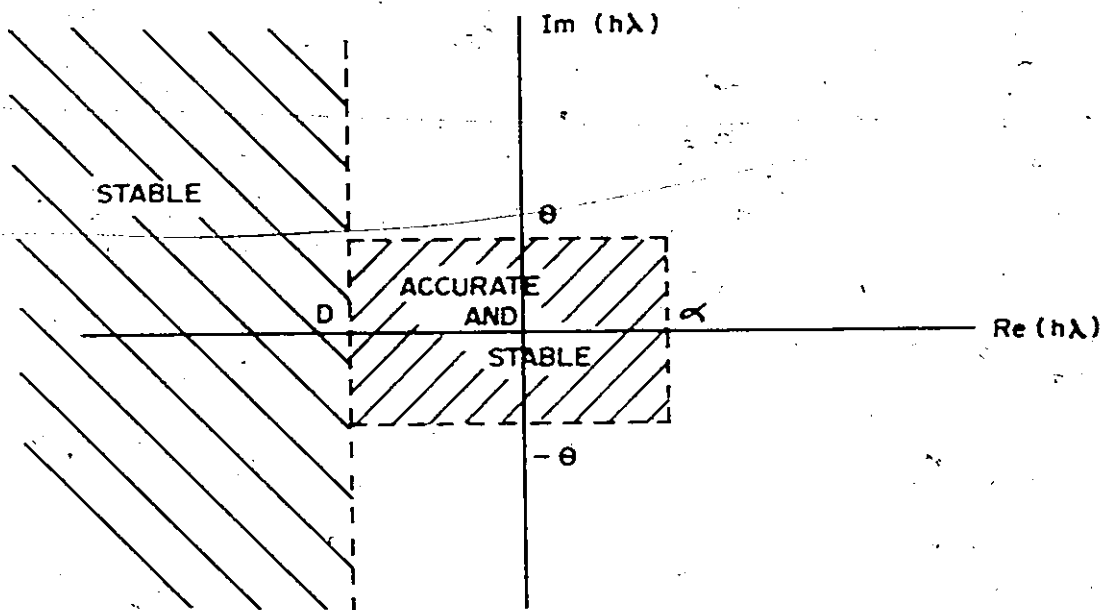


FIGURE A.1: STIFF STABILITY

The Jacobian matrix is needed for the stiff option, as up to three Newton-Raphson iterations of the corrector are performed. The Jacobian may be user-supplied or evaluated numerically. It is re-evaluated only when there is a change in order or the corrector failed to converge.

Much of the algorithm is the same for the nonstiff and stiff option except that different coefficients are used.

### A.3 General Features

Rather than storing back values of the independent variable and the first derivatives, the algorithm uses a variation of the Nordsieck vector (Nordsieck, 1962):

$$a_n = \left[ y_n, h\dot{y}_n, \frac{h^2\ddot{y}_n}{2!}, \dots, \frac{h^k y_n^{(k)}}{k!} \right] \quad (\text{A.5})$$

Gear expresses the predictor-corrector in the form:

$$\text{PREDICTOR EQN.: } a_{n+1,(0)} = \underline{A} a_n \quad (\text{A.6})$$

$$\text{CORRECTOR EQN.: } a_{n+1,(m+1)} = a_{n+1,(m)} + \frac{1}{\underline{L}} \sum_{j=0}^m \underline{A}_{mj} F(a_{n+1,(m)}) \quad (\text{A.7})$$

where  $\underline{A}$  is the Pascal triangle matrix:

$$\begin{aligned} A_{ij} &= \binom{j}{i} \\ &= 0 \text{ if } i > j \end{aligned} \quad (\text{A.8})$$

$\underline{E}(a)$  is the amount by which the differential equation is not satisfied locally by  $a$ :

$$\underline{E}(a) = h [ \underline{f}(t, a) - \underline{a}_1 ] - h \underline{f}(t, y) - h \underline{y} \quad (A.9)$$

For direct iteration as in the nonstiff option

$$\underline{K} = \underline{I} \quad (\text{identity matrix}) \quad (A.10)$$

For Newton-Raphson iteration as in the stiff option

$$\underline{K} = \left[ \begin{array}{c} \underline{I} - h \underline{J} \frac{\partial \underline{f}}{\partial \underline{y}} \\ \underline{0} \end{array} \right] \quad (A.11)$$

where  $\frac{\partial \underline{f}}{\partial \underline{y}}$  is the Jacobian matrix

The coefficients  $\underline{J}$  are given in Gear (1971c).

$\underline{J}$  is chosen to achieve stability and accuracy.

The same order is used on the predictor and corrector steps. The local truncation error is given by

$C_{k+1} h^{k+1} y^{(k+1)} + O(h^{k+2})$  for a  $k$ th order step. The  $C_{k+1}$

values are given in Gear (1971c). The last component of

the Nordsieck vector is  $h^k y_n^{(k)} / k!$  and under suitable

smoothness conditions, its backward difference yields an

estimate of  $h^{k+1} y^{(k+1)} / k!$ . The error is controlled by

keeping the Euclidean norm of the relative local truncation

errors below a specified value.

The method is self-starting, using the first order formula to begin the integration. The order is variable

and is chosen to maximize the step size with some heuristic controls. The algorithm begins with first order and increases quickly to about fourth order where it remains for the bulk of the integration. Then as steady state is approached, the order decreases back to first order.

The nomenclature was first introduced in Gear (1967). Ratliff (1968) does some computations with the method. Gear (1971d) has extended his program to handle large sparse sets of simultaneous differential and algebraic equations; however, the program is in machine language.

Several workers have recommended Gear's method for stiff systems. Nikolai (1973) tells of his experience with it. Walters (1972) provides a detailed discussion of the method. A version of DIFSUB is available in the IMSL library (IMSL, 1973).

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## APPENDIX B: IMP

D.M. Brandon (Brandon, 1972, 1973, 1974a), now with Control Data Corporation, has developed IMP (Implicit Solution Software System). IMP is a software system for the direct or iterative solution of large differential and/or algebraic systems. The package is equation-oriented and contains the author's own A-stable numerical integration technique and sparse matrix routines for solving the corrector equation. IMP is offered as a standard CDC software product through their Application Services Division. For non-CDC computers, the package is available from Brandon himself.

The IMP tests in this thesis were done with an object copy of version 1.1 compiled under FTN version 4.0, optimization level 2, Scope 3.4 operating system.

The IMP user must write his own user subroutine defining the system to be simulated and choosing from the many options available. The problem must be expressed in the form:

$$\dot{y} = A y + B \quad (B.0)$$



where  $A$  is the Jacobian matrix  
and  $B$  is the augmented constant vector.

Both  $A$  and  $B$  may be functions of time or  $y$ . The left hand side is set to zero for algebraic equations. Partial differential equations are reduced to ordinary differential equations by discretization of one or more dependent variables.

### B.1 Brandon's Integration Technique

Brandon's integration procedure (Brandon, 1974f) is a single-step method:

$$y_{n+1} = y_n + h[(1 - \omega)\dot{y}_{n+1} + \omega\dot{y}_n] \quad (B.1)$$

For the equation:

$$\dot{y} = \lambda y \quad (B.2)$$

$$y_{n+1} = y_n e^{h\lambda} \quad (B.3)$$

Substitute (B.2) into (B.1)

$$y_{n+1} = y_n + h[(1 - \omega)\lambda y_{n+1} + \omega\lambda y_n] \quad (B.4)$$

$$y_{n+1}[1 - h\lambda(1 - \omega)] = y_n[1 + h\lambda\omega] \quad (B.5)$$

$$\frac{y_{n+1}}{y_n} = \frac{1 + h\lambda w}{1 - h\lambda(I - w)} = e^{h\lambda} \quad (\text{B.6})$$

Solving for  $w$ :

$$w = -\frac{1}{h\lambda} - \frac{1}{e^{-h\lambda} - 1} \quad (\text{B.7})$$

Liniger and Willoughby (1970) introduced the concept of exponential fitting in that  $w$  can be chosen to make the integration exact for a given value of  $\lambda$ .

For more than one equation, different values of  $w$  may be used:

$$w_i = -\frac{1}{z_i} - \frac{1}{e^{-z_i} - 1} \quad (\text{B.8})$$

There will not be an eigenvalue corresponding to each equation except for the system:

$$\dot{y}_i = \lambda y_i \quad i=1, 2, \dots, N \quad (\text{B.9})$$

Brandon chooses  $z$  to approximate the local gradient:

$$z_i = \frac{\partial \dot{y}_i}{\partial y_i} = h \sum_{j=1}^N a_{ij} \left( \frac{\dot{y}_j}{\dot{y}_i} \right) \quad (\text{B.10})$$

For the single equation  $\dot{y} = \lambda y$ , this reduces to exponential fitting. Thus Brandon has extended the concept of exponential fitting to systems of equations.

Note that no predictor equation is used. The first value for the corrector iteration is taken as  $y_n$ .

The method is A-stable for  $0 \leq w \leq 0.5$ .

There is a very conservative local truncation error estimate. Expanding  $y_{n+1}$  in a Taylor Series:

$$y_{n+1} = y_n + h \dot{y}_n + \frac{h^2}{2!} \ddot{y}_n + \frac{h^3}{3!} \dddot{y}_n + \dots \quad (B.11)$$

$$w \dot{y}_{n+1} = w y_n + h w \dot{y}_n + \frac{h^2}{2!} w \ddot{y}_n + \frac{h^3}{3!} w \dddot{y}_n + \dots \quad (B.12)$$

$$(1-w)y_{n+1} = (1-w)y_n + h(1-w)\dot{y}_n + \frac{h^2}{2!} (1-w)\ddot{y}_n + \frac{h^3}{3!} (1-w)\dddot{y}_n + \dots \quad (B.13)$$

Adding (B.12) and (B.13)

$$y_{n+1} = y_n + h w \dot{y}_n + h(1-w)\dot{y}_{n+1} - h(1-w)\dot{y}_{n+1} + h(1-w)\dot{y}_n + \frac{h^2}{2!} \ddot{y}_n + \frac{h^3}{3!} \dddot{y}_n + \dots \quad (B.14)$$

The error term is:

$$E = -h(1-w)\dot{y}_{n+1} + h(1-w)\dot{y}_n + \frac{h^2}{2!} \ddot{y}_n + \frac{h^3}{3!} \dddot{y}_n + \dots \quad (B.15)$$

Expanding  $\dot{y}_{n+1}$  in a Taylor Series:

$$\dot{y}_{n+1} = \dot{y}_n + h \ddot{y}_n + \frac{h^2}{2!} \dddot{y}_n + \dots \quad (\text{B.16})$$

Substitute (B.16) into (B.15)

$$E = h^2 \left( \omega - \frac{1}{2} \right) \ddot{y}_n + h^3 \left( \frac{\omega}{2} - \frac{1}{3} \right) \dddot{y}_n \quad (\text{B.17})$$

Replace  $\ddot{y}_n$  by  $\frac{\ddot{y}_{n+1} - \ddot{y}_n}{h}$

$$E = \frac{h^2}{12} [(\omega - 3)\ddot{y}_n + (\omega - 4)\ddot{y}_{n+1}] \quad (\text{B.18})$$

Brandon adjusts this term empirically to:

$$E = \left| \frac{h^2 E_R}{\varepsilon(y_n + y_{n+1})} \right| \left( \frac{1}{Y_i} \right) + \left| \frac{h(b_{n+1} - b_n)}{15(y_n + y_{n+1})} \right| \quad (\text{B.19})$$

where

$$E_R = [(1 + d_i)\ddot{y}_{n+1} - (1 - d_i)\ddot{y}_n] \quad (\text{B.20})$$

$$\ddot{y}_i = \sum_{j=1}^N a_{ij} y_j \quad (\text{B.21})$$

$$d_i = 3 \left( 1 - \frac{2\omega_i}{h} \right) \quad (\text{B.22})$$

$$Y_i = 45 \frac{|a_{ii}|}{\sum_{j=1}^N |a_{ij}|} + 5 \quad (\text{B.23})$$

$\bar{z}$  is the augmented constant vector  
 $\bar{z}$  is restricted as:

$$-170 \leq \bar{z}_i \leq -0.015 \quad (B.24)$$

This is the error term for IMP version 1.1; however, Brandon has used in later versions a new error option (Brandon, 1974d). This consists of taking a double step and comparing the solution with two single steps. This approach has been used with Runge-Kutta methods (Carnahan, Luther and Wilkes, 1969).

## B.2 Sparse Linear Equation Solution

Version 1.1 offers many options for solving sparse linear equations. These can occur in solving the corrector or the user can solve his own set of linear equations.

There are two iterative options: Gauss-Seidel double sweep and gradient iteration. Direct elimination using the Crout approach is available. In elimination, the user has options for ordering the system matrix, handling the fill-in and pivoting. Ordering is important, since the order in which the variables are eliminated affects the number of nonzero elements generated (fill-in). However, in normal applications problems, the natural order of the equations would be such that the bandwidth would be minimized.

The storage scheme used is cumulative indexing (section 5.1.3) and quasilinearization (section 4.5) is used in the corrector solution.

Future versions of IMP will include the following additions (Brandon 1974b):

- (1) the conjugate gradient method
- (2) special elimination routines for banded matrices
- (3) special Gauss-Seidel routine for banded matrices
- (4) special elimination routine for symmetric and banded symmetric matrices
- (5) special conjugate gradient method for symmetric matrices
- (6) full matrix elimination for smaller full matrix problems.

## APPENDIX C: SHANNON'S METHOD

Shannon (1971) has developed a technique for the numerical integration of stiff, sensitive and multivalued equations.

For a single equation, the method is very simple.

$$\frac{dy}{dt} = f(t, y) \quad (C.1)$$

If  $f(t, y)$  is less than 1,  $t$  is the dependent variable or the variable of integration. However, if  $f(t, y)$  is or becomes greater than one,  $y$  is made the variable of integration.

i.e.

$$\frac{dt}{dy} = \frac{1}{f(t, y)} \quad (C.2)$$

This may be extended to systems of equations as follows.

For the system:

$$\begin{aligned} \frac{dy_1}{dy_n} &= f_1(y_1, \dots, y_n) \\ \frac{dy_2}{dy_n} &= f_2(y_1, \dots, y_n) \\ &\vdots \\ \frac{dy_n}{dy_n} &= 1 \end{aligned} \quad (C.3)$$

where  $y_n$  is time, the normal variable of integration,  
if we form the matrix of all possible derivatives,

$$D = \begin{bmatrix} \frac{dy_1}{dy_1} & \frac{dy_1}{dy_2} & \cdots & \frac{dy_1}{dy_n} \\ \frac{dy_2}{dy_1} & \frac{dy_2}{dy_2} & \cdots & \frac{dy_2}{dy_n} \\ \vdots & \vdots & \ddots & \vdots \\ \frac{dy_n}{dy_1} & \frac{dy_n}{dy_2} & \cdots & \frac{dy_n}{dy_n} \end{bmatrix} \quad (C.4)$$

It can be shown that there is always at least one column of  $D$  where the values of all derivatives are less than or equal to one in absolute value. If this is the  $k$ th column, then  $y_k$  should be taken as the variable of integration. If the integration proceeds with  $y_k$  as the variable of integration and the largest element  $dy_i/dy_k$  becomes greater than 1, then  $y_i$  will become the independent variable. The derivatives in column  $i$  will then be less than or equal to 1.

Thus the variable of integration is changed as the integration proceeds, so that all the derivatives with respect to that variable are less than or equal to 1. Any method of integration may be used.

Let us try this method on SYSTEM I (Appendix D).



the original system is:

$$\begin{aligned}\frac{dy_1}{dt} &= -500.5y_1 + 499.5y_2 & y_1(0) &= 0 \\ \frac{dy_2}{dt} &= 499.5y_1 - 500.5y_2 & y_2(0) &= 2\end{aligned}\tag{C.5}$$

with analytical solution:

$$\begin{aligned}y_1 &= e^{-t} - e^{-1000t} \\ y_2 &= e^{-t} + e^{-1000t}\end{aligned}\tag{C.6}$$

making time  $y_3$ , the system becomes:

$$\begin{aligned}\frac{dy_1}{dy_3} &= -500.5y_1 + 499.5y_2 & y_1(0) &= 0 \\ \frac{dy_2}{dy_3} &= 499.5y_1 - 500.5y_2 & y_2(0) &= 2 \\ \frac{dy_3}{dy_3} &= 1 & y_3(0) &= 0\end{aligned}\tag{C.7}$$

the matrix of derivatives is:

$$D = \begin{bmatrix} \frac{dy_1}{dy_1} & \frac{dy_1}{dy_2} & \frac{dy_1}{dy_3} \\ \frac{dy_2}{dy_1} & \frac{dy_2}{dy_2} & \frac{dy_2}{dy_3} \\ \frac{dy_3}{dy_1} & \frac{dy_3}{dy_2} & \frac{dy_3}{dy_3} \end{bmatrix}\tag{C.8}$$

$$= \begin{bmatrix} \frac{-500.5y_1 + 499.5y_2}{499.5y_1 - 500.5y_2} & \frac{-500.5y_1 + 499.5y_2}{499.5y_1 - 500.5y_2} \\ \frac{499.5y_1 - 500.5y_2}{-500.5y_1 + 499.5y_2} & \frac{499.5y_1 - 500.5y_2}{-500.5y_1 + 499.5y_2} \\ \frac{-500.5y_1 + 499.5y_2}{499.5y_1 - 500.5y_2} & \frac{-500.5y_1 + 499.5y_2}{499.5y_1 - 500.5y_2} \end{bmatrix} \quad (3.9)$$

With  $y_2$  as the variable of integration, the Jacobian matrix is:

$$J(y_2) = \begin{bmatrix} -500.5 & 499.5 & 0 \\ 499.5 & -500.5 & 0 \\ 0 & 0 & 0 \end{bmatrix} \quad (3.10)$$

The eigenvalues are  $-1$ ,  $-1001$  and  $0$ .

At time  $0$ ,  $y_1 = 0$ ,  $y_2 = 0$

$$D(0) = \begin{bmatrix} 1 & -\frac{999}{1001} & 999 \\ -\frac{1001}{999} & 1 & -1001 \\ \frac{1}{999} & -\frac{1}{1001} & 1 \end{bmatrix} \quad (3.11)$$

Shannon's method say to make  $y_2$  the independent variable.

If we make  $y_2$  the independent variable

$$\begin{aligned}\frac{dy_1}{dy_2} &= \frac{-500.5y_1 + 499.5y_2}{499.5y_1 - 500.5y_2} \\ \frac{dy_2}{dy_2} &= 1 \\ \frac{dy_3}{dy_2} &= \frac{1}{499.5y_1 - 500.5y_2}\end{aligned}\tag{C.12}$$

The Jacobian matrix is:

$$\underline{J}(y_2) = \begin{bmatrix} \frac{1000y_2}{(499.5y_1 - 500.5y_2)^2} & \frac{-1000y_1}{(499.5y_1 - 500.5y_2)^2} & 0 \\ 0 & 0 & 0 \\ \frac{-499.5}{(499.5y_1 - 500.5y_2)^2} & \frac{500.5}{(499.5y_1 - 500.5y_2)^2} & 0 \end{bmatrix}\tag{C.13}$$

The eigenvalues are  $\frac{1000y_2}{(499.5y_1 - 500.5y_2)^2}, 0, 0$ .

at time 0:  $y_1=0, y_2=2$   $\lambda = \frac{2000}{(1001)^2} \approx 2 \times 10^{-5}, 0, 0$  (C.14)

as  $t \rightarrow 0.01$   $y_1 + y_2 = y \rightarrow 0$   $\lambda = \frac{1000}{y} \rightarrow \infty, 0, 0$  (C.15)

To see what happens when  $y_1$  is the variable of integration

$$\frac{dy_1}{dy_1} = 1$$

$$\frac{dy_2}{dy_1} = \frac{499.5y_1 - 500.5y_2}{-500.5y_1 + 499.5y_2} \quad (C.16)$$

$$\frac{dy_3}{dy_1} = \frac{1}{-500.5y_1 + 499.5y_2}$$

The Jacobian matrix  $J(y_1)$  is:

$$J(y_1) = \begin{bmatrix} 0 & 0 & 0 \\ \frac{-1000y_3}{(-500.5y_1 + 499.5y_2)^2} & \frac{1000y_1}{(-500.5y_1 + 499.5y_2)^2} & 0 \\ \frac{500.5}{(-500.5y_1 + 499.5y_2)^2} & \frac{-499.5}{(-500.5y_1 + 499.5y_2)^2} & 0 \end{bmatrix} \quad (C.17)$$

The eigenvalues are  $\frac{1000y_1}{(-500.5y_1 + 499.5y_2)^2}$ , 0, 0.

$$\text{At time } 0: \quad y_1=0, \quad y_2=2 \quad \lambda = 0, 0, 0 \quad (C.18)$$

$$\text{as } t \rightarrow 0.01 \quad y_1+y_2=y+0 \quad \lambda = \frac{1000}{y} \rightarrow \infty, 0, 0 \quad (C.19)$$

If  $y_2$  is made the independent variable, the eigenvalues of the Jacobian at time 0 are 0, 0,  $2 \times 10^{-3}$ . However a very short time later (-0.01), the largest eigenvalue approaches infinity.

Similarly if  $y_1$  is made the variable of integration, the largest eigenvalue also approaches infinity.

As  $t \rightarrow 0.01$ ,  $e^{-1000t} \rightarrow 0$ , and  $y_1 \rightarrow y_2 = y_3 \rightarrow 0$ .

The matrix of derivatives becomes:

$$D = \begin{bmatrix} 1 & 1 & -y \\ 1 & 1 & -y \\ -\frac{1}{y} & -\frac{1}{y} & 1 \end{bmatrix} \rightarrow \begin{bmatrix} 1 & 1 & 0 \\ 1 & 1 & 0 \\ -\infty & -\infty & 1 \end{bmatrix} \quad (C.29)$$

As  $t$  approaches 0.01 approximately,  $y_3$  (time) should be made the variable of integration.

Thus for a very short period of time, the change of independent variable from  $y_3$  to  $y_2$  would speed up the integration; however, after that period we are back where we started with  $y_3$  (time), again the independent variable.

Thus Shannon's method seems to be of limited effectiveness for handling stiff systems.

APPENDIX D: TEST EXAMPLES

Seven stiff examples were tested:

SYSTEM I: Moderately Stiff Linear Real  
Eigenvalue Example

II: Very Stiff Linear Real Eigenvalue  
Example

III: Linear Complex Eigenvalue Example

IV: Krogh's Example

V: Chemistry Example

VI: Polymer Example

VII: Stable Polymer Example

and four nonstiff examples:

SYSTEM VIII: Linear Nonstiff Complex Eigenvalue  
Example

IX: Krogh's Nonstiff Example

X: Nonlinear Reaction Example

XI: Gas Absorber Example

### D.1 SYSTEM I: Moderately Stiff Linear Real Eigenvalue

#### Example

We use the 2-dimensional linear system:

$$\dot{y} = \frac{1}{\Delta} y = \begin{bmatrix} -a & b \\ b & -a \end{bmatrix} y \quad y(0) = \begin{bmatrix} 0 \\ 2 \end{bmatrix} \quad (D.1.1)$$

with analytical solution:

$$\begin{aligned} y_1 &= e^{-(a-b)t} - e^{-(a+b)t} \\ y_2 &= e^{-(a-b)t} + e^{-(a+b)t} \end{aligned} \quad (D.1.2)$$

We choose  $a = 500.5$ ,  $b = 499.5$  to yield the moderately stiff example:

$$\begin{aligned} \dot{y}_1 &= -500.5 y_1 + 499.5 y_2, \quad y_1(0) = 0 \\ \dot{y}_2 &= 499.5 y_1 - 500.5 y_2, \quad y_2(0) = 2 \end{aligned} \quad (D.1.3)$$

with analytical solution:

$$\begin{aligned} y_1 &= e^{-t} - e^{-1000t} \\ y_2 &= e^{-t} + e^{-1000t} \end{aligned} \quad (D.1.4)$$

The range of integration is  $[0,1]$

The eigenvalues are  $-1$  and  $-1000$ .

### D.2 SYSTEM II: Very Stiff Linear Real Eigenvalue Example

As in SYSTEM I we choose  $a = 500000.5$ ,  $b = 499999.5$  to yield the very stiff example:

$$\begin{aligned} \dot{y}_1 &= -500000.5y_1 + 499999.5y_2 & y_1(0) &= 0 \\ \dot{y}_2 &= 499999.5y_1 - 500000.5y_2 & y_2(0) &= 2 \end{aligned} \quad (D.2.1)$$

with analytical solution:

$$\begin{aligned} y_1 &= e^{-t} - e^{-1000000t} \\ y_2 &= e^{-t} + e^{-1000000t} \end{aligned} \quad (D.2.2)$$

The range of integration is  $[0,1]$

The eigenvalues are  $-1$  and  $-1000000$ .

The stiffness ratio of  $10^6$  is as large as has been found in practice.

### D.3 SYSTEM III: Linear Complex Eigenvalue Example

The system:

$$\begin{aligned} \dot{y}_1 &= -Ay_1 + By_2 & + (A - B - 1)e^{-t} & y_1(0)=2 \\ \dot{y}_2 &= -By_2 - Ay_1 & + (A + B - 1)e^{-t} & y_2(0)=2 \\ \dot{y}_3 &= & - Cy_3 + Dy_4 & + (C - D - 1)e^{-t} & y_3(0)=2 \\ \dot{y}_4 &= & - Dy_3 - Cy_4 & + (C + D - 1)e^{-t} & y_4(0)=2 \end{aligned} \quad (D.3.1)$$



is used with analytical solution:

$$\begin{aligned}
 y_1 &= e^{-At} (\cos Bt + \sin Bt) + e^{-t} \\
 y_2 &= e^{-At} (\cos Bt - \sin Bt) + e^{-t} \\
 y_3 &= e^{-Ct} (\cos Dt + \sin Dt) + e^{-t} \\
 y_4 &= e^{-Ct} (\cos Dt - \sin Dt) + e^{-t}
 \end{aligned}
 \tag{D.3.2}$$

We choose  $A = 1000$ ,  $B = C = D = 1$  to yield the moderately stiff example:

$$\begin{aligned}
 \dot{y}_1 &= -1000y_1 + y_2 + 999e^{-t} & y_1(0) &= 2 \\
 \dot{y}_2 &= -y_1 - 1000y_2 + 1000e^{-t} & y_2(0) &= 2 \\
 \dot{y}_3 &= -y_3 + y_4 - e^{-t} & y_3(0) &= 2 \\
 \dot{y}_4 &= -y_3 - y_4 + e^{-t} & y_4(0) &= 2
 \end{aligned}
 \tag{D.3.3}$$

with analytical solution:

$$\begin{aligned}
 y_1 &= e^{-1000t} (\cos t + \sin t) + e^{-t} \\
 y_2 &= e^{-1000t} (\cos t - \sin t) + e^{-t} \\
 y_3 &= e^{-t} (\cos t + \sin t) + e^{-t} \\
 y_4 &= e^{-t} (\cos t - \sin t) + e^{-t}
 \end{aligned}
 \tag{D.3.4}$$

The range of integration is  $[0, 1]$

The eigenvalues are  $-1000 \pm i$ ,  $-1 \pm i$ .

#### D.4 SYSTEM IV: Krogh's Example

A nonlinear example suggested by Krogh was taken from Gear (1971c). The system:

$$\dot{z}_i = -\beta_i z_i + z_i^2 \quad i=1,2,3,4 \quad z_i(0) = -1 \quad (D.4.1)$$

has analytical solution:

$$z_i = \frac{\beta_i}{1 + c_i e^{\beta_i t}} \quad c_i = -(1 + \beta_i) \quad (D.4.2)$$

using the transformation  $y = u z$  (D.4.3)

where

$$u = 1/S \quad \begin{bmatrix} -1 & 1 & 1 & 1 \\ 1 & -1 & 1 & -1 \\ 1 & 1 & -1 & 1 \\ 1 & 1 & 1 & -1 \end{bmatrix} \quad (D.4.4)$$

note that  $z = u^{-1} y$  since  $u = u^{-1}$

we obtain the system:

$$\dot{y} = -u \beta u y + u z^2 \quad (D.4.5)$$

where

$$u \beta u = \begin{bmatrix} \beta_1 & & & \\ & \beta_2 & & \\ & & \beta_3 & \\ & & & \beta_4 \end{bmatrix} \quad (D.4.6)$$

Using  $\beta_1 = 1000$ ,  $\beta_2 = 800$ ,  $\beta_3 = -10$  and  $\beta_4 = 0.001$  we obtain:

$$C_{21} = \frac{1}{2} \begin{bmatrix} -1790.001 & 1809.999 & 189.999 & 210.001 \\ 1809.999 & -1790.001 & -210.001 & -189.999 \\ 189.999 & -210.001 & -1790.001 & -1809.999 \\ 210.001 & -189.999 & -1809.999 & -1790.001 \end{bmatrix}$$

$$+ \frac{1}{1/n} \begin{bmatrix} y_1 + y_2 + y_3 + y_4 + 2y_1y_2 + 2y_1y_3 + 2y_1y_4 - 2y_2y_3 - 2y_2y_4 - 2y_3y_4 \\ " " " " + 2 " - 2 " - 2 " + 2 " + 2 " - 2 " \\ " " " " - 2 " + 2 " - 2 " + 2 " - 2 " + 2 " \\ " " " " - 2 " - 2 " + 2 " - 2 " + 2 " + 2 " \end{bmatrix}$$

(D.4.7)

$$y(0) = \begin{bmatrix} -1 \\ -1 \\ -1 \\ -1 \end{bmatrix}$$

The range of integration is [0,5]

The eigenvalues are time-variant. Initially they are  $2z_i - \beta_i$ , i.e., -1002, -802, 8 and -2.001. As  $t \rightarrow \infty$  the eigenvalues approach  $\beta_i$  (if  $\beta_i < 0$ ) and 0 (if  $\beta_i > 0$ ).

### D.5 SYSTEM V: Chemistry Example

An example which arose in a chemistry problem is taken from Gear (1969a):

$$\begin{aligned}
 \dot{y}_1 &= -0.013 y_1 - 1000 y_1 y_3 & y_1(0) &= 1 \\
 \dot{y}_2 &= -2500 y_2 y_3 & y_2(0) &= 1 \\
 \dot{y}_3 &= -0.013 y_1 - 1000 y_1 y_3 - 2500 y_2 y_3 & y_3(0) &= 0
 \end{aligned} \tag{D.5.1}$$

The range of integration is [0,50]

The eigenvalues are real and initially are about -3500, 0.017 and 0.

### D.6 SYSTEM VI: Polymer Example

A large stiff system of 33 equations representing a polymerization reaction was taken from Hull *et al.* (1972b):

$$\begin{aligned}
 \dot{y}_1 &= k_1 y_2^2 - k_2 y_1 y_3 + \frac{2D}{h^2} [y_2 - (1 + \frac{0.01k}{D}) y_1] \\
 \dot{y}_i &= k_1 y_{11+i}^2 - k_2 y_i y_{22+i} + \frac{D}{h^2} (y_{i-1} - 2y_i + y_{i+1}) \quad i=2, 3, \dots, 10 \\
 \dot{y}_{11} &= k_1 y_{22}^2 - k_2 y_{11} y_{33} + \frac{2D}{h^2} (y_{10} - y_{11}) \\
 \dot{y}_i &= 2k_1 y_i^2 + 2k_2 y_{i-11} y_{i+11} + k_3 y_{i+11} \quad i=12, 13, \dots, 22 \\
 \dot{y}_i &= 2k_1 y_{i-11}^2 - 2k_3 y_{i-22} y_i - 2k_3 y_i \quad i=23, 24, \dots, 33
 \end{aligned} \tag{D.6.1}$$

where

$$k_1 = 0.70$$

$$k_2 = 1.35$$

$$k_3 = 3 \times 10^{-8}$$

$$D = 1.15 \times 10^{-4}$$

$$h = 0.00127$$

(D.6.2)

initial conditions  $y_i(0) = 0 \quad i = 1, \dots, 11$   
 $= 9.0 \quad i = 12, 13, \dots, 22$   
 $= 0.0 \quad i = 23, 24, \dots, 33$

The range of integration was  $[0, 100]$

This example represents a physically unstable system in that some of the variables ( $y_i, i = 23, 24, \dots, 33$ ) become larger and larger as integration proceeds. Mathematically this means that the eigenvalues corresponding to these variables are positive.

#### D.7 SYSTEM VII: Stable Polymer Example

The polymer example in 2.2.5 is an unstable system. If  $k_2$  is changed to  $k_1$  in equations for  $i = 12-22$ , the system becomes a stable fictitious one.

### D.8 SYSTEM VIII: Linear Nonstiff Complex Eigenvalue

#### Example

The system:

$$\begin{aligned}
 \dot{y}_1 &= -Ay_1 + By_2 & y_1(0) &= 1 \\
 \dot{y}_2 &= -By_1 - Ay_2 & y_2(0) &= 1 \\
 \dot{y}_3 &= -Cy_3 + Dy_4 & y_3(0) &= 1 \\
 \dot{y}_4 &= -Dy_3 - Cy_4 & y_4(0) &= 1
 \end{aligned}
 \tag{D.8.1}$$

is used with analytical solution:

$$\begin{aligned}
 y_1 &= e^{-At}(\cos Bt + \sin Bt) \\
 y_2 &= e^{-At}(\cos Bt - \sin Bt) \\
 y_3 &= e^{-Ct}(\cos Dt + \sin Dt) \\
 y_4 &= e^{-Ct}(\cos Dt - \sin Dt)
 \end{aligned}
 \tag{D.8.2}$$

We choose  $A = 0.5$ ,  $B = 0.25$ ,  $C = 0.25$ ,  $D = 0.5$  to yield:

$$\begin{aligned}
 \dot{y}_1 &= -0.5y_1 + 0.25y_2 & y_1(0) &= 1 \\
 \dot{y}_2 &= -0.25y_1 - 0.5y_2 & y_2(0) &= 1 \\
 \dot{y}_3 &= -0.25y_3 + 0.5y_4 & y_3(0) &= 1 \\
 \dot{y}_4 &= -0.5y_3 - 0.25y_4 & y_4(0) &= 1
 \end{aligned}
 \tag{D.8.3}$$

with analytical solution:

$$\begin{aligned}
 y_1 &= e^{-0.5t} (\cos 0.25t + \sin 0.25t) \\
 y_2 &= e^{-0.5t} (\cos 0.25t - \sin 0.25t) \\
 y_3 &= e^{-0.25t} (\cos 0.5t + \sin 0.5t) \\
 y_4 &= e^{-0.25t} (\cos 0.5t - \sin 0.5t)
 \end{aligned}
 \tag{D.8.4}$$

The range of integration is  $[0, 5]$

The eigenvalues are  $-0.5 \pm 0.25i$ ,  $-0.25 \pm 0.5i$ .

#### D.9 SYSTEM IX: Krogh's Nonstiff Example

The equations of SYSTEM IV: Krogh's example are used but with  $\beta_1 = 0.2$ ,  $\beta_2 = 0.2$ ,  $\beta_3 = 0.3$ ,  $\beta_4 = 0.4$  to obtain:

$$\begin{aligned}
 \dot{y} &= \frac{1}{4} \begin{bmatrix} -1.0 & -0.4 & -0.2 & 0.0 \\ -0.4 & -1.0 & 0.0 & 0.2 \\ -0.2 & 0.0 & -1.0 & 0.4 \\ 0.0 & 0.2 & 0.4 & -1.0 \end{bmatrix} y \\
 + \frac{1}{4} & \begin{bmatrix} y_1^2 + y_2^2 + y_3^2 + y_4^2 + 2y_1y_2 + 2y_1y_3 + 2y_1y_4 - 2y_2y_3 - 2y_2y_4 - 2y_3y_4 \\ " & " & " & " + 2" & - 2" & - 2" & + 2" & + 2" & - 2" \\ " & " & " & " - 2" & + 2" & - 2" & + 2" & - 2" & + 2" \\ " & " & " & " - 2" & - 2" & + 2" & - 2" & + 2" & + 2" \end{bmatrix}
 \end{aligned}
 \tag{D.9.1}$$

$$y(0) = \begin{bmatrix} - \\ - \\ - \\ - \end{bmatrix}$$

The range of integration is [0,10]

Here the step size is constrained only by accuracy not stability. The initial eigenvalues are -2.1, -2.2, -2.3 and -2.4. As  $t \rightarrow \infty$ , all the eigenvalues approach zero.

#### 0.10 SYSTEM X: Nonlinear Reaction Example

A system representing a nonlinear reaction was taken from Hull et al. (1972b):

$$\begin{aligned} \dot{y}_1 &= -y_1 & y_1(0) &= 1 \\ \dot{y}_2 &= y_1 - y_2^2 & y_2(0) &= 0 \\ \dot{y}_3 &= y_2^2 & y_3(0) &= 0 \end{aligned} \quad (0.10.1)$$

The integration range was [0,10].



D.11 SYSTEM XI: Gas Absorber Example

A nonlinear system representing the dynamics of a gas absorber was taken from Lapidus and Seinfeld (1971):

$$\begin{aligned} \dot{y}_1 &= \{-[40.8+66.7(M_1+0.08y_1)]y_1 \\ &\quad +66.7(M_2+0.08y_2)y_2\}/s_1+40.8v_1/s_1 \\ \dot{y}_2 &= \{40.8y_{2-1}-[40.8+66.7(M_2+0.08y_2)]y_2 \\ &\quad +66.7(M_{2+1}+0.08y_{2+1})y_{2+1}\}/s_2 \quad i=2,3,4,5 \quad (D.11.1) \\ \dot{y}_6 &= \{40.8y_5-[40.8+66.7(M_6+0.08y_6)]y_6 \\ &\quad +66.7(M_7+0.08y_7)y_7\}/s_6 \end{aligned}$$

where

$$\begin{aligned} v_1 &= v_6 = 0 \\ s_i &= (M_i + 0.16y_i) + 75 \quad i=1,2,\dots,6 \quad (D.11.2) \end{aligned}$$

with initial conditions:

$$y_0 = \begin{bmatrix} -0.08424992 \\ -0.06192031 \\ -0.08368619 \\ -0.10042889 \\ -0.11306320 \\ -0.12243631 \end{bmatrix} \quad M = \begin{bmatrix} 0.73576500 \\ 0.74875687 \\ 0.75929635 \\ 0.76774008 \\ 0.77442637 \\ 0.77971110 \\ 0.78363672 \end{bmatrix} \quad (D.11.3)$$

The range of integration was [0,50].

## APPENDIX E: COMPUTER TIMINGS

In Chapters 5 and 6, various numerical techniques for solving linear algebraic equations and ordinary differential equations were compared by their computer execution times for solving a set of test examples. Only the time to actually solve the problem was measured; time for input-output or problem set-up was not included.

All of the runs were made on the CDC Cyber 73 computer at the University of Western Ontario. The Fortran Extended (FTN), version 4.2 compiler was used with optimization level 1 and Scope 3.4.2 operating system. All of the IMP tests, however, used an object deck supplied by Control Data from FTN version 4.0, optimization level 2 and Scope 3.4 operating system. Some of the longer non-IMP runs were made with both optimization levels 1 and 2, but the difference in execution time was very slight (about 1%).

A rigorous comparison of numerical techniques for solving o.d.e.s should do more than just measure execution time. Hull *et al.* (1972a) also measure the number of derivative evaluations, the time required to evaluate derivatives and the remaining execution time called the overhead. Krogh (1973) discusses in depth, the aspects of

numerically testing a subroutine for the numerical solutions of o.d.e.s.

Execution time itself is not a perfect criteria for comparison as one program may be faster than another on one computer but slower with a different machine. This arises because of different relative computer times for the various arithmetic operations: addition, multiplication and division. For example on the CYBER 73 or CDC 6400, the times for multiplication and division are identical, whereas on other computers, division can be three or four times slower. For the CYBER 73, the ratio of clock pulses for addition, multiplication and division is 11:57:57 while it is 4:5:20 on the Cyber 76 (Brandon, 1974e). For the CYBER 73, this means that an algorithm requiring many divisions rather than multiplications will not appear any slower than another algorithm using few divisions and many multiplications. For example, the IMP algorithm uses fewer divisions than the TRGB algorithm, but this is not reflected in the CYBER 73 execution times. Ideally one should not measure execution time, but the numbers of the different arithmetic operations; these could then be converted into execution times for any desired computer. However, this is very inconvenient.

Another problem is the reproducibility of execution times particularly those below 0.5 seconds. Any program

taking less than 0.5 seconds was run several times to get an average. One program took 0.000, 0.019 and 0.038 on different runs. This corresponds to 0, 1 or 2 pulses of the clock almost on a random basis. If it was convenient, the portion of the program being timed was put in a DO loop and repeated several times to get a larger time which could then be divided by the number of repetitions.

If many other jobs are running in the computer at the same time, it is possible for a program to be taken from core and placed in a waiting queue by another job of higher priority. This may have a very slight effect on the execution time. Ideally the execution times should be measured with no other jobs in the system. This is not very practical; but as much as possible, the times were measured during periods of low usage.

## APPENDIX F: THE DYNYSYS DATA SET

The data required for a simulation is inputted to the DYNYSYS program through the data set. This is described fully in the DYNYSYS Manual (Bobrow, Johnson and Ponton, 1971). We will give here, a brief description of a data set using Example #1 in Chapter 8 as an illustration.

Figure 8.2 contains the dynamic information flow diagram for the example and the data set is at the end of Figure 8.4.

There are four basic sections in a data set: the simulation data, the equipment data, the stream data and the physical properties data. All data is stored in 12 column fields. The first 12 columns contain only alphanumeric data and the numeric data goes into 5 F12 fields starting in column 13.

## F.1 Simulation Data

The simulation data begins with the data word "BEGIN". All cards before this are comments and are ignored by DYNSSYS.

For example #1

IN/OUT	3.0	sets the maximum number of streams entering or leaving any module to three (default = 5)
HMAX	0.01	sets the maximum integration set size to 0.01 (default = 0.05)
TIME	3.0	sets the time of simulation to 3.0 (default = 10.0)
LIBRARY	1.0	means that one new module will be defined immediately after this card. Most common modules such as CONT1, STIR1 and VALV1 are already in a library of module names stored inside DYNSSYS.
DELAY	9.0	DELAY is the new module being defined. It will be accessed as SUBROUTINE TYPE9

Other possible simulation data words are:

COMPS	X	sets the number of components to X (default = 1)
DELTAT	X	sets the initial integration step size to X (default = 0.00001)
HMIN	X	sets the minimum integration step size to X (default = 0.000001)
TOLERANCE	X	sets the error tolerance for the in- tegration to X (default = 0.001)
ORDER	X	sets the maximum permissible order of integration to X (default = 6)
NONSTIFF		causes the nonstiff option to be used (default = stiff option)
MINPIVOT	X	sets the minimum permissible pivot value for TRGB-TRGB2 to X (default = 0.000001)
PRINTING	X	causes output to be printed every X time steps (default = 1)
LINEPLOT		causes output to be printed as a line- plot (in this case additional data must be given after physical proper- ties data)
GRAPH		causes output to be printed as graph or graphs on CALCOMP plotter as well as Lineplot. LINEPLOT must also be used.

FUNCTION X causes X vectors of stream information to be read in as an input function. Stream data follows immediately.

The data word "PROCESS" signifies the beginning of the equipment data and thus the end of the simulation data.

## F.2 Equipment Data

The data word "PROCESS" signifies the beginning of the equipment data. Here data for the various modules is written in their order of execution.

CONT1 1.0 the first module to be executed is CONT1 , referred to as module #1 in the dynamic information flow diagram (Figure 8.2)

After this, the entering and leaving stream numbers are given. A positive number indicates an input stream while a negative number indicates an output stream.

3.0 -4.0 stream 3.0 enters CONT1 and stream 4 leaves it. This can be verified in Figure 8.2.



The following card gives the equipment parameters for the module. This is the data which describes the individual units.

3. 1500. 1000. 2. 3.

For example, the set point for the controller is the third parameter 1000. Up to five parameters may be given. If more are required, the data word "EXTRA" X must follow and the X extra parameters appear on the following data cards. The data set in Figure 8.10 gives an example of this. We continue in this way for all modules. The data word "END" signifies the end of the equipment data.

### F.3 Stream Data

The material flows between equipment are represented by material streams. Information about each stream is stored in the following vector.

position 1 - stream number  
2 - flag  
3 - total mass flow  
4 - temperature  
5 - pressure  
6 - mass fraction of component 1  
7 - mass fraction of component 2,  
etc.

The stream flag serves the dual purpose of identifying the stream type (flags with absolute value  $< 10$  are material flows, those  $> 10$  are information flows) and, if negative, suppresses printing of the stream vector in the output.

The user may choose his own system of units as long as he is consistent throughout. Usually English units are employed.

STREAMS        5.        means that initial information data will be given for five streams.

EXPLICIT        means that the data will be given explicitly.

The following cards give the data. For the first stream

1.    1.    1000.    100.    14.7  
1.

the stream number is 1. The stream is a material flow (the flag is less than 10). The flow rate, temperature and pressure are 1000., 100. and 14.7 respectively. There is only one component so its mass fraction is 1.0.

Data is given for five streams. The data word "END" signifies the end of the stream data.

#### F.4 Physical Properties Data

PROPERTIES -1. means that the physical properties of water are assumed.

The user may provide his own data if desired. See the DYNYSYS Manual for details.

The data word "END" signifies the end of the physical properties data and in this case the end of the data set.

## APPENDIX G: TIME DELAY MODULE

A module for approximating a fixed or variable time delay is available. Time delays may occur in pipelines between equipments or in instruments such as valves and controllers.

Instrument lag is fixed whereas a pipeline delay is usually variable depending on the flow rate of the fluid passing through it. The variable delay is equal to the volume of the delaying apparatus divided by the flow rate through it.

The bucket brigade approach is used to simulate the stream delay. The past times and stream information are stored in vectors and the stream output from the delay is interpolated from these values. For a lengthy delay, the storage space is prohibitive, but later versions of the module may remedy this.

The delayed streams are stored in  $SX(I,J,K)$  where  $I$  represents the vector of past times  
 $J$  represents different delays  
and  $K$  represents the value of the stream vector.

Not all of the stream is necessarily delayed; a fraction of it may be bypassed. Nor is the entire stream vector necessarily delayed; the user may choose the first variable to be delayed and only this variable and all others after it will be delayed. The delay may also simulate a pump by setting the inlet flow rate equal to the output flow rate.

There are five equipment parameters:

- (1) TLAG - if the time delay is fixed, TLAG is the value of the delay. If the delay is variable, TLAG is the negative of the volume of the delaying apparatus or pipeline.
- (2) BYP - fraction of stream not delayed, i.e., bypassed.
- (3) NV - number of storage spaces used in the delay vector. An error message is given if NV is not large enough.
- (4) N1 - first variable to be delayed  
if  $N1 = 3$ , the entire stream is delayed  
if  $N1 = 6$ , only the mass fractions are delayed.

- (5) FLAG - if FLAG is greater than zero, the module simulates a pump; if FLAG = 0, the delay is normal.

7 A listing of the time delay module is given in Figure G.1.

FIGURE G.1: LISTING OF TIME DELAY MODULE

```

C          SUBROUTINE TYPE9                                T09  1
C          THIS MODULE REPRESENTS A FIXED OR VARIABLE TIME DELAY T09  2
C          EQUIPMENT PARAMETERS                            T09  3
C          1 - TLAG = LENGTH OF TIME DELAY, IF TIME DELAY IS FIXED T09  4
C          - NEGATIVE OF VOLUME OF DELAYING EQUIPMENT OR PIPELINE, T09  5
C          IF TIME DELAY IS VARIABLE                      T09  6
C          2 - BYP = FRACTION OF STREAM NOT DELAYED, I.E., BYPASSED T09  7
C          3 - NV = NUMBER OF STORAGE SPACES USED IN DELAY VECTOR T09  8
C          4 - N1 = FIRST VARIABLE TO BE DELAYED 3-TOTAL FLOW, 4-TEMP., ETC T09  9
C          5 - FLAG = FLAG, GT. 0; OUTPUT FLOW CONTROL (PUMP) T09 10
C          FLAG, EQ. 0; NORMAL DELAY                     T09 11
C          COMMON /MAT/ MP(4,5), EP(4,5), S(2,5,7), EX(1) T09 12
C          COMMON /UNIT/ IN, NMP                          T09 13
C          COMMON /CON/ IG, NCOMP, NC4, H, NE, NS, NPR, NPOL, TMAX, IORDER, NGRAPH T09 14
C          COMMON /GEAR2/ EPS, TIME, KFLAG, JSTART, NBVMAX, ICONV, NOPPT, ISTIFF T09 15
C          COMMON /LAG/ ND, NSW                           T09 16
C          DIMENSION MC(10), SX(100,1,8)                 T09 17
C          INTEGER OUT                                     T09 18
C          DATA MC/10*1/                                  T09 19
C          SX IS THE DELAY MATRIX                          T09 20
C          1ST VECTOR - DELAY AT DIFFERENT VALUES OF TIME T09 21
C          2ND VECTOR - DIFFERENT DELAYS                  T09 22
C          3RD VECTOR - VALUES OF STREAM VECTOR          T09 23
C          IN=NMP(IN,3)                                    T09 24
C          OUT=-MP(IN,4)                                    T09 25
C          FIXED TIME DELAY                                T09 26
C          TLAG=EP(IN,1)                                    T09 27
C          VARIABLE TIME DELAY                             T09 28
C          IF (TLAG.LT.0.0) TLAG=ABS(TLAG)/S(2,IN,1)     T09 29
C          BYP=EP(IN,2)                                    T09 30
C          NV=EP(IN,3)                                     T09 31
C          N1=EP(IN,4)                                     T09 32
C          FLAG=1, OUTPUT FLOW CONTROL , 0 , NORMAL     T09 33
C          IF (EP(IN,5).GT.0.0) S(1,IN,2)=S(1,OUT,2)    T09 34
C          FOR FIRST PREDICTOR STEP, SET ALL INITIAL INFORMATION REQUIRED T09 35
C          BY SX=MATRIX, FOR ALL FOLLOWING PREDICTOR STEPS SKIP THE DELAY T09 36
C          IF (IG.LT.2) GO TO 3                            T09 37
C          T09 38
C          T09 39
C          T09 40
C          T09 41
C          T09 42
C          T09 43
C          T09 44
C          T09 45
C          T09 46
C          T09 47
C          T09 48
C          T09 49

```

	IF (JSTART.NE.0) RETURN	T09 50
	ND=ND+1	T09 51
	SX(1,ND,1)=0.0	T09 52
C		T09 53
C	SET INPUT STREAM VALUES IN 1ST VECTOR OF SX-MATRIX	T09 54
C		T09 55
	DO 2 K=N1,NC4	T09 56
	SX(1,ND,K)=S(2,IN,K)	T09 57
C		T09 58
C	SET THE REST OF VALUES IN STORES TO EXIT VALUE	T09 59
C		T09 60
	DO 1 I=2,NV	T09 61
	SX(I,ND,K)=S(1,OUT,K)	T09 62
C		T09 63
C	FILL TIME STORES INITIALLY WITH -1.0 VALUES	T09 64
C		T09 65
	SX(I,ND,1)=-1.	T09 66
1	CONTINUE	T09 67
2	CONTINUE	T09 68
	RETURN	T09 69
C		T09 70
3	ND=ND+1	T09 71
	MC(ND)=MC(ND)+1	T09 72
C		T09 73
C	IF REPEAT THEN PICKED UP VALUES OF TIME STORED IN SX(.,1)	T09 74
C		T09 75
	IF (NSW.EQ.0) GO TO 5	T09 76
	MC(ND)=MC(ND)-1	T09 77
	DO 4 I=2,NV	T09 78
	SX(I,ND,1)=SX(I,ND,8)	T09 79
4	CONTINUE	T09 80
C		T09 81
C	NORMAL OPERATION,STORE A COPY OF TIME VECTOR FOR FUTURE REPEAT	T09 82
C	THEN INCREMENT TIME VALUES AS TIME PROGRESS	T09 83
C		T09 84
5	MD=MC(ND)	T09 85
	IF (MD.GT.NV) MD=NV	T09 86
	DO 6 I=2,NV	T09 87
	SX(I,ND,8)=SX(I,ND,1)	T09 88
6	CONTINUE	T09 89
7	IF (MD.EQ.2) GO TO 8	T09 90
	SX(MD,ND,1)=SX(MD-1,ND,1)+H	T09 91
	MD=MD-1	T09 92
	GO TO 7	T09 93
8	SX(2,ND,1)=H	T09 94
C		T09 95
C	SHIFT VALUES IN THE STORES RIGHT STARTING WITH RIGHT-MOST	T09 96
C		T09 97
	IF (NSW.EQ.1) GO TO 11	T09 98
	NV1=NV-1	T09 99
	DO 10 K=N1,NC4	T09 100
	DO 9 I=1,NV1	T09 101



	II=NV-I-1	T09 102
	L=I-1	T09 103
	SX(I,ND,K)=SX(L,ND,K)	T09 104
9	CONTINUE	T09 105
10	CONTINUE	T09 106
C		T09 107
C	TRANSFER VALUE OF INLET TO 1ST STORAGE OF DELAY VECTOR	T09 108
C		T09 109
11	DO 12 K=N1,NC4	T09 110
	SX(I,ND,K)=S(I,IN,K)	T09 111
12	CONTINUE	T09 112
C		T09 113
C	COMPARE STORED TIME VALUES WITH TIME LAG	T09 114
C		T09 115
	DO 13 I=1,NV1	T09 116
	IF (SX(I,ND,1)-TLAG) 13,14,17	T09 117
13	CONTINUE	T09 118
C		T09 119
C	IF NONE IS GREATER THAN TLAG,EXIT THE LAST VALUE	T09 120
C	IF TIME VALUE EQUALS TLAG,EXIT CORRESPONDING VALUES	T09 121
C		T09 122
	I=NV	T09 123
	IF (SX(NV,ND,1).EQ.-1.0) GO TO 14	T09 124
	PRINT 19	T09 125
	STOP	T09 126
14	DO 15 K=N1,NC4	T09 127
	S(I,OUT,K)=SX(I,ND,K)*(1.-BYP)+S(I,IN,K)*BYP	T09 128
15	CONTINUE	T09 129
	RETURN	T09 130
C		T09 131
C	IF SX(I).GT.TLAG,CHECK IF SX(I-1).GT.TLAG	T09 132
C	IF SO SET THE TIME AT SX(I) TO -1.	T09 133
C		T09 134
16	SX(I,ND,1)=-1.	T09 135
	I=I-1	T09 136
17	L=I-1	T09 137
C		T09 138
C	KEEP TESTING UNTIL ONLY ONE IS LEFT	T09 139
C		T09 140
	IF (SX(L,ND,1).GE.TLAG) GO TO 16	T09 141
C		T09 142
C	INTERPOLATE FOR EXIT VALUE	T09 143
C		T09 144
	A=TLAG-SX(L,ND,1)	T09 145
	B=SX(I,ND,1)-SX(L,ND,1)	T09 146
	DO 18 K=N1,NC4	T09 147
	U=SX(L,ND,K)*(A/B)+(SX(I,ND,K)-SX(L,ND,K))	T09 148
	S(I,OUT,K)=(1.-BYP)*U+BYP*S(I,IN,K)	T09 149
18	CONTINUE	T09 150
	RETURN	T09 151
C		T09 152
19	FORMAT (20H ERROR IN TIME DELAY,/.23H NV MUST BE MADE LARGER)	T09 153

251

END

T09 154

APPENDIX H: WILLIAMS-OTTO PLANT LISTINGS

Figure H.1 gives a listing of all the modules and the data set used in the Williams-Otto plant simulation. The distillation column modules are not included. The version of the executive used allowed for 10 equipment parameters.

FIGURE H.1: LISTING OF MODULES AND DATA SET FOR WILLIAMS-OTTO PLANT

SUBROUTINE TYPE10

SURROUTINE CSTR1

THIS MODULE REPRESENTS THE CONTINUOUS STIRRED TANK REACTOR  
IN THE WILLIAMS-OTTO PLANT  
MODULE IS STIFF

EQUIPMENT PARAMETERS

- 1 - VR - INITIAL HOLDUP IN REACTOR (LB)
- 2 - VW - HOLDUP IN COOLING COIL (LB)
- 3 - HMAW - OVERALL COOLING COIL HEAT TRANSFER COEFFICIENT  
\* EFFECTIVE HEAT TRANSFER AREA (BTU/(DEGREE F\*HR))
- 4 - HMAS - OVERALL STEAM COIL HEAT TRANSFER COEFFICIENT  
\* EFFECTIVE HEAT TRANSFER AREA (BTU/(DEGREE F\*HR))
- 5 - XLHC - LATENT HEAT OF CONDENSATION OF ENTERING STEAM (BTU/LB)

STREAMS

- 1 - INA - A FEED STREAM
- 2 - INB - B FEED STREAM
- 3 - INL - L RECYCLE STREAM
- 4 - INC - COOLING COIL INPUT STREAM
- 5 - INS - STEAM COIL INPUT STREAM
- 6 - IOUT - PRODUCT STREAM
- 7 - IOUTC - COOLING COIL OUTPUT STREAM
- 8 - IOUTS - STEAM COIL OUTPUT STREAM
- 9 - ISIGL - LEVEL CONTROL SIGNAL
- 10 - ISIGB - B FLOW CONTROL SIGNAL (RATIO FA/FB)

NOMENCLATURE

- FA - A FEED STREAM FLOW RATE (LB/HR)
- FB - B FEED STREAM FLOW RATE (LB/HR)
- FL - L RECYCLE STREAM FLOW RATE (LB/HR)
- FR - PRODUCT STREAM FLOW RATE (LB/HR)
- FW - COOLING COIL FLOW RATE (LB/HR)
- TA - A FEED STREAM TEMPERATURE (DEGREES F)
- TB - B FEED STREAM TEMPERATURE (DEGREES F)
- TL - L RECYCLE STREAM TEMPERATURE (DEGREES F)
- TINC - INLET COOLING COIL TEMPERATURE (DEGREES F)
- TS - STEAM COIL TEMPERATURE (DEGREES F)
- SFLOW - STEAM FLOW RATE (LB/HR)
- CPR - HEAT CAPACITY OF REACTOR MIXTURE (BTU/(LB\*DEGREE F))  
(ASSUMED SAME FOR ALL COMPONENTS)
- CPW - HEAT CAPACITY OF COOLANT (BTU/(LB\*DEGREE F))

```

COMMON/MAT/MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF
COMMON /PTAB/ IGFLAG,PP(10,20)
COMMON /UNIT/ IM,NMP
COMMON /MODULE/ IDERY,ITER,ITRI,NZERO
COMMON/ROW/IROW(1000)
COMMON/COLUMN/JCOL(1000)
COMMON/JACOB/XJACOB(1000)
COMMON /RETN/ K1,K2,K3,H1,H2,H3,B1,B2,B3
DIMENSION IROWS(33),JCOLS(33)
DIMENSION Y(9),DERY(9)
REAL K1,K2,K3
DATA IROWS/3*1,4*2,5*3,4*4,4*5,4*6,6*7,2*8,9/
DATA JCOLS/1,2,7,1,2,3,7,1,2,3,6,7,2,3,4,7,3,5,6,7,2,3,6,7,1,2,3,
16,7,8,7,8,9/
DATA CPR,CPW/0.4,1.0/

```

C  
C  
C

CALCULATE MODULE PARAMETERS

```

IF (JSTART.NE.0.OR.IG.EQ.1) GO TO 2
VR=EP(IM,1)
VW=EP(IM,2)
HWA=EP(IM,3)
HSAS=EP(IM,4)
XLHC=EP(IM,5)
INA=MP(IM,3)
INB=MP(IM,4)
INL=MP(IM,5)
INC=MP(IM,6)
INS=MP(IM,7)
IOUT=IABS(MP(IM,8))
IOUTC=IABS(MP(IM,9))
IOUTS=IABS(MP(IM,10))
ISIGL=IABS(MP(IM,11))
ISIGB=IABS(MP(IM,12))
DO 31 I=1,6
31 Y(I)=S(2,IOUT,I+5)
Y(7)=S(2,IOUT,4)
Y(8)=S(2,IOUTC,4)
Y(9)=VR
2 CONTINUE
C USE NEWTON-RAPHSON ITERATION WITH STIFF OPTION
ITER=1
FA=S(IG,INA,3)
FB=S(IG,INB,3)
FL=S(IG,INL,3)
FR=S(IG,IOUT,3)
FW=S(IG,INC,3)
TA=S(IG,INA,4)
TB=S(IG,INB,4)
TL=S(IG,INL,4)

```

TINC=S(I6,INC,4)

C  
C  
C

CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS

IF(I6,EQ,2) GO TO 1

NZERO=33

DO 81 I=1,33

IROW(I)=IROWS(I)

81 JCOL(I)=JCOLS(I)

C  
C  
C

CALL REACT TO GET REACTION RATE COEFFICIENTS AND  
HEATS OF REACTION

TEMP=Y(7)+459.69

CALL REACT(TEMP)

T1=K1\*Y(1)\*Y(2)\*B1/(TEMP\*\*2)

T2=K2\*Y(2)\*Y(3)\*B2/(TEMP\*\*2)

T3=K3\*Y(3)\*Y(6)\*B3/(TEMP\*\*2)

XJACOB(1)=-FR/VR-K1\*Y(2)

XJACOB(2)=-K1\*Y(1)

XJACOB(3)=-T1

XJACOB(4)=-K1\*Y(2)

XJACOB(5)=-FR/VR-K1\*Y(1)-K2\*Y(3)

XJACOB(6)=-K2\*Y(2)

XJACOB(7)=-T1-T2

XJACOB(8)=2.0\*K1\*Y(2)

XJACOB(9)=2.0\*K1\*Y(1)-2.0\*K2\*Y(3)

XJACOB(10)=-FR/VR-2.0\*K2\*Y(2)-K3\*Y(6)

XJACOB(11)=-K3\*Y(3)

XJACOB(12)=2.0\*T1-2.0\*T2-T3

XJACOB(13)=2.0\*K2\*Y(3)

XJACOB(14)=2.0\*K2\*Y(2)

XJACOB(15)=-FR/VR

XJACOB(16)=2.0\*T2

XJACOB(17)=1.5\*K3\*Y(6)

XJACOB(18)=-FR/VR

XJACOB(19)=1.5\*K3\*Y(3)

XJACOB(20)=1.5\*T3

XJACOB(21)=K2\*Y(3)

XJACOB(22)=K2\*Y(2)-0.5\*K3\*Y(6)

XJACOB(23)=-0.5\*K3\*Y(3)-FR/VR

XJACOB(24)=T2-0.5\*T3

XJACOB(25)=-2.0\*K1\*Y(2)\*H1/CPR

XJACOB(26)=- (2.0\*K1\*Y(1)\*H1+3.0\*K2\*Y(3)\*H2)/CPR

XJACOB(27)=- (3.0\*K2\*Y(2)\*H2+1.5\*K3\*Y(6)\*H3)/CPR

XJACOB(28)=-1.5\*K3\*Y(3)\*H3/CPR

XJACOB(29)=- (2.0\*T1\*H1+3.0\*T2\*H2+1.5\*T3\*H3)/CPR

1=(HWAH+HSAS)/(VR\*CPR)-(FL+FA+FB)/VR

XJACOB(30)=HWAH/(VR\*CPR)

XJACOB(31)=HWAH/(VW\*CPW)

XJACOB(32)=- (HWAH+FW\*CPW)/(VW\*CPW)

XJACOB(33)=0.0

GO TO 12

1 CONTINUE

C  
C  
C  
C  
C

CALCULATE DERIVATIVES

CALL REACT TO GET REACTION RATE COEFFICIENTS AND  
HEATS OF REACTION

TEMP=Y(7)\*459.69

CALL REACT(TEMP)

12 CONTINUE

S1=K1\*Y(1)\*Y(2)

S2=K2\*Y(2)\*Y(3)

S3=K3\*Y(3)\*Y(6)

TS=S(IG,INS,4)

C

IF STEAM FLOW IS ZERO, THERE IS NO HEAT TRANSFER

IF(S(IG,INS,3).LT.0.001) TS=Y(7)

C

A MASS BALANCE

DERY(1)=1.0/VR\*(FA+FL\*S(IG,INL,6)-FR\*Y(1))-S1

C

B MASS BALANCE

DERY(2)=1.0/VR\*(FB+FL\*S(IG,INL,7)-FR\*Y(2))-S1-S2

C

C MASS BALANCE

DERY(3)=1.0/VR\*(FL\*S(IG,INL,8)-FR\*Y(3))+2.0\*S1-2.0\*S2-S3

C

E MASS BALANCE

DERY(4)=1.0/VR\*(FL\*S(IG,INL,9)-FR\*Y(4))+2.0\*S2

C

G MASS BALANCE

DERY(5)=1.0/VR\*(FL\*S(IG,INL,10)-FR\*Y(5))+1.5\*S3

C

P MASS BALANCE

DERY(6)=1.0/VR\*(FL\*S(IG,INL,11)-FR\*Y(6))+S2-0.5\*S3

C

TOTAL HEAT BALANCE

DERY(7)=1.0/(VR\*CPR)\*(-2.0\*S1\*H1\*VR-3.0\*S2\*H2\*VR-1.5\*S3\*H3\*VR

1-HWAW\*(Y(7)-Y(8))+HSAS\*(TS-Y(7))-FL\*CPR\*(Y(7)-TL)-FA\*CPR\*(Y(7)-TA)

2-FB\*CPR\*(Y(7)-TB))

C

HEAT BALANCE FOR COOLING COIL

DERY(8)=1.0/(W\*CPW)\*(HWAW\*(Y(7)-Y(8))+FW\*CPW\*(TINC-Y(8)))

C

TOTAL MASS BALANCE

DERY(9)=FA+FB+FL-FR

CALL DIFSUB(9,Y,DERY)

IF(IDERY.NE.0) GO TO 1

C

CALCULATE STREAM OUTPUT

C

PRODUCT STREAM

C

NORMALIZE MASS FRACTIONS

SUM=0.0

DO 55 I=1,6

55 SUM=SUM+Y(I)

DO 6 I=1,6

Y(I)=Y(I)/SUM

6 S(1,IOUT,I+5)=Y(I)

S(1,IOUT,4)=Y(7)

C

COOLING COIL

S(1,IOUTC,3)=S(1,INC,3)

S(1,IOUTC,4)=Y(8)

```
C STEAM COIL
IF(S(IG,INS,3).LT.0.001) GO TO 56
SFLOW=HSAS/XLHC*(S(IG,INS,4)-Y(7))
S(1,INS,3)=SFLOW
56 S(1,IOUTS,3)=S(1,INS,3)
S(1,IOUTS,4)=S(1,INS,4)
C CONTROL SIGNALS
S(1,ISIGL,3)=Y(9)
S(1,ISIGB,3)=FA/FB
RETURN
END
```



## SUBROUTINE REACT(T)

THIS SUBROUTINE SUPPLIES REACTION RATE COEFFICIENTS AND HEATS  
OF REACTION OF THE WILLIAMS-OTTO PLANT REACTION SCHEME BELOW

A + B = C  
C + B = P + E  
P + C = G

T IS DEFINED IN DEGREES RANKINE

COMMON /RETN/ K1,K2,K3,H1,H2,H3,B1,B2,B3  
REAL K1,K2,K3

CONSTANTS OF ARRHENIUS EQUATION :  $K = A \cdot \exp(-B/T)$

DATA A1,A2,A3/5.9755E+09,2.5962E+12,9.6283E+15/  
DATA B1,B2,B3/12000.0,15000.0,20000.0/

ARRHENIUS REACTION COEFFICIENTS

IF(T.LT.500.0) T=500.0  
IF(T.GT.1000.0) T=1000.0  
K1=A1\*EXP(-B1/T)  
K2=A2\*EXP(-B2/T)  
K3=A3\*EXP(-B3/T)

HEATS OF REACTION

H1=-125.0  
H2=-50.0  
H3=-143.0  
RETURN  
END

SUBROUTINE TYPE11

SUBROUTINE EXCH1

THIS MODULE REPRESENTS A SHELL AND TUBE HEAT EXCHANGER  
BOTH THE SHELL AND TUBE SIDE ARE MODELLED AS WELL-STIRRED TANKS  
MODULE IS NONSTIFF

EQUIPMENT PARAMETERS

1 - VHT - TUBE SIDE MASS HOLDUP (LB)  
2 - VHS - SHELL SIDE (COOLANT) MASS HOLDUP (LB H2O)  
3 - AH - AREA OF HEAT TRANSFER (FT\*\*2)  
4 - HH - OVERALL HEAT TRANSFER COEFFICIENT  
(BTU/(DEGREE F\*FT\*\*2\*HR))  
5 - CPW - COOLANT HEAT CAPACITY (BTU/(LB\*DEGREE F))

STREAMS

1 - INT - INLET TUBE SIDE  
2 - INS - INLET SHELL SIDE  
3 - IOUTT - OUTLET TUBE SIDE  
4 - IOUTS - OUTLET SHELL SIDE

NOMENCLATURE

FT - TUBE SIDE FLOW RATE (LB/HR)  
FS - SHELL SIDE FLOW RATE (LB/HR)  
TINT - INLET TUBE SIDE TEMPERATURE (DEGREES F)  
TINS - INLET SHELL SIDE TEMPERATURE (DEGREES F)  
CPR - TUBE SIDE HEAT CAPACITY (BTU/(LB\*DEGREE F))  
TLM - LOG MEAN TEMPERATURE

COMMON/MAT/MP(35,13),EP(35,10),S(2,45,11),EX(1)  
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH  
COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF  
COMMON /PTAB/ IGFLAG,PP(10,20)  
COMMON /UNIT/ IM,NMP  
COMMON /MODULE/ IDERY,ITER,ITRI,NZERO  
DIMENSION Y(8),DERY(8)  
DATA CPR/0.4/

CALCULATE MODULE PARAMETERS

IF(JSTART,NE,0.OR.IG.EQ.1) GO TO 2  
VHT=EP(IM,1)  
VHS=EP(IM,2)  
AH=EP(IM,3)  
HH=EP(IM,4)  
CPW=EP(IM,5)  
INT=MP(IM,3)  
INS=MP(IM,4)

```

      IOUTT=IABS(MP(IM,5))
      IOUTS=IABS(MP(IM,6))
      Y(1)=S(IG,IOUTT,4)
      Y(2)=S(IG,IOUTS,4)
      DO 9 I=1,NCOMP
9     Y(I+2)=S(IG,IOUTT,I+5)
2     CONTINUE
C     USE DIRECT ITERATION IF STIFF OPTION IS USED
      ITER=0
      FT=S(IG,INT,3)
      FS=S(IG,INS,3)
      TINT=S(IG,INT,4)
      TINS=S(IG,INS,4)
1     CONTINUE

C
C     CALCULATE DERIVATIVES
C
C     CALCULATE LOG MEAN TEMPERATURE
      TLM=((TINT-Y(2))+(Y(1)-TINS))/ALOG((TINT-Y(2))/(Y(1)-TINS))
C     HEAT BALANCE (TUBE SIDE)
      DERY(1)=1.0/(VHT*CPR)*(FT*CPR*(TINT-Y(1))-HH*AH*TLM)
C     HEAT BALANCE (SHELL SIDE)
      DERY(2)=1.0/(VHS*CPW)*(FS*CPW*(TINS-Y(2))+HH*AH*TLM)
C     COMPONENT MASS BALANCES (TUBE SIDE)
      DO 10 I=1,NCOMP
10    DERY(I+2)=FT/VHT*(S(IG,INT,I+5)-Y(I+2))
      N=2+NCOMP
      CALL DIFSUB(N,Y,DERY)
      IF(IDERY.NE.0) GO TO 1

C
C     CALCULATE STREAM OUTPUT
C
      S(1,IOUTT,4)=Y(1)
      S(1,IOUTS,4)=Y(2)
      S(1,IOUTS,3)=S(1,INS,3)
      SUM=0.0
      DO 15 I=3,N
15    SUM=SUM+Y(I)
C     PUT NORMALIZED MASS FRACTIONS INTO OUTLET TUBE SIDE STREAM
      DO 16 I=3,N
      Y(I)=Y(I)/SUM
16    S(1,IOUTT,I+3)=Y(I)
      RETURN
      END

```

## SURROUTINE TYPE12

## SUBROUTINE DCAN1

THIS MODULE REPRESENTS A DECANter  
 ONE COMPONENT IS SEPARATED BY GRAVITY FROM THE ENTERING STREAM  
 BOTH TOP AND BOTTOM LAYERS ARE MODELLED AS WELL-STIRRED TANKS  
 FUNCTION X6 GIVES WEIGHT FRACTION OF SEPARATED COMPONENT ENTERING  
 TOP LAYER OF DECANter AS FUNCTION OF TEMPERATURE  
 CONTROL SIGNALS FOR THE HOLDUP IN TOP AND BOTTOM LAYERS ARE  
 EACH SENT TO CONTROLLERS  
 MODULE IS NONSTIFF

## EQUIPMENT PARAMETERS

- 1 - HUPT - INITIAL HOLDUP IN TOP LAYER (LB)
- 2 - HUPB - INITIAL HOLDUP IN BOTTOM LAYER (LB)
- 3 - ISEP - NUMBER OF COMPONENT TO BE SEPARATED FROM ENTERING STREAM

## STREAMS

- 1 - IN - INLET
- 2 - IOUPT - OUTLET FROM TOP OF DECANter
- 3 - IOUTB - OUTLET FROM BOTTOM OF DECANter
- 4 - ISIGT - LEVEL CONTROL SIGNAL (TOP)
- 5 - ISIGB - LEVEL CONTROL SIGNAL (BOTTOM)

## NOMENCLATURE

- FIN - TOTAL INLET FLOW (LB/HR)
- FINT - INLET FLOW ENTERING TOP LAYER (LB/HR)
- FINB - INLET FLOW ENTERING BOTTOM LAYER (LB/HR)
- FOUT - TOTAL OUTLET FLOW (LB/HR)
- FOUPT - OUTLET FLOW FROM TOP LAYER (LB/HR)
- FOUTB - OUTLET FLOW FROM BOTTOM LAYER (LB/HR)
- FT - MASS FRACTIONS ENTERING TOP LAYER
- FEC - MASS FRACTIONS IN TOP LAYER
- WFO - MASS FRACTION OF SEPARATED COMPONENT ENTERING TOP LAYER
- TEMP - DECANter TEMPERATURE (DEGREES F)

```
COMMON/MAT/MP(35,13),EP(35,10),S(2,45,61),EX(1)
COMMON/CON/IG,NCOMP,NC5,H,NE,NS,NPE,NPOL,THAX,IORDER,NGRAPH
COMMON/GEAR2/EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF
COMMON/PTAB/IGFLAG,PP(10,20)
COMMON/UNIT/IM,NMP
COMMON/MODULE/IDERY,ITER,ITRI,NZERO
DIMENSION Y(9),DERY(9),FT(6),FEC(6)
```

## CALCULATE MODULE PARAMETERS

```
IF (JSTART.NE.0.OR.IG.EQ.1) GO TO 2
HUPT=EP(IM,1)
```

```

HUPB=EP(IN,2)
ISEP=EP(IN,3)
IN=MP(IN,3)
IOUTT=IABS(MP(IN,4))
IOUTB=IABS(MP(IN,5))
ISI6T=IABS(MP(IN,6))
ISI6B=IABS(MP(IN,7))
Y(1)=HUPT+HUPB
Y(2)=S(IG,IOUTT,4)
N=NCOMP+2
DO 4 I=3,N
4 Y(I)=HUPT+S(IG,IOUTT,I+3)
Y(N+1)=HUPB
2 CONTINUE
C USE DIRECT ITERATION IF STIFF OPTION IS USED
ITER=0
1 CONTINUE
C
C CALCULATE DERIVATIVES
C
FIN=S(IG,IN,3)
FOUTT=S(IG,IOUTT,3)
FOUTB=S(IG,IOUTB,3)
C TOTAL MASS BALANCE
DERY(1)=FIN-FOUTT-FOUTB
C TOTAL HEAT BALANCE
DERY(2)=1.0/Y(1)*(FIN*S(IG,IN,4)-(FOUTT+FOUTB)*Y(2))
TEMP=Y(2)
C FUNCTION X6 DETERMINES MASS FRACTION OF SEPARATED COMPONENT
C ENTERING TOP LAYER
WFG=X6(TEMP)
C WFG CANNOT BE GREATER THAN ENTERING MASS FRACTION
IF(WFG.GT.S(IG,IN,ISEP+5)) WFG=S(IG,IN,ISEP+5)
FINT=FIN*(1.0-S(IG,IN,ISEP+5))/(1.0-WFG)
FINB=FIN-FINT
DO 101 I=1,NCOMP
101 FT(I)=FIN*S(IG,IN,I+5)/FINT
FT(ISEP)=WFG
SUM=0.0
NF=NCOMP+2
DO 3 I=3,NF
3 SUM=SUM+Y(I)
DO 14 I=1,NCOMP
14 FEC(I)=Y(I+2)/SUM
C COMPONENT MASS BALANCES FOR TOP LAYER
DO 20 I=3,NF
20 DERY(I)=FINT*FT(I-2)-FOUTT*FEC(I-2)
C MASS BALANCE FOR BOTTOM LAYER
N=NCOMP+3
DERY(N)=FINB-FOUTB
CALL DIFSUB(N,Y,DERY)
IF(IDERY.NE.0) GO TO 1

```

```
C
C
C      CALCULATE STREAM OUTPUT
C
C      BOTTOM OUTLET STREAM
C      S(1,IOUTB,4)=TEMP
C      TOP OUTLET STREAM
C      S(1,IOUTT,4)=TEMP
C      SUM=0.0
C      DO 15 I=1,NCOMP
15    SUM=SUM+Y(I,2)
C      DO 16 I=1,NCOMP
16    S(1,IOUTT,I+5)=Y(I,2)/SUM
C      CONTROL SIGNALS
C      S(1,ISIGT,3)=Y(1)
C      S(1,ISIGB,3)=Y(9)
C      RETURN
C      END
```

## FUNCTION XG(T)

THIS FUNCTION CALCULATES THE WEIGHT FRACTION OF SEPARATED COMPONENT (G IN THE WILLIAMS-OTTO PLANT) ENTERING THE TOP LAYER OF DECANTER AS A FUNCTION OF TEMPERATURE

T IS TEMPERATURE IN DEGREES F

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C
C
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C
IF(T.LE.100.0) GO TO 1
IF(T=120.0) 2,3,4
1 XG=0.0
  RETURN
2 XG=0.005769+0.13365*(0.01*T-1.1)+0.7596*(0.01*T-1.1)**2
  IF(XG.LT.0.0) XG=0.0
  RETURN
3 XG=0.050865
  RETURN
4 XG=0.00894+0.3125*(0.01*T-1.24)-0.9*(0.01*T-1.24)**2
  IF(T.GT.128.0) PRINT 10,T
10 FORMAT(23H DECANTER TEMPERATURE =,F6.1,42H REQUIRES EXTRAPOLATION
  1OF SOLUBILITY DATA)
  RETURN
  END

```

SUBROUTINE TYPE13

SUBROUTINE CONV1

THIS MODULE CHANGES MASS FLOW AND MASS FRACTIONS TO MOLAR FLOW  
AND MOLE FRACTIONS OR VICE-VERSA

EQUIPMENT PARAMETERS

I = ICONV = 1, CHANGES MASS TO MOLES  
          = 2, CHANGES MOLES TO MASS

COMMON/MAT/MP(35,13),EP(35,10),S(2,45,11),EX(1)  
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH  
COMMON /PTAB/ IGFLAG,PP(10,20)  
COMMON /UNIT/ IM,NMP  
DIMENSION XMW(10),X(10),F(10),FNEW(10)

GET MODULE PARAMETERS

ICONV=EP(IM,1)  
IF(ICONV.NE.1.AND.ICONV.NE.2) GO TO 75  
IN=MP(IM,3)

IOUT=IABS(MP(IM,4))  
FLOWT=S(IG,IN,3)

DO 10 I=1,NCOMP

GET MOLECULAR WEIGHTS

XMW(I)=PP(I,1)  
X(I)=S(IG,IN,I+5)  
F(I)=FLOWT\*X(I)

10 CONTINUE

IF(ICONV.EQ.2) GO TO 91

CHANGE MASS TO MOLES

DO 11 I=1,NCOMP

11 FNEW(I)=F(I)/XMW(I)

GO TO 90

CHANGE MOLES TO MASS

91 DO 12 I=1,NCOMP

12 FNEW(I)=F(I)\*XMW(I)

90 CONTINUE

SUM=0.0

DO 13 I=1,NCOMP

13 SUM=SUM+FNEW(I)

DO 14 I=1,NCOMP

14 X(I)=FNEW(I)/SUM

CALCULATE STREAM OUTPUT



```
S(I0,IOUT,3)=SUM  
S(I0,IOUT,4)=S(I0,IN,4)  
S(I0,IOUT,5)=S(I0,IN,5)  
DO 15 I=1,NCOMP  
15 S(I0,IOUT,I+5)=X(I)  
RETURN  
75 PRINT 20  
20 FORMAT(37H CONV01 EQUIPMENT PARAMETER NE.1.0R.2)  
STOP  
END
```

SUBROUTINE TYPE14

SUBROUTINE SPLT1

THIS MODULE SPLITS AN INCOMING STREAM INTO 2 EXIT STREAMS  
ACCORDING TO A GIVEN RATIO

EQUIPMENT PARAMETERS

I - RATIO - DESIRED FLOW RATIO OF FIRST EXIT STREAM  
TO ENTERING STREAM

COMMON /MAT/ MP(35,13), EP(35,10), S(2,45,11), EX(1)  
COMMON /CON/ IG, NCOMP, NC5, H, NE, NS, NPR, NPOL, TMAX, IORDER, NGRAPH  
COMMON /UNIT/ IN, NMP

RATIO=EP(IN,1)  
IN=NMP(IN,3)  
IOUT1=IABS(MP(IN,4))  
IOUT2=IABS(MP(IN,5))  
S(1,IOUT1,3)=S(IG,IN,3)\*RATIO  
S(1,IOUT2,3)=S(IG,IN,3)-S(1,IOUT1,3)  
DO 1 I=4,NC5  
S(1,IOUT1,I)=S(IG,IN,I)  
1 S(1,IOUT2,I)=S(IG,IN,I)  
RETURN  
END

SUBROUTINE TYPE1

SUBROUTINE VALV1

V=PORT (PARABOLIC) CONTROL VALVE

EQUIPMENT PARAMETERS

1 - NOT USED

2 - VALVE CONSTANT

3 - ACTION (+=DIRECT, -=REVERSE)

COMMON /UNIT/ IM,NMP

COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)

COMMON /CON/ IG,NGCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH

MI=MP(IM,3)

MO=IABS(MP(IM,4))

CHECK THAT THE VALVE SIGNAL IS IN THE RANGE 0-100

V=S(1,MI,3)

IF (V.LT.0.) V=0.

IF (V.GT.100.) V=100.

ACTION

A=EP(IM,3)

IF (A.LT.0.) S(1,MO,3)=EP(IM,2)\*(100.-V)\*\*2

IF (A.GE.0.) S(1,MO,3)=EP(IM,2)\*V\*\*2

RETURN

END

T01	1
T01	2
T01	3
T01	4
T01	5
T01	6
T01	7
T01	8
T01	9
T01	10
T01	11
T01	12
T01	13
T01	14
T01	15
T01	16
T01	17
T01	18
T01	19
T01	20
T01	21
T01	22
T01	23
T01	24
T01	25
T01	26
T01	27
T01	28



```

NCTR = NCTR+1
RANGE = EP(IM,2)-EP(IM,3)
IF (OUT.LT.0) GO TO 16
SETP = EP(IM,4)*50./RANGE
OLDERR = (S(2,IN,K)*50./RANGE)-SETP
GO TO 18
16 IN2 = -OUT
   OUT = -MP(IM,5)
   IF (S(1,IN2,2).NE.11.) GO TO 15
   IN = IN2
   IN2 = MP(IM,3)
15 SETP = S(1,IN2,3)
   OLDERR = (S(2,IN,K)*50./RANGE)-S(2,IN2,3)
18 CONTINUE
   S(1,IN,K) = S(2,IN,K)
30 ERR = (S(1,IN,K)*50./RANGE)-SETP
   PD = 0.
   PI = 0.
   GO TO (25,26,27,28).N
27 PD = EP(IM,9)*(ERR-OLDERR)/H
   GO TO 25
28 PD = EP(IM,9)*(ERR-OLDERR)/H
26 SIGMA(NCTR) = SIGMA(NCTR)+(H*(ERR+OLDERR)/2.)
   PI = SIGMA(NCTR)/EP(IM,8)
25 S(1,OUT,3) = EP(IM,6)+(EP(IM,7)*(ERR+PI+PD))
   RETURN
200 CONTINUE
   IF(IG.EQ.1) RETURN
   T=S(1,IN,K)
   IF(N.EQ.6) GO TO 201
   IF(T.GE.EP(IM,4)) T=EP(IM,6)
   IF(T.LT.EP(IM,4)) T=0.0
   GO TO 202
201 CONTINUE
   IF(T.LE.EP(IM,4)) T=EP(IM,6)
   IF(T.GT.EP(IM,4)) T=0.0
202 S(1,OUT,3)=T
   RETURN
   END

```

\*\*\*\*\*

SIMULATION OF WILLIAMS-OTTO PLANT

\*\*\*\*\*

BEGIN					
TIME	1.0				
COMPS	6.0				
HMIN	0.00000001				
IN/OUT	10.0				
PRINTING	50.0				
ORDER	2.0				
LIBRARY	4.0				
RDRM2	21.0				
REBO2	22.0				
COLM1	27.0				
COND1	17.0				
PROCESS					
CONT1	1.0				
	6.0	-7.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	3.0	4872.0	4408.0	4640.0	2.0
	50.0	5.0	1.0	0.0	0.0
VALV1	2.0				
	7.0	-14.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	38.3844	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	3.0				
	14.0	-8.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	4.0	0.0	0.0	175.0	6.0
	50.0	0.0	0.0	0.0	0.0
VALV1	4.0				
	8.0	-9.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	0.4	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	5.0				
	14.0	-11.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	4.0	181.0	175.0	180.0	1.0
	10.0	100.0	0.0	0.0	0.0
VALV1	6.0				
	11.0	-12.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	51.148618	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	7.0				
	4.0	-5.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0

	3.0	0.50	0.37	0.43478260872.0	
	50.0	0.1	2.0	0.0	0.0
VALV1	8.0				
	5.0	-2.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	13.34	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CSTR1	9.0				
	1.0	2.0	3.0	12.0	9.0
	-14.0	-13.0	-10.0	-6.0	-4.0
	4640.0	106.08	5000.0	5000.0	945.5
	0.0	0.0	0.0	0.0	0.0
CONT1	10.0				
	18.0	-15.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	4.0	101.0	80.0	100.0	1.0
	50.0	1.0	0.0	0.0	0.0
VALV1	11.0				
	15.0	-16.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	61.40	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
EXCH1	12.0				
	14.0	16.0	-18.0	-17.0	0.0
	0.0	0.0	0.0	0.0	0.0
	274.0	830.0	569.0	82.5	1.0
	0.0	0.0	0.0	0.0	0.0
CONT1	13.0				
	22.0	-23.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	3.0	10481.331	9483.109	9982.22	2.0
	50.0	2.0	1.0	0.0	0.0
VALV1	14.0				
	23.0	-24.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	36.90	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	15.0				
	19.0	-20.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	3.0	3241.35	2932.65	3087.0	2.0
	50.0	2.0	1.0	0.0	0.0
VALV1	16.0				
	20.0	-21.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	1.4848	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
DCAN1	17.0				
	18.0	-24.0	-21.0	-22.0	-19.0
	0.0	0.0	0.0	0.0	0.0
	6895.22	3087.0	5.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0

CONV1	18.0				
	24.0	-25.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	1.0	0.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
REB02	19.0				
	39.0	38.0	-40.0	0.0	0.0
	0.0	0.0	0.0	0.0	1.0
	7682500.0	15750.0	120.0	0.5	13.0
	14.357	-3818.33	227.21	0.0	0.0
RDRM2	20.0				
	35.0	40.0	-38.0	-34.0	-36.0
	0.0	0.0	0.0	0.0	0.0
	70.0	0.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
COLM1	21.0				
	25.0	30.0	34.0	-35.0	-26.0
	0.0	-28.0	0.0	0.0	0.0
	10.0	5.0	0.0	8.0	0.1
	1.0	9.0	1.0	0.0	0.0
COND1	22.0				
	26.0	-27.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	2.0	0.0	0.0	0.0
STIR1	23.0				
	27.0	-30.0	-31.0	-33.0	0.0
	0.0	0.0	0.0	0.0	0.0
	40.0	0.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	24.0				
	31.0	-32.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	3.0	65.0	35.0	50.0	4.0
	50.0	1.5	0.5	0.1	0.0
VALV1	25.0				
	32.0	-33.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	0.01342	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	26.0				
	28.0	-29.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	4.0	110.0	80.0	95.0	2.0
	40.0	1.5	0.5	0.1	0.0
VALV1	27.0				
	29.0	-30.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	0.24644	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONT1	28.0				
	36.0	-37.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0



	3.0	100.0	60.0	80.0	4.0
	50.0	1.5	0.5	0.1	0.0
VALV1	29.0				
	37.0	-38.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0	0.2644	1.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
CONVI	30.0				
	38.0	-41.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	2.0	0.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
SPLT1	31.0				
	41.0	-3.0	-42.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.5	992170270.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
END STREAMS EXPLICIT	42.0				
	1.0	1.0	14500.0	70.0	44.1
	1.0	0.0	0.0	0.0	0.0
	0.0				
	2.0	1.0	33350.0	70.0	44.1
	0.0	1.0	0.0	0.0	0.0
	0.0				
	3.0	1.0	48111.0	100.0	44.1
	.1329111754	.4209654006	.0268382730	.3811766320	0.0
	.0381085190				
	4.0	11.0	0.43478	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0				
	5.0	11.0	50.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0				
	6.0	11.0	4640.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0				
	7.0	11.0	50.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0				
	8.0	11.0	50.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0
	0.0				
	9.0	1.0	951.877	250.0	29.825
	0.0	0.0	0.0	0.0	0.0
	0.0				
	10.0	1.0	951.877	250.0	29.825
	0.0	0.0	0.0	0.0	0.0
	0.0				
	11.0	11.0	0.0	0.0	0.0
	0.0	0.0	0.0	0.0	0.0

0.0				
12.0	1.0	0.0	60.0	14.7
0.0	0.0	0.0	0.0	0.0
0.0				
13.0	1.0	0.0	62.0	14.7
0.0	0.0	0.0	0.0	0.0
0.0				
14.0	1.0	95961.0	70.0	44.1
0.30303	0.69697	0.0	0.0	0.0
0.0				
15.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
16.0	1.0	153500.0	60.0	14.7
0.0	0.0	0.0	0.0	0.0
0.0				
17.0	1.0	153500.0	61.0	14.7
0.0	0.0	0.0	0.0	0.0
0.0				
18.0	1.0	95961.0	62.0	44.1
0.30303	0.69697	0.0	0.0	0.
0.0				
19.0	11.0	3087.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
20.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
21.0	1.0	3712.0	62.0	44.1
0.0	0.0	0.0	0.0	1.0
0.0				
22.0	11.0	9982.22	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
23.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
24.0	1.0	92249.0	62.0	44.1
0.30303	0.69697	0.0	0.0	0.
0.0				
25.0	1.0	649.0	100.0	44.1
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
26.0	1.0	649.0	94.0	270.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
27.0	1.0	649.0	89.0	242.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
28.0	11.0	0.0	80.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				

29.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
30.0	1.0	649.0	89.0	242.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
31.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
32.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
33.0	1.0	33.0	89.0	242.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
34.0	1.0	649.0	121.0	350.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
35.0	1.0	1265.0	121.0	350.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
36.0	11.0	80.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
37.0	11.0	50.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
38.0	1.0	616.0	121.0	350.0
0.1261	0.3992	0.0255	0.3615	0.0
0.0877				
39.0	9.0	487.0	344.35	125.0
0.0	0.0	0.0	0.0	0.0
0.0				
40.0	9.0	7682500.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0				
41.0	1.0	87487.0	100.0	44.1
.1329111754	.4209654006	.0268382730	.3811766320	0.0
.0381085190				
42.0	1.0	39376.0	100.0	44.1
.1329111754	.4209654006	.0268382730	.3811766320	0.0
.0381085190				
END				
PROPERTIES				
20.0				
100.0	60.0	0.0	0.0	0.0
42.24	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
100.0	60.0	0.0	0.0	0.0
42.24	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0

200.0	60.0	0.0	0.0	0.0
62.24	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
200.0	60.0	0.0	0.0	0.0
62.24	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
300.0	60.0	0.0	0.0	0.0
62.24	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
100.0	60.0	0.0	0.0	0.0
62.933147	-123916.59	2096.2	1.0	14250.0
60.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0

END

END

## APPENDIX I: PROGRAM LISTINGS

A listing is given in Figure 1.1 of the DYNYSYS 2.0 executive program except for SUBROUTINES TRGB and TRGB2. These routines were made available to the author for research purposes, but are considered proprietary by their creators. Copies of these may be obtained from:

MR. H.P. HUTCHISON  
LECTURER  
DEPT. OF CHEMICAL ENGINEERING  
CAMBRIDGE UNIVERSITY  
CAMBRIDGE, ENGLAND

A fee will probably be required from commercial users.

### I.1 The DYNYSYS 2.1 Executive Program

Since version 2.0 is not completely available, another version of DYNYSYS, version 2.1, was created to have a sparse matrix oriented package which was completely available to the public. Key's subroutine SIMULT was used in place of TRGB. Version 2.1 is not as efficient as 2.0 since Key's program is not designed for repeated solution of linear equations. The new version, however, does illustrate the usage of DYNYSYS and can be keypunched and

used by anyone.

There are some slight differences in the writings of the modules. A skeleton module is given (Figure 1.2) and also a listing of the executive and a sample module (Figure 1.3) which corresponds to the second example of Chapter 8 (Figure 8.7), a CSTR with first order reversible reaction. The only executive subroutines given in Figure 1.3 are DYN1, DYN2, DIFSUB, SIMULT and RITE. These replace subroutines DYN1, DYN2, DIFSUB, TRGB and TRGB2 in DYN SYS 2.0. Subroutines GET, FETCH, PROPS, TEMP, TRIDAG and GRAPH are identical with those in DYN SYS 2.0. The mainline program and subroutine OUTPUT are identical with DYN SYS 2.0 except for the COMMON/GEAR2/.

The differences in writing of modules are as follows. The Jacobian is stored as described in Section 5.1.2.

COMMON/ROW/IROW(4) is not required.

COMMON/COLUMN/JCOL(4) and COMMON/JACOB/XJACOB(4) are replaced by

COMMON/COLUMN/JCOL(10,2) and COMMON/JACOB(10,2)

i.e. the column numbers and nonzero elements are now stored as two dimensional arrays. Note, however, that the row dimension of JCOL and XJACOB must be the same as

in DIFSUB.

COMMON/MODULE/IDERY,ITER,ITR1,NZERO is replaced by

COMMON/MODULE/IDERY,ITER,ITR1,MC,IPIVOT

NZERO is no longer required, but the user must supply, in SECTION #2 of the skeleton, values of MC and IPIVOT. MC is the maximum number of columns in the XJACOB matrix where the nonzero Jacobian elements are stored. This value can become larger than the maximum number of nonzeros in any Jacobian row since additional nonzero elements can be created during the solution. MC should be constant for any module and may have to be determined by trial and error. It is also a function of the pivot option. If MC is too small, the program will stop and print the message

MC SHOULD BE AT LEAST \_\_\_\_\_

MC should then be set to at least the number in the blank.

IPIVOT is the pivot option (from 1-7) used for the module. The options are described in the skeleton and in section 5.1.2.

The remainder of the module is written exactly as for DYNYSYS 2.0. The only change in the data set is that MINPIVOT is not used.

The difference in execution time between version 2.0 and 2.1 will depend on the size of the system. For very small examples the times will be nearly the same. For large systems, Chapter 5 gives execution times for SIMULT and TRGB-TRGB2 for 50, 100 and 200 equations. For example for 50 equations, bandwidth 9, the best option of Key's method took 0.691 seconds while TRGB2 took 0.049 seconds. Since the linear equations must be solved at each Newton-Raphson iteration of the corrector, the time difference could become substantial for a long simulation with many time steps. For 500 time steps with an average of two Newton-Raphson iterations per step, the execution time would differ by over a minute.



C  
C  
C  
C

FIGURE 1.1: LISTING OF DYNYSYS 2.0 EXECUTIVE

	PROGRAM DYNYSYS(INPUT,OUTPUT,PUNCH,TAPE5=INPUT,TAPE6=OUTPUT,TAPE7=PDYN	DYN	1
	1 UNCH)	DYN	2
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	DYN	3
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DYN	4
	COMMON /PLT/ NPLOTS,PLOTI,PLOTS(20,4),PLOTT	DYN	5
	COMMON /LAG/ NSX,NSW	DYN	6
	COMMON /FTN/ FN(1,1),NFN	DYN	7
	COMMON /GEAR1/ Y(8,222),SAVE(10,222),ERROR(222),YMAX(222),DERY1(222)	DYN	8
	12),DERY2(222)	DYN	9
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	DYN	10
	COMMON /G/ NT,NV,TPLOT(602),V(602),VPLOT(1,602)	DYN	11
1	CALL DYN1	DYN	12
2	PLOTT=0.0	DYN	13
	DO 3 I=1,222	DYN	14
	DERY2(I)=0.0	DYN	15
	YMAX(I)=1.0	DYN	16
3	CONTINUE	DYN	17
	JSTART=0	DYN	18
	NBVMAX=0	DYN	19
	OTIME=TIME	DYN	20
	NSW=0	DYN	21
	IF (NPOL.GT.0) GO TO 4	DYN	22
	GO TO 5	DYN	23
4	CONTINUE	DYN	24
	READ (5,25) X	DYN	25
	NPLOTS=X	DYN	26
	READ (5,25) PLOTI	DYN	27
	READ (5,25) ((PLOTS(I,J),J=1,4),I=1,NPLOTS)	DYN	28
	PRINT 25, ((PLOTS(I,J),J=1,4),I=1,NPLOTS)	DYN	29
5	CONTINUE	DYN	30
C	SET UP INITIAL VALUES FOR GRAPH VECTOR COUNT	DYN	31
	NV=0	DYN	32
	NT=0	DYN	33
	CALL OUTPUT	DYN	34
	KPRINT=0	DYN	35
6	CONTINUE	DYN	36
	TIME=OTIME+H	DYN	37
	KFLAG=1	DYN	38
	ICONV=0	DYN	39
	KPRINT=KPRINT+1	DYN	40
	IG=2	DYN	41
	CALL DYN2	DYN	42
	IG=1	DYN	43
	CALL DYN2	DYN	44
	IF (NPOL.GT.0) GO TO 7	DYN	45
	IF (KPRINT.LT.NPR) GO TO 8	DYN	46
7	CONTINUE	DYN	47
	CALL OUTPUT	DYN	48

8	KPRINT=0	DYN 49
	CONTINUE	DYN 50
	NSW=1	DYN 51
	IF (ICONV.EQ.1.OR.KFLAG.NE.1) GO TO 6	DYN 52
	NSW=0	DYN 53
	OTIME=TIME	DYN 54
C		DYN 55
C	UPDATE STREAM MATRIX	DYN 56
C		DYN 57
	DO 10 I=1,NS	DYN 58
	DO 9 J=3,NC5	DYN 59
	S(2,I,J)=S(1,I,J)	DYN 60
9	CONTINUE	DYN 61
10	CONTINUE	DYN 62
	IF (TIME.LE.TMAX) GO TO 6	DYN 63
	WRITE (6,24)	DYN 64
	CALL OUTPUT	DYN 65
	IF (NGRAPH.NE.1) GO TO 11	DYN 66
	CALL GRAPH (NPLOTS)	DYN 67
11	CONTINUE	DYN 68
	CALL GET (N,X,0)	DYN 69
	IF (N.EQ.3HEND) GO TO 19	DYN 70
	IF (N.EQ.6HREPEAT) GO TO 1	DYN 71
	IF (N.EQ.8HCONTINUE) GO TO 12	DYN 72
	PRINT 20	DYN 73
	STOP	DYN 74
12	H=0.00001	DYN 75
13	CALL GET (N,X,1)	DYN 76
	IF (N.EQ.4HTIME) TMAX=X	DYN 77
	IF (N.EQ.8HLINEPLOT) NPOL=X	DYN 78
	IF (N.EQ.5HGRAPH) NGRAPH=X	DYN 79
	IF (N.EQ.6HALTERS) GO TO 14	DYN 80
	IF (N.EQ.6HALTERE) GO TO 16	DYN 81
	IF (N.EQ.8HFUNCTION) GO TO 17	DYN 82
	IF (N.EQ.3HEND) GO TO 2	DYN 83
	GO TO 13	DYN 84
14	NOS=X	DYN 85
	READ 21, (S(1,NOS,J),J=1,5)	DYN 86
	PRINT 23, (S(1,NOS,J),J=1,5)	DYN 87
	READ 22, (S(1,NOS,J),J=6,NC5)	DYN 88
	PRINT 23, (S(1,NOS,J),J=1,NC5)	DYN 89
	DO 15 J=1,NC5	DYN 90
	S(2,NOS,J)=S(1,NOS,J)	DYN 91
15	CONTINUE	DYN 92
	GO TO 13	DYN 93
16	NOEP=X	DYN 94
	READ 21, (EP(NOEP,J),J=1,5)	DYN 95
	PRINT 23, (EP(NOEP,J),J=1,5)	DYN 96
	GO TO 13	DYN 97
17	NFN=X	DYN 98
	DO 18 I=1,NFN	DYN 99
	READ 21, (FN(I,J),J=1,5)	DYN 100

```
18 READ 22, (FN(I,J),J=6,NC5)
CONTINUE
PRINT 23, ((FN(I,J),J=1,NC5),I=1,NFN)
GO TO 13
19 STOP
C
20 FORMAT (41H LAST CARD MUST BE END,REPEAT OR CONTINUE)
21 FORMAT (12X,5F12.2)
22 FORMAT (12X,4F12.2)
23 FORMAT (1X,9F10.3)
24 FORMAT (720X,23H ***TMAX APPROACHED*** )
25 FORMAT (4F12.4)
END
```

```
DYN 101
DYN 102
DYN 103
DYN 104
DYN 105
DYN 106
DYN 107
DYN 108
DYN 109
DYN 110
DYN 111
DYN 112
DYN 113
```

	SUBROUTINE DYN1	DY1	1
	DATA LOADING AND PRINTING	DY1	2
C	COMMON /UNIT/ IM,NMP	DY1	3
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	DY1	4
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DY1	5
	COMMON /FTN/ FN(1,1),NFN	DY1	6
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	DY1	7
	COMMON /GEAR3/ HMIN,HMAX,XMIN,NOMES	DY1	8
	DIMENSION AM(100)	DY1	9
	DIMENSION ITAG(30)	DY1	10
	DATA ITAG,5HVALV1,5HVALV2,5HSTIR1,5HCONT1,5HSETL1,6HMXPLT1,5HSETT1	DY1	11
	1,6HVESL1,5HDELAY,5HCSTR1,5HEXCH1,5HDCAN1,5HCONV1,5HSPLT1,16*6H	DY1	12
	2 /	DY1	13
	WRITE (6,27)	DY1	14
C	DEFAULT VALUES	DY1	15
	TIME=0.0	DY1	16
	MAXNE=35	DY1	17
	NPR=1	DY1	18
	TMAX=10.0	DY1	19
	NCOMP=1	DY1	20
	NMP=5	DY1	21
	H=0.00001	DY1	22
	HMIN=0.000001	DY1	23
	HMAX=0.05	DY1	24
	EPS=0.001	DY1	25
	NPOL=0	DY1	26
	NGRAPH=0	DY1	27
	IORDER=6	DY1	28
	ISTIFF=1	DY1	29
	NOMES=0	DY1	30
	XMIN=1.0E-06	DY1	31
C	READ AND COPY TITLE	DY1	32
1	READ (5,29) (AM(I),I=1,8)	DY1	33
	WRITE (6,28) (AM(I),I=1,8)	DY1	34
	IF (AM(1).NE.5HBEGIN) GO TO 1	DY1	35
C	READ INITIAL DATA	DY1	36
2	CALL GET (N,X,1)	DY1	37
	IF (N.EQ.5HCOMPS) NCOMP=X	DY1	38
	IF (N.EQ.6HIN/OUT) NMP=X	DY1	39
	IF (N.EQ.6HDELTAT) H=X	DY1	40
	IF (N.EQ.4HHMIN) HMIN=X	DY1	41
	IF (N.EQ.4HHMAX) HMAX=X	DY1	42
	IF (N.EQ.4HTIME) TMAX=X	DY1	43
	IF (N.EQ.8HPRINTING) NPR=X	DY1	44
	IF (N.EQ.9HTOLERANCE) EPS=X	DY1	45
	IF (N.EQ.5HORDER) IORDER=X	DY1	46
	IF (N.EQ.8HLINEPLOT) NPOL=1	DY1	47
	IF (N.EQ.5HGRAPH) NGRAPH=1	DY1	48
	IF (N.EQ.8HNONSTIFF) ISTIFF=0	DY1	49
	IF (N.EQ.9HNOMESSAGE) NOMES=1	DY1	50
	IF (N.EQ.8HMINPIVOT) XMIN=X	DY1	51
	IF (N.EQ.7HLIBRARY) GO TO 3	DY1	52

	IF (N.EQ.7HPROCESS) GO TO 6.	DY1	53
	IF (N.EQ.8HFUNCTION) GO TO 5.	DY1	54
	GO TO 2	DY1	55
3	NLIB=X	DY1	56
	DO 4 I=1,NLIB	DY1	57
	CALL GET (N,X,0)	DY1	58
	J=X	DY1	59
	ITAG(J)=N	DY1	60
4	CONTINUE	DY1	61
	GO TO 2	DY1	62
5	NFN=X	DY1	63
	NC5=NCOMP+5	DY1	64
	READ 39, ((FN(I,J),J=1,NC5),I=1,NFN)	DY1	65
	PRINT 39, ((FN(I,J),J=1,NC5),I=1,NFN)	DY1	66
	GO TO 2	DY1	67
6	CONTINUE	DY1	68
	NMP=NMP+2	DY1	69
	MAXNMP=NMP+1	DY1	70
	NC5=NCOMP+5	DY1	71
	WRITE (6,30)	DY1	72
	DO 9 I=1,MAXNE	DY1	73
	DO 7 J=1,MAXNMP	DY1	74
	MP(I,J)=0	DY1	75
7	CONTINUE	DY1	76
	DO 8 J=1,10	DY1	77
	EP(I,J)=0.0	DY1	78
8	CONTINUE	DY1	79
9	CONTINUE	DY1	80
	NE=0	DY1	81
	NEX=1	DY1	82
	TIME=0.0	DY1	83
10	CALL GET (NM,X,0)	DY1	84
11	IF (NM.EQ.3HEND) GO TO 16	DY1	85
	DO 12 I=1,30	DY1	86
	IF (NM.EQ.ITAG(I)) GO TO 13	DY1	87
12	CONTINUE	DY1	88
	WRITE (6,31) NM	DY1	89
	I=0	DY1	90
13	NT=I	DY1	91
	NE=NE+1	DY1	92
	READ (5,32) (AM(I),I=3,NMP)	DY1	93
	DO 14 J=3,NMP	DY1	94
	MP(NE,J)=AM(J)	DY1	95
14	CONTINUE	DY1	96
	MP(NE,1)=X	DY1	97
	MP(NE,2)=NT	DY1	98
	WRITE (6,33) (MP(NE,I),I=1,NMP)	DY1	99
	READ (5,32) (EP(NE,I),I=1,10)	DY1	100
	WRITE (6,34) (EP(NE,I),I=1,10)	DY1	101
	CALL GET (NM,X,0)	DY1	102
	IF (NM.NE.5HEXTRA) GO TO 15	DY1	103
	NNX=X	DY1	104

	NMX=NNX+NEX-1	DY1 105
	READ (5,32) (EX(J),J=NEX,NMX)	DY1 106
	WRITE (6,34) (EX(J),J=NEX,NMX)	DY1 107
	MP(NE,NMP+1)=NEX	DY1 108
	NEX=NEX+NNX	DY1 109
	CALL GET (NM,X,1)	DY1 110
15	IF (NM,NE,7HRESERVE) GO TO 11	DY1 111
	NNX=X	DY1 112
	NEX=NEX+NNX	DY1 113
	GO TO 10	DY1 114
16	CALL FETCH (7HSTREAMS,NS)	DY1 115
	WRITE (6,35) NS	DY1 116
	DO 18 I=1,NS	DY1 117
	S(1,I,1)=I	DY1 118
	S(1,I,2)=-1.0	DY1 119
	S(1,I,3)=0.0	DY1 120
	S(1,I,4)=530.0	DY1 121
	S(1,I,5)=14.7	DY1 122
	DO 17 J=6,NC5	DY1 123
	S(1,I,J)=0	DY1 124
17	CONTINUE	DY1 125
18	CONTINUE	DY1 126
19	CALL GET (NM,X,-1)	DY1 127
	IF (NM,EQ,8HEXPPLICIT) GO TO 20	DY1 128
	IF (NM,EQ,3HEND) GO TO 25	DY1 129
	IF (NM,EQ,4HSPEC) GO TO 22	DY1 130
	GO TO 19	DY1 131
20	READ (5,36) IT,(AM(I),I=1,5)	DY1 132
	IF (IT,EQ,3HEND) GO TO 25	DY1 133
	READ (5,32) (AM(I),I=6,NC5)	DY1 134
	N=AM(1)	DY1 135
	DO 21 I=1,NC5	DY1 136
	S(1,N,I)=AM(I)	DY1 137
21	CONTINUE	DY1 138
	GO TO 20	DY1 139
22	N1=X	DY1 140
	READ (5,32) (AM(I),I=1,NC5)	DY1 141
	N2=AM(1)	DY1 142
	DO 24 I=N1,N2	DY1 143
	DO 23 J=2,NC5	DY1 144
	S(1,I,J)=AM(J)	DY1 145
23	CONTINUE	DY1 146
	S(1,I,1)=I	DY1 147
24	CONTINUE	DY1 148
	GO TO 19	DY1 149
25	DO 26 I=1,NS	DY1 150
	K=S(1,I,2)	DY1 151
	WRITE (6,38) I,K,(S(1,I,J),J=3,5)	DY1 152
	WRITE (6,37) (S(1,I,J),J=6,NC5)	DY1 153
	DO 26 J=1,NC5	DY1 154
26	S(2,I,J)=S(1,I,J)	DY1 155
	XX=PROPS(0,0)	DY1 156

RETURN

```
27 FORMAT (1H1)
28 FORMAT (1X,8A10)
29 FORMAT (8A10)
30 FORMAT (1H1)
31 FORMAT (18H ***FAIL*** NAME ,A12,8H NOT SET)
32 FORMAT (12X,5F12.0)
33 FORMAT (/,9H UNIT -,I3,9H- TYPE -,I3,1H-,12I5)
34 FORMAT (11X,5F14.5)
35 FORMAT (1H1,I6,8H STREAMS)
36 FORMAT (A3,9X,5F12.0)
37 FORMAT (8X,4F14.5)
38 FORMAT (11H STREAM -,I3,4H- (,I3,1H),3F14.5)
39 FORMAT (12X,5F12.2)
END
```

```
DY1 157
DY1 158
DY1 159
DY1 160
DY1 161
DY1 162
DY1 163
DY1 164
DY1 165
DY1 166
DY1 167
DY1 168
DY1 169
DY1 170
DY1 171
DY1 172
```

```

SURROUTINE GET (NAME,X,IFG)
1  READ (5,6) NAME,X
   IF (NAME.NE.10H          ) GO TO 2
   IF (X.EQ.0.OR.X.EQ.-0) GO TO 1
   WRITE (6,7)
   GO TO 1
2  CONTINUE
   IF (X.LE.0) GO TO 3
   IF (IFG) 3,4,5
3  WRITE (6,9) NAME
   RETURN
4  N=X
   WRITE (6,8) NAME,N
   RETURN
5  WRITE (6,9) NAME,X
   RETURN
C
6  FORMAT (A10,F14.0)
7  FORMAT (48H ---WARN--- DATA SKIPPED WHILE READING KEYWORDS)
8  FORMAT (1X,A10,I12)
9  FORMAT (1X,A10,F14.5)
   END

```

```

GET 1
GET 2
GET 3
GET 4
GET 5
GET 6
GET 7
GET 8
GET 9
GET 10
GET 11
GET 12
GET 13
GET 14
GET 15
GET 16
GET 17
GET 18
GET 19
GET 20
GET 21
GET 22

```



```
1  SURROUTINE FETCH (IWORD,I)
   READ (5,2) IWORDA,AI
   IF (IWORDA.NE.IWORD) GO TO 1 .
   I=AI
   RETURN
C
2  FORMAT (A10,F14.2)
   END
```

```
FET  1
FET  2
FET  3
FET  4
FET  5
FET  6
FET  7
FET  8-
```

	SUBROUTINE DYN2	DY2	1
	COMMON /UNIT/ IM,NMP	DY2	2
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	DY2	3
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DY2	4
	COMMON /BKV/ NBV	DY2	5
	COMMON /CNTR/ NCTR	DY2	6
	COMMON /LAG/ NSX,NSW	DY2	7
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	DY2	8
	NBV=0	DY2	9
	NSX=0	DY2	10
	NOPPT=0	DY2	11
	NCTR=0	DY2	12
	DO 31 IM=1,NE	DY2	13
	IF (ICONV.EQ.1) RETURN	DY2	14
	NTYPE=MP(IM,2)	DY2	15
	GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,22,23,24,25,26,27,28,29,30), NTYPE	DY2	16
1	CALL TYPE1	DY2	17
	GO TO 31	DY2	18
2	CALL TYPE2	DY2	19
	GO TO 31	DY2	20
3	CALL TYPE3	DY2	21
	GO TO 31	DY2	22
4	CALL TYPE4	DY2	23
	GO TO 31	DY2	24
5	CALL TYPE5	DY2	25
	GO TO 31	DY2	26
6	CALL TYPE6	DY2	27
	GO TO 31	DY2	28
7	CALL TYPE7	DY2	29
	GO TO 31	DY2	30
8	CALL TYPE8	DY2	31
	GO TO 31	DY2	32
9	CALL TYPE9	DY2	33
	GO TO 31	DY2	34
10	CALL TYPE10	DY2	35
	GO TO 31	DY2	36
11	CALL TYPE11	DY2	37
	GO TO 31	DY2	38
12	CALL TYPE12	DY2	39
	GO TO 31	DY2	40
13	CALL TYPE13	DY2	41
	GO TO 31	DY2	42
14	CALL TYPE14	DY2	43
	GO TO 31	DY2	44
15	CALL TYPE15	DY2	45
	GO TO 31	DY2	46
16	CALL TYPE16	DY2	47
	GO TO 31	DY2	48
17	CALL TYPE17	DY2	49
	GO TO 31	DY2	50
18	CALL TYPE18	DY2	51
		DY2	52

GO TO 31  
19 CALL TYPE19  
GO TO 31  
20 CALL TYPE20  
GO TO 31  
21 CALL TYPE21  
GO TO 31  
22 CALL TYPE22  
GO TO 31  
23 CALL TYPE23  
GO TO 31  
24 CALL TYPE24  
GO TO 31  
25 CALL TYPE25  
GO TO 31  
26 CALL TYPE26  
GO TO 31  
27 CALL TYPE27  
GO TO 31  
28 CALL TYPE28  
GO TO 31  
29 CALL TYPE29  
GO TO 31  
30 CALL TYPE30  
31 CONTINUE  
RETURN  
END

DY2 53  
DY2 54  
DY2 55  
DY2 56  
DY2 57  
DY2 58  
DY2 59  
DY2 60  
DY2 61  
DY2 62  
DY2 63  
DY2 64  
DY2 65  
DY2 66  
DY2 67  
DY2 68  
DY2 69  
DY2 70  
DY2 71  
DY2 72  
DY2 73  
DY2 74  
DY2 75  
DY2 76  
DY2 77  
DY2 78  
DY2 79

	SURROUTINE OUTPUT	OUT	1
C	THIS ROUTINE PRINTS OUT THE RESULTS	OUT	2
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	OUT	3
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	OUT	4
	COMMON /PLT/ NPLOTS,PLOTI,PLOTS(20,4),PLOTT	OUT	5
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	OUT	6
	COMMON /FTN/ FN(1,1),NFN	OUT	7
	COMMON /FF/ TITLE(11),MPLOT(20,2),PLOT(20,2)	OUT	8
	COMMON /G/ NT,NV,TPLOT(602),V(602),VPLOT(1,602)	OUT	9
	DIMENSION SYMBOL(15),SYMBOL1(4),ALINE(125)	OUT	10
	DATA SYMBOL/1HA,1HB,1HC,1HD,1HE,1HF,1HG,1HH,1HI,1HJ,1HK,1HL,1HM,1HOUT	OUT	11
	1N,1HO/	OUT	12
	DATA SYMBOL1/1H,,1H-,1H ,1H,/	OUT	13
	DATA TITLE/4HSTRM,4HFLAG,4HFLOW,4HTEMP,4HPRES,5HCOMP1,5HCOMP2,5HCO	OUT	14
	1MP3,5HCOMP4,5HCOMP5,5HCOMP6/	OUT	15
	IF (NPOL.GT.0) GO TO 3	OUT	16
	WRITE (6,14) TIME	OUT	17
	WRITE (6,15)	OUT	18
	DO 1 I=1,NS	OUT	19
	IF (S(2,I,2).LT.0.0) GO TO 1	OUT	20
	NN=S(2,I,1)	OUT	21
	WRITE (6,17) NN,(S(1,I,J),J=3,NC5)	OUT	22
1	CONTINUE	OUT	23
	DO 2 I=1,NE	OUT	24
	IF (MP(I,1).GT.0) GO TO 2	OUT	25
	K=IABS(MP(I,1))	OUT	26
	WRITE (6,16) K,(EP(I,J),J=1,5)	OUT	27
2	CONTINUE	OUT	28
	RETURN	OUT	29
3	CONTINUE	OUT	30
	IF (JSTART.NE.0) GO TO 6	OUT	31
C	START PLOTTING RESULTS	OUT	32
	WRITE (6,19)	OUT	33
	DO 4 I=1,NPLOTS	OUT	34
	N1=PLOTS(I,1)	OUT	35
	N2=PLOTS(I,2)	OUT	36
	MPLOT(I,1)=N1	OUT	37
	MPLOT(I,2)=N2	OUT	38
	PLOT(1,1)=PLOTS(I,3)	OUT	39
	PLOT(1,2)=PLOTS(I,4)	OUT	40
	WRITE (6,20) SYMBOL(I),N1,TITLE(N2),PLOTS(I,3),PLOTS(I,4)	OUT	41
4	CONTINUE	OUT	42
	X=0.	OUT	43
	DO 5 I=1,21	OUT	44
	ALINE(I)=X	OUT	45
	X=X+5.	OUT	46
5	CONTINUE	OUT	47
	WRITE (6,21) (ALINE(I),I=1,21)	OUT	48
	WRITE (6,22)	OUT	49
6	CONTINUE	OUT	50
C	CALCULATE NUMBER OF LINES TO SKIP	OUT	51
	XSPACE=(TIME-PLOTT)/PLOTI	OUT	52

	NSPACE=XSPACE	OUT 53
	XXSPACE=NSPACE	OUT 54
	IF ((XSPACE-XXSPACE).GT.0.5) NSPACE=NSPACE+1	OUT 55
	IF (JSTART.EQ.0) NSPACE=2	OUT 56
	IF (NSPACE.LT.1) RETURN	OUT 57
C	SET UP ALINE	OUT 58
	DO 7 I=2,120	OUT 59
	ALINE(I)=SYMBOL1(3)	OUT 60
7	CONTINUE	OUT 61
	ALINE(1)=SYMBOL1(1)	OUT 62
	ALINE(101)=SYMBOL1(1)	OUT 63
	PLOTT=TIME	OUT 64
	NSPACE=NSPACE-1	OUT 65
	IF (NSPACE.EQ.0) GO TO 9	OUT 66
	DO 8 I=1,NSPACE	OUT 67
	WRITE (6,18) (ALINE(J),J=1,101)	OUT 68
8	CONTINUE	OUT 69
C	SET UP NEW LINE()	OUT 70
9	N5=1	OUT 71
	N6=NPLOTS	OUT 72
	NV=NV+1	OUT 73
	IF (NV.GT.602.OR.NV.LT.1) PRINT 24	OUT 74
	DO 13 I=N5,N6	OUT 75
	N1=PLOTS(I,1)	OUT 76
	N2=PLOTS(I,2)	OUT 77
	XN2=(S(1,N1,N2)-PLOTS(I,3))*100./(PLOTS(I,4)-PLOTS(I,3))	OUT 78
	VPLOT(I,NV)=XN2	OUT 79
	XN3=XN2*J.	OUT 80
	N3=XN3	OUT 81
	XXN3=N3	OUT 82
	IF ((XN3-XXN3).GT.0.5) N3=N3+1	OUT 83
	IF (N3.LT.2.OR.N3.GT.100) GO TO 13	OUT 84
	IF (ALINE(N3).NE.SYMBOL1(3)) GO TO 10	OUT 85
	ALINE(N3)=SYMBOL(I)	OUT 86
	GO TO 13	OUT 87
10	DO 11 J=102,108	OUT 88
	IF (ALINE(J).EQ.SYMBOL1(3)) GO TO 12	OUT 89
11	CONTINUE	OUT 90
12	J=109	OUT 91
	ALINE(J)=ALINE(N3)	OUT 92
	ALINE(J+1)=SYMBOL(I)	OUT 93
	ALINE(J+2)=SYMBOL1(4)	OUT 94
13	CONTINUE	OUT 95
	WRITE (6,23) TIME,SYMBOL1(2),(ALINE(J),J=1,110)	OUT 96
	NT=NT+1	OUT 97
	IF (NT.GT.602.OR.NV.LT.1) PRINT 24	OUT 98
	TPLOT(NT)=PLOTT	OUT 99
	RETURN	OUT 100
C		OUT 101
14	FORMAT (////20X,5(1HS),5X,30HPROCESS VARIABLES AT TIME = ,E11.5,	OUT 102
	15X,5(1HS)/)	OUT 103
15	FORMAT (61H STREAM TOT FLOW TEMP PRESS COMP 1 COMP 2	OUT 104

```
1COMP 3,/)
16  FORMAT (1H0,5X,13,5X,6(3X,F12.5))          OUT 105
17  FORMAT (16,F12.3,2F8.1,9F9.5)             OUT 106
18  FORMAT (19X,101A1)                         OUT 107
19  FORMAT (1H1,20X,6HSYMBOL,11X,6HSTREAM,23X,8HVARIABLE,19X,6HRANGES) OUT 108
20  FORMAT (1H0,22X,A2,13X,I3,28X,A7,3X,2(2X,F12.5)) OUT 109
21  FORMAT (////15X,21(1X,F4.0))              OUT 110
22  FORMAT (14X,5(1H.),21(5HI.....))          OUT 111
23  FORMAT (10X,F6.3,2X,112A1)                 OUT 112
24  FORMAT (1X,25H NT,NV,OR N3 OUT OF RANGE)  OUT 113
      END                                       OUT 114-
                                         OUT 115-
```

	SUBROUTINE GRAPH (NPLOTS)	GRA	1
	COMMON /UNIT/ IM,NMP	GRA	2
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	GRA	3
	COMMON /G/ NT,NV,TPLOT(602),V(602),VPLOT(1,602)	GRA	4
	COMMON /FF/ TITLE(11),MPLOT(20,2),PLOT(20,2)	GRA	5
	V(NT+1)=0.0	GRA	6
	V(NT+2)=12.5	GRA	7
C	AAAA=X,AX IS ACTUAL INCHES	GRA	8
	AAAA=10.	GRA	9
	TPLOT(NT+1)=0.0	GRA	10
	TPLOT(NT+2)=TMAX/AAAA	GRA	11
	CALL PLOTS (60,24.,10.75,2)	GRA	12
	CALL PLOT (0.5,0.50,-3)	GRA	13
C		GRA	14
C	SET UP AXIS	GRA	15
	CALL AXIS3 (0.0,0.0,9HTIME(HRS),-9,AAAA,0.0,TPLOT(NT+1),TPLOT(NT+2),	GRA	16
	1),1,0.5,2)	GRA	17
	CALL AXIS3 (0.0,0.0,6HV-AXIS,6.8.00,90.0,V(NT+1),V(NT+2),1,0.4,2)	GRA	18
C		GRA	19
C	PLOT K=1,NPLOTS CURVES,WITH DIFFERENT SYMBOLS	GRA	20
	DO 2 K=1,NPLOTS	GRA	21
	DO 1 J=1,NT	GRA	22
	V(J)=VPLOT(K,J)	GRA	23
1	CONTINUE	GRA	24
	CALL LINE3 (TPLOT,V,NT,1,2,K,0)	GRA	25
2	CONTINUE	GRA	26
C		GRA	27
C	WRITE HEADING	GRA	28
	XD=11.5	GRA	29
	YD=7.0	GRA	30
	CALL SYMBOL (XD,YD,0.2,7HLEGEND ,0.,7)	GRA	31
	XD=10.5	GRA	32
	YD=YD-0.3	GRA	33
	DO 3 K=1,NPLOTS	GRA	34
C		GRA	35
C	WRITE SYMBOL,PEN UP	GRA	36
	YD=YD+0.05	GRA	37
	CALL SYMBOL (XD,YD,0.1,K,0.0,-1)	GRA	38
C		GRA	39
C	WRITE STREAM AND NUMBER	GRA	40
	YD=YD-0.05	GRA	41
	CALL SYMBOL (XD+0.3,YD,0.1,5HSTRM-,0.,5)	GRA	42
	CALL WHERE (P,Q,F)	GRA	43
	STRMNO=MPLOT(K,1)	GRA	44
	CALL NUMBER (P,Q,0.1,STRMNO,0.0,-1)	GRA	45
	CALL WHERE (P,Q,F)	GRA	46
C		GRA	47
C	WRITE VARIABLE NAMES	GRA	48
	CALL SYMBOL (P+0.3,Q,0.1,TITLE(MPLOT(K,2)),0.0,4)	GRA	49
	CALL WHERE (P,Q,F)	GRA	50
C		GRA	51
C	WRITE RANGES FOR VARIABLES	GRA	52

CALL NUMBER (P,0.3,Q,0.1,PLOTR(K,1),0.0,2)  
CALL WHERE (P,Q,F)  
CALL SYMBOL (P,Q,0.1,3H - ,0.0,3)  
CALL WHERE (P,Q,F)  
CALL NUMBER (P,Q,0.1,PLOTR(K,2),0.0,2)  
YD=YD-0.2  
3 CONTINUE  
CALL ENDPLT  
RETURN  
END

GRA 53  
GRA 54  
GRA 55  
GRA 56  
GRA 57  
GRA 58  
GRA 59  
GRA 60  
GRA 61  
GRA 62



	REAL FUNCTIONPROPS(IG,IS)	PRP	1
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	PRP	2
	COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	PRP	3
	COMMON /PTAB/ IGFLAG,PP(10,20)	PRP	4
C	ENTER HERE ON FIRST CALL	PRP	5
	PROPS=0.0	PRP	6
	IGFLAG=0	PRP	7
	CALL GET (NAME,X,0)	PRP	8
	NPP=X	PRP	9
	IF (NPP) 1,3,5	PRP	10
C	-VE,TREAT AS WATER, NO TEMP COEFF.	PRP	11
1	CONTINUE	PRP	12
	DO 2 I=1,NCOMP	PRP	13
	DO 2 J=1,20	PRP	14
	PP(I,J)=0.	PRP	15
	IF (J.EQ.1) PP(I,J)=18.	PRP	16
	IF (J.EQ.2) PP(I,J)=1.	PRP	17
	IF (J.EQ.16) PP(I,J)=62.4	PRP	18
2	CONTINUE	PRP	19
	WRITE (6,7)	PRP	20
	RETURN	PRP	21
3	IGFLAG=1	PRP	22
C	TREAT AS I.G., MW. AS AIR.	PRP	23
	DO 4 I=1,NCOMP	PRP	24
	DO 4 J=1,20	PRP	25
	PP(I,J)=0.	PRP	26
	IF (J.EQ.1) PP(I,J)=.29	PRP	27
	IF (J.EQ.2) PP(I,J)=.26	PRP	28
4	CONTINUE	PRP	29
	WRITE (6,8)	PRP	30
	RETURN	PRP	31
5	WRITE (6,10) NPP	PRP	32
	DO 6 I=1,NCOMP	PRP	33
	READ (5,9) (PP(I,J),J=1,NPP)	PRP	34
	WRITE (6,11) (PP(I,J),J=1,NPP)	PRP	35
6	CONTINUE	PRP	36
C		PRP	37
	RETURN	PRP	38
C		PRP	39
7	FORMAT (20H PROPERTIES AS WATER)	PRP	40
8	FORMAT (18H PROPERTIES AS AIR)	PRP	41
9	FORMAT (12X,5F12.0)	PRP	42
10	FORMAT (15H PROPERTY TABLE,I4,8H ENTRIES).	PRP	43
11	FORMAT (5F12.5)	PRP	44
	END	PRP	45

```
FUNCTION CP (IG,IS)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /PTAB/ IGFLAG,PP(10,20)
IF (IGFLAG.NE.1) GO TO 1
CP=0.26
RETURN
1 SUM=0.0
T=S(IG,IS,4)*PP(1,20)
DO 2 I=1,NCOMP
SUM=SUM+S(IG,IS,I*5)*(((PP(I,5)*T+PP(I,4))*T+PP(I,3))*T+PP(I,2))
2 CONTINUE
CP=SUM
RETURN
END
```

```
CP 1
CP 2
CP 3
CP 4
CP 5
CP 6
CP 7
CP 8
CP 9
CP 10
CP 11
CP 12
CP 13
CP 14
CP 15
```

```
FUNCTION MW (IG,IS)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /PTAB/ IGFLAG,PP(10,20)
DO 1 I=1,NCOMP
SUM=SUM+PP(I,1)*S(IG,IS,I+5)
CONTINUE
MW=SUM
RETURN
END
```

```
MW 1
MW 2
MW 3
MW 4
MW 5
MW 6
MW 7
MW 8
MW 9
MW 10-
```

	FUNCTION SG (IG,IS)	SG	1
	COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)	SG	2
	COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	SG	3
	COMMON /PTAB/ IGFLAG,PP(10,20)	SG	4
	IF (IGFLAG.NE.1) GO TO 2	SG	5
	SUM=0.	SG	6
	DO 1 I=1,NCOMP	SG	7
	SUM=SUM+PP(I,16)*S(IG,IS,I+5)	SG	8
1	CONTINUE	SG	9
	PROPS=S(IG,IS,5)/(10.71*S(IG,IS,4))	SG	10
	SG=PROPS*SUM	SG	11
	RETURN	SG	12
2	SUM=0.	SG	13
	T=S(IG,IS,4)	SG	14
	DO 3 I=1,NCOMP	SG	15
	SUM=SUM+S(IG,IS,I+5)*(PP(I,17)+PP(I,18)*T)	SG	16
3	CONTINUE	SG	17
	SG=SUM	SG	18
	RETURN	SG	19
	END	SG	20



```

FUNCTION ENTHV (IG,IS)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /PTAB/ IGFLAG,PP(10,20)
T=S(IG,IS,4)*PP(1,20)
HV=0.
DO 1 I=1,NCOMP
HV=HV+S(IG,IS,I+5)*((((PP(I,15)*T+PP(I,14))*T+PP(I,13))*T+PP(I,12)
1))*T+PP(I,11))*T+PP(I,10))
CONTINUE
ENTHV=HV
RETURN
END

```

ETV 1  
ETV 2  
ETV 3  
ETV 4  
ETV 5  
ETV 6  
ETV 7  
ETV 8  
ETV 9  
ETV 10  
ETV 11  
ETV 12  
ETV 13

1

```

FUNCTION EQUIL (IL,IV)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,YMAX,IORDER,NGRAPH
COMMON /PTAB/ IGFLAG,PP(10,20)
COMMON /KVALU/ HKK(10)
C DUMMY STATEMENT
EQUIL=0.0
T=S(IG,IL,4)+PP(1,20)
IF (T.EQ.0.) T=100.
K=0
1 CONTINUE
K=K+1
IF (K.GE.50) PRINT 3, T,SUM,IL,IV
IF (K.GE.50) PRINT 4, (S(IG,IL,J),S(IG,IV,J),J=1,10)
IF (K.GE.50) RETURN
DUM=KVAL(IL,IV)
SUM=0.0
SUMY=0.0
DO 2 NC=6,NC5
N=NC-5
S(IG,IV,NC)=HKK(N)*S(IG,IL,NC)
SUM=SUM+S(IG,IV,NC)
DY=-S(IG,IV,NC)*PP(N,7)/(T+PP(N,8))*2
SUMY=SUMY+DY
2 CONTINUE
YER=1.-SUM
T=T+YER/SUMY
S(IG,IL,4)=T-PP(1,20)
IF (ABS(YER).GT.0.01) GO TO 1
S(IG,IL,4)=T-PP(1,20)
S(IG,IV,4)=S(IG,IL,4)
RETURN
C
3 FORMAT (" ITER,SUMY,IL,IV ",2E15.6,2I5)
4 FORMAT (2E15.5)
END

```

```

EQL 1
EQL 2
EQL 3
EQL 4
EQL 5
EQL 6
EQL 7
EQL 8
EQL 9
EQL 10
EQL 11
EQL 12
EQL 13
EQL 14
EQL 15
EQL 16
EQL 17
EQL 18
EQL 19
EQL 20
EQL 21
EQL 22
EQL 23
EQL 24
EQL 25
EQL 26
EQL 27
EQL 28
EQL 29
EQL 30
EQL 31
EQL 32
EQL 33
EQL 34
EQL 35
EQL 36

```

```
REAL FUNCTION KVAL(IL,IV)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /PTAB/ IGFLAG,PP(10,20)
COMMON /KVALU/ HKK(10)
C DUMMY STATEMENT
KVAL=0.0
T=S(IG,IL,4)*PP(I,20)
P=S(IG,IL,5)
DUM=ACTY(IL)
DO 1 I=1,NCOMP
HKK(I)=EXP(PP(I,6)+PP(I,7)/(T+PP(I,8)))*PP(I,9)/P
CONTINUE
RETURN
END
```

KVL	1
KVL	2
KVL	3
KVL	4
KVL	5
KVL	6
KVL	7
KVL	8
KVL	9
KVL	10
KVL	11
KVL	12
KVL	13
KVL	14
KVL	15-



```
FUNCTION ACTY (IS)
COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)
COMMON /CON/ IU,NCOMP,NC5,H,NE,NS,NPR,NPO,THAX,IORDER,NGRAPH.
COMMON /PTAB/ IGFLAG,PP(10,20)
C DUMMY STATEMENT
  ACTY=0.0
  DO 1 I=1,NCOMP
  PP(I,9)=1.
  1 CONTINUE
  RETURN
  END
```

```
ATY 1
ATY 2
ATY 3
ATY 4
ATY 5
ATY 6
ATY 7
ATY 8
ATY 9
ATY 10
ATY 11
```

FUNCTION TEMPE (Q,MASS,N)

REAL MASS

COMMON /MAT/ MP(35,13),EP(35,10),S(2,45,11),EX(1)

COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH

COMMON /PTAB/ IGFLAG,PP(10,20)

T=S(IG,N,4)\*PP(1,20)

IF (Q.LE.0..AND.MASS.LE.0.) GO TO 3

X1=0.

X2=0.

X3=0.

X4=0.

DO 1 I=1,NCOMP

J=I+5

X1=X1+S(IG,N,J)\*PP(I,2)

X2=X2+S(IG,N,J)\*PP(I,3)/2.

X3=X3+S(IG,N,J)\*PP(I,4)/3.

X4=X4+S(IG,N,J)\*PP(I,5)/4.

CONTINUE

CONTINUE

F=Q-(((X4\*T+X3)\*T+X2)\*T+X1)\*T

DF=-(((4.\*X4\*T+3.\*X3)\*T+2.\*X2)\*T+X1)

T2=T-F/DF

IF (ABS(T-T2).LT..01) GO TO 3

T=T2

GO TO 2

TEMPE=T-PP(1,20)

RETURN

END

TMP	1
TMP	2
TMP	3
TMP	4
TMP	5
TMP	6
TMP	7
TMP	8
TMP	9
TMP	10
TMP	11
TMP	12
TMP	13
TMP	14
TMP	15
TMP	16
TMP	17
TMP	18
TMP	19
TMP	20
TMP	21
TMP	22
TMP	23
TMP	24
TMP	25
TMP	26
TMP	27
TMP	28

1  
2

3

	SUBROUTINE DIFSUB (N,YY,DERY)	DIF	1
	COMMON /GEAR1/ Y(8,222),SAVE(10,222),ERROR(222),YMAX(222),DERY1(222)	DIF	2
	12),DERY2(222)	DIF	3
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,NOPPT,ISTIFF	DIF	4
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DIF	5
	COMMON /GEAR3/ HMIN,HMAX,XMIN,NOMES	DIF	6
	COMMON /BKV/ NBV	DIF	7
	COMMON /ROW/ IROW(1887)	DIF	8
	COMMON /COLUMN/ JCOL(1887)	DIF	9
	COMMON /JACOB/ PW(2109)	DIF	10
	COMMON /SUBDI/ ASUB(49)	DIF	11
	COMMON /DIAG/ BDIAG(49)	DIF	12
	COMMON /SUPERD/ CSUP(49)	DIF	13
	COMMON /MODULE/ IDERY,ITER,ITRI,NZERO	DIF	14
	DIMENSION A(8),PERTST(7,2,3)	DIF	15
	DIMENSION NOP(1000),NW(500),PWSAVE(200)	DIF	16
	DIMENSION NOPS(1000)	DIF	17
	DIMENSION YY(222),DERY(222)	DIF	18
	DATA ITRI,IDERY/2*0/	DIF	19
	DATA PERTST/2.0,4.5,7.333,10.42,13.7,17.15,1.0,2.0,12.0,24.0,37.89	DIF	20
	1,53.33,70.08,87.97,3.0,6.0,9.167,12.5,15.98,1.0,1.0,12.0,24.0,37.89	DIF	21
	29,53.33,70.08,87.97,1.0,1.0,1.0,0.5,0.1667,0.04133,0.008267,1.0,1.0	DIF	22
	30,1.0,2.0,1.0,0.3157,0.07407,0.0139/	DIF	23
	DATA A(2)/-1.0/	DIF	24
	IF (IDERY.EQ.1) GO TO 52	DIF	25
	NBV1=NBV+1	DIF	26
	NBV=NBV*N	DIF	27
	IF (NBVMAX.LT.NBV) NBVMAX=NBV	DIF	28
	DO 1 I=NBV1,NBV	DIF	29
	II=I-NBV1+1	DIF	30
	DERY1(I)=DERY(II)	DIF	31
	CONTINUE	DIF	32
1	IF (IG.EQ.1) GO TO 26	DIF	33
C		DIF	34
C	PREDICTOR SECTION	DIF	35
C		DIF	36
	IRET=1	DIF	37
	IF (JSTART.EQ.0) GO TO 4	DIF	38
2	DO 3 I=NBV1,NBV	DIF	39
	DO 3 J=1,K	DIF	40
3	SAVE(J,I)=Y(J,I)	DIF	41
	HOLD=H	DIF	42
	NOOLD=NO	DIF	43
	RACUM=1.0	DIF	44
	IF (JSTART.GT.0) GO TO 23	DIF	45
	GO TO 6	DIF	46
4	CONTINUE	DIF	47
	NO=1	DIF	48
	N3=N	DIF	49
	N4=N*N	DIF	50
	DO 5 I=NBV1,NBV	DIF	51
	II=I-NBV1+1	DIF	52

	Y(1,I)=YY(II)	DIF 53
	Y(2,I)=DERY1(I)*H	DIF 54
5	CONTINUE	DIF 55
	HNEW=H	DIF 56
	K=2	DIF 57
	GO TO 2	DIF 58
6	IF (ISTIFF.EQ.0) GO TO 7	DIF 59
	IF (NQ.GT.6.OR.NQ.LT.1) GO TO 8	DIF 60
	GO TO (16,17,18,19,20,21), NQ	DIF 61
7	IF (NQ.GT.7.OR.NQ.LT.1) GO TO 8	DIF 62
	GO TO (9,10,11,12,13,14,15), NQ	DIF 63
8	PRINT 84, NQ	DIF 64
	STOP	DIF 65
C		DIF 66
C	NONSTIFF COEFFICIENTS,ORDER 1-7	DIF 67
C		DIF 68
9	A(1)=-1.0	DIF 69
	GO TO 22	DIF 70
10	A(1)=-0.500000000	DIF 71
	A(3)=-0.500000000	DIF 72
	GO TO 22	DIF 73
11	A(1)=-0.416666666666667	DIF 74
	A(3)=-0.750000000	DIF 75
	A(4)=-0.166666666666667	DIF 76
	GO TO 22	DIF 77
12	A(1)=-0.375000000	DIF 78
	A(3)=-0.916666666666667	DIF 79
	A(4)=-0.333333333333333	DIF 80
	A(5)=-0.416666666666667E-01	DIF 81
	GO TO 22	DIF 82
13	A(1)=-0.348611111111111	DIF 83
	A(3)=-1.041666666666667	DIF 84
	A(4)=-0.486111111111111	DIF 85
	A(5)=-0.104166666666667	DIF 86
	A(6)=-0.833333333333333E-02	DIF 87
	GO TO 22	DIF 88
14	A(1)=-0.329861111111111	DIF 89
	A(3)=-1.141666666666667	DIF 90
	A(4)=-0.625000000	DIF 91
	A(5)=-0.177083333333333	DIF 92
	A(6)=-0.025000000	DIF 93
	A(7)=-0.138888888888889E-02	DIF 94
	GO TO 22	DIF 95
15	A(1)=-0.31559193121693	DIF 96
	A(3)=-1.235000000	DIF 97
	A(4)=-0.75185185185	DIF 98
	A(5)=-0.255208333333333	DIF 99
	A(6)=-0.486111111111111E-01	DIF 100
	A(7)=-0.486111111111111E-02	DIF 101
	A(8)=-0.1984126984127E-03	DIF 102
	GO TO 22	DIF 103
C		DIF 104

## C STIFF COEFFICIENTS, ORDER 1-6

C		DIF 105
C		DIF 106
16	A(1)=-1.000000000	DIF 107
	GO TO 22	DIF 108
17	A(1)=-0.666666666666667	DIF 109
	A(3)=-0.333333333333333	DIF 110
	GO TO 22	DIF 111
18	A(1)=-0.54545454545455	DIF 112
	A(3)=A(1)	DIF 113
	A(4)=-0.90909090909091E-01	DIF 114
	GO TO 22	DIF 115
19	A(1)=-0.480000000	DIF 116
	A(3)=-0.700000000	DIF 117
	A(4)=-0.200000000	DIF 118
	A(5)=-0.020000000	DIF 119
	GO TO 22	DIF 120
20	A(1)=-0.43795620437956	DIF 121
	A(3)=-0.82116788321169	DIF 122
	A(4)=-0.31021897810219	DIF 123
	A(5)=-0.54744525547445E-01	DIF 124
	A(6)=-0.36496350364964E-02	DIF 125
	GO TO 22	DIF 126
21	A(1)=-0.40816326530612	DIF 127
	A(3)=-0.92063492063492	DIF 128
	A(4)=-0.416666666666666	DIF 129
	A(5)=-0.99206349206349E-01	DIF 130
	A(6)=-0.11904761904762E-01	DIF 131
	A(7)=-0.56689342403628E-03	DIF 132
22	K=NQ+1	DIF 133
	IDOUB=K	DIF 134
	MTYP=2-ISTIFF	DIF 135
	ENQ2=0.5/FLOAT(NQ+1)	DIF 136
	ENQ3=0.5/FLOAT(NQ+2)	DIF 137
	ENQ1=0.5/FLOAT(NQ)	DIF 138
	EUP=(PERTST(NQ,MTYP,2)*EPS)**2	DIF 139
	E=(PERTST(NQ,MTYP,1)*EPS)**2	DIF 140
	EDWN=(PERTST(NQ,MTYP,3)*EPS)**2	DIF 141
	IF (EDWN.EQ.0) GO TO 83	DIF 142
	IF (IRET.EQ.2) GO TO 75	DIF 143
	IF (IRET.EQ.3) RETURN	DIF 144
23	CONTINUE	DIF 145
	DO 24 J=2,K	DIF 146
	DO 24 J1=J,K	DIF 147
	J2=K-J1+J-1	DIF 148
	DO 24 I=NBV1,NBV	DIF 149
24	Y(J2,I)=Y(J2,I)+Y(J2+1,I)	DIF 150
	DO 25 I=1,N	DIF 151
	II=I+NBV1-1	DIF 152
	YY(I)=Y(1,II)	DIF 153
25	CONTINUE	DIF 154
	RETURN	DIF 155
26	CONTINUE	DIF 156

C		DIF 157
C	CORRECTOR SECTION	DIF 158
C		DIF 159
	DO 27 I=NBV1,NBV	DIF 160
	ERROR(I)=0.0	DIF 161
27	CONTINUE	DIF 162
	IWEVAL=ISTIFF	DIF 163
	L=1	DIF 164
28	CONTINUE	DIF 165
C		DIF 166
C	ITER=0 - DIRECT ITERATION OF CORRECTOR	DIF 167
C	ITER=1 - NEWTON-RAPHSON ITERATION OF CORRECTOR	DIF 168
C		DIF 169
	IF (ITER.EQ.0.OR.ISTIFF.EQ.0) GO TO 41	DIF 170
	R=A(1)*H	DIF 171
	IF (ITRI.EQ.1) GO TO 43	DIF 172
C		DIF 173
C	TRGB OPTION	DIF 174
C		DIF 175
	IF (IWEVAL.EQ.-1) GO TO 30	DIF 176
	DO 29 I=1,NZERO	DIF 177
	PW(I)=PW(I)*R	DIF 178
	IF (IROW(I).EQ.JCOL(I)) PW(I)=1.0*PW(I)	DIF 179
29	CONTINUE	DIF 180
	IWEVAL=-1	DIF 181
30	DO 31 I=1,N	DIF 182
	IROW(I+NZERO)=I	DIF 183
	JCOL(I+NZERO)=0	DIF 184
	II=I+NBV1-1	DIF 185
	PW(I+NZERO)=Y(2,II)-DERY1(II)*H	DIF 186
31	CONTINUE	DIF 187
	NEM=NZERO+N	DIF 188
	DO 32 I=1,NEM	DIF 189
	PWSAVE(I)=PW(I)	DIF 190
32	CONTINUE	DIF 191
	IF (JSTART.EQ.0) GO TO 34	DIF 192
	NOP2=NOPS(NOPPT+2)	DIF 193
	DO 33 I=1,NOP2	DIF 194
	II=NOPPT+I	DIF 195
	NOP(I)=NOPS(II)	DIF 196
33	CONTINUE	DIF 197
	CALL TRGB2 (NOP,PW,XMIN,NTEST,NELEM)	DIF 198
	IF (NELEM.EQ.0) GO TO 38	DIF 199
	IF (NOMES.EQ.0) PRINT 85,NELEM,NBV1,NBV	DIF 200
34	CALL TRGB (NOP,NW,PW,JCOL,IROW,N,N,NEM,XMIN)	DIF 201
	IF (JSTART.EQ.0) GO TO 35	DIF 202
	NOP(2)=NOPS(NOPPT+2)	DIF 203
	GO TO 36	DIF 204
35	CONTINUE	DIF 205
C		DIF 206
C	INCREASE LENGTH OF OPERATOR LIST BY 10 PERCENT IN CASE IT MUST	DIF 207
C	BE RE-CREATED	DIF 208

C	NOP(2)=NOP(2)+NOP(2)/10	DIF 209
36	CONTINUE	DIF 210
	NOP2=NOP(2)	DIF 211
	DO 37 I=1,NOP2	DIF 212
	II=NOPPT+I	DIF 213
	NOPS(II)=NOP(I)	DIF 214
37	CONTINUE	DIF 215
38	CONTINUE	DIF 216
	DO 39 I=1,NEM	DIF 217
	PW(I)=PWSAVE(I)	DIF 218
39	CONTINUE	DIF 219
	NOP3=NOP(3)	DIF 220
	DO 40 I=1,N	DIF 221
	II=I+NBV1-1	DIF 222
	SAVE(9,II)=PW(NOP3+I)	DIF 223
40	CONTINUE	DIF 224
	GO TO 49	DIF 225
C		DIF 226
C	DIRECT ITERATION OF CORRECTOR	DIF 227
C		DIF 228
41	CONTINUE	DIF 229
	DO 42 I=NBV1,NBV	DIF 230
	SAVE(9,I)=Y(2,I)-DERY1(I)*H	DIF 231
42	CONTINUE	DIF 232
	GO TO 49	DIF 233
C		DIF 234
C	TRIDIAGONAL OPTION	DIF 235
C		DIF 236
43	CONTINUE	DIF 237
	IF (IWEVAL.EQ.-1) GO TO 46	DIF 238
	DO 44 I=1,N	DIF 239
	ASUB(I)=ASUB(I)*R	DIF 240
	BDIAG(I)=BDIAG(I)*R	DIF 241
	CSUP(I)=CSUP(I)*R	DIF 242
44	CONTINUE	DIF 243
	DO 45 I=1,N	DIF 244
	BDIAG(I)=1.0+BDIAG(I)	DIF 245
45	CONTINUE	DIF 246
	IWEVAL=-1	DIF 247
46	CONTINUE	DIF 248
	DO 47 I=NBV1,NBV	DIF 249
	II=I-NBV1+1	DIF 250
	DERY2(II)=Y(2,I)-DERY1(I)*H	DIF 251
47	CONTINUE	DIF 252
	CALL TRIDAG (N,ASUB,BDIAG,CSUP,DERY2,YY,DERY,PWSAVE)	DIF 253
	DO 48 I=1,N	DIF 254
	SAVE(9,I)=YY(I)	DIF 255
48	CONTINUE	DIF 256
	ITRI=0	DIF 257
49	NT=N	DIF 258
	BND=EPS*ENQ3/FLOAT(N)	DIF 259
		DIF 260

	DO 50 I=NBV1,NBV	DIF 261
	Y(1,I)=Y(1,I)+A(1)*SAVE(9,I)	DIF 262
	Y(2,I)=Y(2,I)-SAVE(9,I)	DIF 263
	ERROR(I)=ERROR(I)+SAVE(9,I)	DIF 264
	IF (ABS(SAVE(9,I)).LE.(BND*YMAX(I))) NT=NT-1	DIF 265
50	CONTINUE	DIF 266
	DO 51 I=1,N	DIF 267
	II=I+NBV1-1	DIF 268
	YY(I)=Y(1,II)	DIF 269
51	CONTINUE	DIF 270
	IF (NT.LE.0) GO TO 56	DIF 271
	IF (L.EQ.3) GO TO 54	DIF 272
C		DIF 273
C	RETURN TO MODULE TO GET NEW DERIVATIVE VALUES	DIF 274
C		DIF 275
	IDERY=1	DIF 276
	RETURN	DIF 277
52	IDERY=0	DIF 278
	DO 53 I=NBV1,NBV	DIF 279
	II=I-NBV1+1	DIF 280
	DERY1(I)=DERY(II)	DIF 281
53	CONTINUE	DIF 282
	L=L+1	DIF 283
	IF (L.LT.4) GO TO 28	DIF 284
54	CONTINUE	DIF 285
	ICONV=1	DIF 286
	RACUM=RACUM*0.25	DIF 287
	IF (NOMES.EQ.0) PRINT 86,NBV1,NBV	DIF 288
	IF (H.LE.(HMIN*1.00001)) GO TO 55	DIF 289
	GO TO 80	DIF 290
55	PRINT 87, NBV1,NBV	DIF 291
	STOP	DIF 292
56	CONTINUE	DIF 293
	IF (ITRI.EQ.0.AND.ITER.EQ.1) NOPPT=NOPPT+NOP(2)	DIF 294
	IF (NBV.NE.NBVMAX) RETURN	DIF 295
C		DIF 296
C	ERROR SECTION	DIF 297
C		DIF 298
	D=0.0	DIF 299
	DO 57 I=1,NBVMAX	DIF 300
	D=D+(ERROR(I)/YMAX(I))**2	DIF 301
57	CONTINUE	DIF 302
	IF (D.GT.E) GO TO 61	DIF 303
	IF (K.LT.3) GO TO 59	DIF 304
	DO 58 J=3,K	DIF 305
	DO 58 I=1,NBVMAX	DIF 306
58	Y(J,I)=Y(J,I)+A(J)*ERROR(I)	DIF 307
59	CONTINUE	DIF 308
	IF (IDOUB.LE.1) GO TO 62	DIF 309
	IDOUB=IDOUB-1	DIF 310
	IF (IDOUB.GT.1) GO TO 77	DIF 311
	DO 60 I=1,NBVMAX	DIF 312



	SAVE(10,I)=ERROR(I)	DIF 313
60	CONTINUE	DIF 314
	GO TO 77	DIF 315
61	KFLAG=KFLAG-2	DIF 316
	IF (NOMES.EQ.0) PRINT 88	DIF 317
	IF (H.LE.(HMIN*1.00001)) GO TO 79	DIF 318
62	PR2=(D/E)**ENQ2*1.2	DIF 319
	PR3=1.0E+20	DIF 320
	IF ((NQ.GE.IORDER).OR.(KFLAG.LE.-1)) GO TO 64	DIF 321
	D=0.0	DIF 322
	DO 63 I=1,NBVMAX	DIF 323
	D=D+((ERROR(I)-SAVE(10,I))/YMAX(I))**2	DIF 324
63	CONTINUE	DIF 325
	PR3=(D/EUP)**ENQ3*1.4	DIF 326
64	PR1=1.0E+20	DIF 327
	IF (NQ.LE.1) GO TO 66	DIF 328
	D=0.0	DIF 329
	DO 65 I=1,NBVMAX	DIF 330
	D=D+(Y(K,I)/YMAX(I))**2	DIF 331
65	CONTINUE	DIF 332
	PR1=(D/EDWN)**ENQ1*1.3	DIF 333
66	CONTINUE	DIF 334
	IF (PR2.LE.PR3) GO TO 72	DIF 335
	IF (PR3.LT.PR1) GO TO 73	DIF 336
67	R=1.0/AMAX1(PR1,1.0E-04)	DIF 337
	NEWQ=NQ-1	DIF 338
68	IDOUB=10	DIF 339
	IF ((KFLAG.EQ.1).AND.(R.LT.1.1)) GO TO 77	DIF 340
	IF (NEWQ.LE.NQ) GO TO 70	DIF 341
	DO 69 I=1,NBVMAX	DIF 342
	Y(NEWQ+1,I)=ERROR(I)*A(K)/FLOAT(K)	DIF 343
69	CONTINUE	DIF 344
70	K=NEWQ+1	DIF 345
	IF (KFLAG.EQ.1) GO TO 74	DIF 346
	RACUM=RACUM*R	DIF 347
	IRET1=3	DIF 348
	GO TO 80	DIF 349
71	CONTINUE	DIF 350
	IRET1=0	DIF 351
	IF (NEWQ.EQ.NQ) RETURN	DIF 352
	NQ=NEWQ	DIF 353
	IRET=3	DIF 354
	GO TO 6	DIF 355
72	IF (PR2.GT.PR1) GO TO 67	DIF 356
	NEWQ=NQ	DIF 357
	R=1.0/AMAX1(PR2,1.0E-04)	DIF 358
	GO TO 68	DIF 359
73	R=1.0/AMAX1(PR3,1.0E-04)	DIF 360
	NEWQ=NQ+1	DIF 361
	GO TO 68	DIF 362
74	CONTINUE	DIF 363
	R=AMIN1(R,HMAX/ABS(H))	DIF 364

	H=H*R	DIF 365
	IF (NQ.EQ.NEWQ) GO TO 75	DIF 366
	NQ=NEWQ	DIF 367
	IRET=2	DIF 368
	GO TO 6	DIF 369
75	R1=1.0	DIF 370
	DO 76 J=2,K	DIF 371
	R1=R1*R	DIF 372
	DO 76 I=1,NBVMAX	DIF 373
76	Y(J,I)=Y(J,I)*R1	DIF 374
	IDOUB=K	DIF 375
77	CONTINUE	DIF 376
	DO 78 I=1,NBVMAX	DIF 377
	YMAX(I)=AMAX1(YMAX(I),ABS(Y(1,I)))	DIF 378
78	CONTINUE	DIF 379
	JSTART=NQ	DIF 380
	RETURN	DIF 381
79	PRINT A9	DIF 382
	STOP	DIF 383
80	CONTINUE	DIF 384
	RACUM=AMAX1(ABS(HMIN/HOLD),RACUM)	DIF 385
	RACUM=AMIN1(RACUM,ABS(HMAX/HOLD))	DIF 386
	H=HOLD*RACUM	DIF 387
	R1=1.0	DIF 388
	DO 81 J=2,K	DIF 389
	R1=R1*RACUM	DIF 390
	DO 81 I=1,NBVMAX	DIF 391
81	Y(J,I)=SAVE(J,I)*R1	DIF 392
	DO 82 I=1,NBVMAX	DIF 393
	Y(1,I)=SAVE(1,I)	DIF 394
82	CONTINUE	DIF 395
	IDOUB=K	DIF 396
	IF (IRET1.EQ.3) GO TO 71	DIF 397
	RETURN	DIF 398
83	PRINT 90	DIF 399
	STOP	DIF 400
C		DIF 401
84	FORMAT (41H THE MAXIMUM ORDER SPECIFIED IS TOO LARGE,I5)	DIF 402
85	FORMAT (43H THE CORRECTOR CANNOT BE SOLVED BECAUSE PWY,I4,47H ) ISDIF 403	DIF 403
	1 A PIVOT ELEMENT WHOSE VALUE IS BELOW XMIN,/,23H DIFFERENTIAL EQUADIF 404	DIF 404
	2TIONS,I4,3H TO,I4,/,33H OPERATOR LIST WILL BE RE-CREATED)	DIF 405
86	FORMAT (49H THE CORRECTOR FAILED TO CONVERGE IN 3 ITERATIONS,23H DDIF 406	DIF 406
	1DIFFERENTIAL EQUATIONS,I4,3H TO,I4)	DIF 407
87	FORMAT (58H CORRECTOR CONVERGENCE COULD NOT BE ACHIEVED FOR H.GT.HDIF 408	DIF 408
	1MIN,/,23H DIFFERENTIAL EQUATIONS,I4,3H TO,I4)	DIF 409
88	FORMAT (30H TRUNCATION ERROR IS TOO LARGE)	DIF 410
89	FORMAT (72H THE STEP WAS TAKEN WITH H=HMIN BUT THE REQUESTED ERRORDIF 411	DIF 411
	1 WAS NOT ACHIEVED)	DIF 412
90	FORMAT (68H THE REQUESTED ERROR IS SMALLER THAN CAN BE HANDLED FORDIF 413	DIF 413
	1 THIS PROBLEM)	DIF 414
	END	DIF 415

C	SUBROUTINE TRIDAG (N,A,B,C,D,V,BETA,GAMMA)	TRI	1
C		TRI	2
C	SUBROUTINE FOR SOLVING A SYSTEM OF N LINEAR SIMULTANEOUS EQUATIONS	TRI	3
C	HAVING A TRIDIAGONAL COEFFICIENT MATRIX.	TRI	4
C		TRI	5
C	SOURCE - APPLIED NUMERICAL METHODS,CARNAHAN LUTHER,& WILKES,1969	TRI	6
C		TRI	7
C	ARGUMENTS	TRI	8
C		TRI	9
C	N - NUMBER OF EQUATIONS	TRI	10
C	A - SUBDIAGONAL COEFFICIENTS ARE STORED IN A(2)...A(N)	TRI	11
C	A(1) IS NOT USED	TRI	12
C	B - DIAGONAL COEFFICIENTS	TRI	13
C	C - SUPERDIAGONAL COEFFICIENTS ARE STORED IN C(1)...C(N-1)	TRI	14
C	C(N) IS NOT USED	TRI	15
C	D - RIGHT HAND SIDE VECTOR	TRI	16
C	V - SOLUTION VECTOR	TRI	17
C	BETA,GAMMA - INTERMEDIATE ARRAYS	TRI	18
C		TRI	19
C	DIMENSIONS - ALL ARGUMENTS EXCEPT N MUST BE DIMENSIONED AT LEAST	TRI	20
C	N IN CALLING PROGRAM	TRI	21
C		TRI	22
C	DIMENSION A(N), B(N), C(N), D(N), V(N), BETA(N), GAMMA(N)	TRI	23
C		TRI	24
C	COMPUTE INTERMEDIATE ARRAYS BETA AND GAMMA	TRI	25
C		TRI	26
C	BETA(1)=B(1)	TRI	27
C	GAMMA(1)=D(1)/BETA(1)	TRI	28
C	DO 1 I=2,N	TRI	29
C	BETA(I)=B(I)-A(I)*C(I-1)/BETA(I-1)	TRI	30
C	GAMMA(I)=(D(I)-A(I)*GAMMA(I-1))/BETA(I)	TRI	31
C	CONTINUE	TRI	32
1		TRI	33
C	COMPUTE FINAL SOLUTION VECTOR V	TRI	34
C		TRI	35
C	V(N)=GAMMA(N)	TRI	36
C	LAST=N-1	TRI	37
C	DO 2 K=1, LAST	TRI	38
C	I=N-K	TRI	39
C	V(I)=GAMMA(I)-C(I)*V(I+1)/BETA(I)	TRI	40
2	CONTINUE	TRI	41
C	RETURN	TRI	42
C	END	TRI	43

FIGURE 1-2: SKELETON OF MODULE FOR DYNYS 2.1

```

SUBROUTINE TYPE2.1
C
C COMMENTS DESCRIBING MODULE
C
C THE FOLLOWING COMMON BLOCKS ARE OR MAY BE REQUIRED:
C MAT,CON,PTAB,UNIT,GEAR2,MODULE,ROW,COLUMN,JACOB,SUBDI,DIAG,SUPERD
C
COMMON/MAT/MP( , ),EP( , ),S( , , ),EX( )
COMMON/CON/IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON/PTAB/IGFLAG,PP( , )
COMMON/UNIT/IM,NMP
COMMON/GEAR2/EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,ISTIFF
COMMON/MODULE/IDERY,ITER,ITRI,MC,IPIVOT
C
C THE ROW DIMENSION OF JCOL AND XJACOB MUST BE THE SAME AS IN DIFSUB
C MC IS MAXIMUM NUMBER OF COLUMNS IN XJACOB AND JCOL
C
COMMON/COLUMN/JCOL( ,MC) (NORMAL OPTION)
COMMON/JACOB/XJACOB( ,MC) (NORMAL OPTION)
C
C N IS NUMBER OF ODES
C
COMMON/SUBDI/A(N) (TRIDIAGONAL OPTION)
COMMON/DIAG/B(N) (TRIDIAGONAL OPTION)
COMMON/SUPERD/C(N) (TRIDIAGONAL OPTION)
C
C *****
C SECTION #1 : PARAMETER CALCULATIONS
C *****
C
C CALCULATE MODULE PARAMETERS : STREAM INPUT,INITIAL CONDITIONS,
C VALUES OF ITER ETC.
C
C INPUT STREAM INFORMATION IS OBTAINED FROM S(IG, , )
C VALUE OF INDEPENDENT VARIABLE (Y) NEED ONLY BE SPECIFIED ON FIRST
C INTEGRATION STEP (I.E. INITIAL CONDITIONS)
C JSTART=0 ON FIRST INTEGRATION STEP
C *CURRENT ORDER OF INTEGRATION TECHNIQUE ON LATER STEPS
C
C IF ISTIFF=0, NONSTIFF COEFFICIENTS WILL BE USED IN INTEGRATION ALGORITHM
C (DIFSUB) FOR ALL MODULES,
C JACOBIAN MATRIX IS NOT REQUIRED
C
C IF ISTIFF=1, STIFF COEFFICIENTS WILL BE USED FOR ALL MODULES
C THEN, IF ITER=0 DIRECT ITERATION OF CORRECTOR WILL BE USED
C (NONSTIFF MODULE)
C JACOBIAN IS NOT REQUIRED
C IF ITER=1 NEWTON-RAPHSON ITERATION OF CORRECTOR WILL
C BE USED (STIFF MODULE)

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JACOBIAN MATRIX MUST BE SUPPLIED

ITER MUST BE SPECIFIED 0 OR 1  
IT IS NOT USED UNLESS ISTIFF=1

IF (IG.EQ.2) GO TO 2

.....  
SECTION #2 : JACOBIAN EVALUATION  
.....

SECTION #2 IS OMITTED FOR NONSTIFF MODULE

JACOBIAN MATRIX IS REQUIRED ONLY IF MODULE EQUATIONS ARE STIFF  
CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS ONLY  
JACOBIAN NEED NOT BE VERY ACCURATE, AS IT IS USED ONLY FOR CONVERGENCE  
OF CORRECTOR

ONLY NONZERO ELEMENTS ARE CALCULATED, BUT ALL DIAGONAL ELEMENTS MUST  
BE STORED WHETHER OR NOT THEY ARE ZERO (NORMAL OPTION)

NORMAL OPTION

IF JACOBIAN MATRIX IS NOT TRIDIAGONAL, KEYS STORAGE SCHEME IS USED  
COLUMN NUMBERS OF NONZERO ELEMENTS IN ROW I ARE STORED IN  
(JCOL(I,J), J=1, NUMBER OF NONZEROS IN THAT ROW)

SIMILARLY THE VALUES OF THE NONZERO ELEMENTS OF ROW I ARE  
STORED IN XJACOB(I, )

REMEMBER TO STORE ALL DIAGONAL ELEMENTS EVEN IF THEY ARE ZERO

MC - MAXIMUM NUMBER OF COLUMNS IN XJACOB MATRIX  
MC CAN BE LARGER THAN THE MAXIMUM NUMBER OF NONZERO ELEMENTS  
IN ANY JACOBIAN ROW SINCE ADDITIONAL NONZERO ELEMENTS CAN BE  
CREATED DURING THE SOLUTION. MC IS CONSTANT FOR ANY SET OF ODES  
AND MAY HAVE TO BE DETERMINED BY TRIAL AND ERROR

IPIVOT - PIVOT OPTION USED IN SIMULT (1-7)

IPIVOT=1 : SIMPLE GAUSS-JORDAN ELIMINATION

IPIVOT=2 : GAUSS-JORDAN PARTIAL PIVOTING

IPIVOT=3 : GAUSS-JORDAN FULL PIVOTING

IPIVOT=4 : MINIMUM ROW-MINIMUM COLUMN

IPIVOT=5 : MINIMUM COLUMN-MINIMUM ROW

IPIVOT=6 : MAXIMUM COLUMN-MINIMUM ROW

IPIVOT=7 : MINIMUM OF ROW ENTRIES TIMES COLUMN ENTRIES

ICOL(I,J) - COLUMN NUMBER OF JTH NONZERO ELEMENT IN ROW I

XJACOB(I,J) - VALUE OF JTH NONZERO ELEMENT OF ROW I

MC=

IPIVOT=

IROW(1,1)=

XJACOB(1,1)=

.

.

IROW(N, )=  
XJACOB(N, )=

TRIDIAGONAL OPTION

FOR TRIDIAGONAL JACOBIAN MATRIX, THE SUBDIAGONAL, DIAGONAL AND SUPERDIAGONAL  
ELEMENT VALUES ARE STORED IN ARRAYS A, B AND C FROM COMMON BLOCKS SUBDI,  
DIAG AND SUPERD RESPECTIVELY

N - NUMBER OF ODES

A - VALUES OF SUBDIAGONAL ELEMENTS ARE STORED IN A(2)...A(N)  
A(1) IS NOT USED

B - VALUES OF DIAGONAL ELEMENTS

C - VALUES OF SUPERDIAGONAL ELEMENTS ARE STORED IN C(1)...C(N-1)  
C(N) IS NOT USED

B(1)=

C(1)=

A(2)=

B(2)=

C(2)=

.

.

A(N-1)=

B(N-1)=

C(N-1)=

A(N)=

B(N)=

1 CONTINUE

IF TRIDIAGONAL OPTION IS BEING USED, ITRI MUST BE SET TO 1 HERE.  
IF NORMAL OPTION IS USED ITRI MAY BE IGNORED

ITRI=1

2 CONTINUE

SECTION #3 : DERIVATIVE CALCULATION

CALCULATE DERIVATIVES

DERY(1)=

.

.

.

DERY(N)=



FIGURE I.3: LISTING OF DYNYSYS 2.1 EXECUTIVE  
(ROUTINES IDENTICAL OR NEARLY IDENTICAL WITH THOSE  
IN DYNYSYS 2.0 ARE NOT LISTED)

C  
C  
C  
C  
C

```

SUBROUTINE DYN1
DATA LOADING AND PRINTING
COMMON /UNIT/ IM,NMP
COMMON /MAT/ MP(15,5),EP(15,5),S(2,16,7),EX(1)
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH
COMMON /FTN/ FN(1,1),NFN
COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,ISTIFF
COMMON /GEAR3/ HMIN,HMAX,NOMES
DIMENSION AM(100)
DIMENSION ITAG(30)
DATA ITAG/5HVALV1,5HVALV2,5HSTIR1,5HCONT1,5HSETL1,6HMXPLT1,5HSETT1,
1,6HVESL1,5HDELAY,5HCSTR1,5HEXCH1,5HDCAN1,5HCONV1,5HSPLT1,16*6H
2 /
WRITE (6,27)
DEFAULT VALUES
TIME=0.0
MAXNE=20
NPR=1
TMAX=10.0
NCOMP=1
NMP=5
H=0.00001
HMIN=0.000001
HMAX=0.05
EPS=0.001
NPOL=0
NGRAPH=0
IORDER=6
ISTIFF=1
NOMES=0
READ AND COPY TITLE
1 READ (5,29) (AM(I),I=1,8)
WRITE (6,28) (AM(I),I=1,8)
IF (AM(1).NE.5HBEGIN) GO TO 1
C READ INITIAL DATA
2 CALL GET (N,X,1)
IF (N.EQ.5HCOMPS) NCOMP=X
IF (N.EQ.6HIN/OUT) NMP=X
IF (N.EQ.6HDELTAT) H=X
IF (N.EQ.4HHMIN) HMIN=X
IF (N.EQ.4HHMAX) HMAX=X
IF (N.EQ.4HTIME) TMAX=X
IF (N.EQ.8HPRINTING) NPR=X
IF (N.EQ.9HTOLERANCE) EPS=X
IF (N.EQ.5HORDER) IORDER=X
IF (N.EQ.8HLINEPLOT) NPOL=1
IF (N.EQ.5HGRAPH) NGRAPH=1

```

DY1 1  
DY1 2  
DY1 3  
DY1 4  
DY1 5  
DY1 6  
DY1 7  
DY1 8  
DY1 9  
DY1 10  
DY1 11  
DY1 12  
DY1 13  
DY1 14  
DY1 15  
DY1 16  
DY1 17  
DY1 18  
DY1 19  
DY1 20  
DY1 21  
DY1 22  
DY1 23  
DY1 24  
DY1 25  
DY1 26  
DY1 27  
DY1 28  
DY1 29  
DY1 30  
DY1 31  
DY1 32  
DY1 33  
DY1 34  
DY1 35  
DY1 36  
DY1 37  
DY1 38  
DY1 39  
DY1 40  
DY1 41  
DY1 42  
DY1 43  
DY1 44  
DY1 45  
DY1 46  
DY1 47

C

C  
1

C  
2



	IF (N.EQ.8HNONSTIFF) ISTIFF=0	DY1 48
	IF (N.EQ.9HNOMESSAGE) NOMES=1	DY1 49
	IF (N.EQ.7HLIBRARY) GO TO 3	DY1 50
	IF (N.EQ.7HPROCESS) GO TO 6	DY1 51
	IF (N.EQ.8HFUNTION) GO TO 5	DY1 52
	GO TO 2	DY1 53
3	NLIB=X	DY1 54
	DO 4 I=1,NLIB	DY1 55
	CALL GET (N,X,0)	DY1 56
	J=X	DY1 57
	ITAG(J)=N	DY1 58
4	CONTINUE	DY1 59
	GO TO 2	DY1 60
5	NFN=X	DY1 61
	NC5=NCOMP+5	DY1 62
	READ 39, ((FN(I,J),J=1,NC5),I=1,NFN)	DY1 63
	PRINT 39, ((FN(I,J),J=1,NC5),I=1,NFN)	DY1 64
	GO TO 2	DY1 65
6	CONTINUE	DY1 66
	NMP=NMP+2	DY1 67
	MAXNMP=NMP+1	DY1 68
	NC5=NCOMP+5	DY1 69
	WRITE (6,30)	DY1 70
	DO 9 I=1,MAXNE	DY1 71
	DO 7 J=1,MAXNMP	DY1 72
	MP(I,J)=0	DY1 73
7	CONTINUE	DY1 74
	DO 8 J=1,5	DY1 75
	EP(I,J)=0	DY1 76
8	CONTINUE	DY1 77
9	CONTINUE	DY1 78
	NE=0	DY1 79
	NEX=1	DY1 80
	TIME=0.0	DY1 81
10	CALL GET (NM,X,0)	DY1 82
11	IF (NM.EQ.3HEND) GO TO 16	DY1 83
	DO 12 I=1,30	DY1 84
	IF (NM.EQ.ITAG(I)) GO TO 13	DY1 85
12	CONTINUE	DY1 86
	WRITE (6,31) NM	DY1 87
	I=0	DY1 88
13	NT=I	DY1 89
	NE=NE+1	DY1 90
	READ (5,32) (AM(I),I=3,NMP)	DY1 91
	DO 14 J=3,NMP	DY1 92
	MP(NE,J)=AM(J)	DY1 93
14	CONTINUE	DY1 94
	MP(NE,1)=X	DY1 95
	MP(NE,2)=NT	DY1 96
	WRITE (6,33) (MP(NE,I),I=1,NMP)	DY1 97
	READ (5,32) (EP(NE,I),I=1,5)	DY1 98
	WRITE (6,34) (EP(NE,I),I=1,5)	DY1 99

	CALL GET (NM,X,0)	DY1 100
	IF (NM,NE.5HEXTRA) GO TO 15	DY1 101
	NNX=X	DY1 102
	NMX=NNX+NEX-1	DY1 103
	READ (5,32) (EX(J),J=NEX,NMX)	DY1 104
	WRITE (6,34) (EX(J),J=NEX,NMX)	DY1 105
	MP(NE,NMP+1)=NEX	DY1 106
	NEX=NEX+NNX	DY1 107
	CALL GET (NM,X,1)	DY1 108
15	IF (NM,NE.7HRESERVE) GO TO 11	DY1 109
	NNX=X	DY1 110
	NEX=NEX+NNX	DY1 111
	GO TO 10	DY1 112
16	CALL FETCH (7HSTREAMS,NS)	DY1 113
	WRITE (6,35) NS	DY1 114
	DO 18 I=1,NS	DY1 115
	S(1,I,1)=I	DY1 116
	S(1,I,2)=-1.0	DY1 117
	S(1,I,3)=0.0	DY1 118
	S(1,I,4)=530.0	DY1 119
	S(1,I,5)=14.7	DY1 120
	DO 17 J=6,NC5	DY1 121
	S(1,I,J)=0	DY1 122
17	CONTINUE	DY1 123
18	CONTINUE	DY1 124
19	CALL GET (NM,X,-1)	DY1 125
	IF (NM,EQ.8HEXPPLICIT) GO TO 20	DY1 126
	IF (NM,EQ.3HEND) GO TO 25	DY1 127
	IF (NM,EQ.4HSPEC) GO TO 22	DY1 128
	GO TO 19	DY1 129
20	READ (5,36) IT,(AM(I),I=1,5)	DY1 130
	IF (IT,EQ.3HEND) GO TO 25	DY1 131
	READ (5,32) (AM(I),I=6,NC5)	DY1 132
	N=AM(1)	DY1 133
	DO 21 I=1,NC5	DY1 134
	S(1,N,I)=AM(I)	DY1 135
21	CONTINUE	DY1 136
	GO TO 20	DY1 137
22	N1=X	DY1 138
	READ (5,32) (AM(I),I=1,NC5)	DY1 139
	N2=AM(1)	DY1 140
	DO 24 I=N1,N2	DY1 141
	DO 23 J=2,NC5	DY1 142
	S(1,I,J)=AM(J)	DY1 143
23	CONTINUE	DY1 144
	S(1,I,1)=I	DY1 145
24	CONTINUE	DY1 146
	GO TO 19	DY1 147
25	DO 26 I=1,NS	DY1 148
	K=S(1,I,2)	DY1 149
	WRITE (6,38) I,K,(S(1,I,J),J=3,5)	DY1 150
	WRITE (6,37) (S(1,I,J),J=6,NC5)	DY1 151

```
DO 26 J=1,NC5
26 S(2,I,J)=S(1,I,J)
   XX=PROPS(0,0)
   RETURN
C
27 FORMAT (1H1)
28 FORMAT (1X,8A10)
29 FORMAT (8A10)
30 FORMAT (1H1)
31 FORMAT (1BH ***FAIL*** NAME ,A12,8H NOT SET)
32 FORMAT (12X,5F12.0)
33 FORMAT (/,9H UNIT -,I3,9H- TYPE -,I3,1H-,12I5)
34 FORMAT (11X,5F14.5)
35 FORMAT (1H1,I6,8H STREAMS)
36 FORMAT (A3,9X,5F12.0)
37 FORMAT (8X,4F14.5)
38 FORMAT (11H STREAM -,I3,4H- (,I3,1H),3F14.5)
39 FORMAT (12X,5F12.2)
   END
```

DY1 152  
DY1 153  
DY1 154  
DY1 155  
DY1 156  
DY1 157  
DY1 158  
DY1 159  
DY1 160  
DY1 161  
DY1 162  
DY1 163  
DY1 164  
DY1 165  
DY1 166  
DY1 167  
DY1 168  
DY1 169  
DY1 170

	SUBROUTINE DYN2	DY2	1
	COMMON /UNIT/ IM,NMP	DY2	2
	COMMON /MAT/ MP(15,5),EP(15,5),S(2,16,7),EX(1)	DY2	3
	COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DY2	4
	COMMON /BKV/ NBV	DY2	5
	COMMON /CNTR/ NCTR	DY2	6
	COMMON /LAG/ NSX,NSW	DY2	7
	COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,ISTIFF	DY2	8
	NBV=0	DY2	9
	NSX=0	DY2	10
	NCTR=0	DY2	11
	DO 31 IM=1,NE	DY2	12
	IF (ICONV.EQ.1) RETURN	DY2	13
	NTYPE=MP(IM,2)	DY2	14
	GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,22,23,24,25,26,27,28,29,30), NTYPE	DY2	15
1	CALL TYPE1	DY2	16
	GO TO 31	DY2	17
2	CALL TYPE2	DY2	18
	GO TO 31	DY2	19
3	CALL TYPE3	DY2	20
	GO TO 31	DY2	21
4	CALL TYPE4	DY2	22
	GO TO 31	DY2	23
5	CALL TYPE5	DY2	24
	GO TO 31	DY2	25
6	CALL TYPE6	DY2	26
	GO TO 31	DY2	27
7	CALL TYPE7	DY2	28
	GO TO 31	DY2	29
8	CALL TYPE8	DY2	30
	GO TO 31	DY2	31
9	CALL TYPE9	DY2	32
	GO TO 31	DY2	33
10	CALL TYPE10	DY2	34
	GO TO 31	DY2	35
11	CALL TYPE11	DY2	36
	GO TO 31	DY2	37
12	CALL TYPE12	DY2	38
	GO TO 31	DY2	39
13	CALL TYPE13	DY2	40
	GO TO 31	DY2	41
14	CALL TYPE14	DY2	42
	GO TO 31	DY2	43
15	CALL TYPE15	DY2	44
	GO TO 31	DY2	45
16	CALL TYPE16	DY2	46
	GO TO 31	DY2	47
17	CALL TYPE17	DY2	48
	GO TO 31	DY2	49
18	CALL TYPE18	DY2	50
	GO TO 31	DY2	51
		DY2	52

19 CALL TYPE19  
GO TO 31  
20 CALL TYPE20  
GO TO 31  
21 CALL TYPE21  
GO TO 31  
22 CALL TYPE22  
GO TO 31  
23 CALL TYPE23  
GO TO 31  
24 CALL TYPE24  
GO TO 31  
25 CALL TYPE25  
GO TO 31  
26 CALL TYPE26  
GO TO 31  
27 CALL TYPE27  
GO TO 31  
28 CALL TYPE28  
GO TO 31  
29 CALL TYPE29  
GO TO 31  
30 CALL TYPE30  
31 CONTINUE  
RETURN  
END

DY2 53  
DY2 54  
DY2 55  
DY2 56  
DY2 57  
DY2 58  
DY2 59  
DY2 60  
DY2 61  
DY2 62  
DY2 63  
DY2 64  
DY2 65  
DY2 66  
DY2 67  
DY2 68  
DY2 69  
DY2 70  
DY2 71  
DY2 72  
DY2 73  
DY2 74  
DY2 75  
DY2 76  
DY2 77  
DY2 78-

SUBROUTINE DIFSUB (N,YY,DERY)	DIF	1
COMMON /GEAR1/ Y(8,222),SAVE(10,222),ERROR(222),YMAX(222),DERY1(222)	DIF	2
12),DERY2(222)	DIF	3
COMMON /GEAR2/ EPS,TIME,KFLAG,JSTART,NBVMAX,ICONV,ISTIFF	DIF	4
COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH	DIF	5
COMMON /GEAR3/ HMIN,HMAX,NOMES	DIF	6
COMMON /BKV/ NBV	DIF	7
COMMON /COLUMN/ JCOL(10,10)	DIF	8
COMMON /JACOB/ PW(10,10)	DIF	9
COMMON /SUBDI/ ASUB(49)	DIF	10
COMMON /DIAG/ BDIAG(49)	DIF	11
COMMON /SUPERD/ CSUP(49)	DIF	12
COMMON /MODULE/ IDERY,ITER,ITRI,MC,IPIVOT	DIF	13
DIMENSION IRC(2), ICC(2)	DIF	14
DIMENSION PWSAVE(2,2), JCOLS(2,2)	DIF	15
DIMENSION A(8), PERTST(7,2,3)	DIF	16
DIMENSION YY(222), DERY(222)	DIF	17
DATA MR/10/	DIF	18
DATA ITRI,IDERY/2*0/	DIF	19
DATA PERTST/2.0,4.5,7.333,10.42,13.7,17.15,1.0,2.0,12.0,24.0,37.8	DIF	20
1,53.33,70.08,87.97,3.0,6.0,9.167,12.5,15.98,1.0,1.0,12.0,24.0,37.8	DIF	21
29,53.33,70.08,87.97,1.0,1.0,1.0,0.5,0.1667,0.04133,0.008267,1.0,1.0	DIF	22
30,1.0,2.0,1.0,0.3157,0.07407,0.0139/	DIF	23
DATA A(2)/-1.0/	DIF	24
IF (IDERY.EQ.1) GO TO 47	DIF	25
NBV1=NBV+1	DIF	26
NBV=NBV+N	DIF	27
IF (NBVMAX.LT.NBV) NBVMAX=NBV	DIF	28
DO 1 I=NBV1,NBV	DIF	29
II=I-NBV1+1	DIF	30
DERY1(II)=DERY(II)	DIF	31
CONTINUE	DIF	32
IF (IG.EQ.1) GO TO 26	DIF	33
PREDICTOR SECTION	DIF	34
IRET=1	DIF	35
IF (JSTART.EQ.0) GO TO 4	DIF	36
DO 3 I=NBV1,NBV	DIF	37
DO 3 J=1,K	DIF	38
SAVE(J,I)=Y(J,I)	DIF	39
HOLD=H	DIF	40
NQOLD=NQ	DIF	41
RACUM=1.0	DIF	42
IF (JSTART.GT.0) GO TO 23	DIF	43
GO TO 6	DIF	44
CONTINUE	DIF	45
NQ=1	DIF	46
N3=N	DIF	47
N4=N*N	DIF	48
DO 5 I=NBV1,NBV	DIF	49
II=I-NBV1+1	DIF	50
	DIF	51
	DIF	52

```

Y(1,I)=YY(II)
Y(2,I)=DERY1(I)*H
5 CONTINUE
HNEW=H
K=2
GO TO 2
6 IF (ISTIFF.EQ.0) GO TO 7
IF (NQ.GT.6.OR.NQ.LT.3) GO TO 8
GO TO (16,17,18,19,20,21), NQ
7 IF (NQ.GT.7.OR.NQ.LT.1) GO TO 8
GO TO (9,10,11,12,13,14,15), NQ
8 PRINT 79, NQ
STOP
C
C NONSTIFF COEFFICIENTS,ORDER 1-7
C
9 A(1)=-1.0
GO TO 22
10 A(1)=-0.500000000
A(3)=-0.500000000
GO TO 22
11 A(1)=-0.416666666666667
A(3)=-0.750000000
A(4)=-0.166666666666667
GO TO 22
12 A(1)=-0.375000000
A(3)=-0.916666666666667
A(4)=-0.333333333333333
A(5)=-0.416666666666667E-01
GO TO 22
13 A(1)=-0.348611111111111
A(3)=-1.041666666666667
A(4)=-0.486111111111111
A(5)=-0.104166666666667
A(6)=-0.833333333333333E-02
GO TO 22
14 A(1)=-0.329861111111111
A(3)=-1.141666666666667
A(4)=-0.625000000
A(5)=-0.177083333333333
A(6)=-0.025000000
A(7)=-0.138888888888889E-02
GO TO 22
15 A(1)=-0.31559193121693
A(3)=-1.235000000
A(4)=-0.75185185185
A(5)=-0.255208333333333
A(6)=-0.486111111111111E-01
A(7)=-0.486111111111111E-02
A(8)=-0.1984126984127E-03
GO TO 22
C

```

```

DIF 53
DIF 54
DIF 55
DIF 56
DIF 57
DIF 58
DIF 59
DIF 60
DIF 61
DIF 62
DIF 63
DIF 64
DIF 65
DIF 66
DIF 67
DIF 68
DIF 69
DIF 70
DIF 71
DIF 72
DIF 73
DIF 74
DIF 75
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DIF 77
DIF 78
DIF 79
DIF 80
DIF 81
DIF 82
DIF 83
DIF 84
DIF 85
DIF 86
DIF 87
DIF 88
DIF 89
DIF 90
DIF 91
DIF 92
DIF 93
DIF 94
DIF 95
DIF 96
DIF 97
DIF 98
DIF 99
DIF 100
DIF 101
DIF 102
DIF 103
DIF 104

```

## C STIFF COEFFICIENTS, ORDER 1-6

C		DIF 105
C		DIF 106
16	A(1)=-1.000000000 GO TO 22	DIF 107
17	A(1)=-0.666666666666667 A(3)=-0.333333333333333 GO TO 22	DIF 108
18	A(1)=-0.545454545454545 A(3)=A(1) A(4)=-0.90909090909091E-01 GO TO 22	DIF 109
19	A(1)=-0.480000000 A(3)=-0.700000000 A(4)=-0.200000000 A(5)=-0.020000000 GO TO 22	DIF 110
20	A(1)=-0.43795620437956 A(3)=-0.82116788321169 A(4)=-0.31021897810219 A(5)=-0.54744525547445E-01 A(6)=-0.36496350364964E-02 GO TO 22	DIF 111
21	A(1)=-0.40816326530612 A(3)=-0.92063492063492 A(4)=-0.416666666666666 A(5)=-0.99206349206349E-01 A(6)=-0.11904761904762E-01 A(7)=-0.56689342403628E-03	DIF 112
22	K=NQ+1 IDOUB=K MTYP=2-ISTIFF ENQ2=0.5/FLOAT(NQ+1) ENQ3=0.5/FLOAT(NQ+2) ENQ1=0.5/FLOAT(NQ) EUP=(PERTST(NQ,MTYP,2)*EPS)**2 E=(PERTST(NQ,MTYP,1)*EPS)**2 EDWN=(PERTST(NQ,MTYP,3)*EPS)**2 IF (EDWN.EQ.0) GO TO 78 IF (IRET.EQ.2) GO TO 70 IF (IRET.EQ.3) RETURN	DIF 113
23	CONTINUE DO 24 J=2,K DO 24 J1=J,K J2=K-J1+J-1 DO 24 I=NBV1,NBV	DIF 114
24	Y(J2,I)=Y(J2,I)+Y(J2+1,I) DO 25 I=1,N II=I+NBV1-1 YY(I)=Y(1,II)	DIF 115
25	CONTINUE	DIF 116
26	RETURN	DIF 117
	CONTINUE	DIF 118
		DIF 119
		DIF 120
		DIF 121
		DIF 122
		DIF 123
		DIF 124
		DIF 125
		DIF 126
		DIF 127
		DIF 128
		DIF 129
		DIF 130
		DIF 131
		DIF 132
		DIF 133
		DIF 134
		DIF 135
		DIF 136
		DIF 137
		DIF 138
		DIF 139
		DIF 140
		DIF 141
		DIF 142
		DIF 143
		DIF 144
		DIF 145
		DIF 146
		DIF 147
		DIF 148
		DIF 149
		DIF 150
		DIF 151
		DIF 152
		DIF 153
		DIF 154
		DIF 155
		DIF 156



C		DIF 157
C	CORRECTOR SECTION	DIF 158
C	DO 27 I=NBV1,NBV	DIF 159
	ERROR(I)=0.0	DIF 160
27	CONTINUE	DIF 161
	IWEVAL=ISTIFF	DIF 162
	L=1	DIF 163
28	CONTINUE	DIF 164
C		DIF 165
C	ITER=0 - DIRECT ITERATION OF CORRECTOR	DIF 166
C	ITER=1 - NEWTON-RAPHSON ITERATION OF CORRECTOR	DIF 167
C		DIF 168
	IF (ITER.EQ.0.OR.ISTIFF.EQ.0) GO TO 36	DIF 169
	R=A(1)*H	DIF 170
	IF (ITRI.EQ.1) GO TO 38	DIF 171
C		DIF 172
C	SIMULT OPTION	DIF 173
C		DIF 174
	IF (IWEVAL.EQ.-1) GO TO 31	DIF 175
	DO 30 J=1,MC	DIF 176
	DO 29 I=1,N	DIF 177
	PW(I,J)=PW(I,J)*R	DIF 178
	IF (JCOL(I,J).EQ.I) PW(I,J)=1.0+PW(I,J)	DIF 179
	PWSAVE(I,J)=PW(I,J)	DIF 180
	JCOLS(I,J)=JCOL(I,J)	DIF 181
29	CONTINUE	DIF 182
30	CONTINUE	DIF 183
	IWEVAL=-1	DIF 184
31	DO 32 I=NBV1,NBV	DIF 185
	II=I-NBV1+1	DIF 186
	DERY2(II)=Y(2,I)-DERY1(I)*H	DIF 187
32	CONTINUE	DIF 188
	CALL SIMULT (N,MR,MC,PW,DERY2,JCOL,1.0E-10,ICC,IRC,IPIVOT)	DIF 189
	DO 33 I=1,N	DIF 190
	II=I+NBV1-1	DIF 191
	SAVE(9,II)=PW(I,I)	DIF 192
33	CONTINUE	DIF 193
	DO 35 J=1,MC	DIF 194
	DO 34 I=1,N	DIF 195
	PW(I,J)=PWSAVE(I,J)	DIF 196
	JCOL(I,J)=JCOLS(I,J)	DIF 197
34	CONTINUE	DIF 198
35	CONTINUE	DIF 199
	GO TO 44	DIF 200
C		DIF 201
C	DIRECT ITERATION OF CORRECTOR	DIF 202
C		DIF 203
36	CONTINUE	DIF 204
	DO 37 I=NBV1,NBV	DIF 205
	SAVE(9,I)=Y(2,I)-DERY1(I)*H	DIF 206
37	CONTINUE	DIF 207
		DIF 208

	GO TO 44	DIF 209
C		DIF 210
C	TRIDIAGONAL OPTION	DIF 211
C		DIF 212
38	CONTINUE	DIF 213
	IF (IWEVAL.EQ.-1) GO TO 41	DIF 214
	DO 39 I=1,N	DIF 215
	ASUB(I)=ASUB(I)*R	DIF 216
	BDIAG(I)=BDIAG(I)*R	DIF 217
	CSUP(I)=CSUP(I)*R	DIF 218
39	CONTINUE	DIF 219
	DO 40 I=1,N	DIF 220
	BDIAG(I)=1.0*BDIAG(I)	DIF 221
40	CONTINUE	DIF 222
	IWEVAL=-1	DIF 223
41	CONTINUE	DIF 224
	DO 42 I=NBV1,NBV	DIF 225
	II=I-NBV1+1	DIF 226
	DERY2(II)=Y(2,I)-DERY1(I)*H	DIF 227
42	CONTINUE	DIF 228
	CALL TRIDAG (N,ASUB,BDIAG,CSUP,DERY2,YY,DERY,PMSAVE)	DIF 229
	DO 43 I=1,N	DIF 230
	SAVE(9,I)=YY(I)	DIF 231
43	CONTINUE	DIF 232
	ITRI=0	DIF 233
44	NT=N	DIF 234
	BND=EPS*ENG3/FLOAT(N)	DIF 235
	DO 45 I=NBV1,NBV	DIF 236
	Y(1,I)=Y(1,I)+A(1)*SAVE(9,I)	DIF 237
	Y(2,I)=Y(2,I)-SAVE(9,I)	DIF 238
	ERROR(I)=ERROR(I)+SAVE(9,I)	DIF 239
	IF (ABS(SAVE(9,I)).LE.(BND*YMAX(I))) NT=NT-1	DIF 240
45	CONTINUE	DIF 241
	DO 46 I=1,N	DIF 242
	II=I+NBV1-1	DIF 243
	YY(I)=Y(1,II)	DIF 244
46	CONTINUE	DIF 245
	IF (NT.LE.0) GO TO 51	DIF 246
	IF (L.EQ.3) GO TO 49	DIF 247
C		DIF 248
C	RETURN TO MODULE TO GET NEW DERIVATIVE VALUES	DIF 249
C		DIF 250
	IDERY=1	DIF 251
	RETURN	DIF 252
47	IDERY=0	DIF 253
	DO 48 I=NBV1,NBV	DIF 254
	II=I-NBV1+1	DIF 255
	DERY1(I)=DERY(II)	DIF 256
48	CONTINUE	DIF 257
	L=L+1	DIF 258
	IF (L.LT.4) GO TO 28	DIF 259
49	CONTINUE	DIF 260

	ICONV=1	DIF 261
	RACUM=RACUM*0.25	DIF 262
	IF (NOMES.EQ.0) PRINT 80, NBV1,NBV	DIF 263
	IF (H.LE.(HMIN*1.00001)) GO TO 50	DIF 264
	GO TO 75	DIF 265
50	PRINT 81, NBV1,NBV	DIF 266
	STOP	DIF 267
51	CONTINUE	DIF 268
	IF (NBV.NE.NBVMAX) RETURN	DIF 269
C		DIF 270
C	ERROR SECTION	DIF 271
C		DIF 272
	D=0.0	DIF 273
	DO 52 I=1,NBVMAX	DIF 274
	D=D+(ERROR(I)/YMAX(I))**2	DIF 275
52	CONTINUE	DIF 276
	IF (D.GT.E) GO TO 56	DIF 277
	IF (K.LT.3) GO TO 54	DIF 278
	DO 53 J=3,K	DIF 279
	DO 53 I=1,NBVMAX	DIF 280
53	Y(J,I)=Y(J,I)+A(J)*ERROR(I)	DIF 281
54	CONTINUE	DIF 282
	IF (IDOUB.LE.1) GO TO 57	DIF 283
	IDOUB=IDOUB-1	DIF 284
	IF (IDOUB.GT.1) GO TO 72	DIF 285
	DO 55 I=1,NBVMAX	DIF 286
	SAVE(10,I)=ERROR(I)	DIF 287
55	CONTINUE	DIF 288
	GO TO 72	DIF 289
56	KFLAG=KFLAG-2	DIF 290
	IF (NOMES.EQ.0) PRINT 82	DIF 291
	IF (H.LE.(HMIN*1.00001)) GO TO 74	DIF 292
57	PR2=(D/E)**ENQ2*1.2	DIF 293
	PR3=1.0E+20	DIF 294
	IF ((NO.GE.IORDER).OR.(KFLAG.LE.-1)) GO TO 59	DIF 295
	D=0.0	DIF 296
	DO 58 I=1,NBVMAX	DIF 297
	D=D+((ERROR(I)-SAVE(10,I))/YMAX(I))**2	DIF 298
58	CONTINUE	DIF 299
	PR3=(D/EUP)**ENQ3*1.4	DIF 300
59	PR1=1.0E+20	DIF 301
	IF (NQ.LE.1) GO TO 61	DIF 302
	D=0.0	DIF 303
	DO 60 I=1,NBVMAX	DIF 304
	D=D+(Y(K,I)/YMAX(I))**2	DIF 305
60	CONTINUE	DIF 306
	PR1=(D/EDWN)**ENQ1*1.3	DIF 307
61	CONTINUE	DIF 308
	IF (PR2.LE.PR3) GO TO 67	DIF 309
	IF (PR3.LT.PR1) GO TO 68	DIF 310
62	R=1.0/AMAX1(PR1,1.0E-04)	DIF 311
	NEWQ=NQ-1	DIF 312

63	IDOUB=10	DIF 313
	IF ((KFLAG.EQ.1).AND.(R.LT.1.1)) GO TO 72	DIF 314
	IF (NEWQ.LE.NQ) GO TO 65	DIF 315
	DO 64 I=1,NBVMAX	DIF 316
	Y(NEWQ+1,I)=ERROR(I)*A(K)/FLOAT(K)	DIF 317
64	CONTINUE	DIF 318
65	K=NEWQ+1	DIF 319
	IF (KFLAG.EQ.1) GO TO 69	DIF 320
	RACUM=RACUM*R	DIF 321
	IRET1=3	DIF 322
	GO TO 75	DIF 323
66	CONTINUE	DIF 324
	IRET1=0	DIF 325
	IF (NEWQ.EQ.NQ) RETURN	DIF 326
	NQ=NEWQ	DIF 327
	IRET=3	DIF 328
	GO TO 6	DIF 329
67	IF (PR2.GT.PR1) GO TO 62	DIF 330
	NEWQ=NQ	DIF 331
	R=1.0/AMAX1(PR2,1.0E-04)	DIF 332
	GO TO 63	DIF 333
68	R=1.0/AMAX1(PR3,1.0E-04)	DIF 334
	NEWQ=NQ+1	DIF 335
	GO TO 63	DIF 336
69	CONTINUE	DIF 337
	R=AMIN1(R,HMAX/ABS(H))	DIF 338
	H=H*R	DIF 339
	IF (NQ.EQ.NEWQ) GO TO 70	DIF 340
	NQ=NEWQ	DIF 341
	IRET=2	DIF 342
	GO TO 6	DIF 343
70	R1=1.0	DIF 344
	DO 71 J=2,K	DIF 345
	R1=R1*R	DIF 346
	DO 71 I=1,NBVMAX	DIF 347
71	Y(J,I)=Y(J,I)*R1	DIF 348
	IDOUB=K	DIF 349
72	CONTINUE	DIF 350
	DO 73 I=1,NBVMAX	DIF 351
	YMAX(I)=AMAX1(YMAX(I),ABS(Y(1,I)))	DIF 352
73	CONTINUE	DIF 353
	JSTART=NQ	DIF 354
	RETURN	DIF 355
74	PRINT #3	DIF 356
	STOP	DIF 357
75	CONTINUE	DIF 358
	RACUM=AMAX1(ABS(HMIN/HOLD),RACUM)	DIF 359
	RACUM=AMIN1(RACUM,ABS(HMAX/HOLD))	DIF 360
	H=HOLD*RACUM	DIF 361
	R1=1.0	DIF 362
	DO 76 J=2,K	DIF 363
	R1=R1*RACUM	DIF 364

	DO 76 I=1,NBVMAX	DIF 365
76	Y(J,I)=SAVE(J,I)*R1	DIF 366
	DO 77 I=1,NBVMAX	DIF 367
	Y(1,I)=SAVE(1,I)	DIF 368
77	CONTINUE	DIF 369
	IDOUB=K	DIF 370
	IF (IRET1.EQ.3) GO TO 66	DIF 371
	RETURN	DIF 372
78	PRINT 84	DIF 373
	STOP	DIF 374
C		DIF 375
79	FORMAT (41H THE MAXIMUM ORDER SPECIFIED IS TOO LARGE,I5)	DIF 376
80	FORMAT (49H THE CORRECTOR FAILED TO CONVERGE IN 3 ITERATIONS,/,23HDIF 377	
	1 DIFFERENTIAL EQUATIONS,I4,3H TO,I4)	DIF 378
81	FORMAT (58H CORRECTOR CONVERGENCE COULD NOT BE ACHIEVED FOR H,GT,MDIF 379	
	1MIN,/,23H DIFFERENTIAL EQUATIONS,I4,3H TO,I4)	DIF 380
82	FORMAT (30H TRUNCATION ERROR IS TOO LARGE)	DIF 381
83	FORMAT (72H THE STEP WAS TAKEN WITH H=HMIN BUT THE REQUESTED ERRORDIF 382	
	1 WAS NOT ACHIEVED)	DIF 383
84	FORMAT (68H THE REQUESTED ERROR IS SMALLER THAN CAN BE HANDLED FORDIF 384	
	1 THIS PROBLEM)	DIF 385
	END	DIF 386

SUBROUTINE SIMULT (NU,MR,MC,A,P,ICOL,ZTEST,ICC,IRC,IPIVOT)

ARGUMENTS

NU : NUMBER OF UNKNOWNNS  
 MR : MAXIMUM ROW DIMENSION OF MATRIX A IN CALLING PROGRAM  
 MC : MAXIMUM NUMBER OF COLUMNS OF MATRIX A  
 A : MATRIX CONTAINING NONZERO ELEMENTS  
 P : RIGHT HAND SIDE VECTOR  
 ICOL : RECORDS POSITION OF NONZERO ELEMENTS  
 ICC,IRC : DIMENSIONED IN CALLING PROGRAM AND USED ONLY IN  
 SUBROUTINE SIMULT  
 IPIVOT : PIVOT OPTION 1-7  
     1 : SIMPLE GAUSS-JORDAN ELIMINATION  
     2 : GAUSS-JORDAN PARTIAL PIVOTING  
     3 : GAUSS-JORDAN FULL PIVOTING  
     4 : MINIMUM ROW-MINIMUM COLUMN  
     5 : MINIMUM COLUMN-MINIMUM ROW  
     6 : MAXIMUM COLUMN-MINIMUM ROW  
     7 : MINIMUM OF ROW ENTRIES TIMES COLUMN ENTRIES

SOLUTION VECTOR RETURNED IN FIRST COLUMN OF A : A(I,1)

INTEGER PIVROW,PIVCOL,OPROW  
 DIMENSION IRC(NU), ICC(NU), A(MR,MC), ICOL(MR,MC), P(NU)

INITIALIZE ROW ENTRY COUNTER

DO 1 I=1,NU

IRC(I)=0

CONTINUE

COUNT ROW AND COLUMN ENTRIES

JELE=0

DO 3 I=1,NU

DO 2 J=1,MC

IC=ICOL(I,J)

IF (IC,EQ.0) GO TO 3

JELE=JELE+1

ICC(I)=J

IRC(IC)=IRC(IC)+1

CONTINUE

CONTINUE

JCOL=0

JELIM=0

DO 800 I=1,NU

800 IF (ICC(I).GT.JCOL) JCOL=ICC(I)

LKJ=1

GO TO 26

CONTINUE

NORMALIZE PIVROW

X=A(PIVROW,IY)

IC=ICC(PIVROW)

DO 5 J=1,IC

SIM 1  
 SIM 2  
 SIM 3  
 SIM 4  
 SIM 5  
 SIM 6  
 SIM 7  
 SIM 8  
 SIM 9  
 SIM 10  
 SIM 11  
 SIM 12  
 SIM 13  
 SIM 14  
 SIM 15  
 SIM 16  
 SIM 17  
 SIM 18  
 SIM 19  
 SIM 20  
 SIM 21  
 SIM 22  
 SIM 23  
 SIM 24  
 SIM 25  
 SIM 26  
 SIM 27  
 SIM 28  
 SIM 29  
 SIM 30  
 SIM 31  
 SIM 32  
 SIM 33  
 SIM 34  
 SIM 35  
 SIM 36  
 SIM 37  
 SIM 38  
 SIM 39  
 SIM 40  
 SIM 41  
 SIM 42  
 SIM 43  
 SIM 44  
 SIM 45  
 SIM 46  
 SIM 47  
 SIM 48  
 SIM 49  
 SIM 50  
 SIM 51  
 SIM 52

	A(PIVROW,J)=A(PIVROW,J)/X	SIM	53
5	CONTINUE	SIM	54
	A(PIVROW,IY)=1.0	SIM	55
	P(PIVROW)=P(PIVROW)/X	SIM	56
C	SELECT ROWS THAT CAN BE OPERATED ON	SIM	57
	DO 21 I=1,NU	SIM	58
	IF (IRC(PIVCOL).EQ.1) GO TO 22	SIM	59
	IF (I.EQ.PIVROW) GO TO 21	SIM	60
	IC=IABS(ICC(I))	SIM	61
	DO 20 J=1,IC	SIM	62
	IF (ICOL(I,J)-PIVCOL) 20,6,21	SIM	63
C	IF YOU CAN GET TO THIS POINT OPROW CONTAINS PIVOTAL ELEMENT	SIM	64
6	OPROW=I	SIM	65
	JKOP=1	SIM	66
	JKPI=1	SIM	67
	C=-A(OPROW,J)	SIM	68
	P(OPROW)=P(PIVROW)*C+P(OPROW)	SIM	69
7	CONTINUE	SIM	70
	IF (ICOL(PIVROW,JKPI).EQ.0) GO TO 21	SIM	71
	IF (ICOL(OPROW,JKOP).EQ.0) GO TO 8	SIM	72
	IF (ICOL(PIVROW,JKPI)-ICOL(OPROW,JKOP)) 8,12,19	SIM	73
C	OPROW DOES NOT CONTAIN THIS ELEMENT,ADD ELEMENT TO OPROW	SIM	74
8	ICC(I)=ICC(I)+1	SIM	75
	IF (ICC(I).LE.0) ICC(I)=ICC(I)-2	SIM	76
	II=IABS(ICC(I))	SIM	77
C	IF (II.GT.JCOL) JCOL=II	SIM	78
	IF (II.GT.MC) GO TO 9	SIM	79
	GO TO 10	SIM	80
9	CONTINUE	SIM	81
	PRINT 47, II	SIM	82
	STOP	SIM	83
10	CONTINUE	SIM	84
	JKL=JKOP+1	SIM	85
11	IX=II-1	SIM	86
	A(OPROW,II)=A(OPROW,IX)	SIM	87
	ICOL(OPROW,II)=ICOL(OPROW,IX)	SIM	88
	II=IX	SIM	89
	IF (II.GE.JKL) GO TO 11	SIM	90
	A(OPROW,JKOP)=A(PIVROW,JKPI)*C	SIM	91
	ICOL(OPROW,JKOP)=ICOL(PIVROW,JKPI)	SIM	92
	IX=ICOL(OPROW,JKOP)	SIM	93
	IRC(IX)=IRC(IX)+1	SIM	94
	GO TO 18	SIM	95
C	PIVROW AND OPROW CONTAIN THIS ELEMENT *SHIFT BOTH AND OPERATE ON	SIM	96
C	OPR	SIM	97
12	IX=ICOL(OPROW,JKOP)	SIM	98
	IF (IX.EQ.PIVCOL) GO TO 13	SIM	99
	X=A(PIVROW,JKPI)*C+A(OPROW,JKOP)	SIM	100
	A(OPROW,JKOP)=X	SIM	101
C	TEST OPROW TO SEE IF ANY ELEMENTS WERE ELIMINATED OTHER THAN	SIM	102
C	THOSE IN THE PIVOTAL COLUMN	SIM	103
	ATEST=ABS(X)-ZTEST	SIM	104

	IF (ATEST,GT.0.) GO TO 18	SIM 105
13	IRC(IX)=IRC(IX)-1	SIM 106
C	JELIM=JELIM+1	SIM 107
	ICC(OPROW)=ICC(OPROW)-1	SIM 108
	IF (ICC(OPROW)) 15,14,16	SIM 109
14	GO TO 24	SIM 110
15	CONTINUE	SIM 111
	ICC(OPROW)=ICC(OPROW)+2	SIM 112
16	IX=IABS(ICC(OPROW))	SIM 113
	DO 17 NK=JKOP,IX	SIM 114
	A(I,NK)=A(I,NK+1)	SIM 115
	ICOL(I,NK)=ICOL(I,NK+1)	SIM 116
17	CONTINUE	SIM 117
	IX=IX+1	SIM 118
	ICOL(I,IX)=0	SIM 119
	JKPI=JKPI+1	SIM 120
	GO TO 7	SIM 121
18	JKPI=JKPI+1	SIM 122
C	PIVROW DOES NOT CONTAIN THIS ELEMENT*SHIFT OPROW AND CONTINUE	SIM 123
19	JKOP=JKOP+1	SIM 124
	GO TO 7	SIM 125
20	CONTINUE	SIM 126
21	CONTINUE	SIM 127
22	CONTINUE	SIM 128
C	ELIMINATES PIVROW AND PIVCOL FROM BEING CONSIDERED AGAIN	SIM 129
	ICC(PIVROW)=-ICC(PIVROW)	SIM 130
	IRC(PIVCOL)=-IRC(PIVCOL)	SIM 131
	LKJ=LKJ+1	SIM 132
	IF (LKJ.LE.NU) GO TO 26	SIM 133
C	UNSCRAMBLE AND STORE SOLUTION IN FIRST COLUMN OF A	SIM 134
	DO 23 I=1,NU	SIM 135
	II=ICOL(I,1)	SIM 136
	A(II,1)=P(I)	SIM 137
23	CONTINUE	SIM 138
	RETURN	SIM 139
24	CONTINUE	SIM 140
C	SUBROUTINE SINGLR	SIM 141
	IB=1	SIM 142
	DO 25 M=1,NU	SIM 143
	II=ICC(M)	SIM 144
	IF (II,GT.IB) IB=II	SIM 145
25	CONTINUE	SIM 146
	WRITE (6,48) LKJ,PIVROW,PIVCOL,OPROW	SIM 147
	WRITE (6,49) (M,IRC(M),ICC(M),M=1,NU)	SIM 148
	WRITE (6,50)	SIM 149
	CALL RITE (1,NU,IB,MR,MC,A,ICOL)	SIM 150
	WRITE (6,51)	SIM 151
	CALL RITE (2,NU,IB,MR,MC,A,ICOL)	SIM 152
	STOP	SIM 153
26	CONTINUE	SIM 154
	GO TO (27,28,30,33,36,40,44), IPIVOT	SIM 155
27	CONTINUE	SIM 156



C		SIM 157
C	PIVOT OPTION 1 : SIMPLE GAUSS-JORDAN ELIMINATION	SIM 158
C		SIM 159
C	SUBROUTINE PIVSEL	SIM 160
C	GAUSS ELIMINATION	SIM 161
	PIVROW=LKJ	SIM 162
	PIVCOL=LKJ	SIM 163
	IY=1	SIM 164
C	END GAUSS ELIMINATION	SIM 165
C	RETURN	SIM 166
	GO TO 4	SIM 167
28	CONTINUE	SIM 168
C		SIM 169
C	PIVOT OPTION 2 : GAUSS-JORDAN PARTIAL PIVOTING	SIM 170
C		SIM 171
C	SUBROUTINE PIVSEL	SIM 172
C	GAUSS JORDAN-PARTIAL PIVOTING	SIM 173
	PIVCOL=LKJ	SIM 174
	ATEST=0.	SIM 175
	DO 29 I=1,NU	SIM 176
	IC=ICC(I)	SIM 177
	IF (IC.LE.0) GO TO 29	SIM 178
	II=ICOL(I,1)	SIM 179
	IF (II.GT.PIVCOL) GO TO 29	SIM 180
	AA=ABS(A(I,1))	SIM 181
	IF (AA.LE.ATEST) GO TO 29	SIM 182
	ATEST=AA	SIM 183
	PIVROW=I	SIM 184
29	CONTINUE	SIM 185
	IY=1	SIM 186
C	END GAUSS JORDAN-PARTIAL PIVOTING	SIM 187
C	RETURN	SIM 188
	GO TO 4	SIM 189
30	CONTINUE	SIM 190
C		SIM 191
C	PIVOT OPTION 3 : GAUSS-JORDAN FULL PIVOTING	SIM 192
C		SIM 193
C	SUBROUTINE PIVSEL	SIM 194
C	GAUSS JORDAN FULL PIVOTING	SIM 195
	ATEST=0.	SIM 196
	DO 32 I=1,NU	SIM 197
	IC=ICC(I)	SIM 198
	IF (IC.LE.0) GO TO 32	SIM 199
	DO 31 J=1,IC	SIM 200
	II=ICOL(I,J)	SIM 201
	IR=IRC(II)	SIM 202
	IF (IR.LE.0) GO TO 31	SIM 203
	AA=ABS(A(I,J))	SIM 204
	IF (AA.LE.ATEST) GO TO 31	SIM 205
	ATEST=AA	SIM 206
	PIVROW=I	SIM 207
	PIVCOL=II	SIM 208

	IY=J	SIM 209
31	CONTINUE	SIM 210
32	CONTINUE	SIM 211
C	END GAUSS JORDAN FULL PIVOTING	SIM 212
C	RETURN	SIM 213
	GO TO 4	SIM 214
33	CONTINUE	SIM 215
C		SIM 216
C	PIVOT OPTION 4 : MINIMUM ROW-MINIMUM COLUMN	SIM 217
C		SIM 218
C	SUBROUTINE PIVSEL	SIM 219
C	SELECT FIRST MIN ROW THEN FIRST MIN COL	SIM 220
C	SELECT ROW WITH MINIMUM ENTRIES	SIM 221
	IK=100000	SIM 222
	DO 34 I=1,NU	SIM 223
	IC=ICC(I)	SIM 224
	IF (IC.GE.IK.OR.IC.LE.0) GO TO 34	SIM 225
	PIVROW=I	SIM 226
	IK=IC	SIM 227
34	CONTINUE	SIM 228
C	SELECT SMALLEST AVAILABLE COLUMN FROM PIVROW	SIM 229
	IK=100000	SIM 230
	IC=ICC(PIVROW)	SIM 231
	DO 35 I=1,IC	SIM 232
	II=ICOL(PIVROW,I)	SIM 233
	IR=IRC(II)	SIM 234
	IF (IR.GE.IK.OR.IR.LE.0) GO TO 35	SIM 235
	PIVCOL=II	SIM 236
	IK=IR	SIM 237
	IY=I	SIM 238
35	CONTINUE	SIM 239
C	END FIRST MIN ROW THEN FIRST MIN COL	SIM 240
C	RETURN	SIM 241
	GO TO 4	SIM 242
36	CONTINUE	SIM 243
C		SIM 244
C	PIVOT OPTION 5 : MINIMUM COLUMN-MINIMUM ROW	SIM 245
C		SIM 246
C	SUBROUTINE PIVSEL	SIM 247
C	SELECT FIRST MIN COL THEN FIRST MIN ROW	SIM 248
	IK=100000	SIM 249
	DO 37 I=1,NU	SIM 250
	IR=IRC(I)	SIM 251
	IF (IR.GE.IK.OR.IR.LE.0) GO TO 37	SIM 252
	PIVCOL=I	SIM 253
	IK=IR	SIM 254
37	CONTINUE	SIM 255
	IK=100000	SIM 256
	DO 39 I=1,NU	SIM 257
	IC=ICC(I)	SIM 258
	IF (IC.LE.0) GO TO 39	SIM 259
	DO 38 J=1,IC	SIM 260

	IF (ICOL(I,J).LT.PIVCOL) GO TO 38	SIM 261
	IF (ICOL(I,J).GT.PIVCOL.OR.IC.GE.IK) GO TO 39	SIM 262
	IK=IC	SIM 263
	PIVROW=I	SIM 264
	IY=J	SIM 265
38	CONTINUE	SIM 266
39	CONTINUE	SIM 267
C	END SELECT FIRST MIN COL THEN FIRST MIN ROW	SIM 268
C	RETURN	SIM 269
	GO TO 4	SIM 270
40	CONTINUE	SIM 271
C		SIM 272
C	PIVOT OPTION 6 : MAXIMUM COLUMN-MINIMUM ROW	SIM 273
C		SIM 274
C	SUBROUTINE PIVSEL	SIM 275
C	SELECT FIRST MAX COL THEN FIRST MIN ROW	SIM 276
	IK=-1	SIM 277
	DO 41 I=1,NU	SIM 278
	IR=IRC(I)	SIM 279
	IF (IR.LE.IK.OR.IR.LE.0) GO TO 41	SIM 280
	PIVCOL=I	SIM 281
	IK=IR	SIM 282
41	CONTINUE	SIM 283
	DO 43 I=1,NU	SIM 284
	IC=ICC(I)	SIM 285
	IF (IC.LE.0) GO TO 43	SIM 286
	DO 42 J=1,IC	SIM 287
	IF (ICOL(I,J).LT.PIVCOL) GO TO 42	SIM 288
	IF (ICOL(I,J).GT.PIVCOL.OR.IC.GE.IK) GO TO 43	SIM 289
	IK=IC	SIM 290
	PIVROW=I	SIM 291
	IY=J	SIM 292
42	CONTINUE	SIM 293
43	CONTINUE	SIM 294
C	END SELECT FIRST MAX COL THEN FIRST MIN ROW	SIM 295
C	RETURN	SIM 296
	GO TO 4	SIM 297
44	CONTINUE	SIM 298
C		SIM 299
C	PIVOT OPTION 7 : MINIMUM OF ROW ENTRIES TIMES COLUMN ENTRIES	SIM 300
C		SIM 301
C	SUBROUTINE PIVSEL	SIM 302
C	SELECT FIRST MIN(ROW*COL)	SIM 303
	IK=100000	SIM 304
	DO 46 I=1,NU	SIM 305
	IC=ICC(I)	SIM 306
	IF (IC.LE.0) GO TO 46	SIM 307
	DO 45 J=1,IC	SIM 308
	II=ICOL(I,J)	SIM 309
	IR=IRC(II)	SIM 310
	IF (IR.LE.0) GO TO 45	SIM 311
	III=IC*IR	SIM 312

	IF (III,GE,IK) GO TO 45	SIM 313
	PIVROW=I	SIM 314
	PIVCOL=II	SIM 315
	IK=III	SIM 316
	IY=J	SIM 317
45	CONTINUE	SIM 318
46	CONTINUE	SIM 319
C	END SELECT FIRST MIN(ROW*COL)	SIM 320
C	RETURN	SIM 321
	GO TO 4	SIM 322
C		SIM 323
47	FORMAT (22H MC SHOULD BE AT LEAST,I3)	SIM 324
48	FORMAT (*1 MATRIX SINGULAR*,//,* NO. CYCLES COMPLETED**,I5,* P1	SIM 325
	IVROW**,I5,* PIVCOL=*I5,* OPROW**I5)	SIM 326
49	FORMAT (*0 COL:/ROW NO.,NO. COL. ENTRIES,NO. ROW ENTRIES*/(6(I7,2I	SIM 327
	15)))	SIM 328
50	FORMAT (*1 COEFFICIENT MATRIX*)	SIM 329
51	FORMAT (*1 COLUMN IDENTIFICATION*)	SIM 330
	END	SIM 331-

```

SUBROUTINE RITE (IDUM, NR, NC, MR, MC, A, ICOL)
DIMENSION A(MR, MC), ICOL(MR, MC)
IPRINT=12
IF (IDUM.NE.1) IPRINT=30
IPR=IPRINT-1
DO 4 K=1, NC, IPRINT
MAX=K+IPR
IF (MAX.GT.NC) MAX=NC
IF (K.NE.1) WRITE (6,7)
IF (IDUM.EQ.1) GO TO 2
WRITE (6,5) (I, I=K, MAX)
DO 1 J=1, NR
WRITE (6,8) J, (ICOL(J, I), I=K, MAX)
CONTINUE
GO TO 4
WRITE (6,6) (I, I=K, MAX)
DO 3 J=1, NR
WRITE (6,9) J, (A(J, I), I=K, MAX)
CONTINUE
CONTINUE
RETURN

C
5 FORMAT (6X, 30I4)
6 FORMAT (6X, I2I10)
7 FORMAT (1H1)
8 FORMAT (1X, I5, 30I4)
9 FORMAT (1X, I5, 12G10.3)
END

```

RIT 1  
RIT 2  
RIT 3  
RIT 4  
RIT 5  
RIT 6  
RIT 7  
RIT 8  
RIT 9  
RIT 10  
RIT 11  
RIT 12  
RIT 13  
RIT 14  
RIT 15  
RIT 16  
RIT 17  
RIT 18  
RIT 19  
RIT 20  
RIT 21  
RIT 22  
RIT 23  
RIT 24  
RIT 25  
RIT 26  
RIT 27  
RIT 28

1  
2  
3  
4  
C  
5  
6  
7  
8  
9

SUBROUTINE TYPE10

SUBROUTINE REAC1

THIS MODULE REPRESENTS A CSTR WITH A FIRST ORDER REVERSIBLE REACTION, 1 INPUT STREAM, 1 OUTPUT STREAM, CONSTANT TEMPERATURE.

EQUIPMENT PARAMETERS

1 - REACTOR VOLUME - FT\*\*3  
 2 - K1 - FORWARD REACTION RATE CONSTANT - MIN\*\*-1  
 3 - K2 - BACKWARD REACTION RATE CONSTANT - MIN\*\*-1  
 4 - ITER - ITER=0 - USE DIRECT ITERATION WITH STIFF OPTION  
 ITER=1 - USE NEWTON-RAPHSON ITERATION WITH STIFF OPTION  
 FOR NONSTIFF OPTION ITER IS NOT USED

COMMON /MAT/ MP(15,5),EP(15,5),S(2,16,7),EX(1)  
 COMMON /CON/ IG,NCOMP,NC5,H,NE,NS,NPR,NPOL,TMAX,IORDER,NGRAPH  
 COMMON /PTAB/ IGFLAG,PP(10,10)  
 COMMON /UNIT/ IM  
 COMMON /COLUMN/ JCOL(10,10)  
 COMMON /JACOB/ XJACOB(10,10)  
 COMMON /MODULE/ IDERY,ITER,ITRI,MC,IPIVOT  
 DIMENSION Y(2), DERY(2)  
 REAL K1,K2

CALCULATE MODULE PARAMETERS.

GET VOLUME OF REACTOR FROM EP - VOL

VOL=EP(IM,1)

GET REACTION RATE CONSTANTS FROM EP - K1,K2

K1=EP(IM,2)

K2=EP(IM,3)

GET ITERATION OPTION FROM EP

ITER=EP(IM,4)

GET STREAM NUMBER OF INPUT STREAM FROM MP - IN

IN=MP(IM,3)

GET STREAM NUMBER OF OUTPUT STREAM FROM MP - IOUT

IOUT=IABS(MP(IM,4))

GET DENSITY OF INPUT STREAM FROM PP - DENS

DENS=PP(1,2)

CALCULATE REACTOR TIME CONSTANT - TAU

TAU=VOL/(S(IG,IN,3)/DENS)

GET INITIAL REACTOR CONCENTRATIONS FROM OUTPUT STREAM

Y(1)=S(IG,IOUT,6)

Y(2)=S(IG,IOUT,7)

CALCULATE JACOBIAN MATRIX ON CORRECTOR PASS

IF (IG.EQ.2) GO TO 1

MC=2

IPIVOT=3

T10 1  
 T10 2  
 T10 3  
 T10 4  
 T10 5  
 T10 6  
 T10 7  
 T10 8  
 T10 9  
 T10 10  
 T10 11  
 T10 12  
 T10 13  
 T10 14  
 T10 15  
 T10 16  
 T10 17  
 T10 18  
 T10 19  
 T10 20  
 T10 21  
 T10 22  
 T10 23  
 T10 24  
 T10 25  
 T10 26  
 T10 27  
 T10 28  
 T10 29  
 T10 30  
 T10 31  
 T10 32  
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 T10 35  
 T10 36  
 T10 37  
 T10 38  
 T10 39  
 T10 40  
 T10 41  
 T10 42  
 T10 43  
 T10 44  
 T10 45  
 T10 46  
 T10 47  
 T10 48  
 T10 49  
 T10 50  
 T10 51  
 T10 52

	JCOL (1,1)=1	T10	53
	JCOL (2,1)=1	T10	54
	JCOL (1,2)=2	T10	55
	JCOL (2,2)=2	T10	56
	XJACOB (1,1)=- (K1+1.0/TAU)	T10	57
	XJACOB (2,1)=K1	T10	58
	XJACOB (1,2)=K2	T10	59
	XJACOB (2,2)=- (K2+1.0/TAU)	T10	60
1	CONTINUE	T10	61
C		T10	62
C	CALCULATE DERIVATIVES	T10	63
C		T10	64
C	MASS BALANCE FOR COMPONENT A (UNITS=MIN**=-1)	T10	65
C	RATE OF CHANGE OF MASS FRACTION OF COMPONENT A	T10	66
C	= RATE OF INPUT OF A	T10	67
C	-RATE OF OUTPUT OF A	T10	68
C	+RATE AT WHICH B REACTS INTO A	T10	69
C	-RATE AT WHICH A REACTS INTO B	T10	70
C	DERY (1)=- (K1+1.0/TAU)*Y (1)+K2*Y (2)+S (I0,IN,6)/TAU	T10	71
C	MASS BALANCE FOR COMPONENT B (UNITS=MIN**=-1)	T10	72
C	RATE OF CHANGE OF MASS FRACTION OF COMPONENT B	T10	73
C	= RATE OF INPUT OF B	T10	74
C	-RATE OF OUTPUT OF B	T10	75
C	+RATE AT WHICH A REACTS INTO B	T10	76
C	-RATE AT WHICH B REACTS INTO A	T10	77
C	DERY (2)=K1*Y (1)- (K2+1.0/TAU)*Y (2)+S (I0,IN,7)/TAU	T10	78
C		T10	79
C	CALL DIFSUB TO SOLVE ODES FOR MODULE	T10	80
C		T10	81
C	CALL DIFSUB (2,Y,DERY)	T10	82
C	IF (IDERY.NE.0) GO TO 1	T10	83
C		T10	84
C	CALCULATE STREAM OUTPUT	T10	85
C		T10	86
C	NORMALIZE CONCENTRATIONS OF A AND B	T10	87
C	SUM=Y (1)+Y (2)	T10	88
C	Y (1)=Y (1)/SUM	T10	89
C	Y (2)=Y (2)/SUM	T10	90
C	PUT MASS FRACTIONS INTO OUTPUT STREAMS	T10	91
C	S (1,IOUT,6)=Y (1)	T10	92
C	S (1,IOUT,7)=Y (2)	T10	93
C	RETURN	T10	94
C	END	T10	95

.....  
 REACTOR SIMULATION - 2 CSTRS  
 .....


```

BEGIN
TIME      1.0
HMAX      1.0
COMPS     2.0
LIBRARY   1.0
REAC1     10.0
PROCESS
REAC1     1.0
          1.0      -2.0
          10.0     4.90909   4.09091   1.0
REAC1     2.0
          2.0      -3.0
          10.0     686.81   312.19   1.0
END
STREAMS
EXPLICIT  3.0
          1.0      1.0      624.0   60.0   14.7
          1.0      0.0
          2.0      1.0      624.0   70.0   14.7
          1.0      0.0
          3.0      1.0      624.0   120.0  14.7
          1.0      0.0
END
PROPERTIES -1.0
END

```

END





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